

PERFORMANCE AND NO_x MODELLING IN A
DIRECT INJECTION STRATIFIED CHARGE ENGINE

by

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ABSTRACT

A model has been developed to predict the performance and NO_x emission of the Texaco stratified charge engine. This complete engine cycle program starts from engine geometry, valve and fuel injection timing, fuel characteristics and operating conditions and includes the following phases: valve overlap, intake, compression, combustion, expansion and exhaust.

The fuel mixing process is described by a jet mixing model and the equations are solved in a cylindrical coordinate system with a non-uniform pressure field. The jet shape changes were calculated for the jet impingement effect against a solid wall. The air motion inside the cylinder is determined by a detailed model to specify the field for the fuel jet.

During combustion fuel jet is divided into many elements and the combustion process of each element is analyzed as a mixing process between the jet and surrounding air, entrainment into a flame front and subsequent combustion.

For heat transfer the walls of the combustion chamber are divided into five regions; intake valve, exhaust valve, cylinder head, cylinder wall and piston top. Each region is assumed to have a different temperature and different gas velocity.

Nitric oxide emissions are calculated by using the extended Zel'dovich kinetic scheme, with the steady state assumption for the N concentration and equilibrium values used for H, O, O₂ and OH concentrations.

The model computes combustion rates, heat transfer and NO_x based on the same jet mixing assumptions without appeal to separate mixing processes at each stage.

Comparison of the model prediction with the available experimental data shows reasonably good agreement.

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SYMBOLS

A	= heat transfer area
Bore	= cylinder bore diameter
b	= jet radius in free jet
b_c	= jet radius when contacting with wall
C_p	= specific heat at constant pressure
E	= internal energy
H	= specific enthalpy
h	= distance from cylinder head to piston top
L_{cnr}	= length of connecting rod
M	= mass
N	= engine speed
Nu	= Nusselt number
P	= pressure
Q	= heat transfer to system
R	= radial distance of jet location
R_c	= radius of piston cup
Re	= Reynolds number
R_g	= gas constant
R_p	= radius of piston
R_{cr}	= radius of crank
s	= distance along the jet trajectory
S_e	= plume surface area
T	= temperature
u	= jet velocity

u_e	= turbulent eddy entrainment velocity
V	= volume
V_c	= piston cup volume
v_i	= parallel component of cross flow to u
v_n	= normal component of cross flow to u
v_r	= radial component of jet (swirl) velocity
v_z	= vertical component of jet (swirl) velocity
w	= angular velocity of jet (swirl)
W	= work done by system
X_f	= mass fraction of fuel in jet
Z	= axial distance of jet location
α	= entrainment parameter for the parallel flow
β	= entrainment parameter for the normal flow
ρ	= density
ϕ	= equivalence ratio
θ	= tangential component of jet location
θ_i	= fuel injection timing
θ_j	= jet contacting angle with wall (see Fig. 4)
θ_{cr}	= crank angle (0 at top dead center)
λ	= Taylor microscale
t_c	= characteristic reaction time for the microscale

Subscripts

i = element number i

in = inflow

out = outflow

e = entrained gas in the plume

b = burned gas in the plume

u = unburned gas in the plume

∞ = the gas surrounding the jet

I Introduction

The name, Texaco Controlled Combustion System (TCCS) is used to describe a spark ignited, direct injection, stratified charge engine. The combustion chamber is a deep cup in the piston; relatively higher air swirl is used. This engine may be thought of as a cross between the homogeneous charge, spark-ignition gasoline engine, and the heterogeneous charge, compression-ignition diesel engine. However, since combustion timing is controlled by fuel injection rates; the combustion process more nearly approximates that of an open chamber diesel engine. This engine concept appears to have the well controlled, soft combustion of the spark-ignition engine and, thereby, the lightweight structure of the gasoline engine. The TCCS concept also demonstrates excellent fuel economy, broad fuel tolerance, (multi-fuel capability) and relatively low emissions. The basic conceptual details behind the Texaco Controlled Combustion System are described below while additional hardware details can be found in the references on the TCCS engine (1-4).

TCCS Engine Description

The Texaco Controlled Combustion System, as illustrated in Fig. 1, requires coordination of air swirl, fuel-injection and positive ignition. The high air swirl is obtained from a shrouded inlet valve and is amplified during compression by being forced into a deep narrow

combustion chamber. The combustion chamber is essentially a cup having a cylindrical upper section with toroidal bottom cast into the head of the piston. The diameter of the cup is approximately one half of the cylinder diameter.

The high pressure injection system is based on standard diesel practice and uses a special version of a standard Roosa Master Pencil Nozzle. The distinguishing feature of this nozzle is a special flat seat and a single-hole orifice instead of the more usual conical seating, multi-hole sac-tip design. Valve opening pressure is usually set at 1500-2000 psi. For full load, fuel injection duration corresponds approximately to the time for one air swirl and the overall fuel-air ratio is near stoichiometric. At lower loads, obtained by decreased fuel injection duration and quantity, the operation is lean of stoichiometric and over all air to fuel ratio approaches 100:1 at idle conditions.

The usual diesel engine problems of long ignition delay, high rates of pressure rise and high peak pressures with low cetane fuels are avoided in TCCS operation by providing a positive ignition source. The Texaco Ignition System (TTIS) is a high energy, multi-spark unit with controlled duration. The ignition is triggered near the start of injection and continues through part or all of the injection process. The problems of spontaneous ignition and octane requirement associated with conventional gasoline engines is eliminated with TCCS since the residence time of combustible fuel-air mixtures is extremely short.

The TCCS mode of operation thus results in unique characteristics;

high part-load thermal efficiency at lean mixtures and inherently low hydrocarbon and carbon monoxide emissions resulting from excess air operation and controlled combustion rates. In addition, with the high pressure injection system, a wide range of fuel volatility can be tolerated. High compression ratio and/or inlet super charging can be used to produce good operation and performance on low cost fuels. These characteristics, in addition to quick warm-up and excellent driveability are significant factors in achieving an automotive engine with good performance and low exhaust emissions.

The purpose of the research work to be described in the following pages has been to develop a model to predict the performance of the TCCS, stratified charge engine given the engine geometry, operating parameters and the fuel properties. Once the model is shown to be effective in predicting the performance and NO_x , it can then be used as a design tool. With the brief description of the geometry and the mode of operation of the engine given above, we will now proceed to a discussion of the model.

II. Direct Injection Engine Simulation Model

1. Background

In the direct fuel injection engine, a distribution of temperature and equivalence ratio exists inside the combustion chamber. Since formation of NO_x is very much dependent on the local equivalence ratio and temperature, the key problem for this engine simulation is to define the spacial and temporal distribution of these parameters.

Early diesel engine combustion models include work by Lyn (6) and he developed an empirical calculation of the heat release rate by using triangles to simulate mixing and combustion of successive burned elements. Shahed, et. al. (7) integrated this idea with assumptions that have been widely used in spark ignition engine models (8,9) to compute NO_x . In this model, the fuel is assumed to mix with a stoichiometric amount of air before combustion and the mixed gas is divided into many packaged elements which are not mixing with each other. The temperature distribution inside the combustion chamber was explained by the different combustion time of each element. Bastress (34) took into consideration the distribution of equivalence ratio inside the combustion chamber, but this model is not based on physical mixing and combustion processes.

Following this work much effort was placed on developing physical arguments to compute the heat release rate proposed by Lyn. Shipinski (10), Whitehouse (11) and Bracco (12) have explained this delay by using fuel droplet evaporation models. But it is apparent that the combustion

process in direct injection engines cannot be predicted only by evaporation time.

Spray formation and mixing in engines was studied using continuum models by Adler (13) and Rife (14). Rife concludes that the motion of the fuel jet can be satisfactorily analyzed with such a model, but this model was not integrated into a combustion calculation.

Hiroyasu (15) made a complete heterogeneous mixture combustion model by assuming a simple cone shaped spray model and a more extensive jet mixing model, which included the effects of cross flow and jet tip shape, was developed by Chiu (16). These models do not include any adjustments of the fuel jet on the wall of the combustion.

Much of the work on heterogeneous combustion is based on gas turbine engine models. Stochastic techniques appear to be a powerful method (17, 18) for including the mixing phenomena in a statistical way. Further, the mixing rates have a physical basis. However, the models lack the geometric detail required for design and consequently have somewhat limited utility. In addition, these models require an enormous computing time in the case of reciprocating engines because of the way the combustion chamber air motion changes with crank angle. It is also possible that air motion inside the reciprocating engine is complicated enough to require more than a single mixing parameter.

In previous work, a model of the TCCS engine was developed by Jain (19). In our first extension of this model to compute NO_x , the

combustion chamber is divided into two areas; one a hot burned gas area, the other cold air. Each zone was considered to have uniform equivalence ratio and temperature. NO_x was calculated by using extended Zel'dovich equations. Values for the (NO_x) concentration obtained from this calculation are very low and this result was assumed to be due to the following reasons:

- a) During the rapid combustion process, fixed equivalence ratio (1.4) for the combustion zone was assumed. (NO_x) concentration and one way reaction rates decrease sharply as equivalence ratio goes above 1.1 ~ 1.2.

- b) The model assumed temperature constant inside the plume thereby making the (NO_x) concentration even lower, when compared with the actual case where a temperature distribution is known to exist.

Consequently we focused our attention on a new model that would include the mixing process through appeal to the equations of motion for a turbulent jet.

2. The New Model

The model divides the complete engine cycle into the following periods:

- i) valve overlap period
- ii) intake period
- iii) compression period
- iv) combustion period
- v) expansion period
- vi) exhaust period

Also the engine cycle is divided into the following items based on physical phenomena:

- i) gas exchange
- ii) heat transfer
- iii) air motion inside the cylinder
- iv) fuel jet dynamics
- v) heat release
- vi) NO_x formation
- vii) friction loss

The great benefit of this program is to be able to calculate the jet

dynamics coupled with the surrounding air motion, as a basis for calculation of the heat release rate, the NO_x formation and heat transfer between jet elements and combustion wall.

The computer program is based on the following assumptions;

- i) Temperatures in the cylinder are functions of time and location.
- ii) Pressures in the cylinder are functions of time only.
- iii) The charge is assumed homogeneous during intake, compression and exhaust process. (During combustion, the fuel jet is divided into 7-15 elements and each element has a different location and different equivalence ratio.)
- iv) The individual elements in the gas mixture are homogeneous but each has two different temperature zones, i.e., burned and unburned mixture. The unburned gas is composed of a low temperature mixture of air, non-reacting fuel and residual gases. The combustion zone is composed of high temperature unburned mixture and combustion products.
- v) There is mixing between jet elements and their environment, but no mixing between jet elements themselves.
- vi) Quasi-steady adiabatic and isentropic flow is assumed for mass flows past the valves.
- vii) The intake and exhaust manifolds are treated as infinite plenums having specified pressure and temperature histories, except during reversed flow past the intake valve, when plug flow is assumed to occur.

viii) Heat transfer is predicted with a model based on concepts introduced by Woschni(20) that have been extended to use local velocities in the combustion chamber.

ix) Thermodynamic characteristics of gas are based on a model proposed by Martin and Heywood(21).

x) Nitric oxide emissions are calculated by using the extended Zel'dovich kinetic scheme, with the steady state assumption for the N concentration and equilibrium values used for H, O, O₂, and OH concentrations in the adiabatic core.

3. Systems and Thermodynamic Equations

The early complete engine cycle simulation by Borman(22) assumed that the combustion gas as one uniform mixed element. In our model air and fuel are divided into many elements and every element has the same set of system and thermodynamic equations. A schematic of our model is shown in Fig. 2. Every element is viewed as an open system. *

i) State equation:

$$PV_i = M_i R_{gi} T_i$$

$$\frac{\dot{P}}{P} + \frac{\dot{V}_i}{V_i} = \frac{\dot{M}_i}{M_i} + \frac{\dot{T}_i}{T_i} \quad (3-1)$$

$$R_{gi} = R_{gi}(T_i, P, \phi_i) - \text{constant during each computing interval}$$

ii) Mass conservation:

$$\dot{M}_i = \dot{M}_{in_i} - \dot{M}_{out_i} \quad (3-2)$$

iii) Energy conservation for open system:

$$\dot{E}_i = \dot{Q}_i - \dot{W}_i + \dot{M}_{in_i} H_{in_i} - \dot{M}_{out_i} H_{out_i} \quad (3-3)$$

*All symbols are defined on pages 10 and 11.

iv) Internal energy equations:

$$E_i = M_i H_i - PV_i$$

$$\dot{E}_i = \dot{M}_i H_i + M_i \dot{H}_i - \dot{P}V_i - P\dot{V}_i \quad (3-4)$$

v) Work definition:

$$\dot{W}_i = P\dot{V}_i \quad (3-5)$$

vi) Enthalpy definition:

$$\dot{H}_i = C_{pi} \dot{T}_i$$

$$C_{pi} = C_{pi}(T_i, P, \phi_i) - \text{constant during each computing interval} \quad (3-6)$$

vii) Volume constraint:

$$\dot{V}_1 + \dot{V}_2 + \dots + \dot{V}_n = \dot{V} \quad (3-7)$$

viii) From geometry:

$$\dot{V} = R_{cr} \times \sin\theta_{cr} + L_{cnr} \times \frac{(R_{cr}/L_{cnr})^2 \times \sin\theta_{cr} \times \cos\theta_{cr}}{\sqrt{1 - (R_{cr} \times \sin\theta_{cr}/L_{cnr})^2}} \quad (3-8)$$

$$\times \text{Bore}^2 \times \frac{\pi}{4} \times \frac{d\theta_{cr}}{dt}$$

Values of \dot{Q}_i , M_{in_i} and M_{out_i} are given from several subprograms in advance before these differential equations will be solved. Total number of unknowns is N elements x 6 values (\dot{E}_i , \dot{M}_i , \dot{H}_i , \dot{V}_i , \dot{W}_i and \dot{T}_i) + 2 (\dot{P} and \dot{V}). Since the total number of differential equations is N elements x 6 (Eq.(1) ~ Eq.(6)) + (Eq. (17) and Eq. (8)), these differential equations can be solved.

In the computer program these differential equations are solved according to the following procedure:

We get \dot{V}_i from Eq. (1) ~ Eq. (6)

$$\begin{aligned} \dot{V}_i = & \frac{V_i}{M_i C_{pi} T_i} (\dot{Q}_i + M_{in_i} H_{in_i} - M_{out_i} H_{out_i} - (M_{in_i} - M_{out_i}) H_i) \\ & + \frac{M_{in_i} - M_{out_i}}{M_i} V_i + \left(\frac{V_i^2}{M_i C_{pi} T_i} - \frac{V_i}{P} \right) \dot{P} \end{aligned} \quad (3-9)$$

Substituting \dot{V}_i in Eq. (7) by Eq. (9), we get

$$\dot{P} = (\dot{V} - A)/B \quad (3-10)$$

where

$$\begin{aligned} A = & \sum_{i=1}^n \frac{V_i}{M_i C_{pi} T_i} (\dot{Q}_i + M_{in_i} H_{in_i} - M_{out_i} H_{out_i} - (M_{in_i} - M_{out_i}) H_i) \\ & + \frac{M_{in_i} - M_{out_i}}{M_i} V_i \end{aligned} \quad (3-11)$$

$$B = \sum_{i=1}^n \left(\frac{v_i^2}{M_i C_{pi} T_i} - \frac{v_i}{P} \right) \quad (3-12)$$

4. Jet Model

From the study of diesel combustion in a rapid compression machine by Rife and Heywood(14), it can be concluded that the fuel jet breaks into droplets near the nozzle orifice and the relative velocity of droplets in the jet flow is small. This means the mixing between packaged jet elements themselves is not important and only mixing between jet elements and surrounding air is taken into consideration. From the characteristics lengths analysis by Jain(19) he concluded that most of the droplets can evaporate before reaching the spark plug. Thus the model that fuel ignition is initiated by the spark plug and heat release is controlled only by mixing is now justified.

The mixing model is based on the turbulent entrainment assumptions of Hoult and Weil(23). The rate of entrainment for turbulent plumes introduced by Hoult and Weil have been modified to include the effects of large density variations, as suggested by Ricou and Spalding(24) and Escudier(25).

For the application of these turbulent entrainment theories to reciprocal engines, three more new aspects were added to them.

1) The NO_x value is not determined only by the present gas characteristics, but also it is very much dependent on the previous histories of gas characteristics. For this purpose jet model is solved by using the packaging method in order to identify every jet element at any instant.

2) All the previous jet models were set in a two-dimensional cartesian coordinate system. However, a cylindrical coordinate system is much more appropriate for reciprocal engines. The most important difference between these two systems is that the pressure field is not constant any more in the cylindrical coordinate system, especially in high swirl ratio engines. Jet elements, thus, are affected by the surface forces from the non-uniform pressure field and these surface forces have the same magnitude but opposite direction of the centrifugal forces when the jet elements have the same density and angular velocity as the surrounding air.

3) Restrictions of the combustion wall were taken into account. The cross section geometry of the jet was assumed to be in the shape of a D (Fig. 4) when it is contacting with the wall with the same centerline position and cross section area of the cylindrical jet. This consideration proved to have a good advantage for determining jet trajectory and heat transfer rate between jet elements and the combustion wall.

Let us consider a jet element "1" into a quiescent atmosphere of

element " ∞ " in the absence of any chemical reaction (Fig. 3). Then, the general equations of the jet model reduce to the following.

i) Conservation of mass

when jet is not contacting with the wall

$$\frac{1}{\frac{ds}{dt}} \cdot \frac{d}{dt}(\rho\pi b^2 u) = (\rho/\rho_\infty)^{1/2} \rho_\infty 2\pi b(\alpha|u-v_i| + \beta|v_n|) \quad (4-1)$$

when jet is contacting with the wall

$$\frac{1}{\frac{ds}{dt}} \cdot \frac{d}{dt}(\rho\pi b^2 u) = (\rho/\rho_\infty)^{1/2} \rho_\infty 2\pi b_c \left(1 - \frac{\theta_j}{2\pi}\right) (\alpha|u-v_i| + \beta|v_n|) \quad (4-1)$$

θ_j : jet contacting angle with the wall (see Fig. 4)

$$\text{Max. of } \theta_j = \pi$$

ii) Conservation of angular momentum

$$\frac{d}{dt}(\rho\pi b^2 u R w) = R w_s \frac{d}{dt}(\rho\pi b^2 u) - \rho\pi b^2 u v_r w \quad (4-2)$$

w_s : angular velocity of swirl

$\rho\pi b^2 u v_r w$: coriolis force

iii) Conservation of radial direction momentum

$$\frac{d}{dt}(\rho\pi b^2 u v_r) = \rho\pi b^2 u R w^2 - \rho_\infty \pi b^2 u R w_s^2 + v_{R\infty} \frac{d}{dt}(\rho\pi b^2 u) \quad (4-3)$$

$\rho\pi b^2 u R w^2$: centrifugal force

$\rho_\infty \pi b^2 u R w_s^2$: force arising from pressure gradient swirling flow

iv) Conservation of vertical direction momentum

$$\frac{d}{dt}(\rho\pi b^2 u v_z) = v_{z\infty} \frac{d}{dt}(\rho\pi b^2 u) \quad (4-4)$$

v) From geometry

$$u \frac{du}{dt} = R^2 w \frac{dw}{dt} + R w^2 \frac{dR}{dt} + v_r \frac{dv_r}{dt} + v_z \frac{dv_z}{dt} \quad (4-5)$$

$$\frac{dR}{dt} = v_r \quad (4-6)$$

vi) Conservation of fuel

$$\frac{d}{dt}(\rho\pi b^2 u X_f) = 0 \quad (4-7)$$

vii) Jet is in the combustion chamber

$$\text{Jet location } (R, \theta, Z) \leq \text{ combustion chamber} \quad (4-8)$$

viii) For contacting angle θ_j

$$\pi b^2 = \pi b_c^2 \times \left(1 - \frac{\theta_j}{2\pi}\right) + \frac{b_c^2}{2} \sin\theta_j \quad (4-9)$$

5. Air Motion Surrounding the Jet

Calculation of gas velocity surrounding the jet is required to define the jet trajectory and mixing ratio between the jet and gas. A first order calculation was made by M. Martin(21). This model is limited to the regions close to the cylinder wall or piston by the assumptions and the model can be extended to the whole region inside the cylinder for our purpose. Since an exact solution including viscous effects is difficult and time-consuming, several assumptions are necessary for simplification of this model.

i) The combustion chamber is divided into three regions (Fig. 6);

Inside each region, axial velocity is uniform on the same horizontal plane, radial velocity and angular velocity is uniform on the same radius.

ii) In region (3) there is no radial velocity.

This assumption is necessary in order to avoid a large velocity discontinuity at the piston cup edge due to a large area discontinuity there.

iii) Boundary conditions.

Axial velocity = 0 at cylinder head,
 radial velocity = 0 at cylinder wall,
 and axial velocity is the same as piston at piston surface.

iv) Swirl at bottom center is solid body rotation.

v) Angular momentum is not conserved.

After precisely checking the pictures (26) of the rapid compression machine it was found that the swirl amplitude (the ratio of swirl at TDC to swirl at BDC) decreases when swirl magnitude goes up. Under normal Texaco engine operation (swirl ratio is 3.65 x rpm at BDC), this swirl decaying is about 10%. The following equation is used for estimating swirl decaying:

$$\frac{\text{swirl momentum decayed}}{\text{initial swirl momentum}} = C_{DE} \times w^2 \times \text{Time}$$

C_{DE} : swirl decaying factor 6.5×10^{-6} for Texaco engine

w: angular velocity (rad/sec)

Time: time since intake valve closes (sec)

(1) Calculation of Radial Velocities (Fig. 6)

(a) $\underline{R_c < r < R_p}$

$$\frac{dM_2}{dt} = - \frac{dM_1}{dt} = 2\pi r h \rho \bar{v}_r \quad (5-1)$$

$$M_1 = \pi(R_p^2 - r^2) h \rho \quad (5-2)$$

$$M_2 = (\pi r^2 h + V_c) \rho \quad (5-3)$$

M_i : the mass in control volume i

t : time

ρ : density of the charge

V_c : piston cup volume

Combining (5-1), (5-2) and (5-3)

$$(\pi r^2 h + V_c) \rho' + \pi r^2 \rho h' = - \pi(R_p^2 - r^2) (h' \rho + \rho' h)$$

$$\rho' = - \left(\frac{h'}{K_1 + h} \right) \rho \quad (5-4)$$

when $K_1 = \frac{V_c}{\pi R_p^2}$ (5-5)

Let $X = \frac{h}{K_1}$ (5-6)

Then solving for ρ in terms of initial conditions at time t_0 yields

$$\rho(t) = \rho(t_0) \left(\frac{X(t_0) + 1}{X(t) + 1} \right) \quad (5-7)$$

From Eq. (5-1) and (5-2) we have

$$\bar{v}_r = \frac{-\frac{dM_1}{dt}}{2\pi r h \rho} = \frac{-\pi(R_p^2 - r^2)(h'\rho + \rho'h)}{2\pi r h \rho}$$

Using (5-4) and letting

$$v_p = \frac{dh}{dt} = \text{piston velocity (upward velocity has negative sign)}$$

We find

$$\bar{v}_r(r, t) = - \left(\frac{r}{2K_1} \right) \left[\left(\frac{R_p^2}{r^2} - 1 \right) \frac{v_p(t)}{X(t)(1+X(t))} \right] \quad (5-8)$$

(b) $0 < r < R_c$

The radial velocity inside the piston cup is assumed to be zero.

$$\frac{dM_2}{dt} = - \frac{dM_1}{dt} = 2\pi r h \rho \bar{v}_r \quad (5-9)$$

$$k_2 = V_c / (\pi R_c^2) \quad (5-10)$$

$$M_1 = \pi(R_p^2 - r^2) h\rho + \pi(R_c^2 - r^2) K_2\rho \quad (5-11)$$

$$M_2 = \pi(h + k_2)r^2\rho \quad (5-12)$$

Combining (5-9), (5-11) and (5-12)

$$\begin{aligned} \pi(h + k_2)r^2\rho' + \pi r^2\rho h' &= -\pi(R_p^2 - r^2)(h'\rho + h\rho') \\ &\quad - \pi(R_c^2 - r^2)k_2\rho' \end{aligned}$$

$$\rho' = -\left(\frac{h'}{k_1 + h}\right)\rho \quad (5-13)$$

From Eq. (5-9) and (5-12) we have

$$\bar{v}_r(r,t) = \left(\frac{r}{2k_1}\right) \frac{\left(\frac{k_2}{k_1} - 1\right) v_p(t)}{X(t)(1 + X(t))} \quad (5-14)$$

(2) Calculation of Axial Velocities

(a) 0 < Z < h

$$-\frac{dM_1}{dt} = \frac{dM_2}{dt} = \pi R_p^2 \bar{v}_z \rho \quad (5-15)$$

$$M_1 = \pi R_p^2 \rho Z \quad (5-16)$$

$$M_2 = \pi R_p^2 \rho (h-Z) + V_c \rho \quad (5-17)$$

We assume

$$\bar{v}_{z1}(Z,t) = v_p Z/h \quad \text{for } R_c < r < R_p \quad (5-18)$$

From (5-15), (5-16), (5-17) and (5-18) we have

$$\bar{v}_{z2}(Z,t) = v_p Z/h \left\{ 1 - \left(\frac{R_p}{R_c} \right)^2 \frac{1}{(X(t) + 1)} \right\} \quad \text{for } 0 < r < R_c \quad (5-19)$$

(b) $h < Z$

$$- \frac{dM_1}{dt} = \frac{dM_2}{dt} = \pi R_c^2 \bar{v}_z \rho \quad (5-20)$$

$$M_1 = \pi R_p^2 h + \pi R_c^2 (Z-h) \quad (5-21)$$

$$M_2 = \pi R_c^2 (k_2 - (Z-h)) \quad (5-22)$$

From (5-20), (5-21) and (5-22) we have

$$\bar{v}_z(Z,t) = \frac{k_1 - k_2 + Z}{k_1 + h} v_p \quad (5-23)$$

(3) Calculation of Angular Velocity

The gas angular velocity inside the cylinder is assumed to be a solid body rotation.

From momentum equation

$$\begin{aligned}
 & \underbrace{\int_0^{R_p} \int_0^h \int_0^{2\pi} \rho w r^3 d\theta dz dr}_{\text{momentum in region (1)}} + \underbrace{\int_0^{R_c} \int_0^{k_2} \int_0^{2\pi} \rho w r^3 d\theta dz dr}_{\text{momentum in region (2)}} \\
 = & (1 - \text{DECAY}) \times \left[\int_0^{R_p} \int_0^h \int_0^{2\pi} \rho_o w_o r^3 d\theta dz dr + \int_0^{R_c} \int_0^{k_2} \int_0^{2\pi} \rho_o w_o r^3 d\theta dz dr \right] \quad (5-24) \\
 & \underbrace{\hspace{15em}}_{\text{initial momentum}}
 \end{aligned}$$

DECAY: swirl decaying ratio

From Eq. (5-24)

$$\begin{aligned}
 w(t) &= (1 - \text{DECAY}) \times \frac{\rho_o}{\rho} \times \frac{k_2 R_c^4 + h_o R_p^4}{k_2 R_c^4 + h R_p^4} w_o \\
 &= (1 - \text{DECAY}) \times \frac{(h + k_1)}{(h_o + k_1)} \times \frac{k_2 R_c^4 + h_o R_p^4}{k_2 R_c^4 + h R_p^4} w_o \quad (5-25)
 \end{aligned}$$

Figure 7 and Fig. 8 are the calculation results using these assumptions. At -50 BTDC gas movement is dominated by the piston velocity and swirl. The radial velocity is almost negligible. At -30 BTDC, the radial velocity begins to grow as the gas in the squish area is forced into the piston cup. At -10 BTDC, the piston is almost stopped and the radial velocity, especially at the edge of the piston cup, is very high. Squish effect can be computed from this study. This more detailed study in general confirms the basic conclusions of Martin but provides more detailed specification of the flows for jet mixing analysis.

6. Combustion Model

From the observation of pictures in the rapid compression machine it was found that the unburned gas mixture hitting the spark plug has a small plume first and this plume grows, entraining surrounding unburned gas mixture. The main flame can be seen a little bit downstream from the spark plug and this ignition delay time is about 0.7 - 0.8 msec at 2000 rpm. These phenomena are also verified by log P-log V diagrams shown in Marsh thesis(27). A mixing and chemical induction time seems to be associated with this delay and the order of magnitude is similar to the delay in a spark ignition engine.

To explain the combustion delay mentioned above, an analytical model for combustion reaction time was developed. In this model a fuel-air

element starts to burn when it hits the spark plug but the fuel in the jet element doesn't burn instantaneously because of finite flame speed and a typical induction period. In the early stages of combustion growth, this burning nucleus is so small that this stage can be treated as if there were no combustion. The schematic of this model applied to the TCCS engine is shown in Fig. 9.

1) Entrainment of unburned mixture

The entrainment of the surrounding gas mixture into the burning nucleus has been treated with an analysis based on the work of Blizard and Keck(28) which relates the turbulent eddy entrainment velocity to the engine speed.

The mass rate of entrainment of surrounding gas mixture can be given as

$$\frac{dM_e}{dt} = \sqrt{\rho_u \rho_b} * S_e * U_e \quad (6-1)$$

where S_e is plume surface area and U_e the turbulent eddy entrainment velocity. In the Blizard and Keck model, the entrainment velocity is based on the velocity of the inlet jet. However, in the TCCS geometry, entrainment into the spark nucleus is dominated by the velocity of the fuel jet

$$U_e \approx 0.22u \quad (6-2)$$

ii) Combustion delay

Tabaczynski(29) computed a characteristic reaction time (τ) for a large eddy using a characteristic reaction time (τ_c) defined for an eddy of the order of the turbulent microscale. The rate of mass burned in an individual eddy is assumed to be proportional to the mass of unburned gas that exists in the plume

$$\frac{dM_b}{dt} = \frac{M_u}{\tau_c} = \frac{M_e - M_b}{\tau_c}$$

M_e : entrained gas in the plume

M_b : burned gas in the plume

M_u : unburned gas in the plume

$\tau_c = \lambda/S_e$ characteristic reaction time for the microscale

λ : Taylor microscale

$$\lambda = 0.17 \times \frac{\text{intake valve lift}}{\text{compression ratio}} \quad (\text{Ref. (28) and (30)})$$

S_e : laminar flame speed (Ref. (28))

7. NO_x Model

The NO_x formation model is based on the same model as combustion mentioned before which is composed of unburned gas mixture outside flame front, burning zone and burned gas. But the temperatures of these zones are defined by only two typical temperatures: burned and unburned gas temperatures, by decoupling the burning zone into these two temperature zones. Figure 12 shows the mass histories of these zones during the period from the fuel injection to fully developed plume. As can be seen, the burning zone is very thin and disappears quickly, and this zone is proved not to have much effect on NO_x amount by computer simulation. This model also calculates the transferred amount of NO_x between elements by mixing. The NO formation model described by Lavoie, Heywood and Keck (32), and Komiyama and Heywood (8) is used, i.e., NO formation is governed by the extended Zel'dovich mechanism:



The hydrocarbon oxidation reactions are fast relative to the NO formation process and a reasonable approximation is that the species O, O₂, H, OH and N₂ are in equilibrium. A steady-state assumption is made for [N]. The rate of change of NO mass fraction {NO} due to chemical reaction;

$$\frac{d\{NO\}}{dt} = \frac{2M_{no}}{\rho} \frac{(1 - (\{NO\}/\{NO\}_e)^2)R_1}{(1 + K\{NO\}/\{NO\}_e)} \quad (7-4)$$

{NO} = NO mass fraction

M_{no} = molecular weight of NO

ρ = gas density

$$R_1 = K_{+1} (O)_e (N_2)_e = K_{-1} (N)_e (NO)_e$$

()_e = mole concentration in equilibrium

$$K = \frac{R_1}{R_2 + R_3} = \frac{K_{-1} (NO)_e}{K_{+2} (O_2)_e + K_{+3} (OH)_e}$$

8. Heat Transfer Model

Heat transfer calculation of the TCCS engine seems to require a more complicated model than that of the spark ignition engine, because the heterogeneous mixture causes a large temperature distribution inside the combustion chamber, and also heat transfer between the jet and piston top has an important effect on the early stage of combustion like the impingement effect. The wall inside the TCCS engine, thus, is divided into five regions as many diesel engines are so, i.e., piston top, cylinder wall, cylinder head, intake valve and exhaust valve (see Fig. 10). These temperatures are estimated as 650°K, 600°K, 650°K, 500°K and 700°K respectively. Each wall of the combustion chamber is in contact with the hot, burned gas plume while the remaining are in contact with relatively cold unburned gas mixture.

All the necessary data for the heat transfer calculation like the heat transfer area of the jet elements and gas velocity at the wall are provided by the jet model calculation.

The conventional engine heat transfer correlations of Woschni(20) have been used to model the heat transfer process. The Nusselt number is expressed as;

$$Nu = 0.035 R_e^{0.8} \quad (8-1)$$

Using Woschni's relationships for gas velocities, density, viscosity and thermal conductivity, the convective heat transfer coefficient is expressed as;

$$\mu = 7014 B_{ore}^{-0.2} P^{0.8} G^{-0.53} [C_1 + 100C_2 \frac{VT_o}{P_o V_o} (P - P_{is}) + C_3 v_s]^{0.8} \quad (8-2)$$

$$Q = C_{\mu} A (T - T_w) \quad (8-3)$$

B_{ore} : bore

P : pressure

T : temperature

A : heat transfer area

N : engine speed

V : total volume

v_s : swirl velocity

P_o, T_o and V_o : pressure, temperature and volume when intake valve closes

P_{is} : when compression and expansion are isentropic

Woschni used an average piston speed $(\frac{SN}{30})$ as a gas velocity at the wall in his equations but the swirl velocity has a much higher magnitude than the piston speed for the Texaco engine and the swirl velocity term (v_s) was added to them. Heat transfer is not assumed to occur between jet elements and air.

III Comparison of Simulation Predictions with Experimental Data and Conclusions

Experimental works were performed on a TCCS engine by Marsh (27) and Fly (33). A 3-7/8" x 3-7/8" single cylinder engine designed by Texaco has been set up at the Sloan Automotive Laboratory at MIT to carry out experimental investigations on the TCCS, stratified charge concept. A brief description of the engine set up and cross-cut distillate fuel characteristics are given in Table 1-3. Additional details regarding the engine set up, data taking and data reduction can be found in their theses (27) (33).

The following comparisons of calculation results with experimental data were conducted while changing injection timing and equivalence ratio;

- i) Mass fraction burned (Figs. 15,16)
- ii) Cylinder pressure (Figs. 17,18)
- iii) NO versus injection timing (Fig. 19)
- iv) Volumetric efficiency, indicated mean effective pressure and peak pressure versus injection timing (Fig. 20)
- v) Exhaust temperature versus injection timing (Fig. 21)
- vi) NO concentration histories in each element (Fig. 23) (prediction only).
- vii) Swirl effects on indicated mean effective pressure and NO (Fig. 24) (prediction only).

As shown in these figures, every parameter except exhaust temperature was predicted accurately by this simulation model. The

major conclusions which have been drawn from this study are:

1. Ignition delays were calculated successfully by the travelling time from fuel nozzle to spark plug and induction time of unburned mixture to the burning zone.
2. The burning rate was accurately predicted by the jet mixing process based on turbulent entrainment assumptions.
3. Good agreement in cylinder pressure during compression and expansion process means that the heat transfer amount between gas and wall was accurately estimated by the model, which calculates the local gas velocities and jet contacting areas with the wall based on the Woschni correlations.
4. NO_x values were predicted within practical errors. NO_x concentration is very sensitive to the mixing process. The figure of NO_x concentration histories in each element shows that exhaust NO_x can be reduced drastically by rapid mixing after the spark plug.
5. The model predicted higher exhaust temperatures than actual data. Heat transfer calculations from gas to port seems to be necessary for precise exhaust temperature prediction.

6. Higher swirl ratio has a good effect on NO concentrations at high equivalence ratios due to the rapid cooling of burned gas. At low equivalence ratios, however, higher swirl ratio has an unfavorable effect on NO concentration due to the coincidence of higher gas temperatures and peak cylinder pressures.
7. Some more detailed calculation about air motion inside the cylinder is recommended to get more accurate NO prediction over wide range engine operations.
8. It is recommended that this model be applied for calculating another size engine performance to check this model validity.

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TABLE 1
Engine Specifications

Dimensions		
bore		3.875 in.
stroke		3.875 in.
connecting rod		6.625 in.
clearance volume		4.570 in. ³
Valve Timings		
	Opens	Closes
inlet valve	10 BTCD (1)	55 ABDC (2)
	0 (2)	45 (1)
exhaust valve	55 BBDC (1)	10 ATDC (2)
	45 (2)	0 (1)

(1) at 0.006 in. valve lift

(2) valve face flush with head

TABLE 2
Summary of Instrumentation

<u>Temperatures</u>		
Air Orifice Inlet	Water Outlet	Bearing Oil
Air Inlet	Exhaust	Fuel Inlet
Water Inlet	Crankcase Oil	Fuel Returns

Instrument - All Points

Chromel - Alumel Thermocouple

Omega DS - 500 Digital Readout

Resolution 1^oF

<u>Pressures</u>		
	Method	Resolution
Inlet Air	Water Manometer	.1 in.
Crankcase Vacuum	Water Manometer	.1 in.
Oil Pressure	Panel Gage	2 PSI
Exhaust	Mercury Manometer	.1 in.
Dynamometer Load	Mercury Manometer	.1 in.
Injection Line	Kistler 601 Piezoelectric transducer, Kistler 504E	30 PSI*
	Charge Amplifier	
Combustion Chamber	Kistler 609A Piezoelectric Transducer, Kistler 503D Charge Amplifier	.2 PSI**

TABLE 3

Summary of Fuel Properties

	Cross-cut Distillate
Molecular Wt.	125
H:C Ratio	1.825
Specific Gravity	.80
Boiling Point °F	106-648
Lower Heating Value (Btu/lbm)	18038
Stoichiometric F/A Ratio	.0694
FIA - %	
Aromatics	29.5
Olefins	3.0
Saturates	67.5
Octane No. RON	76.6
Cetane No.	28.3

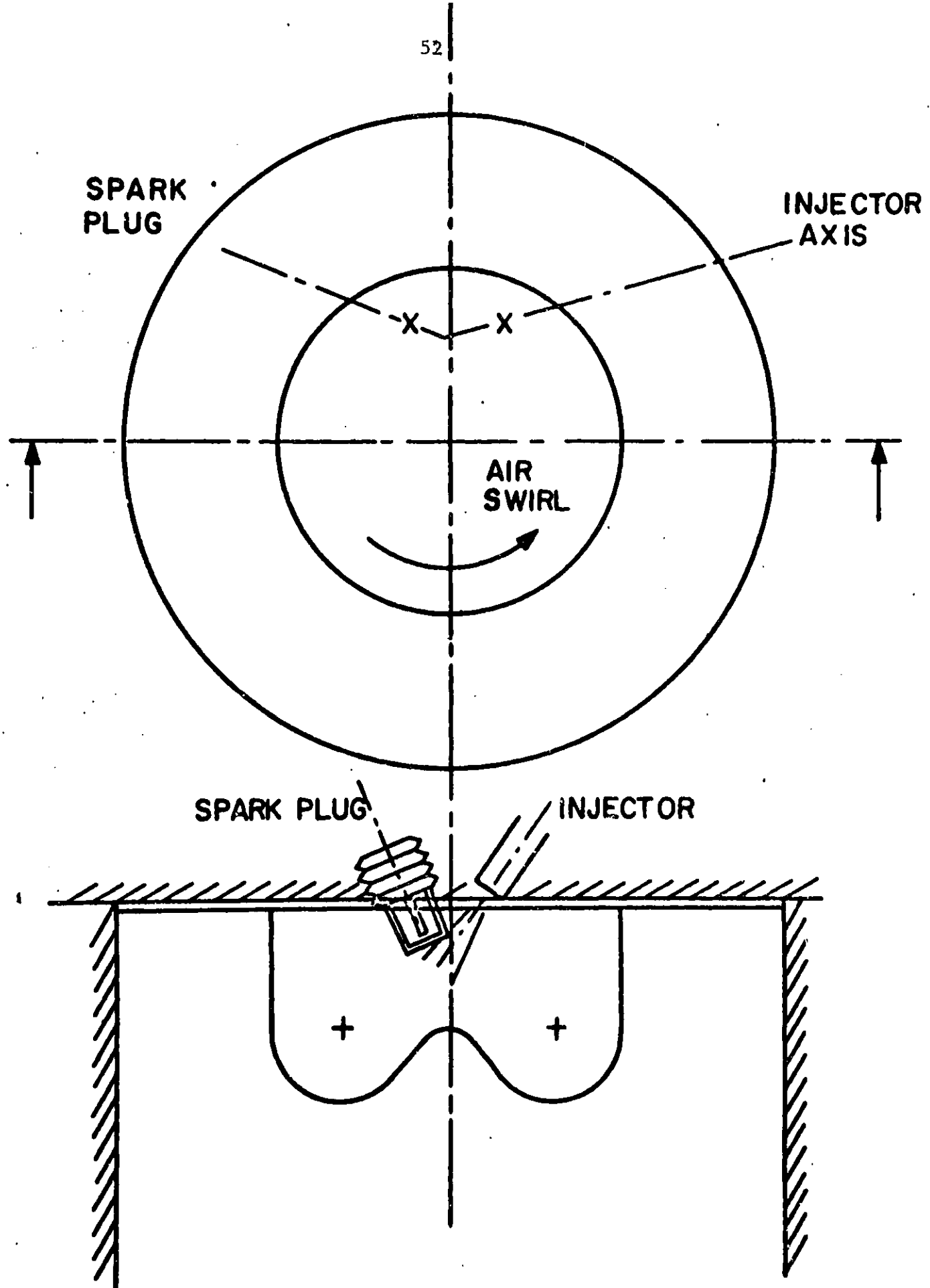


Fig. 1 TEXACO CONTROLLED COMBUSTION SYSTEM (SCHEMATIC)

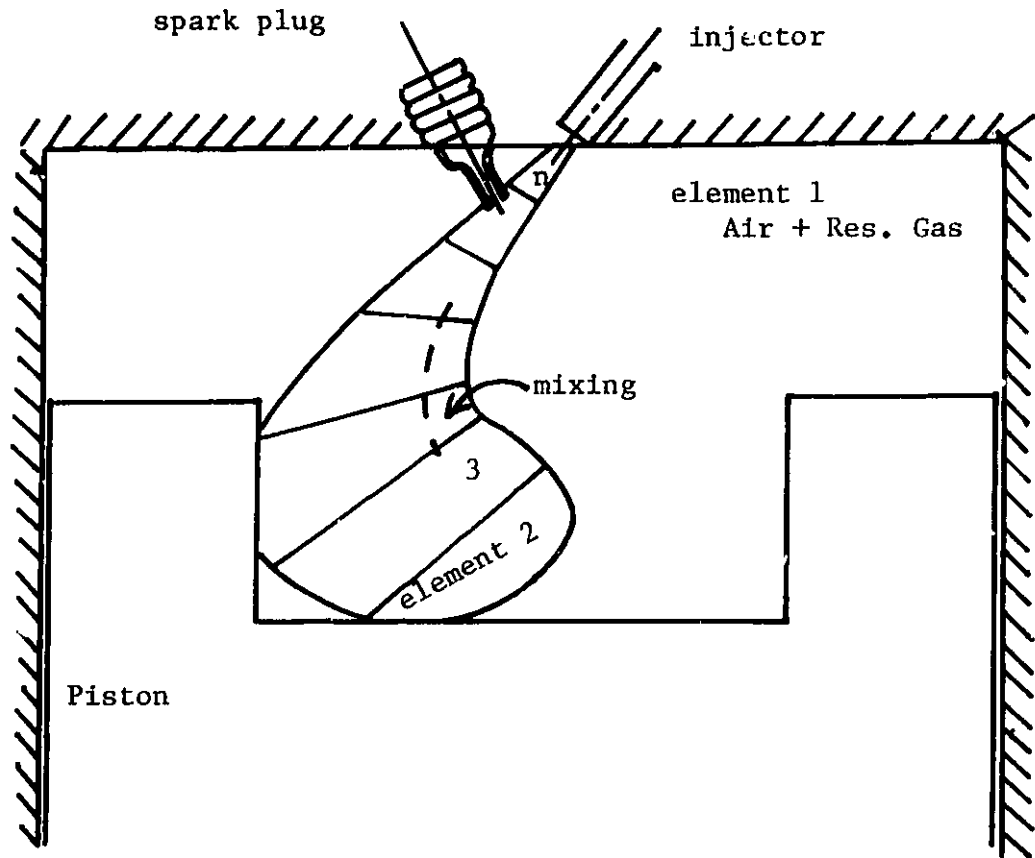
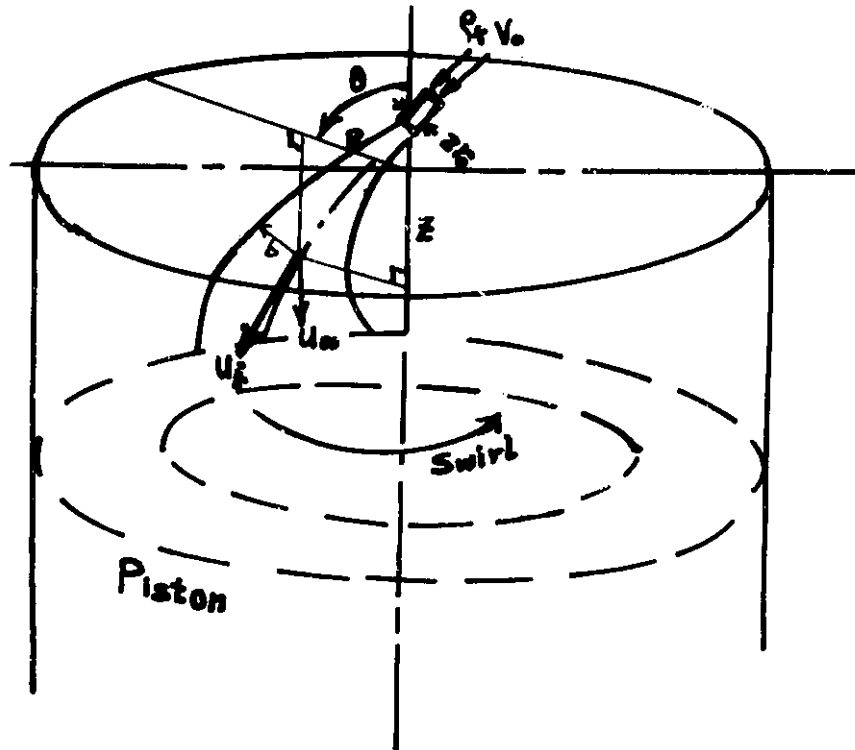


Fig. 2 System Model



Jet Velocity $U_j = U_j (v_R, w, V_z)$

Air Velocity $U_\infty = U_\infty (v_{R\infty}, w_s, V_{z\infty})$

Fig. 3 Jet Model with Swirl

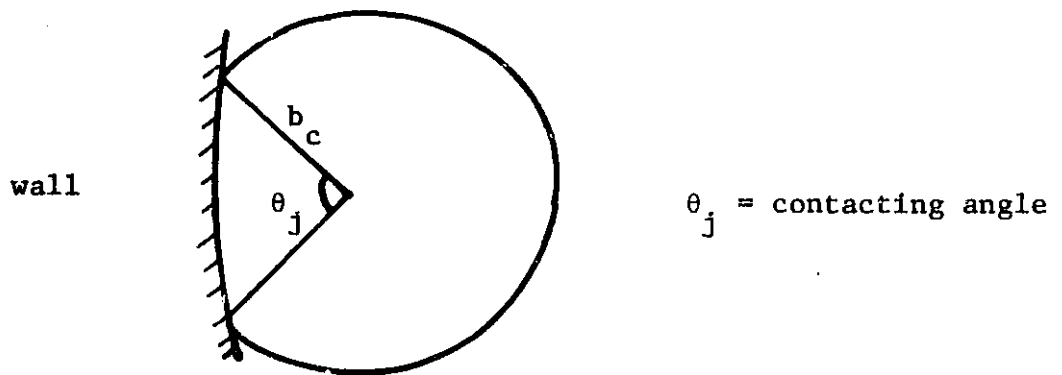
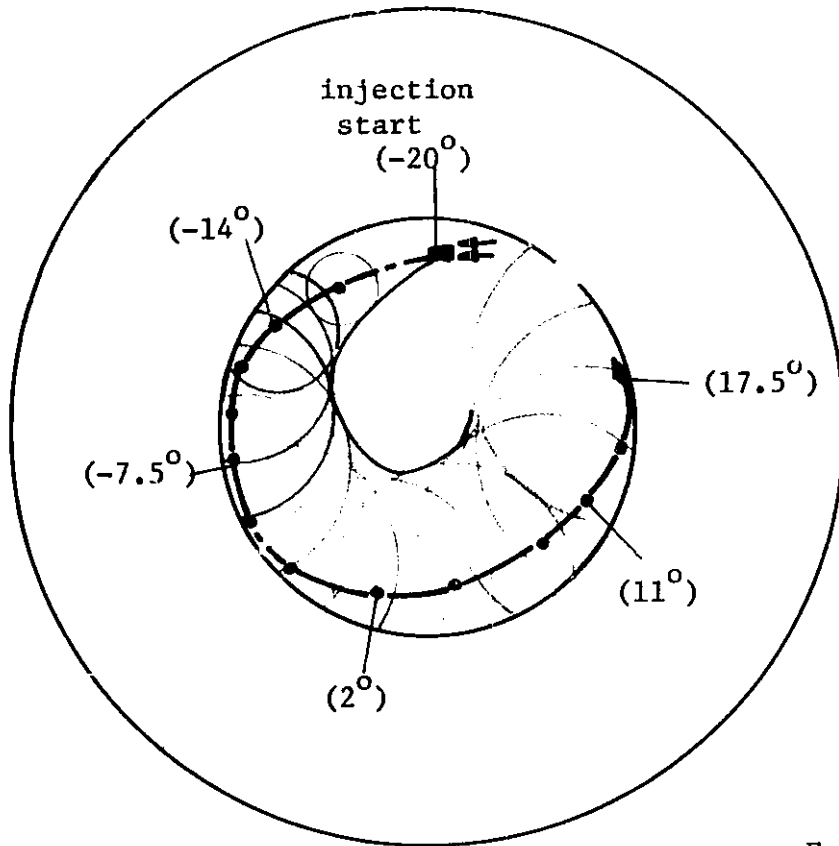
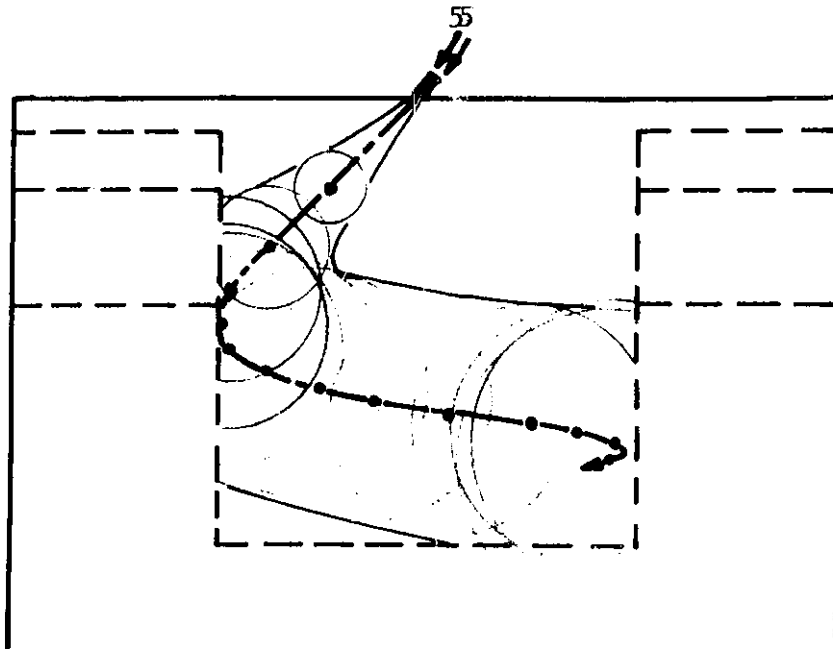


Fig. 4 Jet Cross Cur. Shape When Contacting with Wall



Engine Speed = 2000 rpm

Swirl Ratio = 3.65 x Engine Speed (at

$\phi = 0.545$

Fig. 5 Calculation Results of Jet Model
(First Jet Element Trajectory)

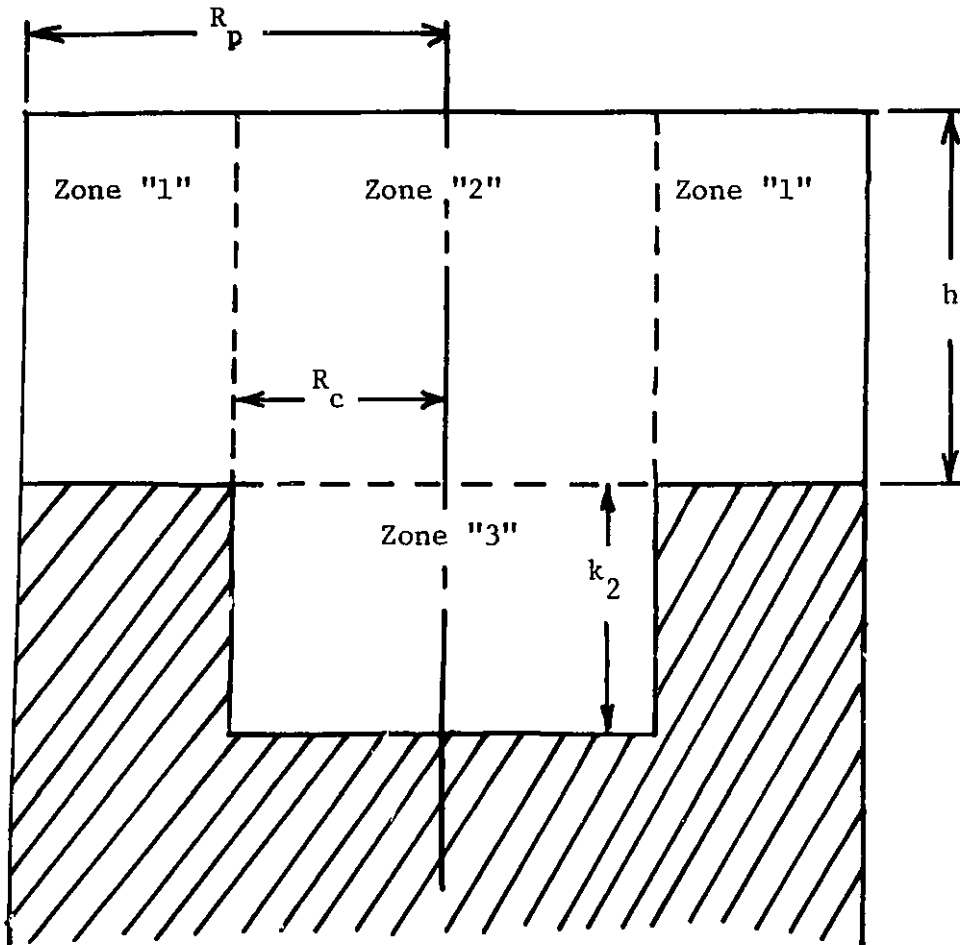
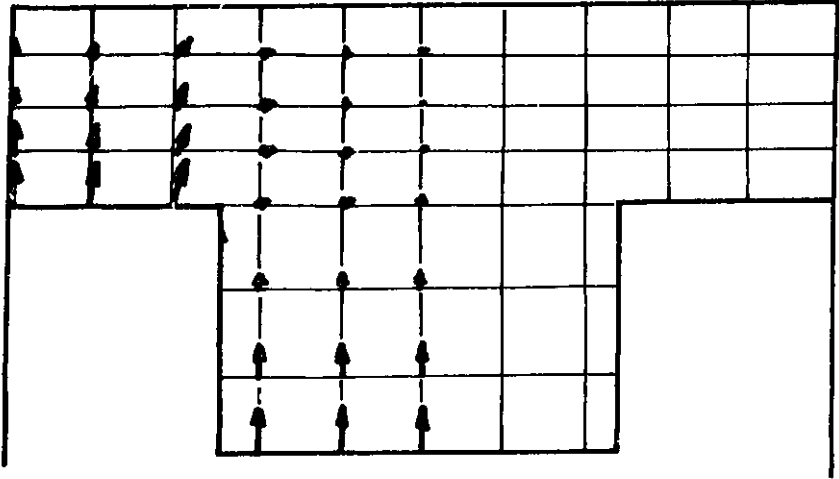


Fig. 6 Model of Air Motion Calculation

RPM = 2500 rpm
 SCALE 20 m/s \rightarrow 1 cm
 (velocity)

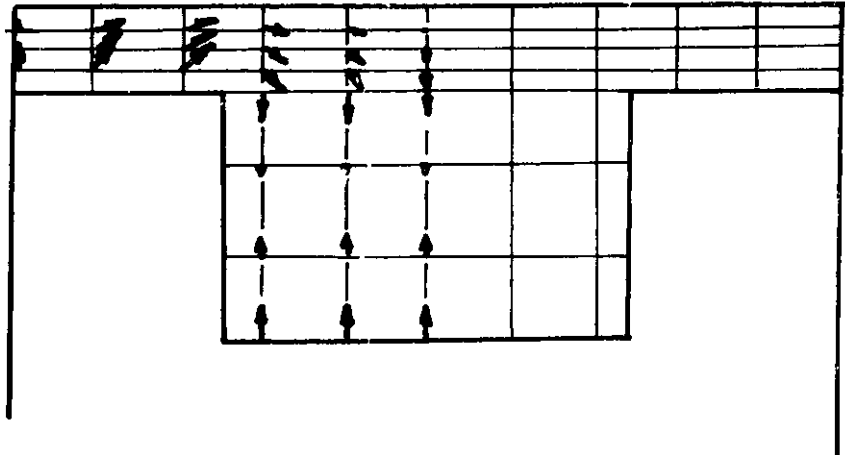
i) at -50° before TDC

piston speed
 = 11.8 m/s



ii) at -30° before TDC

piston speed
 = 8.1 m/s



iii) at -10° before TDC

piston speed
 = 2.9 m/s

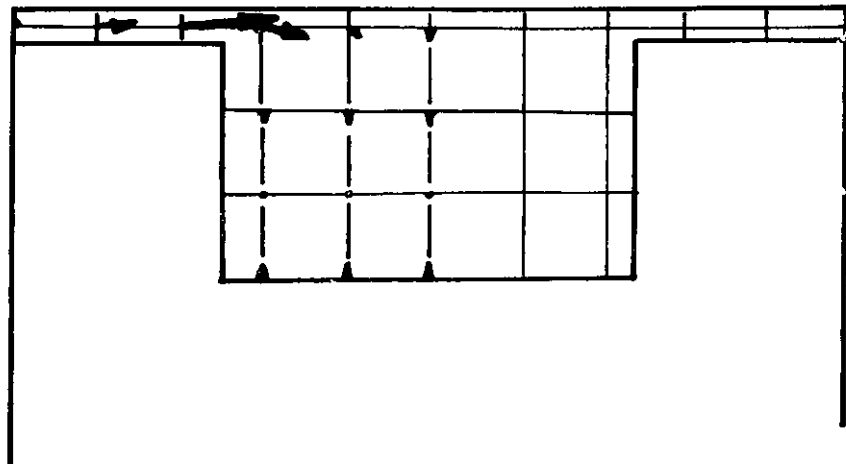


Fig. 7 Air Motion by Calculation

(Radial and Axial Components)

RPM = 2500 rpm

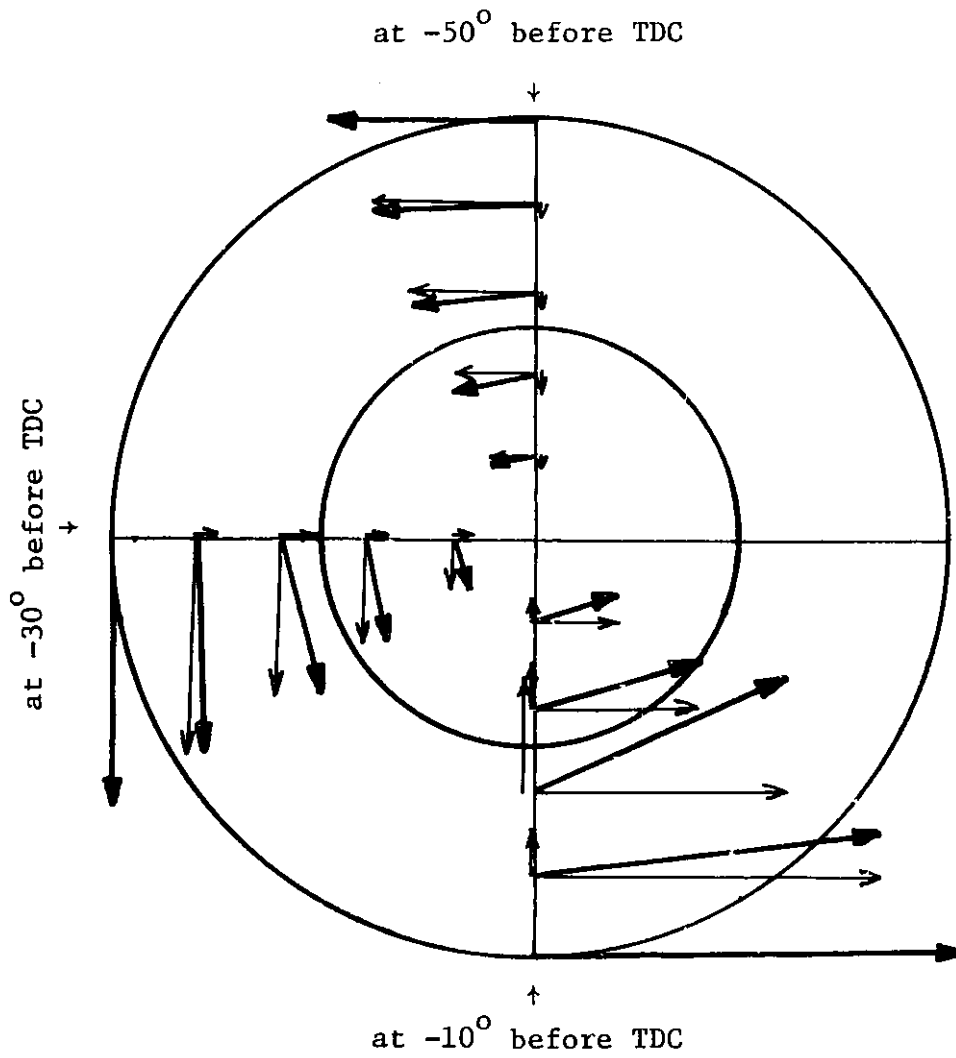
Scale 20m/s \rightarrow 1 cm (velocity)

Fig. 8 Air Motion (Prediction)
(Radial and Tangential Components)

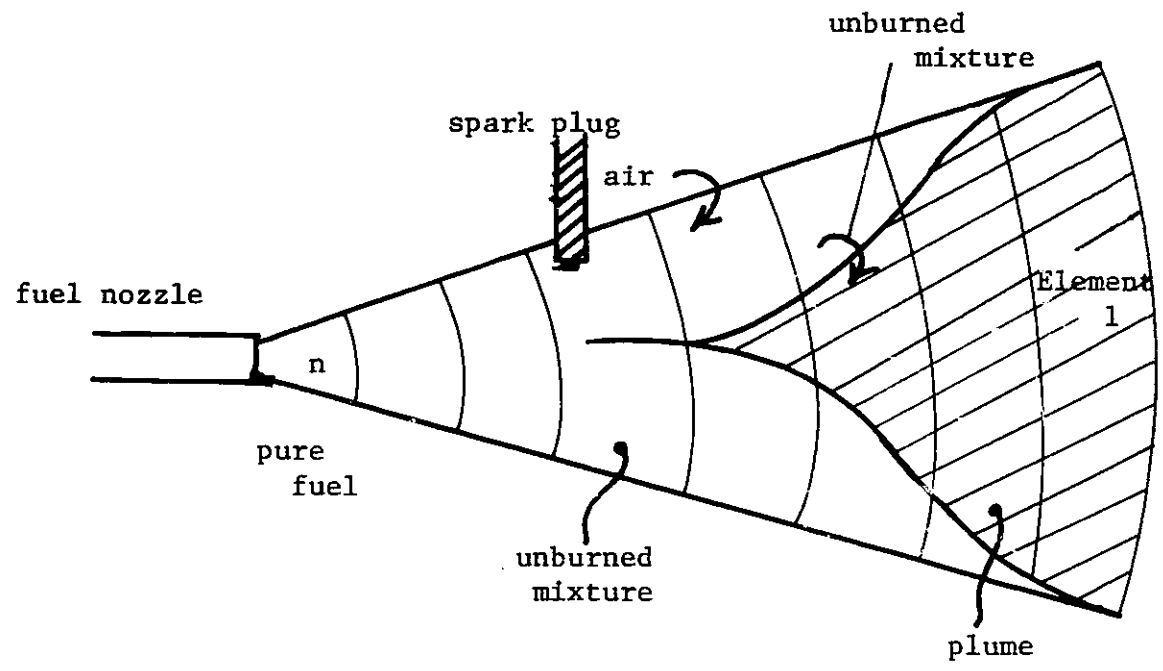


Fig. 9 The Schematic of Ignition Model

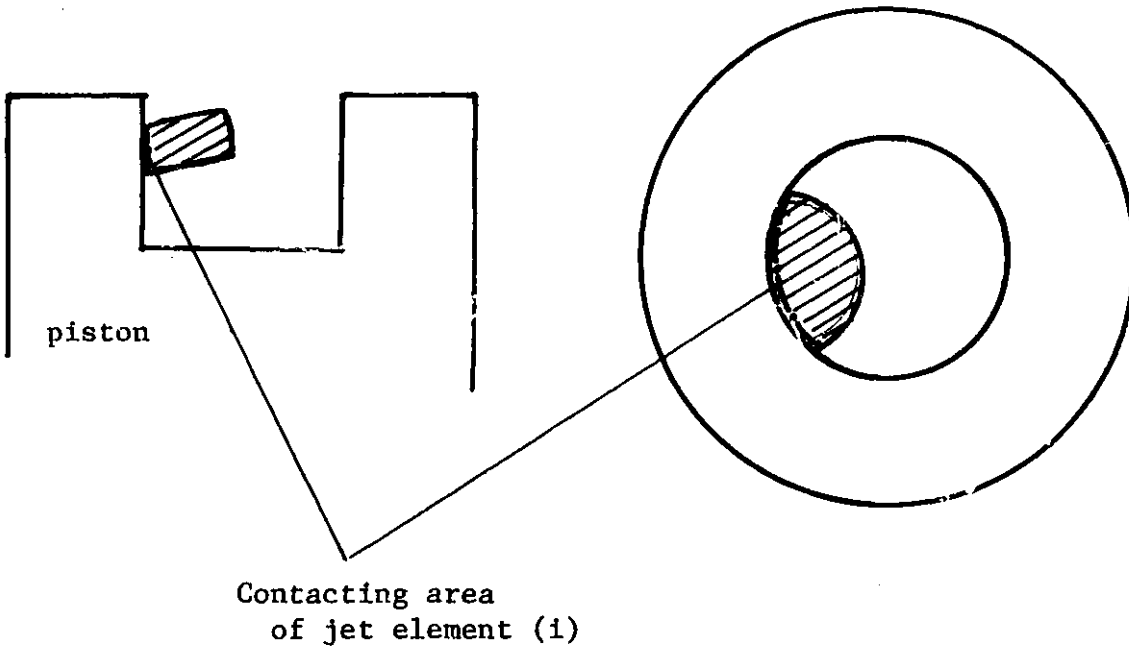
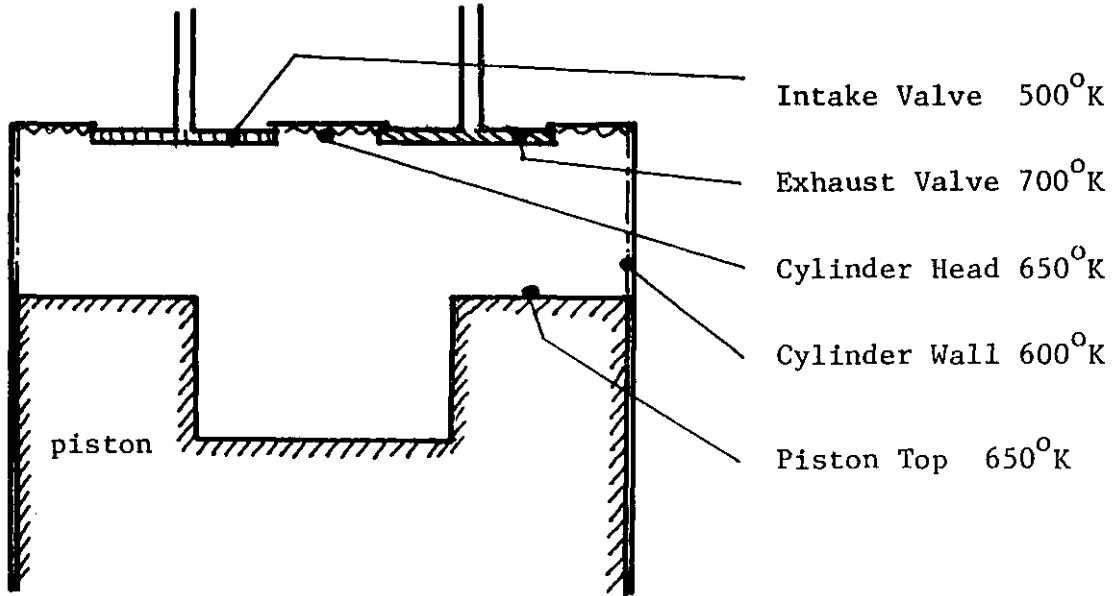


Fig. 10 Heat Transfer Model

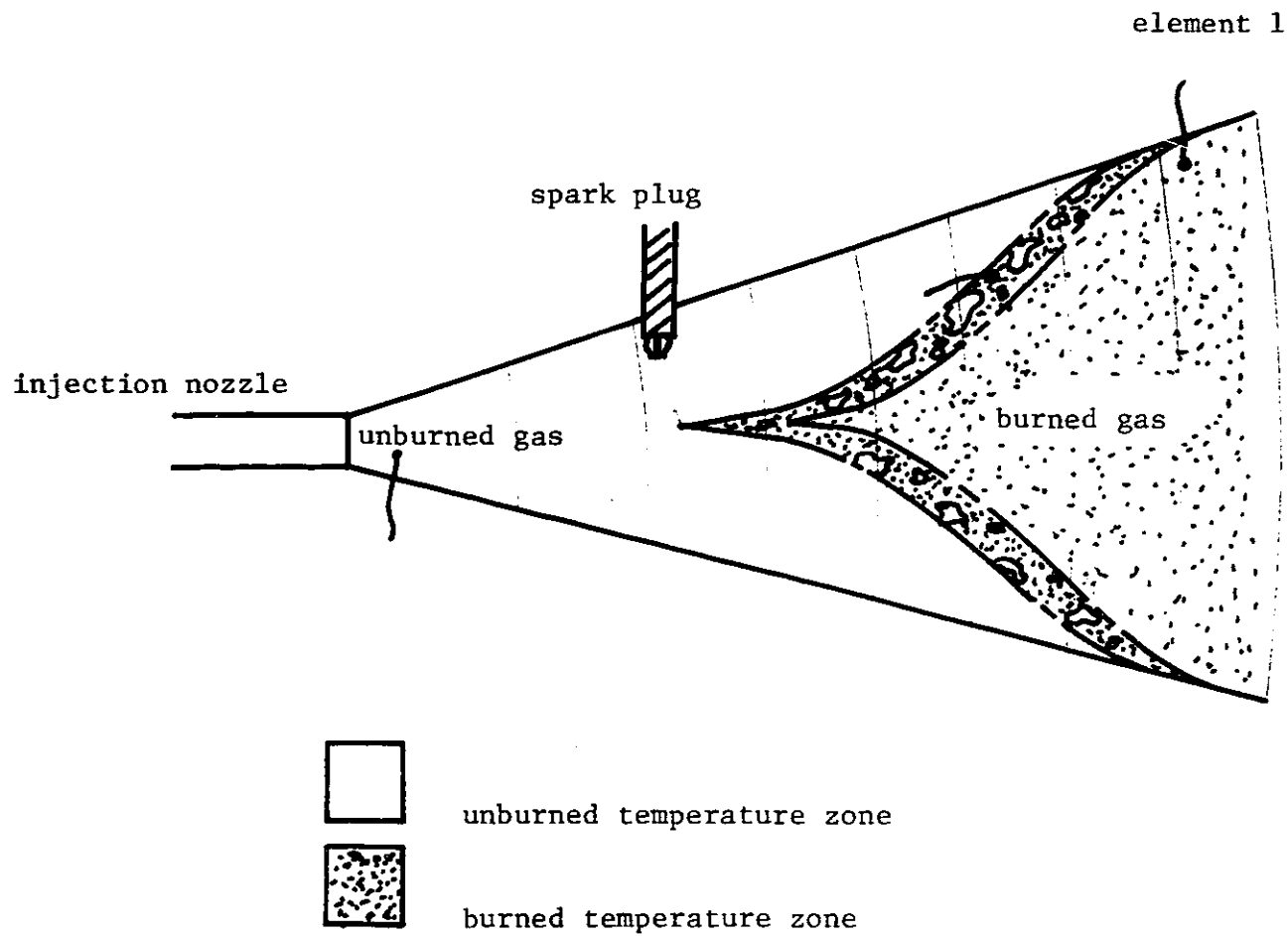


Fig. 11 The Schematic of NO_x Model

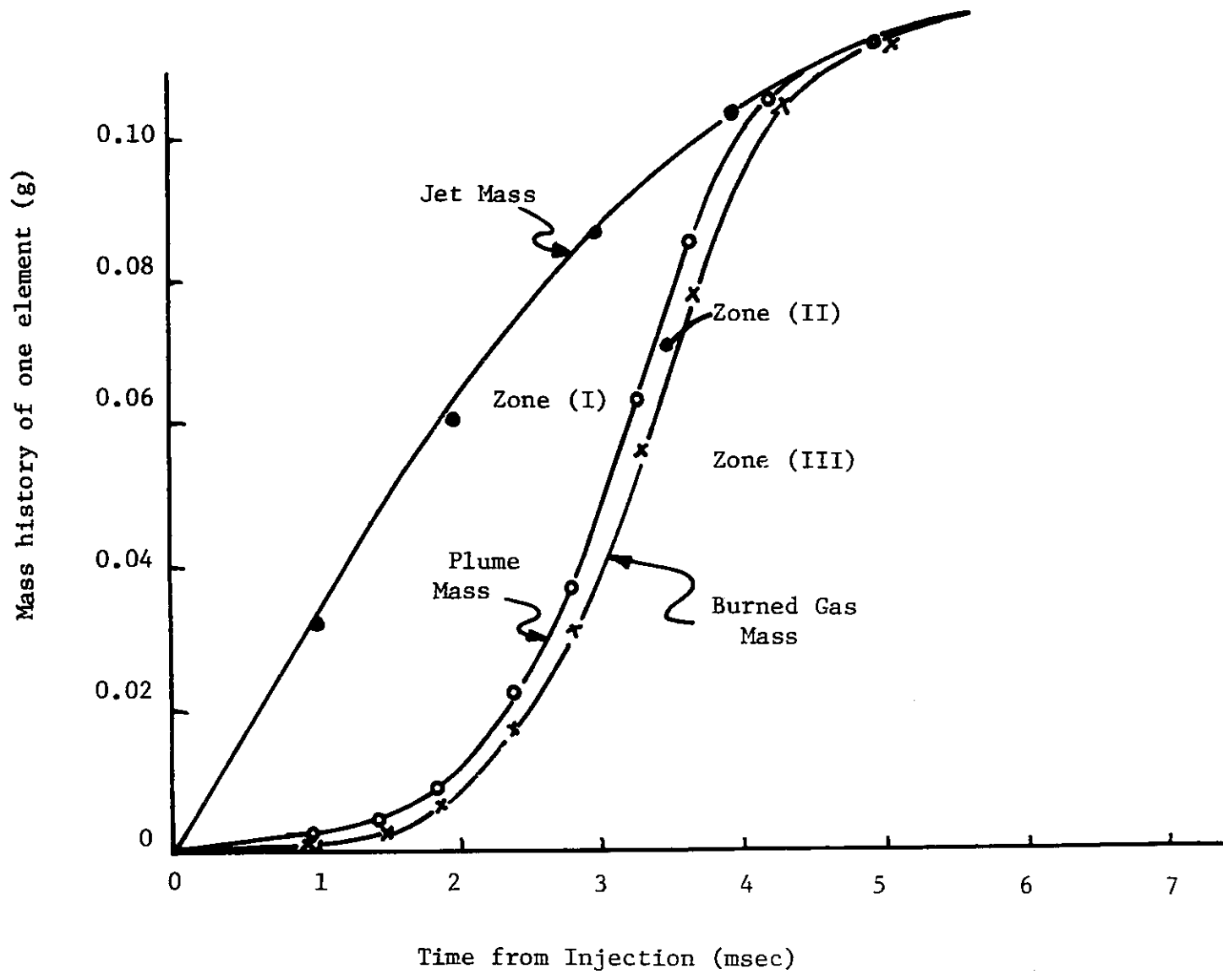


Fig. 12 Mass History of One Element (prediction)

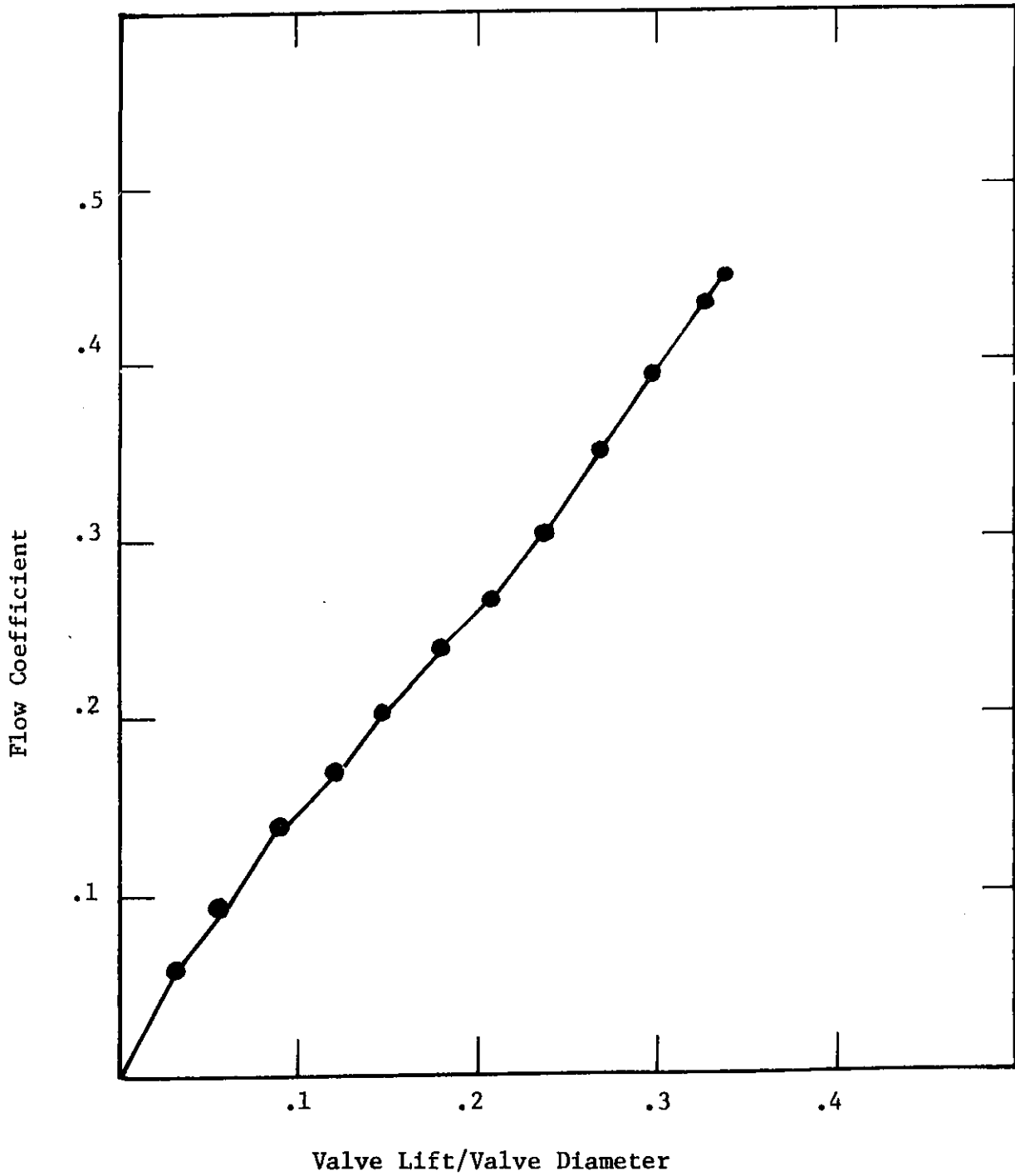


Fig. 13 Valve Flow Coefficient for the Intake Valve

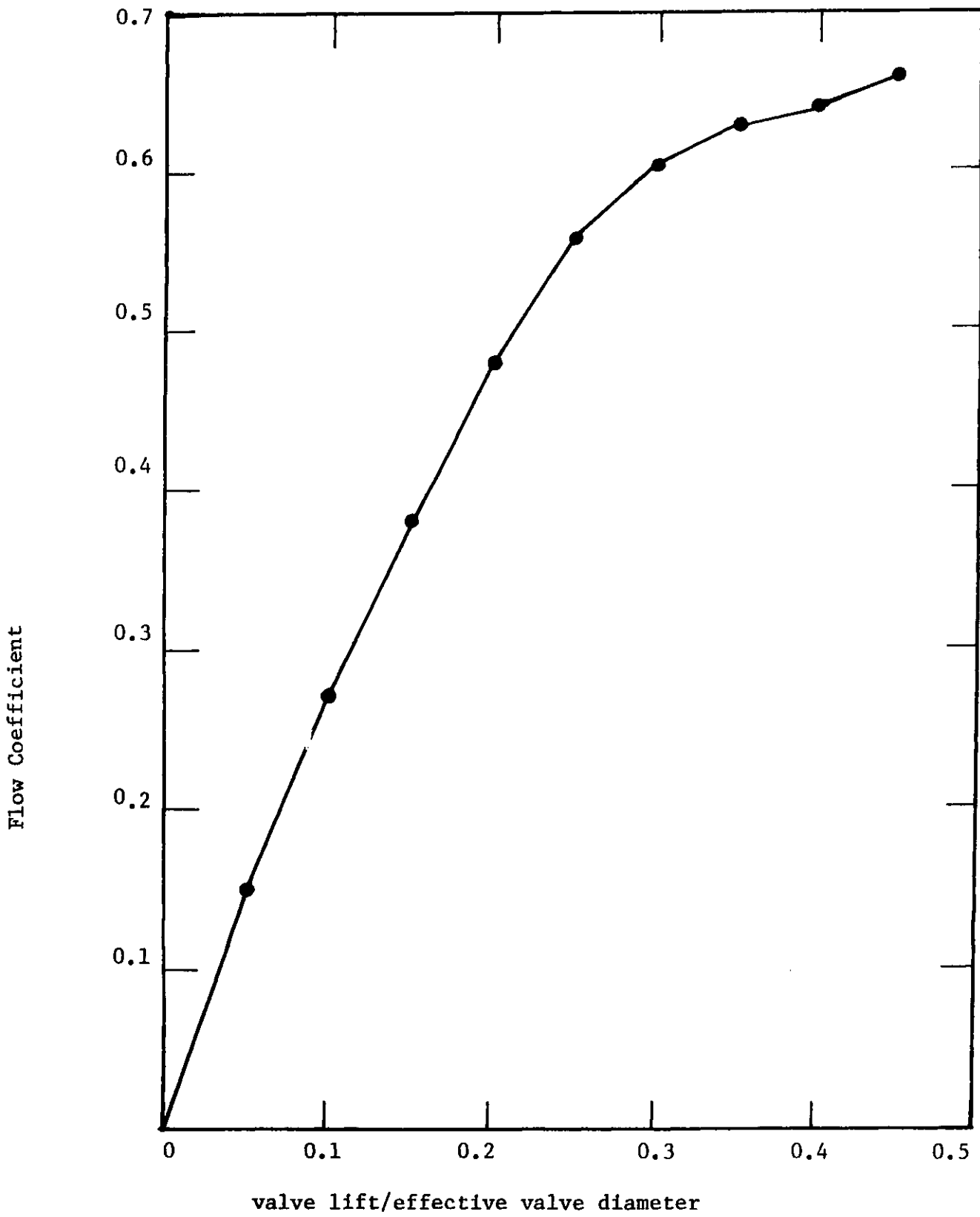


Fig. 14 Valve Flow Coefficient for the Exhaust Valve

Fig. 15 Normalized Mass Fraction

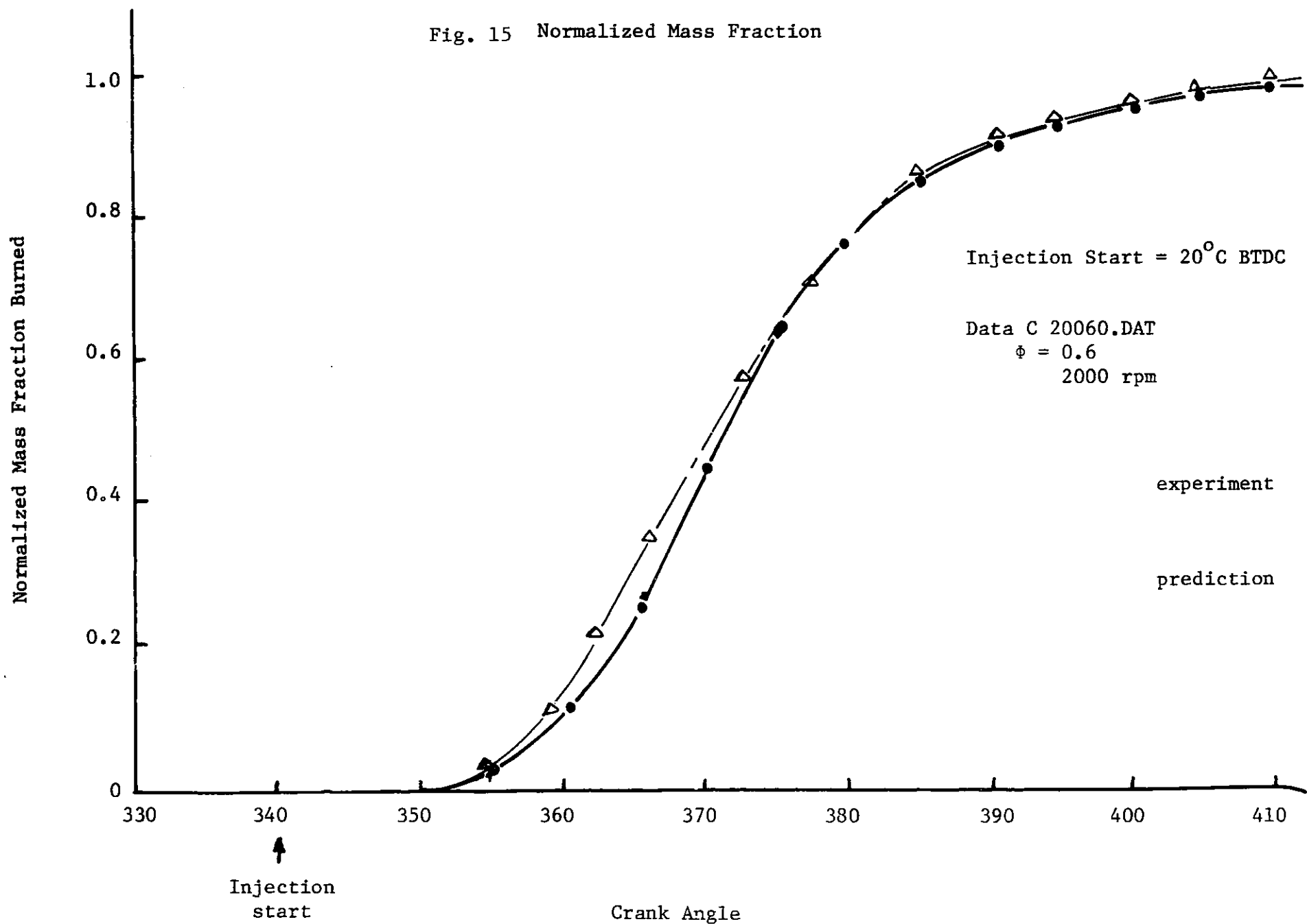
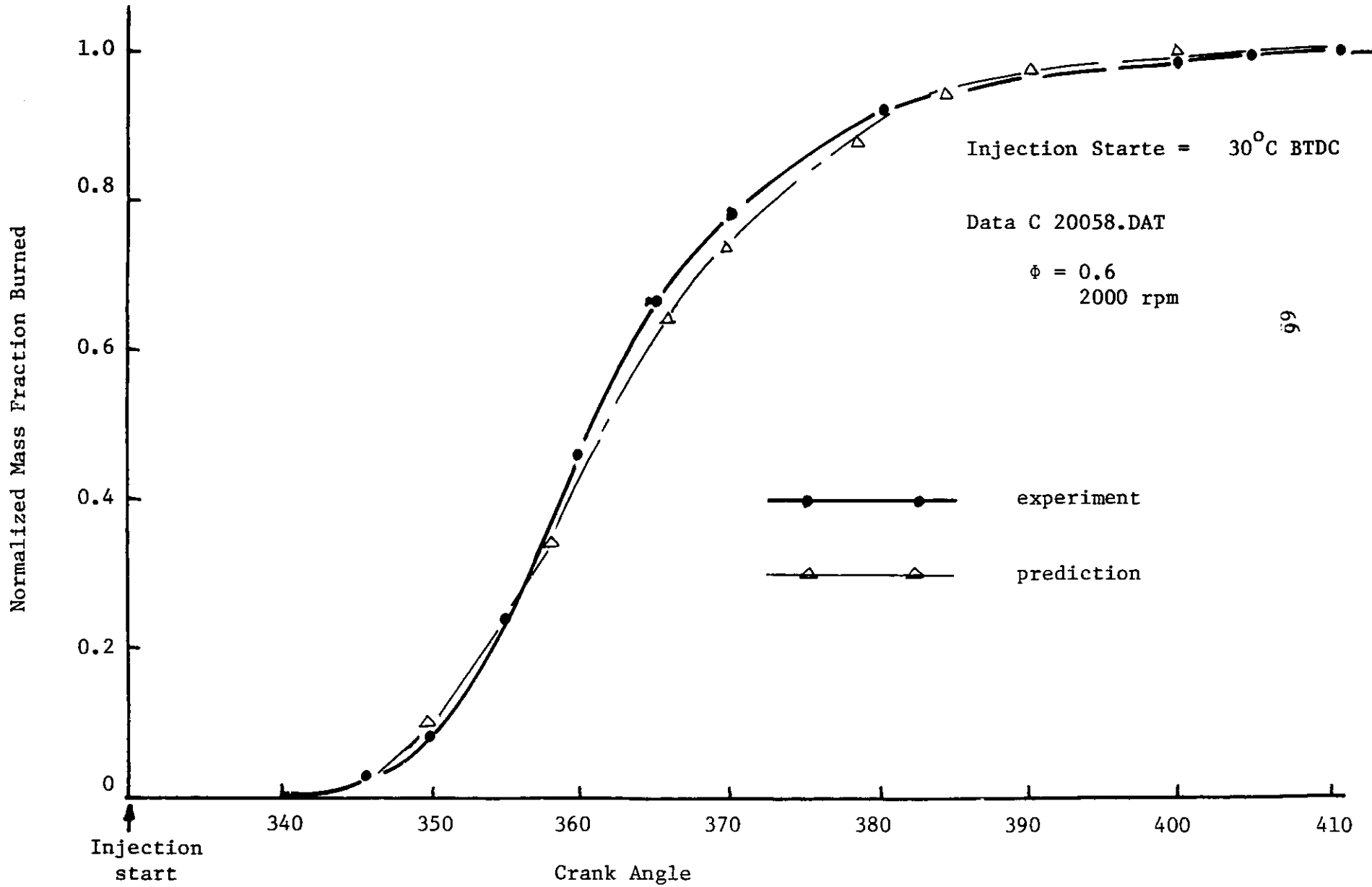


Fig. 16 Normalized Mass Fraction Burned



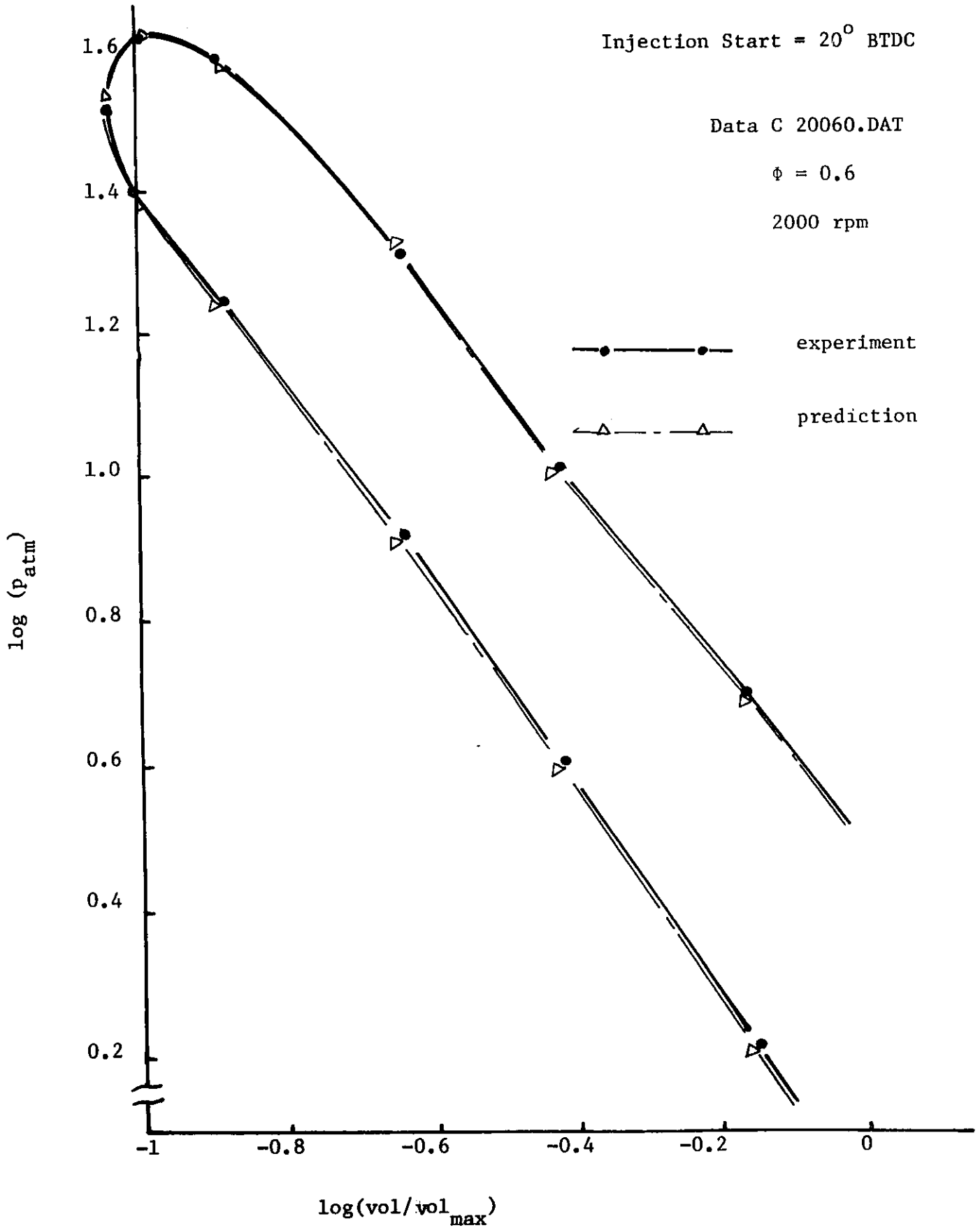


Fig. 17 $\log P - \log V$ Diagram ($\theta_1 = -20^{\circ}$)

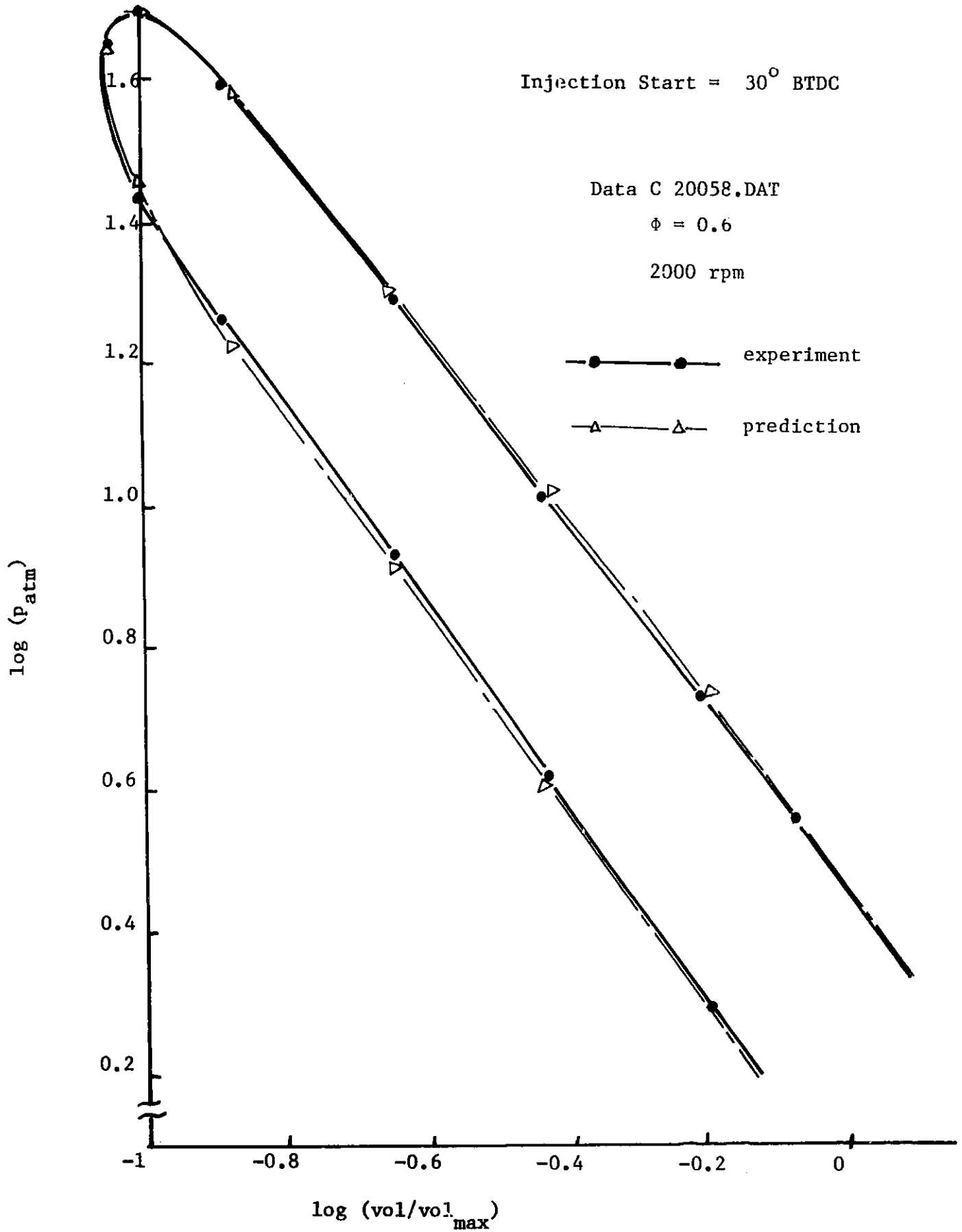


Fig. 18 log P - log V Diagram ($\theta_1 = -30^\circ$)

$\phi = 0.6$
2000 rpm

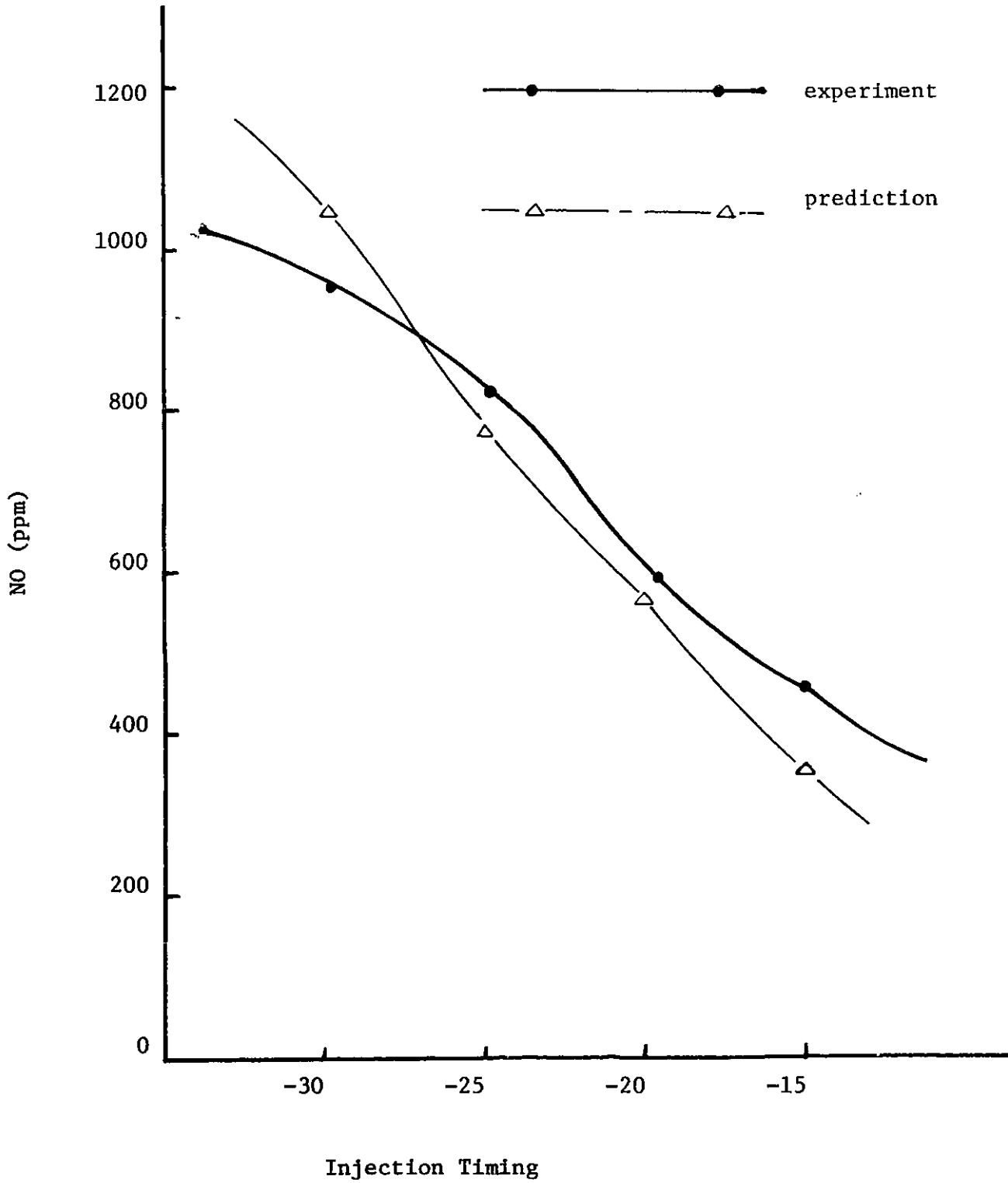


Fig. 19 Exhaust NO

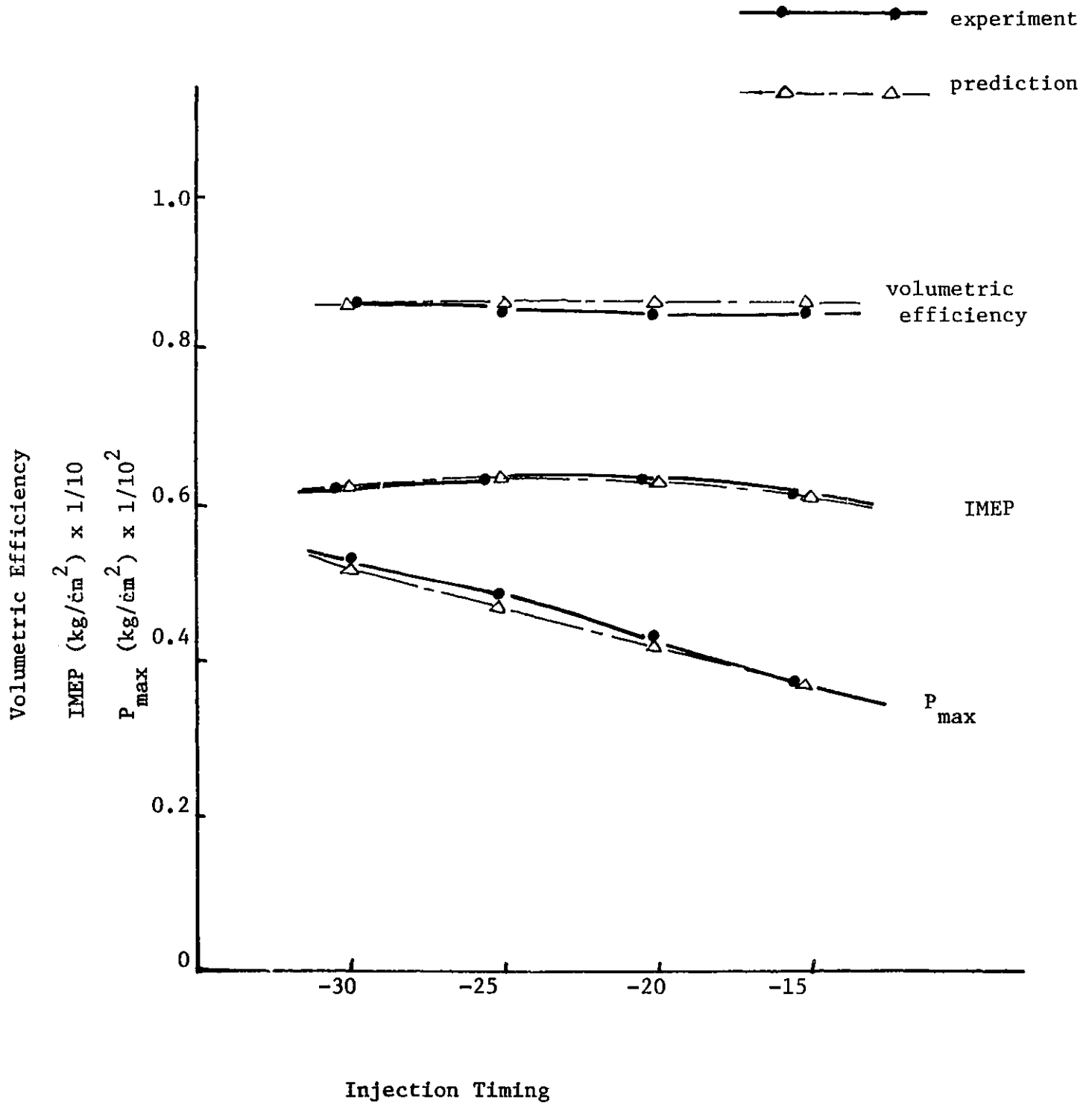


Fig. 20 Volumetric Efficiency, IMEP and P_{max}

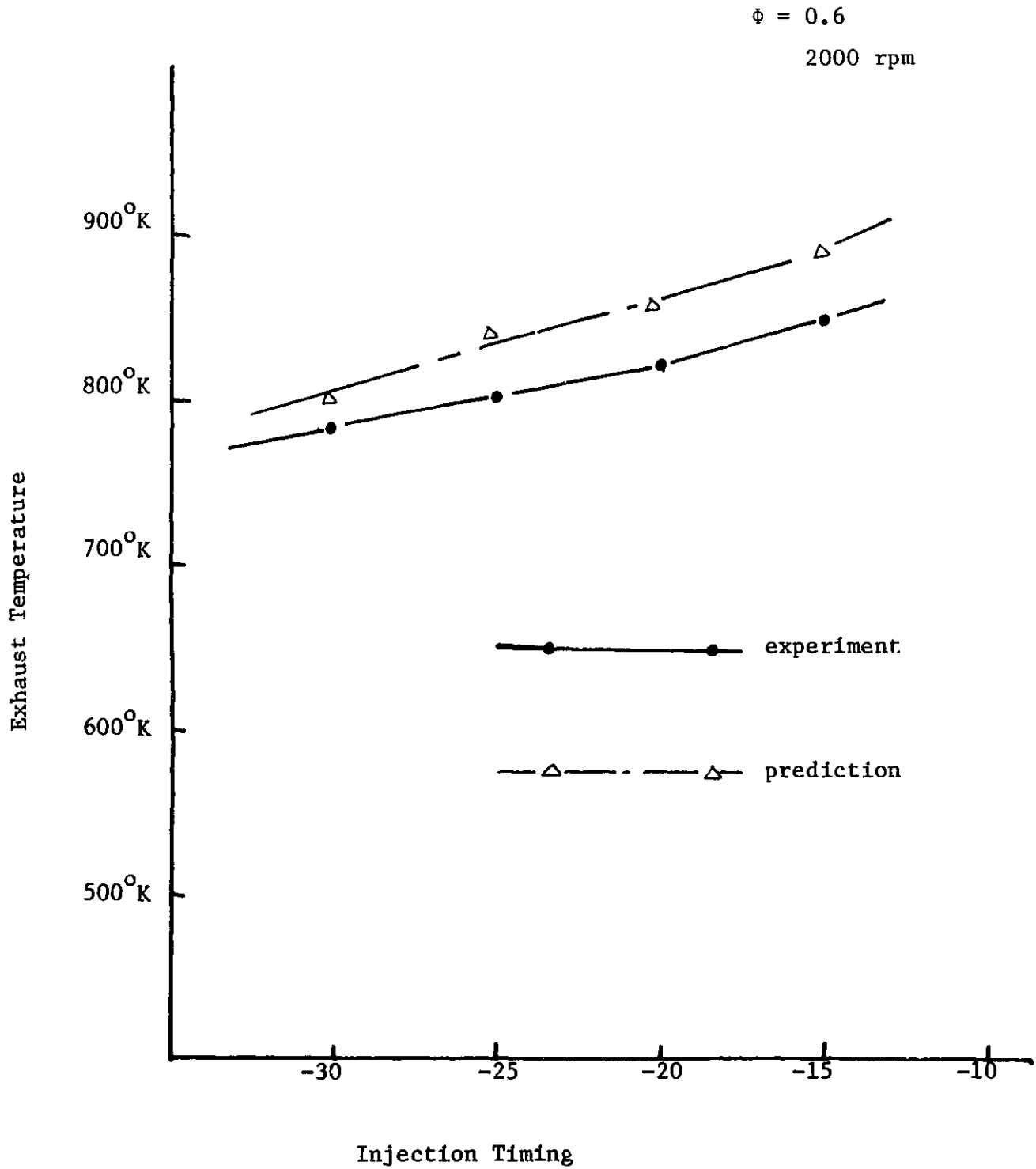


Fig. 21 Exhaust Temperature

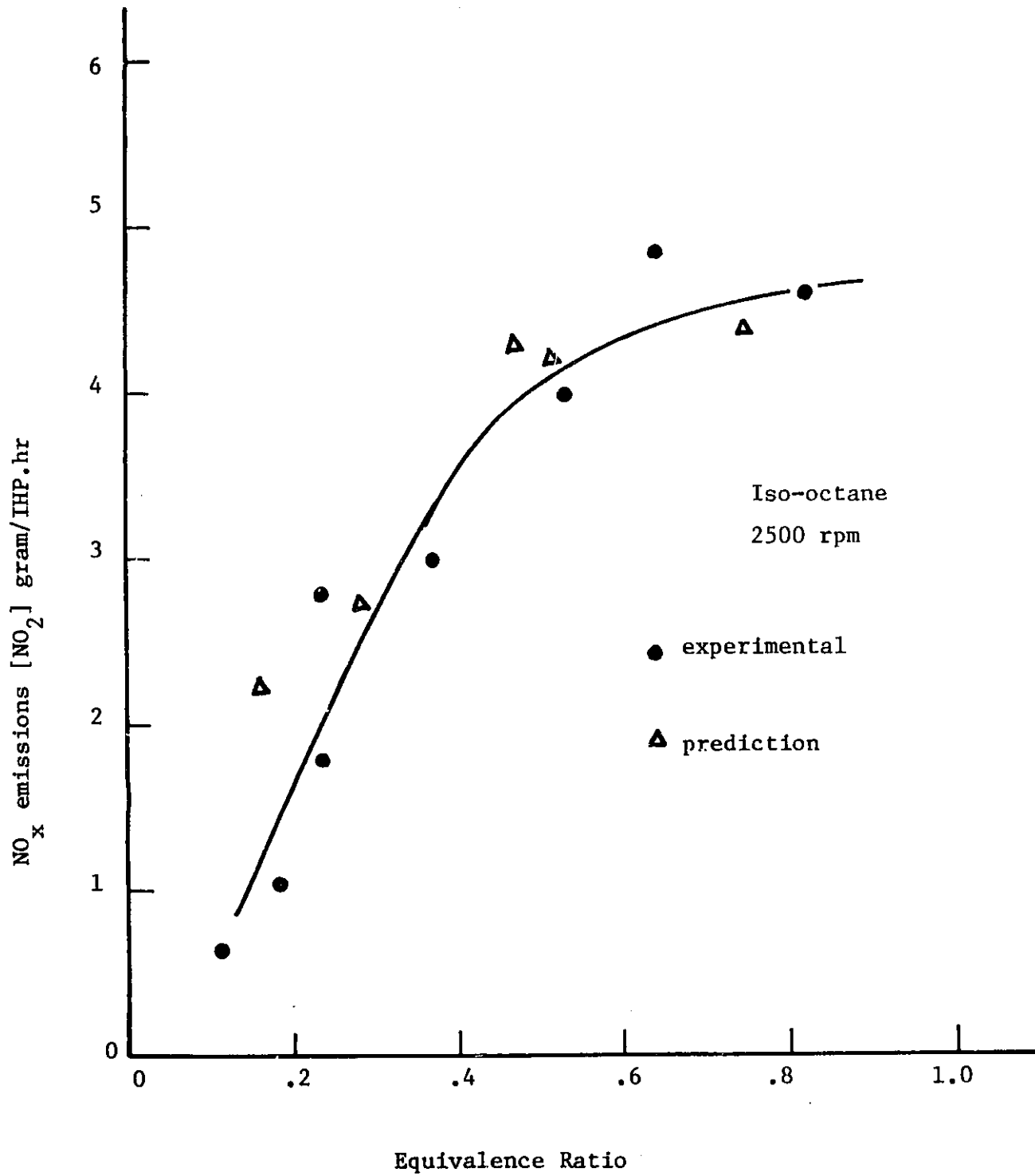


Fig. 22 NO_x vs. Equivalence Ratio

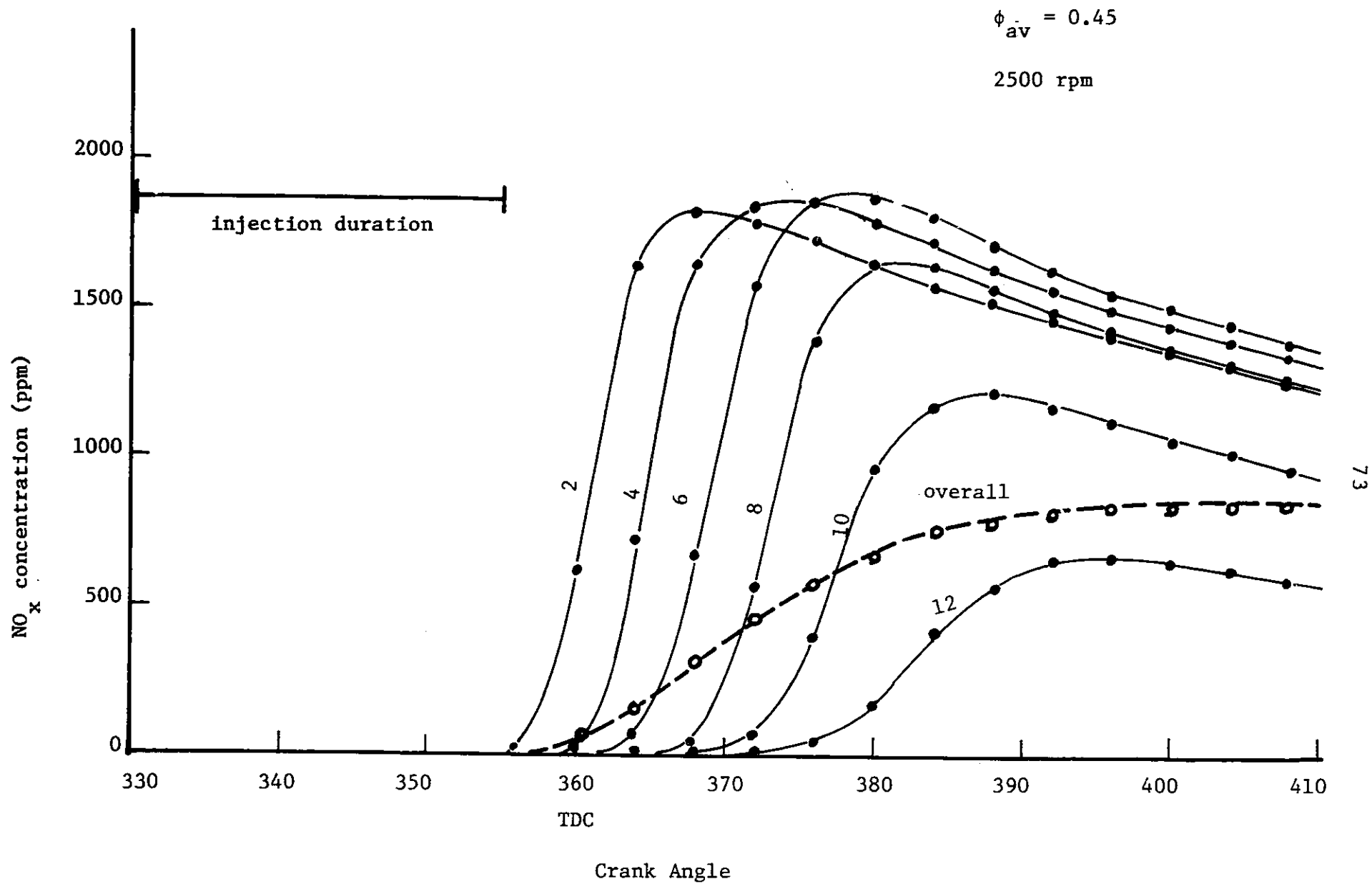


Fig. 23 NO Concentration in Each Element

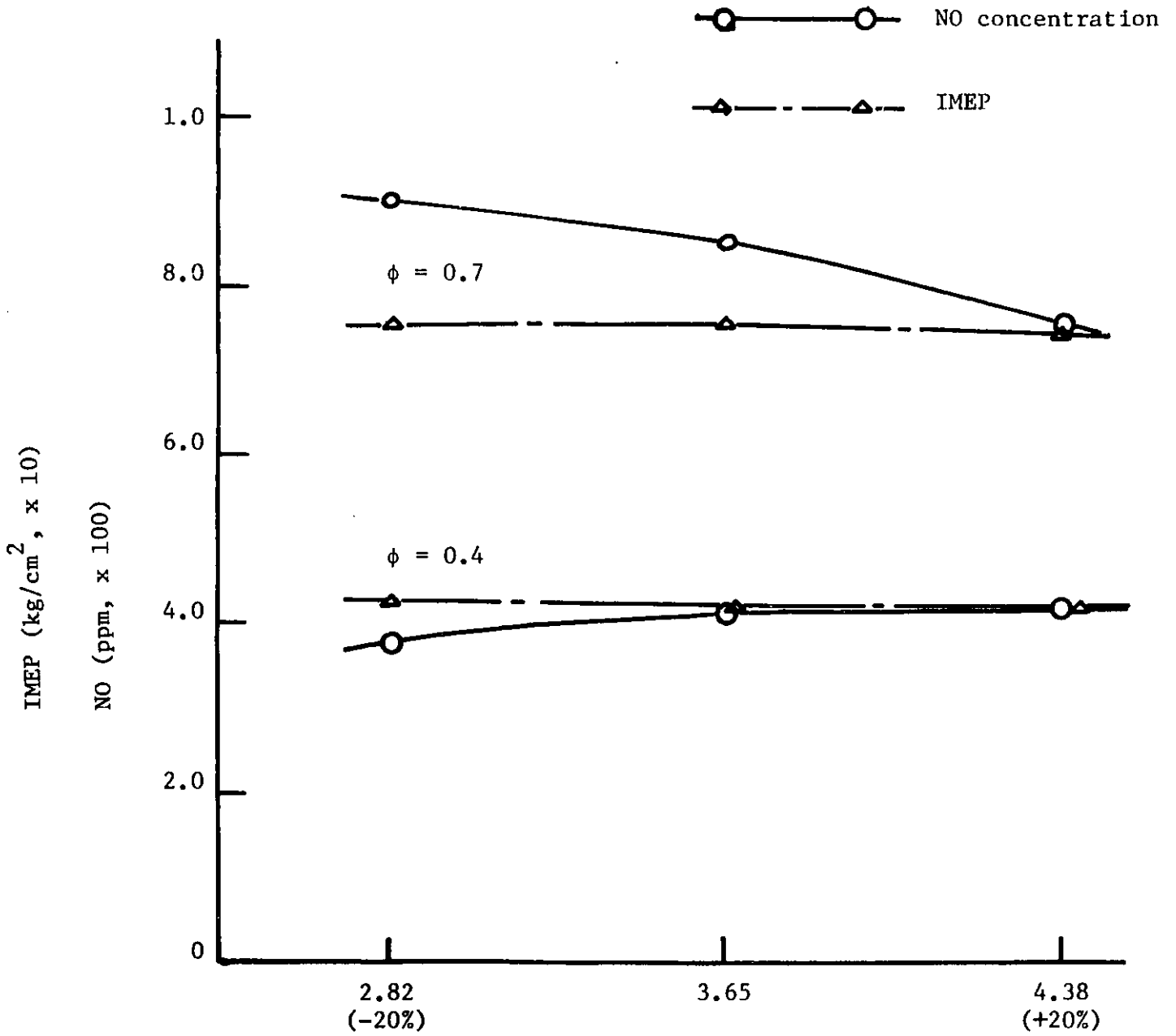


Fig. 24 The Effects of Swirl Ratio on Indicated Mean Effective Pressure and NO Concentration

Appendix A

Gas Exchange Process

During a discussion of exhaust and intake phases in the model description, it was seen that a knowledge of discharge-coefficient and the effective flow areas for both exhaust and intake valves were required. In this section, the experimental data on engine port steady flow tests (obtained from Texaco) will be reported.

Figure 13 and 14 shows the results from the tests mentioned above for the intake and exhaust port in the form of a plot of flow coefficient vs. valve lift to effective valve diameter ratio (L/D).

The mass flow rates through exhaust and intake valves have been treated as quasi-steady flow through a restriction. The governing equations can be written as follows:

$$\dot{M} = C_v A_v \left(\frac{RT_o}{P_o} \right) (\gamma RT_o)^{1/2} \left\{ \frac{2}{\gamma-1} \right\} \left[\left(\frac{P_2}{P_o} \right)^{2/\gamma} - \left(\frac{P_2}{P_o} \right)^{\frac{\gamma+1}{2(\gamma-1)}} \right]^{1/2}$$

For choked flow, with the condition for choked flow being:

$$\frac{P_o}{P_s} > \left(\frac{\gamma+1}{2} \right)^{\frac{\gamma}{\gamma-1}}$$

and

$$\dot{M} = C_v A_v \left(\frac{P_{oc}}{RT_{oc}} \right) (\gamma RT_o)^{1/2} \left\{ \left[\frac{2}{\gamma+1} \right]^{\frac{\gamma+1}{2(\gamma-1)}} \right\}^{1/2}$$

where C_v = valve discharge coefficient
 A_v = valve flow area
 P_o, T_o = upstream stagnation pressure and temperature
 P_2 = downstream static pressure

The logic to take care of the reverse flows through the valves has been put into the computer program. When there is reverse flow into the intake system (during valve overlap), the mass that flows past the intake valve is recorded. Its composition and temperature while in the intake system are assumed to be the same as those in the combustion chamber gas when intake valve opens. It has been assumed that, before any fresh charge can enter the chamber, all the mass that left the chamber during reverse flow is pulled back in first. For NO_x calculation NO_x mass transfer through the intake valve is taken into consideration.

Appendix B

Friction Loss

The following procedure for determining FMEP for a conventional spark ignition engine is taken from Bishop(33). The equations have been empirically correlated with data from engines ranging from single to eight cylinders, 4:1 to 17:1 compression ratio, and 0.75 to 2.5 bore to stroke ratio.

1. $FMEP_{\text{misc \& pumps}}$ = $0.39(N/1000)^{1.5}$
2. $FMEP_{\text{cam gear}}$ = $[30 - (4N/1000)]GH^{1.75}/B^2S$
3. $FMEP_{\text{bearing}}$ = $(KB/S)N/1000$
4. $FMEP_{\text{crank case}}$ = Σ Eqns. 1,2,3
5. $FMEP_{\text{blowby}}$ = $[(P_a - P_1)/14.2]^{1/2} [1.72 R_c^{0.4} - (0.49 + 0.015 R_c) (N/1000)^{1.185}]$
6. $V_p/1000$ = $SN/6000$
7. $FMEP_{\text{viscous piston}}$ = $(21.93M/BS) V_p/1000$

8. $FMEP_{\text{static ring tension}} = 2.11 S_n / B^2$
9. $FMEP_{\text{ring gas pressure}} = [(P_a - P_1) / 14.2] [2.35 S / B^2] [0.088 R_c + 0.182 R_c (1.33 - (0.121 V_P / 1000))]]$
10. $FMEP_{\text{total}} = \Sigma \text{ Eqns. } 4, 5, 7, 8, 9$

Symbols

- B: cylinder bore diameter (in)
- C: number of cylinder
- D: total engine displacement (in³)
- G: number of intake valves per cylinder
- H: intake valve head diameter (in)
- K: journal bearing size coefficient
- M: equivalent length of piston skirt (projected area of skirt ÷ cylinder bore diameter)
- N: revolutions per minute of crankshaft
- P_a : dry atmospheric pressure (psia)
- P: intake manifold vacuum (positive, when less than atmospheric pressure) (psig)
- R_c : compression ratio
- S: piston stroke (in)

V_p : mean piston velocity (ft/min)

n: total number of piston rings per cylinder

Appendix C

Thermodynamic Properties (hydrocarbon-air combustion)

The thermodynamic properties of the unburned and burned gases must be determined continuously during the entire cycle analysis. In most cases, the assumption of local chemical equilibrium can be justified and; in principal, for a given pressure and temperature, the mass action equations could be solved directly for species concentrations from which the desired thermodynamic properties could be derived. Alternatively, the species concentration which minimized the Gibbs free energy of the system could be calculated. Both approaches are quite involved and the calculations are expensive and time-consuming.

Our calculations use a computational method which is based on curve fitting data obtained from detailed thermochemical calculations to a functional form obtained from a consideration of carbon-air combustion.

This method was developed by M. Martin(21) and additional work done by S.D. Hires(34).

The following effects are taken into consideration by this method:

- i) Fuel vapor coefficient for unburned, air and residual gas mixtures.

- ii) Chemical dissociation effect above 1000°K for burned gases.

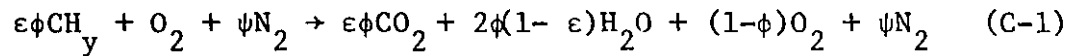
I Composition

- . unburned gas
 - i) fuel vapor
 - ii) burned gas (residual gas)
 - iii) air

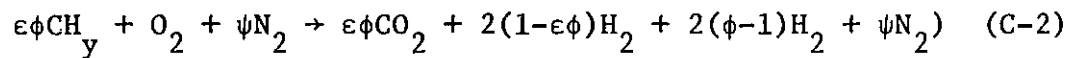
- . burned gas
 - i) burned gas

II Combustion reaction equations (hydrocarbon-air, under 1000°K)

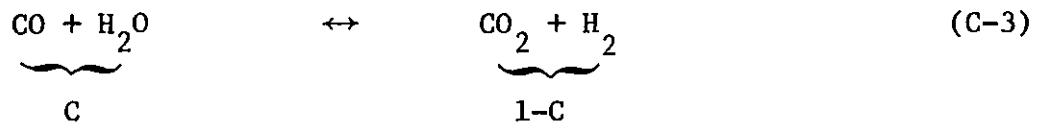
- i) lean mixtures ($\phi < 1$)



- ii) rich mixtures ($\phi > 1$)



and water gas reaction



This reaction is in equilibrium with equilibrium constant $K(T)$.

"C" is obtained by solving the following equation for its positive root.

$$(1-K) C^2 + 2[1-\epsilon\phi + K(\phi-1 + \epsilon\phi)]C - 2K\epsilon\phi(\phi-1) = 0 \quad (C-4)$$

$K(T)$ is determined by curve fitting JANAF Table data(35) over the temperature range 400-3200^oK and is given by:

$$\ln(K(T)) = 2.743 - 1.761/t - 1.611/t^2 + 0.2803/t^3 \quad (C-5)$$

where

$$t = T/1000 \quad (T = \text{ }^{\circ}\text{K})$$

The coefficients for species are shown in Table C-1.

TABLE C-1

i	Species	$\phi \leq 1$	$\phi > 1$
1	CO ₂	$\epsilon\phi$	$\epsilon\phi - C$
2	H ₂ O	$2(1-\epsilon)\phi$	$2(1-\epsilon\phi) + C$
3	CO	0	C
4	H ₂	0	$2(\phi-1) - C$
5	O ₂	$1-\phi$	0
6	N ₂	ψ	ψ
	sum	$(1-\epsilon) + 1 + \psi$	$(2-\epsilon)\phi + \psi$

where

ψ = the molar N:O ratio of the product

y = the molar H:C ratio of the fuel

$\epsilon = 4/(4+y)$

ϕ = the fuel-air equivalence ratio

III Grams of product per mole of O₂ reactant

$$\tilde{M} = \phi(8\epsilon+4) + 32 + 28\psi \quad (C-6)$$

IV Specific enthalpy and specific heat at constant pressure

$$\tilde{M} C_p = \sum_{i=1}^6 x_i \sum_{j=1}^4 (a_{ij} t^{j-1} + a_{5j}/t^2) \quad (C-7)$$

$$\tilde{M} h = \sum_{i=1}^6 x_i \sum_{j=1}^4 (a_{ij} t^j/j - a_{5j}/t + a_{6j}) \quad (C-8)$$

The coefficients a_{ij} are obtained by curve fitting JANAF Table data to the above functional form. The values of a_{ij} are given in Table C-2. The resultant C_p is in cal/g - °K and h is in Kcal/g.

TABLE C-2

Coefficients for Polynomial Fit to Thermodynamic Properties

Coefficients for $100^{\circ}\text{K} < T < 500^{\circ}\text{K}$

i	Species	a_{i1}	a_{i2}	a_{i3}	a_{i4}	a_{i5}	a_{i6}^*
1	CO ₂	4.7373	16.653	-11.232	2.8280	.006767	-93.75793
2	H ₂ O	7.8097	-.20235	3.4187	-1.1790	.001436	-57.0800
3	CO	6.9738	-.82383	2.9420	-1.1762	.0004132	-27.196
4	H ₂	6.9919	.16170	-.21821	.29682	-.016252	-.11819
5	O ₂	6.2957	2.3884	-.031479	-.32674	.004359	.103637
6	N ₂	7.0922	-1.2958	3.2069	-1.2022	-.0003458	-.013967
7	C ₈ H ₁₈ **	-.55313	181.62	-97.787	20.402	-.03095	-60.518

Coefficients for $500^{\circ}\text{K} < T < 6000^{\circ}\text{K}$

i	Species	a_{i1}	a_{i2}	a_{i3}	a_{i4}	a_{i5}	a_{i6}^*
1	CO ₂	11.940	2.0886	-.47029	.037363	-.58945	-97.1418
2	H ₂ O	6.1391	4.6078	-.93560	.066695	.03358	-56.6259
3	CO	7.0996	1.2760	-.28775	.022356	-.15987	-27.7346
4	H ₂	5.5557	1.7872	-.28813	.019515	.16118	.76498
5	O ₂	7.8658	.68837	-.031944	-.002687	-.20139	-.89346
6	N ₂	6.8078	1.4534	-.32899	.025610	-.11895	-.33184
7	C ₈ H ₁₈ **	.55313	181.62	-97.787	20.402	-.03095	-60.518

*picked to give enthalpy datum at 0°K

**The coefficients apply to 2,2,4 trimethyl pentene (iso-octane). Different equations may be needed for different fuels.

V) Molecular weight of the burned mixture

$$\bar{M} = \begin{cases} \tilde{M}/((1-\varepsilon)\phi+1 + \psi) & \phi \leq 1 \\ \tilde{M}/((2-\varepsilon)\phi + \psi) & \phi > 1 \end{cases}$$

APPENDIX D Computer Program

Program Listing

```
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
C
C *** PREDICTING THE DETAILED PERFORMANCE AND EMISSION CHARACTERISTICS*** C
C *** OF THE TEXACO CONTROLLED-COMBUSTION+STRATIFIED CHARGE ENGINE *** C
C
C *** THIS PROGRAM HAS BEEN DEVELOPED IN ORDER TO PREDICT THE *** C
C *** DETAILED PERFORMANCE AND THE NOX EMISSION CHARACTERISTICS OF THE ** C
C *** TCCS, STRATIFIED CHARGE ENGINE, GIVEN ENGINE GEOMETRY, FUEL PROPERTIES C
C *** AND THE OPERATING CONDITIONS. C
C *** THE DETAILS OF THE THEORY C
C *** AND ANY OTHER INFORMATION REGARDING THIS PROGRAM CAN BE HAD FROM C
C *** SAB(H).HIRAKI.31-159, 253-2388 *** C
C
C*** ALL RIGHTS RELATED TO THIS PROGRAM ARE RESERVED BY SLOAN AUTOMOTIVE
C*** LABORATORY, M.I.T.
C
C***WARNING*** THIS PROGRAM IS NOT FINAL VERSION. THIS PROGRAM WILL
C BE CHANGED WITHOUT NOTICING IN FUTURE.
C
C INPUT VARIABLES
C SEE INPUT SUBROUTINE
C
C OUTPUTS
C ANGLE= CRANK ANGLE DEG
C P,PRESS= CYLINDER PRESSURE ATA
C OPM= NOX CONCENTRATION PPM
C WNO= NOX MASS G*10**6
C NUM=ELEMENT NUMBER
C *1 NOT YET BURNED
C 2 PLUME IS DEVELOPING
C 3 PLUME HAS FULLY DEVELOPED
C T,T,AV= TEMPERATURE OF THE ELEMENT
C UTEM= UNBURNED TEMPERATURE K(FOR JET ELEMENT)
C BTEM= BURNED TEMPERATURE K(FOR JET ELEMENT)
C MASS= TOTAL MASS OF THE ONE ELEMENT MG
C PMASS= PLUME MASS MG(FOR JET ELEMENT)
C RMASS= BURNED GAS MASS MG(FOR JET ELEMENT)
C V = VOLUME CM**3
C EQUI= EQUIVALENC RATIO
C CP= SPECIFIC HEAT AT CONSTANT PRESSURE CAL/G K
C ENTH= ENTHALPY CAL/G
C OPM= NO CONCENTRATION PPM
C WNO= MASS OF NO G*10**6
C R= JET LOCATION RADIAL CM
C Z= JET LOCATION AXIAL CM.
C DEG= JET LOCATION TANGENTIAL DEG
C HLOSS= HEAT LOSS CAL
C RATIO= BURNED MASS FRACTION
C PS= OUTPUT POWER TOTAL SINCE INTAKE VALVE OPENS PS
C AEFF= INTAKE EFFECTIVE FLOW AREA CM**2
C GWI= INTAKE MASS FLOW G/S
C AEFF= EXHAUST EFFECTIVE FLOW AREA CM**2
C GWE= EXHAUST MASS FLOW G/S
C
C
C SUBROUTINES REQUIRED
C 1. DUNGE
C 2. OPFR
C 3. INPOP
C 4. EXPOR
```

C 5. HEATR
 C 6. INPUT
 C 7. CALCU
 C 8. DELTA
 C 9. DCAL
 C 10. JFT
 C 11. VGAS
 C 12. CONT
 C 13. FNT
 C 14. VOLCY
 C 15. HPRD
 C 16. CLDPRD
 C 17. DEPIVS
 C 18. D'IODT
 C 19. FOT3
 C 20. TWDDIM
 C 21. UPRD
 C 22. PTCHEM
 C 23. NXNSOL
 C 24. FRICT FRICTIONS

C
C

```

COMMON/EPROP/ IECHF(100),EFUEL(100),EANGL(100),FMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DUEMAS(100),
1 ERGAS(100),ECP(100),FENTH(8,100),FMASS(8,100),EEQUI(8,100),
1 EVOL(8,100),ETEMP(8,100),PCYL(8)
COMMON /STATE/ ANGLE1,ANGLE,ICOND,ASTEP,NSTEP,IF
COMMON /STAT1/ PPM,ILAST,NJET,PHIO,FGX
COMMON /TIMIG/ AINT0,AEXT0,AINTC,AFXTC,AFUELS,AFUFLE
COMMON /GEOMT/ STROKE,RORE,CONL,VCLFAR,VCUP,RCUP,CYLN
COMMON /SIM1/ STGINT,STGEXT,STOEXT
COMMON /HETPS/ NDIVID,TMETAL(10),QHTRC(10),ARF(10)
COMMON /HETR1/ PRASE,TRASE,VRASE,COHT1,COHT2,COHT,RHR(10)
COMMON /SIM2/ TTGINT,TTGEXT,TTOEXT,TTQHT(10),TOHT(10),PS
COMMON /FUFLP/ FUEL,FKCAL,ZM,ZN,PSI,SAFC,VACAL
COMMON /JETS/ RJET(50),HJET(50),AJET(50),BJET(50),THER(50),
1 VRJET(50),VZJET(50),WJET(50),VJET(50),AINJ(50),RHJET(50),
2 PITCH(50)
COMMON /PLIME/ PMASS(50),BMASS(50),RPLUM(50),HRMASS(50)
COMMON /JETDAT/ NSWIR0,ALPHA,RFTA,PHGAS
COMMON /SUPGAS/ FSWIR,WSWIR,SDECAY
COMMON /COMR/ RPLUM0,FNTIR,RLFSP,ASPARC,TSCALF
COMMON /OSOPE/ LPT,INP,LINCO,LINFST,ITEL,LITEL
COMMON /NOXSP/ CONNO(100),PPMNO(100),*NO(100),SIMDM(100),PPMEXT
DIMENSION ZMDFR(5),TMSPI(100)
REAL*4 M:R2
COMMON /FUFL/ AF(16),FMW,CX,HY,0Z,QLDVER
COMMON/OXIDANT/XI/CMPSTN/X(7),WRAR
COMMON/CONTL/ ASTIN,ASTCL,ASTCR,ASTEX,ASTOV,PRININ,PRINCL,PRINFX,
1 PRINCV
COMMON/INFLO/ NINEF,XINEF(20),YINEF(20),AREIND,PINT,TINT,GINT,
1 EQUINT,AREINT,XINV
COMMON/EXFLO/NEXFF,XEYFF(20),YEYFF(20),AREEXD,PEXT,TEXT,GEXT,
1 IGLFT,ILFFT,DEXT,DFLOW,ELEFT,AREEXT,XEXV
COMMON /AMPJE/ TAMR,PAVR
COMMON /FRIDT/ RSKTPT,FCDE,RINGN,FMER(10)
DIMENSION HRTMP(50),PLTEMP(50),PLFNT(50)
DIMENSION YTEMP(10),YCP(10)
DIMENSION YPDM(2),YPRM(2)

```

C


```

C DATA OF SPECIFIC HEAT AT CONSTANT PRESSURE VS TEMPERATURE
C DATA OF TEMPERATURE
C DATA XTEMP/
  1 500.,1000.,1500.,2000.,2250.,2500.,2750.,3000.,3250.,3500./
C DATA OF SPECIFIC HEAT AT CONSTANT PRESSURE
C DATA YCP/
  1 0.26,0.3,0.34,0.37,0.43,0.51,0.72,0.84,1.24,1.6/
C
C
C CARD READER AND LINE PRINTER
  INP=5
  LPT=6
C
C JCOND=1 VALVE OVERLAP PERIOD
C JCOND=2 INTAKE PROCESS
C JCOND=3 COMPRESSION PERIOD
C JCOND=4 COMBUSTION AND EXPANSION PERIOD
C JCOND=5 EXHAUST PROCESS
C
9001 CONTINUE
  ASTEP=2.0
  PSI=3.76
C
C INITIALIZE COMBUSTION CONDITION
C IFCHE=1 NOT YET BURNED
C IFCHE=2 PLUME IS DEVELOPING IN THE JET
C IFCHE=3 PLUME IS FULLY DEVELOPED
  DO 1010 I=1,100
    IFCHE(I)=1
    COMNO(I)=0.
1010 CONTINUE
  EPMAX=10.
  MAXITS=15
  JLAST=1
  LITFL=1
  RESFRK=0.
C
C INPUT SUBROUTINE
C
  CALL INPUT
C
C
C NOCUL=0 PERFORMANCE ONLY
C NOCUL=1 (NO) CALCULATION WILL BE DONE
  NOCUL=1
C DO YOU WANT NOX CALCULATION?
  READ(INP,302) NOCUL.PPMEXT
C
C CALCULATION STARTING CRANK ANGLE AND ENDING CRANK ANGLE
  READ(INP,29) ANGLES.ALAST
  DEL=ZN/ZM
  EFUEL(1)=0.
  ENTINI=ENT(TINT,FOUJNT,SAFC)
  IF=1
  ANGLE=ANGLES
  ANGLE1=ANGLES
  CALL VOLCY(ANGLE,FVOL(2,1),DVCYL,STROKE,BORE,CONL,VCLFAR,VCIIP)
  VCULO=EVOL(2,1)
  FENTH(2,1)=ENT(ETEMP(2,1),EEQUI(2,1),SAFC)
  ENTIN2=EENTH(2,1)

```

```

CALL HPR0D(PCYL(2),FTEMP(2,1),FFOHT(2,1),DEL,PST,DJM,DJM,DUM,DUM,
1 DUM,DUM,EMWT(1),FRGAS(1),CVRG,DUM)
FCP(1)=FRGAS(1)/41.443+CVHG
EMASS(2,1)=PCYL(2)*EVOL(2,1)/(FRGAS(1)*FTEMP(2,1))

```

C

C INITIAL CONDITION

```

100 TTGINI=0.
    TTGEXT=0.
    TTDEXT=0.
    TGLEFT=0
    TMAX=0.
    PMAX=0.
    PS=0.
    PSTH=0.
    PSEX=0.
    JCOND=0
    ENTINT=ENTIN?
200 CONTINUE
    DO 10 I=1,NDIVID
    TQHT(I)=0.
10 TQHT(I)=0.

```

C

C SWIRL DECAYING FACTOR FSWIR=1. AT THE TIME OF INTAKE VALVE CLOSING

```

    FSWIP=1.
3000 CONTINUE
    PCYLO=PCYL(2)

```

C

CALL OPERA

C

C RINGE KUTTA METHOD

```

CALL PUNGF(ENTINT)
ANGLE1=ANGLE1+ASTEP
TTGINI=TTGINI+STGINI*ASTEP
TTGEXT=TTGEXT+STGEXT*ASTEP
TTDEXT=TTDEXT+STDEXT*ASTEP
DO 3900 I=1,NDIVID
3900 TQHT(I)=TQHT(I)+TQHT(I)*ASTEP

```

C

C CHECK THE INTAKE FLOW

```

    IF(JCOND.GE.3) GO TO 570
    IF(TTGINI.GT.0.) GO TO 572

```

C

C INTAKE FLOW IS RESIDUAL GAS

```

    ENTINT=ENTIN?
    GO TO 570

```

C

C INTAKE FLOW IS FRESH AIR

```

572 ENTINT=ENTIN1
570 CONTINUE
    CALL VOLCY(ANGLE1,VCYLN,VCYLO,STROKE,BORF,CONL,VCLEAR,VCUP)
    DPS=((PCYL(2)+PCYLO)/2.-PAMB)*(VCYLN-VCYLO)*CYLN*RPM/900000.
    VCYLO=VCYLN
    PS=PS+DPS
    IF(JCOND=2) 580,580,590
580 PSIN=PSIN+DPS
    GO TO 530
590 IF(JCOND=5) 530,595,530
595 PSEX=PSEX+DPS
530 DO 16 I=1,NDIVID
16 TQHT(I)=TQHT(I)+TQHT(I)*ASTEP

```

```

      IF (IGLFFT) 500,500,510
500  GLEFFT=GLEFT-STGEXT
      IF (GLEFFT) 520,520,510
520  IGLFFT=1
510  CONTINUE
      IF (JCOND.NF.4) GO TO 4500
C
C  COMBUSTION ROUTINE
C
C  ESTIMATE THE PLUME TEMPERATURE
DO 3700 I=1,IF
  IF (IECHE(I).EQ.2) GO TO 3400
  URTEMP(I)=FTEMP(2,I)
  PLTEMP(I)=FTEMP(2,I)
  PLENT(I)=EFNTH(2,I)
  GO TO 3700
3400  ENTJET=EFNTH(2,1)
      CALL TWODIM(10,XTEMP,YCP,PLTEMP(I),PLCPI)
      PLCP=PLCPI
      ENTPL=(EFNTH(2,1)*EMASS(2,I)-ENTJET*(EMASS(2,I)-PMASS(I)))/
1  PMASS(I)
      DPLT=(ENTPL-PLENT(I))/PLCP
      DO 3820 J=1,3
      PLT=PLTEMP(I)+DPLT
      TDUM2=PLTEMP(I)+DPLT/3.
      TDUM3=PLTEMP(I)+DPLT*2./3.
      TDUM4=PLTEMP(I)+DPLT
      CALL TWODIM(10,XTEMP,YCP,TDUM2,CPDUM2)
      CALL TWODIM(10,XTEMP,YCP,TDUM3,CPDUM3)
      CALL TWODIM(10,XTEMP,YCP,TDUM4,CPDUM4)
      DPLT=(ENTPL-PLENT(I))*(1./PLCP+2./CPDUM2+2./CPDUM3+1./CPDUM4)/6.
3820  CONTINUE
      PLENT(I)=ENTPL
      PLTEMP(I)=PLT
3700  CONTINUE
      IF (NOCUL.F0.0) GO TO 4500
C
C  CALCULATION OF NOX CHANGE BY MASS TRANSFER
C
DO 4012 I=2,IF
  CONNO(I)=CONNO(I)+CONNO(1)*SUMDM(I)/(EMASS(2,1)-SUMDM(1))
4012  CONTINUE
  CONNO(1)=CONNO(1)+CONNO(1)*SUMDM(1)/(EMASS(2,1)-SUMDM(1))
  DO 4010 I=1,IF
  TMSPI(I)=EMASS(2,I)/EMWT(I)
  IF (IECHE(I).EQ.1) GO TO 4020
  IF (IECHE(I).EQ.3) GO TO 4040
  PLSPI=PMASS(I)/EMWT(I)
  GO TO 4050
4040  PLSPI=TMSPI(I)
4050  CONTINUE
  CALL EQIB(PLTEMP(I),PCY(2),FFQUI(2,I),IFLAG,ZMOFR)
  IF (PLTEMP(I).GT.2400.) GO TO 4022
  IINOM=1
  GO TO 4024
4022  IINOM=4
4024  ASTNO=ASTEP/FLOAT(IINOM)
  DO 4026 IIN0=1,IINOM
  CALL DPROD(PLTEMP(I),PCY(2),PLSPI,ZMOFR,IIFLAG,CONNO(I),
1  DCONNO)

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      CONNO(I)=CONNO(I)+DCONNO*ASTNO/(RPM*6.)
4075 CONTINUE
4080 PDMNO(I)=1000000.*CONNO(I)/TMSPI(I)
      WNO(I)=30.*CONNO(I)
4010 CONTINUE
      TMSPI=0.
      TCONNO=0.
      TTMSPI=0.
      TWNO=0.
      DO 4030 I=1,IF
      TCONNO=TCONNO+CONNO(I)
      TTMSPI=TTMSPI+TMSPI(I)
      TWNO=TWNO+WNO(I)
4030 CONTINUE
      TDCM=1000000.*TCONNO/TTMSPI
4500 CONTINUE
      LINECO=L INFCO-1
      IF (L INECO) 2000,2000,2010
C
C PRINT OUT
2000 GO TO (2100,2200,2300,2400,2500),JCONO
C
C VALVE OVERLAP PERIOD
2100 GGNT=STGINT*CYLN*6.*PDM
      GGXT=STGEXT*CYLN*6.*RPM
      HTRS1=-TQHTR(1)
      HTRS2=-TTQ-TR(1)
      WRITE(LPT,2900) ANGLE1,PCYL(2),ETEMP(2,1),EVOL(2,1),PS,
1 FMASS(2,1),HTRS1 ,HTRS2 ,AREINT,GGNT,AREEXT,GGXT
      GO TO 9100
C
C INTAKE PROCESS
2200 GGNT=STGINT*CYLN*6.*PDM
      HTRS1=-TQHTR(1)
      HTRS2=-TTQ-TR(1)
      WRITE(LPT,2910) ANGLE1,PCYL(2),ETEMP(2,1),EVOL(2,1),PS,
1 FMASS(2,1),HTRS1 ,HTRS2 ,AREINT,GGNT
      GO TO 9100
C
C COMPRESSION PROCESS
2300 SPAT=WSWIR*180./((3.1416*6.*RPM)
      HTRS1=-TQHTR(1)
      HTRS2=-TTQ-TR(1)
      WRITE(LPT,2920) ANGLE1,PCYL(2),ETEMP(2,1),EVOL(2,1),PS,
1 FMASS(2,1),HTRS1 ,HTRS2 ,FSWIR,SRAT
      GO TO 9100
C
C COMBUSTION PERIOD
C EXPANSION PERIOD
2400 CONTINUE
      IF (IE.EQ.1) GO TO 2300
      HTRS1=-TQHTR(1)
      HTRS2=-TTQ-TR(1)
      TRATBI=0.
      DO 2410 I=2,IF
2410 TRATBI=TRATBI+HRMASS(I)
      TRATBI=TRATBI/FUFL
      SPAT=WSWIR*180./((3.1416*6.*RPM)
      IF (NOCIL.EQ.0) GO TO 5010
      TWNOXX=TWNO*10.**6

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WRITE(LPT,2930) ANGLE1,PCYL(2),PS,FSWIR,SRAT,HTRS2
WRITE(LPT,2931) TRATRI,VCYLN,PHIO,TPPM,TWNXXX,HTRS1
GO TO 5015
5010 WRITE(LPT,2930) ANGLE1,PCYL(2),PS,FSWIR,SRAT,HTRS2
WRITE(LPT,2932) TRATRI,VCYLN,PHIO, HTRS1
5015 DO 5020 I=1,IF
HHL0S=HHL0S(I)
IF(NOCUL.EQ.0) GO TO 5022
IF(I.EQ.1) GO TO 5024
C
C NOX AND JET ELEMENT
AJF=AJET(I)*180./3.1416
ACQ=THEP(I)*180./3.1416
EMASMG=EMASS(2,I)*1000.
PMASMG=PMASS(I)*1000.
RMASMG=RMASS(I)*1000.
RATRU=HRMASS(I)/EFUFL(I)
WNOIMG=WNO(I)*10.**6
WRITE(LPT,2936) I,IFCHE(I),ETEMP(2,I),PLTEMP(I),RATRU
1 EMASMG ,PMASMG ,RMASMG ,EVOL(2,I),EEQUI(2,I),ECP(I),
2 EENTH(2,I),PPMNO(I),WNOIMG,RJET(I),AJET(I),AJE,BJET(I),ACQ,HHL0S
GO TO 5020
C
C NOX AND AIR
5024 EMASMG=EMASS(2,I)*1000.
WNOIMG=WNO(I)*10.**6
WRITE(LPT,2937) I,IFCHE(I),ETEMP(2,I),
1 EMASMG, EVOL(2,I),EEQUI(2,I),ECP(I),
2 EENTH(2,I),PPMNO(I),WNOIMG, HHL0S
GO TO 5020
C
C PERFORMANCE AND JET
5022 IF(I.EQ.1) GO TO 5026
AJF=AJET(I)*180./3.1416
ACQ=THEP(I)*180./3.1416
EMASMG=EMASS(2,I)*1000.
PMASMG=PMASS(I)*1000.
RMASMG=RMASS(I)*1000.
RATRU=HRMASS(I)/EFUFL(I)
WRITE(LPT,2934) I,IFCHE(I),ETEMP(2,I),PLTEMP(I),RATRU
1 EMASMG ,PMASMG ,RMASMG ,EVOL(2,I),EEQUI(2,I),ECP(I),
2 EENTH(2,I), RJET(I),AJET(I),AJE,BJET(I),ACQ,HHL0S
GO TO 5020
C
C PERFORMANCE AND AIR
5026 EMASMG=EMASS(2,I)*1000.
WRITE(LPT,2938) I,IFCHE(I),ETEMP(2,I),
1 EMASMG, EVOL(2,I),EEQUI(2,I),ECP(I),
2 EENTH(2,I), HHL0S
5020 CONTINUE
GO TO 4100
C
C EXHAUST PROCESS
2500 GGXT=STGEAT*CYLN*6.*PPM
HTRS1=-TQTR(1)
HTRS2=-TQTR(1)
WRITE(LPT,2940) ANGLE1,PCYL(2),FTEMP(2,1),EVOL(2,1),PS,
1 FMSS(2,1),HTRS1 ,HTRS2 ,AREFFI,GGXT
GO TO 2100
2900 FORMAT(14,F5.1)

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1          1X,F6.2,1X,F5.0,1X,F6.0,1X,F6.2,2X,F8.3,
1          E11.3 ,E11.3 ,1X,F6.2,1X,F7.2,2X,F6.2,1X,F7.2)
2920 FORMAT(1H ,F5.1,
1          1X,F6.2,1X,F5.0,1X,F6.0,1X,F6.2,2X,F8.3,
1          F11.3 ,F11.3 ,1X,F6.2,1X,F7.2 )
2920 FORMAT(1H ,F5.1,
1          1X,F6.2,1X,F5.0,1X,F6.0,1X,F6.2,2X,F8.3,
1          F11.3,F11.3,1X,F6.3,1X,F6.2)
2930 FORMAT(/,1X,F5.1,2X,IPRFSS,=1,F6.2,1X,IPSE=1,F6.1,1X,IPSWIP=1,F6.3,
1          1X,IPRT=1,F6.3,1X,IPJMH,REFJ=1,E9.3)
2931 FORMAT(2X,5HTOTAL,14X,F7.3,21X,2F7.2,14X,F7.0,F7.0,30X,E9.2)
2932 FORMAT(2X,5HTOTAL,14X,F7.3,21X,2F7.2,14X,          44X,E9.2)
2934 FORMAT(2X,I2,1X,I2,2F7.0,F7.3,3F7.1,2F7.2,F7.3,F7.1,14X,
1          F6.1,F6.2,F6.0,F6.3,F6.0,E9.2)
2936 FORMAT(2X,I2,1X,I2,2F7.0,F7.3,3F7.1,2F7.2,F7.3,F7.1,F7.0,F7.2,
1          F6.1,F6.2,F6.0,F6.3,F6.0,E9.2)
2937 FORMAT(2X,I2,1X,I2, F7.0,14X,F7.1,14X,2F7.2,F7.3,F7.1,F7.0,
1          F7.0,          18X,12X,E9.2)
2938 FORMAT(2X,I2,1X,I2, F7.0,14X,F7.1,14X,2F7.2,F7.3,F7.1,
1          22X,12X,E9.2)
2940 FORMAT(1H ,F5.1,
1          1X,F6.2,1X,F5.0,1X,F6.0,1X,F6.2,2X,F8.3,
1          F11.3, E11.3 ,1X,F6.2,1X,F7.2)
9100 CONTINUE
      LTNFCO=LINFST
2010 IF(PMAX-PCYL(2)) 7000,7030,7030
7000 PMAX=PCYL(2)
      PMAXAN=ANGLEF1
7030 CONTINUE
C
C IF THE TEMPERATURES OF EVERY ELEMENTS ARE BELOW TEMLOW(AROUND 1700K).
C THERE WILL BE NO CHANGE IN NO VALUE.
C WE WILL CONSIDER ONLY ONE AVERAGE ELEMENT.
      TEMLOW=2200.
      IF(JCOND.NE.4) GO TO 7300
      AAN=ANGLE1-(AFUELE+30.)
      IF(AAN.LT.0.) GO TO 7300
      IF(IE.FO.1) GO TO 7300
      IF(TEMP(2,2).GT.TEMLOW) GO TO 7300
      IF(TEMP(2,IE).GT.TEMLOW) GO TO 7300
C
C
C CALCULATE THE TOTAL MASS AND ENTHALPY
      TENTH=0.
      TMASS=0.
      DO 7320 I=1,IF
      TENTH=TENTH+FEENTH(2,I)*FMASS(2,I)
      TMASS=TMASS+EMASS(2,I)
7320 CONTINUE
C
C CALCULATE THE AVERAGE ENTHALPY
      AVENT=TENTH/TMASS
C
C CALCULATE THE AVERAGE TEMPERATURE
C FIRST FIND OUT THE ELEMENT WHICH ENTHALPY IS CLOSE TO THE
C AVERAGE ENTHALPY
      DEFTM=1000.
      IT=1
      DO 7330 I=1,IF
      DEFT=ABS(AVENT-FEENTH(2,I))
      IF(DEFT.LT.DEFT) GO TO 7330

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      DENTM=DENT
      IT=1
7330 CONTINUE
C
C DEFINE THE NEW PROPERTIES
      FTEMP(2,1)=FTEMP(2,IT)+(AVENT-FFNTH(2,IT))/FCP(IT)
      FEQUI(2,1)=PHIN
      FFNTH(2,1)=AVFNT
      FMASS(2,1)=TMASS
      CALL HPROD(PCYL(2),FTEMP(2,1),FEQUI(2,1),DFL,PST,ENTHLP,CSURP,
1 CSURT,RHO,DRHODT,DRHODP,EMWT(1),FRGAS(1),CVBG,FBGR)
      FCP(1)=FRGAS(1)/41.447+CVBG
      EVOL(2,1)=VCYLN
      IF=1
      WRITE(LPT,7340)
      WRITE(LPT,7342) PCYL(2),FTEMP(2,1),FEQUI(2,1),FFNTH(2,1),
1 FMASS(2,1)
      IF(NOCUL.EQ.0) GO TO 7331
C
C NOX AMOUNT WILL BE DEFINED.
C EXHAUST NOX ON MASS BASE
      ANOXEX=TWNQ*ITGINT/TMASS*RPW/120.
      WRITE(LPT,7450) TPRM,ANOXEX
7331 CONTINUE
      WRITE(LPT,7420)
      WRITE(LPT,7430)
7340 FORMAT(/,10X,***EXPANSION PROCESS IS CALCULATED AS ONE ELEMENT**
1*)
7342 FORMAT(15X,18HPRESSURE          = ,F10.2,'ATA',/,
1 15X,18HAV. TEMPERATURE           = ,F10.1,'K DEG',/,
2 15X,18HAV. EQUI. RATIO           = ,F10.3,/,
3 15X,18HAV. ENTHALPY              = ,F10.2,'CAL/G',/,
4 15X,18HTOTAL MASS                = ,F10.4,'G')
7420 FORMAT(/2X,4HTHEO,3X,4H P ,3X,4H T ,3X,4H V ,3X,4HWORK.
1 3X,7HCYL.MAS,3.  H HEAT TRANSFER,7X,4HFSWR,3X,4HSP)
7430 FORMAT(2X ,4HDFG ,3X,4HATA ,3X,4H K ,2X,5HCM**3,3X,4H PS .
1 3X,7H G ,3X,15H CAL/A CAL ,11X,3X,4H*ROM)
7450 FORMAT(/,10X,***EXHAUST NOX***,/,
1 15X,15HNOX CONCEN. = ,F10.1,'PPM',/,
2 15X,15HTOTAL MASS = ,E12.3,'G')
7300 CONTINUE
      IF(ANGLE1-ALAST) 7040,7050,7050
7040 GO TO 3000
7050 CONTINUE
      ITFXT=1
      QMANI=TTQEXT/TTGEXT
      TEXT=700.
7240 QQ=ENT(TEXT,FEQUI(2,1),SAFC)-QMANI
      PQ=QQ/QMANI
      IF(ABS(PQ)-0.003) 7220,7220,7230
7230 IF(ITFXT-2) 7200,7210,7210
7200 ITFXT=2
      XROM(1)=700.
      YROM(1)=PQ
      TEXT=800.
      GO TO 7240
7210 CALL CONT(TEXT,PQ,0.,0.,XROM,YROM,ITFXT)
      ITFXT=ITFXT+1
      IF(ITFXT-20) 7240,7220,7220
7220 CONTINUE

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ITFL=ITFL+1
WRITE(LPT,9090) TEXT
GO TO 7050
7070 CONTINUE
IF (ITFL-LITEL) 7260,7260,7040
7260 ANGLE1=ANGLE1-720.
GO TO 100
7040 CONTINUE
VSTROK=RORF*BORE*STROKE/4.*3.1416
CALL FRICT
WRITE(LPT,9092) FMFP(2),FMFP(3),FMFP(4),FMFP(5),FMFP(6),
1 FMFP(7),FMFP(8),FMFP(1)
AIMFP=PS/VSTROK*7500000./(RPM/120.)
PIMFP=PSIN/VSTROK*7500000./(RPM/120.)
PFMFP=PSFX/VSTROK*7500000./(RPM/120.)
RMFP=AIMFP-FMFP(1)
RHP=RMFP*VSTROK/7500000.*(RPM/120.)
AISFC= FUEL*3600./PS*PPM/120.
PSFC= FUEL*3600./RHP*PPM/120.
WRITE(LPT,9000) PIMFP,PSIN
WRITE(LPT,9010) PFMFP,PSFX
WRITE(LPT,9020) AIMFP,PS,AISFC
WRITE(LPT,9030) RMFP,RHP,PSFC
RHERE=TTQHTR(1)/(FIUFL*(FKCAL-VACAL))
PERMI=CYLN*PPM/2.
DO 17 I=1,NDIVID
17 TTQHTR(I)=TTQHTR(I)*PERMI
WRITE(LPT,9055)
WRITE(LPT,9056) (TTQHTR(I),I=1,NDIVID),RHERE
WRITE(LPT,9070) PMAX,PMAXAN
23 FORMAT(9F10.0)
301 FORMAT(I2)
302 FORMAT(I2,F10.0)
9000 FORMAT(///,10X,'PIMFP=',F8.2,'G/CM**2',5X,'PSIN=',F8.2,'HP')
9010 FORMAT(1H ,10X,'PFMFP=',F8.2,'G/CM**2',5X,'PSFX=',F8.2,'HP')
9020 FORMAT(1H ,10X,'AIMFP=',F8.0,'G/CM**2',5X,'IHP=',F8.2,'HP',
1 5X,'AISFC=',F8.2,'G/HP.HR')
9030 FORMAT(1H ,10X,'RMFP=',F8.0,'G/CM**2',5X,'RHP=',F8.2,'HP',
1 5X,'PSFC=',F8.2,'G/HP.HR')
9055 FORMAT(1H ,10X,'HEAT REJECTION= CAL/MIN')
9056 FORMAT(1H , 9X,15HTOTAL H. REJ = ,F10.4,/,
1 10X,15HCYL. LINER =,F10.4,/,
2 10X,15HPISTON TOP =,F10.4,/,
3 10X,15HPISTON CUP =,F10.4,/,
4 10X,15HCYL. HEAD =,F10.4,/,
5 10X,15HINT. VALVE =,F10.4,/,
6 10X,15HFXH. VALVE =,F10.4,/,
7 7X,18HHEAT REJ./FUFL =,F10.3)
9070 FORMAT(1H ,10X,'MAX CYL PRESS=',F10.1,'AT',F5.1,'DEG. CA')
9090 FORMAT(1H ,10X,'EXHAUST TEMPERATURE=',F8.1,'K(°F)')
9092 FORMAT(///,10X,'PREDICTION OF THE FRICTION LOSS**',/,
1 5X,16HISC. AND PUMPS = , F12.3,7HG/CM**2,/,
1 5X,16HCAM GEAR = , F12.3,7HG/CM**2,/,
1 5X,16HREADING = , F12.3,7HG/CM**2,/,
1 5X,16HRLD. BY = , F12.3,7HG/CM**2,/,
1 5X,16HVISCOUS PISTON = , F12.3,7HG/CM**2,/,
1 5X,16HPING TENSION = , F12.3,7HG/CM**2,/,
1 5X,16HPING GAS PRESS = , F12.3,7HG/CM**2,/,
14X,1-----,/,
1 5X,16HTOTAL LOSS = , F12.3,7HG/CM**2)

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```
4001 FORMAT(2X,2I2,7E12.4)
4010 FORMAT(//,F10.1)
4000 FORMAT(2X,2I2,9E12.4)
4050 FORMAT(2E12.4)
4080 FORMAT(6F10.2,2F10.0,3F10.4)
9000 READ(INP,301) NEXT0
      IF(NEXT0.EQ.1) GO TO 9001
      CALL EXIT
      END
```

SUBROUTINE DCAL

```

C
C
C CALCULATION OF HEAT RELEASE FOR EACH ELEMENT
COMMON/EPROP/ IFCHE(100),EFJEL(100),EANGL(100),FMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DOEMAS(100),
1 ERGAS(100),ECP(100),FENTH(8,100),FMASS(8,100),FEQUI(8,100),
1 FVOL(8,100),FTEMP(8,100),PCYL(8)
COMMON /STATE/ ANGLE1,ANGLE,COND,ASTEP,NSTEP,IE
COMMON /STAT1/ RPM,JLAST,NJET,PHIO,EGR
COMMON /FUFLP/ FJEL,FYCAL,Z4,ZN,PSI,SAFC,VACAL
COMMON /JETS/ RJET(50),HJET(50),AJET(50),BJET(50),THER(50),
1 VRJET(50),VZJET(50),WJET(50),VJET(50),AINJ(50),RHJET(50),
2 PITCH(50)
COMMON /PLUME/ PMASS(50),HMASS(50),RPLUM(50),HRMASS(50)
COMMON /JETDAT/ WSWIRD,ALPHA,BETA,RHGAS
COMMON/COMP/ RPLUM0,ENTUR,RLFSP,ASPARK,TSCALE
DFL=ZN/ZM
IF(IE.EQ.1) GO TO 300
DO 100 I=2,IE
AJE=AJET(I)*180./3.1416
C HEAT RELEASE = 0 UNLESS SPECIFIED LATER
DHFUEL(I)=0.
C
C CHECK WHETHER THE FUEL ELEMENT REACHES THE SPARK PLUG OR NOT.
IF(AJE.LT.ASPARK) GO TO 100
IF(PMASS(I).GT.1.E-7) GO TO 400
C
C SET THE INITIAL PLUME MASS AND GURNED MASS
RPLUM(I)=RPLUM0
PMASS(I)=RHJET(I)*4./3.*3.1416*RPLUM0**3
C
C SCALE FACTOR
PMASS(I)=PITCH(I)/2.*PMASS(I)
BMASS(I)=PMASS(I)**2./3.
IFCHE(I)=2
400 DO 410 IP=1,10
C
C TURBULENT ENTRAINMENT RATE
UP=FENTUR*VJET(I)
DPMASS= SQRT(RHJET(I)*RHJET(I))*4.*3.1416*RPLUM(I)**2*UP
DPMASS=DPMASS*ASTEP/(6.*RPM)/10.
C
C SCALE FACTOR
DPMASS=DPMASS*PITCH(I)/2.
C
C EFFECT OF FLAME COMING BACK
IF(IECHE(2).NE.3) GO TO 402
DPMASS=DPMASS*2.
402 CONTINUE
PMASS(I)=PMASS(I)+DPMASS
IF(PMASS(I).LT.FMASS(2,I)) GO TO 420
PMASS(I)=FMASS(2,I)
IFCHE(I)=3
420 TCHA=TSCALE/RLFSP
DPMASS=(PMASS(I)-BMASS(I))/TCHA*ASTEP/(6.*RPM)/10.
C
BMASS(I)=BMASS(I)+DPMASS
C NEXT EQUATION INCLUDES THE SCALE FACTOR ABOUT CALCULATION PITCH
RPLUM(I)=(PMASS(I)*3./2/PITCH(I)/(RHJET(I)*4.*3.1416))**0.333

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410 CONTINUE
   CALL UPROP(PCYL(1),FTEMP(1,I),FFQUI(2,I),DEL,PSI,0.,ENTAIR,
  1 DUM,DUM,DUM,DUM,DUM,DUM,DUM,DUM)
   CALL HPROD(PCYL(1),FTEMP(1,I),FFQUI(2,I),DEL,PSI,FNTGAS,
  1 DUM,DUM,DUM,DUM,DUM,DUM,DUM,DUM)

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C

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C INCLUDING THE FORMATION ENTHALPY
  XX=(ENTAIR-FNTGAS)*1000./FKCAL/FEQUI(2,I)*(SAFC+FEQUI(2,I))
  IF (XX-1.) 430,440,440

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440 XX=1.
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430 HRMASO=HRMASS(I)
   HRMASS(I)=FFUEL(I)*RMASS(I)/EMASS(2,I)*XX
   DHFUEL(I)=(HRMASS(I)-HRMASO)*FKCAL/ASTEP
   TRMASS=TRMASS+HRMASS(I)
   TORMAS=TORMAS+ORMASS

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```
100 CONTINUE
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300 DHFUEL(1)=0.
```

```
RETURN
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```
END
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SUBROUTINE INPIIT
COMMON/EPROP/ IECHF(100),EFJEL(100),EANGL(100),EMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DOEMAS(100),
1 ERGAS(100),ECP(100),FENTH(8,100),FMASS(8,100),FEQUI(A,100),
1 EVOL(A,100),FTEMP(8,100),PCYL(8)
COMMON /STAT1/ RPM, JLAST, NJET, PHIO, EGR
COMMON /TIMIG/ AINTO, AFXTO, AINTC, AFXTC, AFUELS, AFUELE
COMMON /GFDMT/ STROKE, BORE, CONL, VCLFAR, VCUP, RCUP, CYLN
COMMON /HETRS/ NDIVID, TMETAL(10), QHTRC(10), ARE(10)
COMMON /STATE/ ANGLE1, ANGLE, JCOND, ASTEP, NSTEP, IF
COMMON /FUFLP/ FUEL, FKCAL, ZM, ZN, PST, SAFC, VACAL
COMMON /JETS/ RJET(50), HJET(50), AJET(50), BJET(50), THER(50),
1 VRJET(50), VZJET(50), WJET(50), VJET(50), AINJ(50), RHJET(50),
2 PITCH(50)
COMMON /PLUME/ PMASS(50), BMASS(50), RPLUM(50), HRMASS(50)
COMMON /JETDAT/ WSWIR0, ALPHA, BETA, RHGAS
COMMON /SURGAS/ FSWIR, WSWIR, SDFCAY
COMMON /COMR/ RPLUM0, FNTIR, RLFSP, ASPARK, TSCALE
COMMON /OSQPE/ LPT, JNP, LINECO, LINEST, ITEL, LJTEL
REAL*4 MBAR
COMMON /FUFL/ AF(6), FNW, CX, HY, OZ, QLOWER
COMMON /OXDANT/ XI /CMPSTN/ X(7), MBAR
COMMON /CONTL/ ASTIN, ASTCL, ASTCR, ASTEX, ASTOV, PRININ, PRINCL, PRINEX,
1 PRINOV
COMMON /INFI O/ NINEF, XINEF(20), YINEF(20), AREIND, PINT, TINT, GINT,
1 EQUINT, ARFINT, XINV
COMMON /FXFLO/ NEXEF, XEXEF(20), YEXEF(20), AREEXD, PEXT, TEXT, GEXT,
1 IGLEFT, TLFFT, QEXT, QFLOW, ELEFT, AREEXT, XEXV
COMMON /AMRJE/ TAMR, PAMR
COMMON /HETR1/ PRASE, TRASE, VRASE, COHT1, COHT2, COHT, -HR(10)
COMMON /FRIDT/ RSKIRT, FCDE, RINGN, FMEP(10)
DIMENSION VINLI(20), XCOEI(20), YCOFI(20), YEXLI(20), XCOFE(20),
1 YCOEE(20)
JNP=5
LPT=6
RFSFRK=0.
DFL=ZN/ZM
WRITE(LPT,3000)
3000 FORMAT(1H1,2X,1**CYCLE SIMULATION FOR DIRECT INJECTION ENGINE**
1VERSION 1***)
READ(INP,3010)
WRITE(LPT,3010)
3010 FORMAT(80H
1
WRITE(LPT,3020)
3020 FORMAT(1H ,//,1H ,33X,1TABLE OF INPUT DATA AND INITIAL ENGINE COND
1ITIONS',/,5X,120(1H*))
C
C READ AND WRITE THE DATA OF ENGINE GEOMETRY
C
C STROKE=STROKE(CM)
C CONL=CONNECTING ROD(CM)
C VCLFAR=CLEARANCE VOLUME ( EXCLUDING CUP VOLUME,CM**3)
C VCUP=CUP VOLUME ( CM**3)
C RCUP= CUP RADIUS (CM)
C CYLN= NUMBER OF CYLINDER
READ(INP,23) STROKE,BORE,CONL,VCLFAR,VCUP,RCUP,CYLN
C
C READ AND WRITE THE DATA OF VALVE AND FUEL INJECTION TIMING
C

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C  AINTO=INTAKE VALVE STARTS TO OPEN (CRANK DEGREE, AROUND -10)
C  AEXTO=EXHAUST VALVE STARTS TO OPEN (AROUND 50)
C  AINTC=INTAKE VALVE STARTS TO CLOSE (AROUND 200)
C  AEXTC=EXHAUST VALVE STARTS TO CLOSE (AROUND 10)
C  AFUELS=FUEL INJECTION STARTS (AROUND 330)
C  AFUELE= FUEL INJECTION ENDS (AROUND 355)
C  READ(INP,23) AINTO,AEXTO,AINTC,AEXTC,AFUELS,AFUELE

C
C  READ AND WRITE FUEL PROPERTIES
C
C  ZM= CHEMICAL FORMULA, NUMBER OF H
C  ZN= CHEMICAL FORMULA, NUMBER OF C
C  SAFC= CHEMICALLY CORRECT AIR-FUEL RATIO
C  FKCAL= LOWER HEATING VALUE OF THE FUEL
C  VACAL=LATENT HEAT VALUE (CAL/G)
C  READ(INP,23) ZM,ZN,SAFC,FKCAL,VACAL
C  CX=ZN
C  HY=ZM
C  QLOWER=FKCAL/1000.
C  AF(1)=-0.553
C  AF(2)=182.
C  AF(3)=-97.8
C  AF(4)=20.4
C  AF(5)=-0.0309
C  AF(6)=-60.5
C  XI=3.76
C  ENW=0.
C  O7=0.

C
C  READ AND WRITE THE DATA OF THE ENGINE OPERATING CONDITION
C
C  FUEL=TOTAL FUEL AMOUNT/ 1 SHOT (GRAM)
C  RPM=ENGINE SPEED (RPM)
C  EGR=EXHAUST GAS RECIRCULATION
C  READ(INP,23) FUEL,RPM,EGR

C
C  HOW MANY REGIONS WILL YOU TAKE FOR HEAT TRANSFER AREA?(NDIVID=6)
C  NDIVID=6

C
C  ELEMENT 1 IS FOR TOTAL
C  NDIVID=NDIVID+1

C
C  READ AND WRITE THE DATA OF THE TEMPERATURE INSIDE ENGINE
C
C  TP= TEMPERATURE OF PISTON TOP
C  TINT= TEMPERATURE OF INTAKE VALVE
C  TEXV= TEMPERATURE OF EXHAUST VALVE
C  TW= TEMPERATURE OF CYLINDER WALL
C  TCYLH= TEMPERATURE OF CYLINDER HEAD
C  READ(INP,23) TP,TINT,TEXV,TW,TCYLH
C  TMETAL(2)=TW
C  TMETAL(3)=TP
C  TMETAL(4)=TP
C  TMETAL(5)=TCYLH
C  TMETAL(6)=TINT
C  TMETAL(7)=TEXV

C
C  CALCULATIONS FOR THE DISTANCES FROM THE CENTER POINT
C  THESE DATA WILL BE USED TO CALCULATE THE VELOCITIES AT THE WALL
C  RHP(2)=RORF/2.
C  RHP(3)=(RORF/2.+RCUP)/2.

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PHR(4)=RCUP
PHR(5)=RORF/(2.*1.414)
PHR(6)=RORF/4.
PHR(7)=RORF/4.
C
C HEAT TRASFER COEFFICIENTS
C
C COHT1,COHT2= HEAT TRANSFER COEFFICIENT
C COHT1=2.28
C COHT2=0.0072
C COHT=ADJUSTING TOTAL HEAT TRANSFER AMOUNT(MAYBE 1)
C READ(INP,23) COHT1,COHT2,COHT
C
C READ AND WRITE THE DATA OF INITIAL JET CONDITIONS
C
C AINJV=JET DIRECTION ANGLE FROM VERTICAL LINE (DEGREE)
C AINJT=JET DIRECTION ANGLE FROM TANGENTIAL LINE (DEGREE)
C RINIT= JET RADIAL LOCATION WHEN FUEL IS INJECTED (CM)
C RINIT= JET INITIAL RADIUS (CM)
C READ(INP,23) AINJV,AINJT,RINIT,RINIT
C AINJV= AINJV*3.1416/180.
C AINJT= AINJT*3.1416/180.
C
C READ AND WRITE DATA OF AIR ENTRAINMENT PARAMETER
C
C WSRATO= SWIRL RATIO AT ROC (*RPM)
C ALPHA= ENTRAINMENT PARAMETER FOR PARALLEL VELOCITY DIFFERENCE
C (0.11 IS RECOMMENDED)
C BETA= ENTRAINMENT PARAMETER FOR NORMAL VELOCITY DIFFERENCE
C (BETA=0.6 IS RECOMMENDED)
C RHFUEL= DENSITY OF FUEL (GRAM/CM**3)
C RHGAS= DENSITY OF SURROUNDING GAS (GRAM/CM**3)
C SDECAY = SWIRL DECAYING FACTOR (6.5E-6 IS RECOMMENDED FOR
C TCCS ENGINE)
C READ(INP,23) WSRATO,ALPHA,BETA,RHFUEL,RHGAS,SDECAY
C WSWIR0=3.1416/180.*RPM*6.*WSRATO
C
C READ DATA OF COMBUSTION PARAMETER
C
C RPLUM0= INTIAL PLUME RADIUS(CM)
C ENTUR= TURBULENCE ENTRAINMENT PARAMETER
C RLFSF= LAMINAR FLAME SPEED(CM/SFC)
C ASPARK=SPARK PLUG LOCATION FROM FUEL INJECTION NOZZLE(DEG)
C READ(INP,23) RPLUM0,ENTUR,RLFSF,ASPARK
C
C
C DATA OF INTAKE AND EXHAUST SYSTEM
C DINV= DIAMETER OF INTAKE VALVE
C SINV=MAXIMUM LIFT OF INTAKE VALVE
C XINV=NUMBER OF INTAKE VALVE
C DEXV= DIAMETER OF EXHAUST VALVE
C SEXV= MAXMUM LIFT OF EXHAUST VALVE
C XEXV=NUMBER OF EXHAUST VALVE
C READ(INP,23) DINV,SINV,DEXV,SEXV,XINV,XEXV
C APEIN0=3.1416*DINV*SINV*XINV
C APFEX0=3.1416*DEXV*SEXV*XEXV
C APE(3)=3.1416/4.*(RORF*RORF-RCUP*RCUP)*4.)
C APE(4)=VCUP*2./RCUP
C APE(6)=3.1416/4.*DINV*DINV
C APE(7)=3.1416/4.*DEXV*DEXV

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ARE(5)=3.1416/4.*BORE*BORE-ARE(6)-ARE(7)
C
C
C DATA OF FRICTIONS
C RSKIRT= EQUIVALENT LENGTH OF PISTON SKIRT( PROJECTED ARE OF SKIRT
C / CYLINDER BORE DIAMETER)
C FCOE= JOURNAL BEARING SIZE COEFFICIENT
C RINGN= TOTAL NUMBER OF PISTON RINGS PER CYLINDER
C READ(INP,23) RSKIRT,FCOE,RINGN
C READ DATA OF VALVE LIFT VS. NORMALIZED CRANK ANGLE FOR INTAKE VALVE
C READ(INP,301) NINEF
C READ(INP,301) NINCO
C READ(INP,23) (XINEF(I),I=1,NINEF)
C READ(INP,23) (YINLI(I),I=1,NINEF)
C READ(INP,23) (XCOEI(I),I=1,NINCO)
C READ(INP,23) (YCOEI(I),I=1,NINCO)
C DO 1000 I=1,NINEF
C VLIFR=SINV*YINLI(I)/DINV
C CALL TWODIM(NINCO,XCOEI,YCOEI,VLIFR,COEF)
C YINEF(I)=YINLI(I)*COEF
1000 CONTINUE
C READ DATA OF VALVE LIFT VS. NORMALIZED CRANK ANGLE FOR EXHAUST VALVE
C READ(INP,301) NEXEF
C READ(INP,301) NEXCO
C READ(INP,23) (XEXEF(I),I=1,NEXEF)
C READ(INP,23) (YEXLI(I),I=1,NEXEF)
C READ(INP,23) (XCOEE(I),I=1,NEXCO)
C READ(INP,23) (YCOEE(I),I=1,NEXCO)
C DO 1010 I=1,NEXEF
C VLIFR=SEXV*YEXLI(I)/DEXV
C CALL TWODIM(NEXCO,XCOEE,YCOEE,VLIFR,COEF)
C YEXEF(I)=YEXLI(I)*COEF
1010 CONTINUE
C
C HOW MANY ELEMENTS DO YOU WANT TO DIVIDE THE INJECTED FUEL INTO ?
C 10 ELEMENTS ARE RECOMMENDED
C READ(INP,301) NJET
C
C AMBIENT CONDITION
C TAM3= AMBIENT TEMPERATURE
C PAMB= AMBIENT PRESSURE
C READ(INP,23) TAM3,PAMB
C
C INTAKE AND EXHAUST PRESSURE
C PINT=INTAKE PRESSURE (ATA)
C PEXT=EXHAUST PRESSURE (ATA)
C READ(INP,23) PINT,PEXT,TINT
C
C COMPUTING INCREMENT
C READ(INP,23) ASTIN,ASTCL,ASTCR,ASTEX,ASTOV
C
C PRINT CONTROL DATA
C READ(INP,23) PRININ,PRINCL,PRINEX,PRINOV
C
C INITIAL ENGINE CONDITION
C READ(INP,23) PCYL(2),FTEMP(2,1),FEQU(2,1),TEXT
C ELEFT=FEQU(2,1)
C TLEFT=ETEMP(2,1)
C
C COMPRESSION RATIO

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DISP=3.1416*BORE*ROPE/4.*STROKE
CR=(VCLEAR+VCUP+DISP)/(VCUP+VCLEAR)
C
C TAYLOR MICRO SCALE
TSCALE=SINV/CP*0.17
VINIT=FUFL/RHFUEL/(3.1416*BINIT**2*(AFUELE-AFUELS))*(6.*RPM)
WRITE(6,5200)
WRITE(6,5210) VINIT
5200 FORMAT(2X,'INITIAL JET SPEED')
5210 FORMAT(10X,'VINIT=',F12.1,'CM/SFC')
C
C SET THE INITIAL CONDITIONS FOR ALL JET ELEMENTS
ASTCB=(AFUELE-AFUELS)/FLOAT(NJET)
ASTEP=ASTCB
NNJET=NJET+1
DO 100 I=2,NNJET
  BJFT(I)=BINIT
  THET(I)=0.
  VJET(I)=VINIT
  WJET(I)=VINIT*SIN(AINJV)*COS(AINJT)/RINIT
  VRJET(I)=-VINIT*SIN(AINJV)*SIN(AINJT)
  VZJET(I)=VINIT*COS(AINJV)
  RJFT(I)=RINIT
  HJET(I)=0.
  AJET(I)=0.
  RHJET(I)=RHFUEL
  RPLUM(I)=RPLUM0
  PMASS(I)=0.
  BMASS(I)=0.
  HRMASS(I)=0.
  AINJ(I)=AFUELS+FLOAT(I-2)*ASTEP
C NEXT ASSUMPTION( FFOUJI=3. ) WAS MADE FOR GETTING APPROPRIATE
C THERMODYNAMIC PROPERTIES
FFOUI(2,I)=3.
DD=AFUELE-((FLOAT(I)-1.)*ASTEP+AFUELS)
IF(DD.LT.0.) GO TO 90
EMASS(2,I)=3.1416*BINIT**2.*VINIT*ASTEP/(6.*RPM)*RHFUEL
EFUEL(I)=EMASS(2,I)
PITCH(I)=ASTEP
GO TO 100
90 EMASS(2,I)=3.1416*BINIT**2.*VINIT*(AFUELE-((FLOAT(I)-2.)*ASTEP+
1 AFUELS))/(6.*RPM)*RHFUEL*ASTEP
EFUEL(I)=EMASS(2,I)
PITCH(I)=AFUELE-((FLOAT(I)-2.)*ASTEP+AFUELS)
GO TO 110
100 CONTINUE
110 CONTINUE
C
C CALCULATION OF INTAKE AIR PROPERTIES
EQUINT=EGR*FELEFT*SAFC/((1.-EGR)*(ELEFT+SAFC)+EGR*SAFC)
WRITE(LPT,7020)
WRITE(LPT,7010) STROKE,AINT0,7M,TP,
1 CONL,AFXTO,ZV,TW,
2 ROPE,AINTC,SAFC,TCYLT,
3 VCLFAP,AEXTC,FKCAL,TINV
WRITE(LPT,7011) VCUP,AFUELS,VACAL,TEXV,
5 AFUELF,RHFUEL
WRITE(LPT,7020)
AINVV=AINJV*180./3.1416
AINTT=AINJT*180./3.1416

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WRITE(LPT,7030) WSRAT0,PPLUM0.
1   AINVV,ENTUR.
1   AINTT,ALPHA,RLFSP.
3   RINIT,9FTA,ASPAK.
4   RINIT,RHGAS
WRITE(LPT,7031) RCIP,SDFCAY,TSCALE
WRITE(LPT,7020)
WRITE(LPT,7040) RPM,PIINT,ASTOV,PRINDV.
1   FUEL,TINT,ASTIN,PRININ.
2   EGR,EQUINT,ASTCL,PRINCL
WRITE(LPT,7041) ASTCB,PRINCL.
1   PAMB,PEXT,ASTCL,PRINCL.
2   TAMP,TEXT,ASTEX,PRINEX.
3   ELEFT
WRITE(LPT,7020)
WRITE(LPT,7050) DINV,ARF(6),PCYL(2).
1   SINV,ARF(7),COHT,FTEMP(2,1).
2   XINV,ARF(3),COHT1,FFQUI(2,1)
WRITE(LPT,7051) ARE(4),COHT2,TEXT.
4   DEXV,ARF(5).
5   SEXV.
6   XEXV
WRITE(LPT,7020)
WRITE(LPT,7060) RSKIRT.
1   FCOE.
2   RINGN
WRITE(LPT,7020)
WRITE(LPT,7100)
WRITE(LPT,7110)
WRITE(LPT,7000) (XINEF(I),I=1,NINEF)
WRITE(LPT,7120)
WRITE(LPT,7000) (YINEF(I),I=1,NINEF)
WRITE(LPT,7140)
WRITE(LPT,7110)
WRITE(LPT,7000) (XEXEF(I),I=1,NEXEF)
WRITE(LPT,7120)
WRITE(LPT,7000) (YEXEF(I),I=1,NEXEF)
RETURN
23  FORMAT(8F10.0)
6020 FORMAT(5E12.4)
6030 FORMAT(6E12.4)
6040 FORMAT(4E12.4)
6050 FORMAT(2E12.4)
6060 FORMAT(I2)
6070 FORMAT(10E12.4)
301  FORMAT(I2)
7000 FORMAT(10X,10F10.3)
7010 FORMAT(32H          ENGINE GEOMETRY          *,
1     32H          TIMING                          *,
2     36H          FUEL PROPERTIES                 *,
3     30H          ESTIMATE TEMPERATURES          /*,
4 2X,12HSTROKE      =,F10.3,84CM          *,
5 17H INT.VAL.OPEN  =,F9.1,64DEG          *,
6 19H 7N(N0. OF H)  =,F10.1,7H           /*,
7 18H PISTON TOP(TP) =,F10.1,1HK,/.
8 2X,12HC0N.ROD     =,F10.2,84CM          *,
9 17H EXH.VAL.OPEN  =,F9.1,64DEG          *,
1 19H 7N(N0. OF C)  =,F10.1,7H           /*,
2 18H CYL.WALL(TW)  =,F10.1,1HK,/.
3 2X,12HRORE        =,F10.3,84CM          *.

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4 17H INT.VAL.CLOSF =, F9.1.6HDEG *,
5 19H SAFC(STOTCH.A/F)=, F10.1.7H *,
6 18H CYL.HEAD(TCYLH)=, F10.1.1HK,/,
7 2X.12HCLEAR.VOL. =, F10.2.8HCM**3 *,
8 17H EXH.VAL.CLOSF =, F9.1.6HDEG *,
9 19H LOWER HEATING =, F10.0.7H CAL/G*,
5 18H INT.WALVF(TINV)=, F10.1.1HK)
7011 FORMAT(2X,
2 12HCUP VOL. =, F10.2.8HCM**3 *,
3 17H FUEL INJ.START=F9.1.6HDEG *,
4 19H VAPOIZING HEAT =, F10.0.7H CAL/G*,
5 18H EXH.VALVE(TEXV)=,F10.1.1HK,/,
6 32X,
7 17H FUEL INJ.END =, F9.1.6HDEG *,
8 19H DENSITY =, F9.3.7HG/CM**3 .1H*)
7020 FORMAT(2X,128H-----)
)-----
2--- )
7030 FORMAT(42H INITIAL JET DATA *,
1 44H JET MIXING PARAMETER *,
2 43H PLUME MODEL ,/,
3 2X,20HJET DIRECTION** ,19X,1H*,
4 27H SWIRL RATIO AT PDC =,F10.3.7H**RPM *,
5 23H INITIAL PLUME RADIIJS =,F10.4.7H CM *,/,
6 2X,20H ANGLE(VERTICAL) =,F10.2.10H DEG *,
7 27H AIR ENTRAIN. PARAMETER , 16X,1H*,
8 23H ENTRAIN. PARAMETER =, F10.3.9X,1H*,/,
9 2X,20H ANGLE(TANGENTIAL)= .F10.2.10H DEG *,
1 27H ALPHA =, F10.3.6X,1H*,
1 23H LAM. FLAME SPEED =, F10.2.10H CM/SEC *,/,
2 22H JET RADIUS =,F10.5.10H CM *,
3 27H BETA =, F10.3.6X,1H*,
4 23H SPARK PLUG LOCATION =, F10.2.10H DEG *,/,
5 22H JET LOCATION(R) =, F10.3.10H CM *,
6 27H DENSITY OF SURROUND. GAS =,F10.5.7H G/CC *,
7 8X,19H(FROM INJ. NOZZLE) )
7031 FORMAT(
8 22H PISTON CUP RADIUS =, F10.3.10H CM *,
9 27H SWIRL DECAY FACTOR =, E10.3.6X,1H*,
1 23H TAYLOR MICRO SCALE =, F10.4.10H CM )
7040 FORMAT(2X,31H ENGINE CONDITION *,
1 31H INTAKE CONDITIONS *,
2 31H COMPUTING INTERVALS *,
3 31H PRINT CONTROL ,/,
4 2X,15HENGINE SPEED =, F8.0.8H RPM *,
5 15H PRESSURE =, F8.3.8H ATA *,
6 15H VALVE OVERLAP=, F8.2. 8H DEG *,
6 15H VALVE OVERLAP=, F8.2.8X,/,
8 2X,15HFUEL AMOUNT =, F8.5. 8H G/SHOT*,
9 15H TEMPERATURE =, F8.2.8H K *,
9 15H INTAKE =, F8.2.8H DEG *,
1 15H INTAKE =, F8.2.8X,/,
2 2X,15HFCR =, F8.3.7X,1H*,
3 15H EQUI. RATIO =,F8.3.7X,1H*,
4 15H COMPRESSION =, F8.2.8H DEG *,
5 15H COMPRESSION =, F8.2 )
7041 FORMAT(2X,
6 31H AMBIENT CONDITIONS *,
7 31H EXHAUST CONDITIONS(ASSUMF) *,
8 15H COMBUCTION =, F8.2.8H DEG *,

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9 15H COMBUSTION      =. F8.2.8X./,
9 2X.15HPRESSURE     =. F8.2.8H ATA  *,
9 15H PRESSURE       =. F8.2.8H ATA  *,
2 15H EXPANSION      =. F8.2.8H DEG  *,
3 15H EXPANSION      =. F8.2.8X./,
42X.15HTEMPERATURE  =. F8.2.8H K    *,
5 15H TEMPERATURE   =. F8.2.8H K    *,
6 15H EXHAUST        =. F8.2.8H DEG  *,
7 15H EXHAUST        =. F8.2.8X./,
4 32X.1H*,
5 15H EQUI. RATIO   =. F8.3.7X.1H*)
7050 FORMAT(2X,31H      INTAKE VALVE      *,
1 31H      HEAT TRANS. AREA      *,
2 31H      HEAT TRANS. COEFF      *,
3 31H      INITIAL CONDITION      /,
4 2X.15HDIAETER      =. F9.3.7H CM  *,
5 15H INTAKE VALVE = .F9.3.7H CM**2*,
6 31H      (WOSCHNI'S EQ.) *,
7 15H CYL. PRESS.    =. F9.2.7H ATA  /,
8 2X.15HLIFT         =. F9.3.7H CM  *,
9 15H EXHAUST VALVE = .F9.3.7H CM**2*,
9 15H COHT           =. F9.5.6X.1H*,
1 15H CYL. TEMP.     =. F9.1.7H DEG  /,
2 2X.15HNUMPR        =. F9.1.6X.1H*,
3 15H PISTON TOP     =. F9.3.7H CM**2*,
4 15H COHT1          =. F9.5.6X.1H*,
5 15H CYL. EQUI.     = .F9.3)
7051 FORMAT(2X,31H      EXHAUST VALVE     *,
1 15H PISTON TOP     = .F9.3.7H CM**2*,
2 15H COHT2          =. F9.5.6X.1H*,
3 15H EXHAUST TEMP. =. F9.1.7H DEG  *,/,
4 2X.15HDIAETER     = .F9.3.7H CM  *,
5 15H CYL. HEAD      =. F9.3.7H CM**2*,/,
6 2X.15HLIFT         =. F9.3.7H CM  *,
7 15H CYL. WALL      =. 7X.9H VARIABLE* /,
8 2X.15HNUMPR        =. F9.1.6X.1H*)
7060 FORMAT(2X,31H      FRICTION LOSS     *,/,
1 2X.15HPISTON SKIRT = . F9.3.7H CM  *,/,
2 2X.15HJOU. BEAR. COE = , F9.3.6X.1H*,/,
3 2X.15HNO. OF RINGS = .F9.1.6X.1H*)
7100 FORMAT(1H1,/,2X,
1 *INTAKE VALVE--EFFECTIVE FLOW AREA VS. CRANK ANGLE*)
7110 FORMAT(5X,*NORMALIZED CRANK ANGLE*)
7120 FORMAT(5X,*NORMALIZED EFFECTIVE FLOW AREA*)
7140 FORMAT(/,2X,
1 *EXHAUST VALVE--EFFECTIVE FLOW AREA VS. CRANK ANGLE*)
END

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SUBROUTINE RUNGE(FNTINT)

THIS PROGRAM WAS MADE FOR OPERATING RUNGE-KUTTA METHOD

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COMMON/PROP/ JFCHE(100),EFUEL(100),EANGL(100),FMWT(100),
1 DFUEL(100),DHLOS(100),DIEMAS(100),DIEMAS(100),
1 FMGAS(100),ECP(100),FENTH(5,100),FMASS(8,100),EEDUT(8,100),
1 EVOL(8,100),FTEMP(8,100),PCYL(8)
COMMON /TIMIG/ AINTD,AFXTD,AINTC,AFXTC,AFUELS,AFUELE
COMMON /STATE/ ANGLE1,ANGLE,JCOND,ASTEP,NSTEP,IE
COMMON /STAT1/ RPM,ILAST,NJET,PHIO,FSR
COMMON /GEOM1/ STROKE,BORE,CONL,VCLEFR,VCUP,RCUP,CYLN
COMMON /METS1/ NOIVID,IMETAL(10),QHTRC(10),ARE(10)
COMMON /HETR1/ PRASE,THASE,VRASE,COHT1,COHT2,COHT,PHR(10)
COMMON /SUM1/ STGINT,STGEXT,STOEXT
COMMON /SUM2/ TTGINT,TTGEXT,TTOEXT,TTQHTR(10),TQHTR(10),PC
COMMON /FUFL1/ FUEL,FKCAL,ZM,ZN,PST,SAFC,VACAL
2FAL*4 M3A3
COMMON /FUFL/ AF(6),ENW,CX,HY,OZ,OLWER
COMMON/OXDANT/XT/CMPSTN/X(7),MRAR
COMMON /JETS/ PJET(50),HJET(50),AJFT(50),BJFT(50),THER(50),
1 VPJET(50),VZJFT(50),WJET(50),VJET(50),AINJ(50),RHJET(50),
2 PITCH(50)
COMMON /PLJME/ PMASS(50),BMASS(50),RPLUM(50),HRMASS(50)
COMMON /JETDAT/ VSWTR,ALPHA,BETA,PHGAS
COMMON /SUGAS/ FSWTR,WSWIR,SDFCAY
COMMON/COMP/ RPLUMD,ENTIR,RLFSP,ASPAK,TSCALE
COMMON/INFL0/ NINEF,XTNEF(20),YINEF(20),AREIND,PINT,TINT,GINT,
1 EQUINT,AREINT,XINV
COMMON/FAFL0/NFXEF,XEYEF(20),YFXEF(20),AREEXD,PEXT,TEXT,GEXT,
1 TGLEFT,TLFFT,DEXT,OFLOW,ELEET,AREEXT,XEXV
COMMON /NOYSP/ CONNO(100),PPMNO(100),WNO(100),SUMNO(100),PPMEXT
DIMENS10W SUMHT(30)
DEL=ZM/ZM
STGINT=0.
STGEXT=0.
STOEXT=0.
GINT=0.
GEXT=0.
DEXT=0.
RESERK=0.0
NSTEP=1
IFE=IE+1
DO 31 I=1,IFE
SUMHT(I)=0.
21 CONTINUE
DO 24 I=1,NOIVID
22 TQHTK(I)=0.
ANGLE=ANGLE1
30 IU(20,20,22,22,22),JCOND
22 CALL VGAS(0.001,0.001,RPM,ANGLE1,DUM,0JM,WSWIR,DUM,FSWTR)
FSWIR=FSWIR-SDFCAY*WSWIR*WSWIR*ASTEP/(6.*RPM)
30 IU 21
20 FSR)=1.0
FSWIR=.SWIR0
21 CONTINUE
IF(JCOND.EQ.4) G) TO 30
30 IU 40

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```

C CALCULATION OF JET MODEL IN ORDER TO GET THE ENTRAINMENT AIR
3) DOFMAS(1)=0.
C CHECK WHETHER NEW FUEL ELEMENT IS INJECTED OR NOT
  NUJET=NUJET+1
  IF(IE=NUJET) 1010,1020,1020
1010 AXZ=AFUELE+10.
  IF(ANGLE.GT.AXZ) GO TO 40
  IE=IE+1
  FTEMP(2,IE)=FTEMP(2,1)
  CALL UPRUP(PCYL(2),FTEMP(2,IE),FEQUI(2,IE),DEL,PSI,
  1 RESPRK,EFNTH(2,IE),CSURP,CSURT,RHO,DRHODT,DRHODP,CHI,
  2 FEGAS(IF),CVBG)
  FFNTH(2,IE)=EFNTH(2,IF)*1000.
  EMWT(IE)=MPAR
  FCP(IE)=FEGAS(IE)/4).447+CVBG
  EVOL(2,IE)=FMASS(2,IE)*FEGAS(IE)*FTEMP(2,IE)/PCYL(2)
1020 CONTINUE
  DO 32 I=2,IF
  FEQUI0=FEQUI(2,I)
  EMASS0=FMASS(2,I)
  IF(EMASS(2,1).LT.0.040) GO TO 35
  IF(EMASS(2,1).GT.DOFMAS(1)) GO TO 37
C
C ALL AIR HAS BEEN ENTRAINED BY JET
35 ALPHA=0.
  BETA=0.
  37 CONTINUE
  IF(HJET(1).LT.0.2) GO TO 220
  IF(RJET(1).LT.1.0) GO TO 222
  JSTEP=4
  GO TO 230
222 JSTEP=8
  GO TO 230
220 JSTEP=50
230 CALL JET(RJET(1),HJET(1),AJET(1),RJET(1),THER(1),VRJET(1),
  1 VZJET(1),WJET(1),TOM,EMASS(2,1),ASTEP,JSTEP,RRM,ANGLE,PITCH(1),
  1 FSWIP,MSWIP)
  VJET(1)=SQRT(VRJET(1)**2+VZJET(1)**2+(RJET(1)*WJET(1))**2)
  VOL=(3.1416*RJET(1)**2*(1.-THER(1))/(2.*3.1416))+3.1416*AJET(1)**2
  1 *SIN(THER(1)/2.)*VJET(1)*PITCH(1)/(6.*RRM)
  EFUJ(1)=FMASS(2,1)/VOL
  FEQUI(2,1)=FEFUEL(1)*(SAPC+EEQUI(2,1))/(EMASS(2,1)-EFUEL(1))
  1 +FEQUI(2,1)
  FEQUI(3,1)=(FEQUI(2,1)-FEQUI0)/ASTEP
  DIEMAS(1)=TOM/ASTEP
  SINDM(1)=TOM
  DOFMAS(1)=0.
  DOFMAS(1)=DOFMAS(1)+DIEMAS(1)
  FMASS(2,1)=EMASS0
  FEQUI(2,1)=FEQUI0
  32 CONTINUE
  SINDM(1)=-DOFMAS(1)*ASTEP
  DIEMAS(1)=0.
  FEQUI(3,1)=0.
33 CONTINUE
  PCYL(1)=PCYL(2)
  DO 100 I=1,IE
  FMASS(1,I)=FMASS(2,I)
  FEQUI(1,I)=FEQUI(2,I)
  EVOL(1,I)=EVOL(2,I)

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      ETEMP(1,I)=ETEMP(2,I)
      FENTH(1,I)=FENTH(2,I)
100 CONTINUE
      ANGLE=ANGLE+1
      CALL CALCQ(DV CYL,ENTINT)
      CALL DELTA(DV CYL,ENTINT)
      STGINT=STGINT+GINT/6.
      STGEXT=STGEXT+GEXT/6.
      STQEXT=STQEXT+QEXT/6.
      DO 120 I=1,NQIM1
120 TQHT(I)=TQHT(I)+QHTRC(I)/6.

      SECOND STEP

      NSTEP=NSTEP+1
      PCYL(4)=PCYL(3)*ASTEP
      PCYL(1)=PCYL(2)+PCYL(4)/2.
      DO 200 I=1,IE
      FMASS(4,I)=FMASS(3,I)*ASTEP
      FEQUI(4,I)=FEQUI(3,I)*ASTEP
      EVOL(4,I)=EVOL(3,I)*ASTEP
      ETEMP(4,I)=ETEMP(3,I)*ASTEP
      FENTH(4,I)=FENTH(3,I)*ASTEP
      FMASS(1,I)=FMASS(2,I)+FMASS(4,I)/2.
      FEQUI(1,I)=FEQUI(2,I)+FEQUI(4,I)/2.
      EVOL(1,I)=EVOL(2,I)+EVOL(4,I)/2.
      ETEMP(1,I)=ETEMP(2,I)+ETEMP(4,I)/2.
      FENTH(1,I)=FENTH(2,I)+FENTH(4,I)/2.
      IF(IECHE(I),EQ,1) GO TO 210
      CALL HPRUD(PCYL(1),ETEMP(1,I),FEQUI(1,I),DEL,PSI,FNTHLP,
1 CSUBP,CSURT,RHO,DPHDT,DKHDDP,FMWT(I),ERGAS(I),CVBG,FRG)
      GO TO 212
210 CALL UPRUD(PCYL(1),ETEMP(1,I),FEQUI(1,I),DEL,PSI,RESFRK,ENTHLP,
1 CSUBP,CSURT,RHO,DPHDT,DKHDDP,CH1,FRGAS(I),CVBG)
      FMWT(I)=WRAP
212 CONTINUE
      FRG(I)=FRGAS(I)/41.447+CVBG
      SUPHT(I)=SIMPHT(I)-DHLOS(I)/6.
220 CONTINUE
      ANGLE=ANGLE+ASTEP/2.
      CALL CALCQ(DV CYL,ENTINT)
      CALL DELTA(DV CYL,ENTINT)
      STGINT=STGINT+GINT/3.
      STGEXT=STGEXT+GEXT/3.
      STQEXT=STQEXT+QEXT/3.
      DO 225 I=1,NQIM1
225 TQHT(I)=TQHT(I)+QHTRC(I)/3.

      THIRD STEP

      PCYL(5)=PCYL(3)*ASTEP
      PCYL(1)=PCYL(2)+PCYL(5)/2.
      DO 300 I=1,IE
      FMASS(5,I)=FMASS(3,I)*ASTEP
      FEQUI(5,I)=FEQUI(3,I)*ASTEP
      EVOL(5,I)=EVOL(3,I)*ASTEP
      ETEMP(5,I)=ETEMP(3,I)*ASTEP
      FENTH(5,I)=FENTH(3,I)*ASTEP
      FMASS(1,I)=FMASS(2,I)+FMASS(5,I)/2.
      FEQUI(1,I)=FEQUI(2,I)+FEQUI(5,I)/2.

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PCYL (1) = PCYL (2) * ASTEP
PCYL (2) = PCYL (2) + (PCYL (4) + 2. * PCYL (5) + 2. * PCYL (6) + PCYL (7)) / 6.
DO 500 I = 1, IE
FMASS (1, I) = FMASS (3, I) * ASTEP
FEQUI (1, I) = FEQUI (3, I) * ASTEP
FVOL (1, I) = FVOL (3, I) * ASTEP
FTEMP (1, I) = FTEMP (3, I) * ASTEP
FENTH (1, I) = FENTH (3, I) * ASTEP
FMASS (2, I) = FMASS (2, I) + (FMASS (4, I) + 2. * FMASS (5, I) + 2. * FMASS (6, I)
+ FMASS (7, I)) / 6.
FEQUI (2, I) = FEQUI (2, I) + (FEQUI (4, I) + 2. * FEQUI (5, I) + 2. * FEQUI (6, I)
+ FEQUI (7, I)) / 6.
FVOL (2, I) = FVOL (2, I) + (FVOL (4, I) + 2. * FVOL (5, I) + 2. * FVOL (6, I)
+ FVOL (7, I)) / 6.
FTEMP (2, I) = FTEMP (2, I) + (FTEMP (4, I) + 2. * FTEMP (5, I) + 2. * FTEMP (6, I)
+ FTEMP (7, I)) / 6.
FENTH (2, I) = FENTH (2, I) + (FENTH (4, I) + 2. * FENTH (5, I) + 2. * FENTH (6, I)
+ FENTH (7, I)) / 6.
IF (IECHE (I), EQ, 1) GO TO 510
CALL MPROP (PCYL (1), FTEMP (1, I), FEQUI (1, I), DEL, PSI, ENTHLP,
1 CSUBP, CSURT, PHO, DRHOUT, DRHODP, EMWT (I), ERGAS (I), CVBG, EBGR)
GO TO 512
510 CALL UPROP (PCYL (1), FTEMP (1, I), FEQUI (1, I), DEL, PSI, RESFRK, ENTHLP,
1 CSUBP, CSURT, PHO, DRHOUT, DRHODP, CHI, ERGAS (I), CVBG)
EMWT (I) = MRAP
512 CONTINUE
ECP (I) = ERGAS (I) / 41.447 + CVBG
DHLOS (I) = (SUMHT (I) - DHLOS (I) / 6.) * ASTEP
500 CONTINUE
C INJECTED FUEL HAS SAME TEMP AND ENTHALPY AS AIR. ( THIS ASSUMPTION IS
C NECESSARY FOR THE ACCURATE CALCULATION. BECAUSE THE MASS JUST AFTER
C INJECTED IS SO SMALL THAT IT WILL CAUSE A LARGE AMOUNT OF FPROP.)
IF (ANGLE.GT. AFUELE) GO TO 530
IF (IC.EQ.1) GO TO 530
FTEMP (2, IE) = FTEMP (2, I)
FENTH (2, IE) = FENTH (2, I)
530 CONTINUE
RETURN
END

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SUBROUTINE CALC(DV CYL,FNTINT)
C
C THIS SUBROUTINE IS TO SET THE DATA FOR SUBROUTINE RUNGE
COMMON/EPROP/ TECH(100),EFUEL(100),EANG(100),EMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DOEMAS(100),
1 FRGAS(100),ECP(100),FENTH(8,100),EMASS(8,100),EEQUI(8,100),
1 EVOL(8,100),ETEMP(8,100),PCYL(8)
COMMON /TIMIG/ AINT0,AEXT0,AINTC,AEXTC,AFUELS,AFUELE
COMMON /STATE/ ANGLF1,ANGLE,JCOND,ASTEP,NSTEP,IE
COMMON /GEOMT/ STROKE,BORE,CONL,VCLEAR,VCUP,RCUP,CYLN
COMMON /HETRS/ NDIVID,TMETAL(10),QHTRC(10),ARF(10)
COMMON /HETR1/ PRASF,TBASE,VBASE,COHT1,COHT2,COHT,RHR(10)
COMMON /SUM2/ TTGINT,TTGEXT,TTQEXT,TTQHTR(10),TQHTR(10),PS
COMMON /STAT1/ RPM,JLAST,NJET,PHIO,EGR
COMMON /FUELP/ FUEL,FKCAL,ZM,ZN,PSI,SAFC,VACAL
COMMON /JETS/ RJET(50),HJET(50),AJET(50),BJET(50),THER(50),
1 VRJET(50),VZJET(50),WJET(50),VJET(50),AINJ(50),RHJET(50),
2 PITCH(50)
COMMON /PLIME/ PMASS(50),BMASS(50),RPLUM(50),HRMASS(50)
COMMON /JETDAT/ WSWIRG,ALPHA,BETA,RHGAS
COMMON/INFLO/ NINEF,XINEF(20),YINEF(20),AREIND,PINT,TINT,GINT,
1 EQUINT,AREFINT,XINV
COMMON/EXFLO/NFEXF,XEXEF(20),YEXEF(20),AREEXD,PEXT,TEXT,GEXT,
1 IGLFT,TLFT,QEXT,QFLOW,ELEFT,AREEXT,XEXV
COMMON/COMR/ RPLUM0,ENTUR,RLFSP,ASPARK,TSCALE
C
C CYLINDER VOLUME
CALL VOLCY(ANGLE,VCYL,DV CYL,STROKE,BORE,CONL,VCLEAR,VCUP)
C
C HEAT TRANSFER
CALL HEATR
C
C NO HEAT RELEASE FOR ELEMENT 1(AIR)
DHFUEL(1)=0.
GO TO (100,200,300,400,600),JCOND
C
C WHEN THE INTAKE AND EXHAUST VALVE OPEN
C INTAKE PORT CALCULATION
100 CALL INPOR
DIEMAS(1)=GINT
IF(GINT) 700,710,710
C
C INTAKE FLOW IS FROM PORT TO CYLINDER
710 EEQ0=EEQUI(1,1)
EEQN=(EMASS(1,1)*EFQUI(1,1)*(EQUINT+SAFC)+GINT*EQUINT*
1 (EEQUI(1,1)+SAFC))/(FMASS(1,1)*(EQUINT+SAFC)+GINT*(EFQUI(1,1)
2 +SAFC))
EEQUI(3,1)=EEQN-EEQ0
QFLOW=GINT*FNTINT
GO TO 720
C
C INTAKE FLOW IS FROM CYLINDER TO PORT (BACK FLOW)
700 EFQUI(3,1)=0.
QFLOW=GINT*FNT*(ETEMP(1,1)*EEQUI(1,1)*PCYL(1))
C
C EXHAUST PORT CALCULATION
720 CALL EXPOR
DOEMAS(1)=-GEXT
EMASS(3,1)=GINT+GEXT
QFLOW=QFLOW+GEXT*FENTH(1,1)

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      GO TO 2000
C
C WRPN THE INTAKE PROCESS CALCULATION
C INTAKE PORT CALCULATION
200 CALL INPOR
    DIEMAS(1)=GINT
    IF(GINT) 800,810,810
C
C INTAKE FLOW IS FROM PORT TO CYLINDER
810 EEQO=EEQUI(1,1)
    EEON=(EMASS(1,1)*EEOUT(1,1)*(EQUINT+SAFC)+GINT*EQUINT*
    ) (EEQUI(1,1)+SAFC))/(EMASS(1,1)*(EQUINT+SAFC)+GINT*(EEQUI(1,1)
    2 +SAFC))
    EEQUI(3,1)=EEON-EEQO
    QFLOW=GINT*FNTINT
    GO TO 820
C
C INTAKE FLOW IS FROMCYLINDER TO PORT (BACK FLOW)
800 FFQUI(3,1)=0.
    QFLOW=GINT*EENTH(1,1)
820 GEXT=0.
    QEXT=0.
    EMASS(3,1)=GINT
    GO TO 2000
C
C COMPRESSION AND EXPANSION
300 EEQUI(3,1)=0.
    EMASS(3,1)=0.
    QFLOW=0.
    GINT=0.
    GEXT=0.
    QEXT=0.
    GO TO 2000
C
C COMBUSTION AND EXPANSION PROCESS
400 CONTINUE
    IF(IE.EQ.1) GO TO 300
    IF(NSTEP.GE.2) GO TO 410
    CALL DCAL
C
C THE EFFECT OF PUSHING THE FJEL ELEMENT INTO CYLINDER
    IF(ANGLE.GF.AFUELE) GO TO 410
    DDV=EVOL(1,IE)/ASTEP
410 CONTINUE
    QFLOW=-DOEMAS(1)*EENTH(1,1)
    GINT=0.
    GEXT=0.
    QEXT=0.
    IF(ANGLE.GF.AFUELE) GO TO 420
    DVCYL=DVCYL-DDV
420 CONTINUE
    GO TO 2000
C
C EXHAUST VALVE OPEN
600 EFQUI(3,1)=0.
    CALL EXPOR
    DOEMAS(1)=-GEXT
    EMASS(3,1)=GEXT
    QFLOW=GEXT*FENTH(1,1)
2000 RETURN
    END

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SUBROUTINE FRICT

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C
C CALCULATION OF THE FRICTION LOSSES
C THIS PROGRAM WAS ORIGINALLY MADE BASED ON IN.-LB UNITS.
COMMON /STAT1/ RPM,ILAST,NJET,PHIO,FSR
COMMON /GEOMT/ STROKE,BORE,CONL,VCLEAR,VCUP,RCUP,CYLN
COMMON /INFLO/ NINEF,XINEF(20),YINEF(20),AREIND,PINT,TINT,GINT,
1 FQUINT,ARFINT,XINV
COMMON /AMR1E/ TAMR,PAMR
COMMON /FRIDT/ RSKIRT,FCOE,RINGN,FMEP(10)
FAC1=2.54
FAC2=1000./14.2
BOREX=BORE/FAC1
STROX=STROKE/FAC1
RSKIX=RSKIRT/FAC1
CR=(3.1416*BORE*BORE*STROKE/4.+(VCUP+VCLEAR)/(VCUP+VCLEAR))

C
C MISCELLANEOUS AND PUMPS
FMEP(2)=FAC2*0.39*(RPM/1000.)**1.5

C
C CAM GEAR
DINTV=SQRT(AREIND*4./(3.1416*XINV))/FAC1
FMEP(3)=FAC2*(30.-(4.*RPM/1000.))*XINV*DINTV**1.75/(BOREX*BOREX
1 *STROX)

C
C BEARING
FMEP(4)=FAC2*(FCOE*BOREX/STROX)*RPM/1000.

C
C BLOWBY
XX=ABS(PAMR-PINT)
IF (XX.LT.0.) XX=0.
FMEP(5)=FAC2*SQRT(XX)*(1.72*CR**0.4-(0.49+0.015*CR)*(RPM/1000.))**
1 1.185)

C
C VISCOUS PISTON
FMEP(6)=FAC2*(21.93*RSKIX/(BOREX*STROX)*STROX*RPM/5000.)

C
C STATIC RING TENSION
FMEP(7)=FAC2*2.11*STROX*RINGN/(BOREX*BOREX)

C
C RING GAS PRESSURE
FMEP(8)=FAC2*XX*(2.35*STROX/(BOREX*BOREX))*(0.088*CR+0.182*CR**
1 (1.33-(0.121*STROX*RPM/5000.)))

C
C TOTAL FRICTION LOSS
FMEP(1)=0.
DO 100 I=2,8
FMEP(1)=FMEP(1)+FMEP(I)
100 CONTINUE
RETURN
END

```

SUBROUTINE VGAS(RJET,HJET, RPM, ANGLE,VRGAS,VZGAS,WSWIR,VPIS,
1 FSWIR)

```

C
C
C CALCULATION OF THE GAS VELOCITY SURROUNDING THE JET
COMMON /GEOMT/ STROKE,BORE,CONL,VCLEAR,VCUP,RCUP,CYLN
COMMON /JETDAT/ WSWIR0,ALPHA,BETA
C RJET=JET RADIUS FROM CENTER(CM) (GIVEN)
C HJET=JET VERTICAL LOCATION FROM CYLINDER HEAD (GIVEN)
C RPM= ENGINE SPEED (GIVEN)
C ANGLE= CRANK ANGLE (DEG),TDC=360 (GIVEN)
C VRGAS=VELOCITY COMP. OF RADIAL DIRECTION (GIVEN)
C VZGAS=VELOCITY COMP. OF VERTICAL DIRECTION (GIVEN)
C WSWIR=ANGULAR VELOCITY(RAD/SEC) (GIVEN)
C VPIS= PISTON SPEED(CM/SEC) (GIVEN)
C FSWIR=CONSERVATION RATIO OF SWIRL (GIVEN)
C STROKE ,BORE,CONL,VCLEAR,VCUP,RCUP,WSWIR0 ARE GIVEN IN COMMON
C (SEE THE INPUT ROUTINE TO CHECK THE MEANINGS OF THESE VALUES)
C NO SUBROUTINES ARE REQUIRED
  RCYL=BORE/2.
  AK1=VCUP/(3.1416*RCYL*RCYL)
  AK2=VCUP/(3.1416*RCUP*RCUP)
  ANG=ANGLE*3.1416/180.
  ANG0=3.1416
C
C PISTON HIGHT FROM THE PISTON TO THE CYLINDER HEAD
HIGH=SQRT(1.-(STROKE/2.*SIN(ANG)/CONL)**2 )
HIGH=VCLEAR/(3.1416*BORE*BORE/4.)+STROKE/2.*(1.-COS(ANG))
1 *CONL*(1.-HIGH)
C
C PISTON HIGHT AT BDC
HIGH0=SQRT(1.-(STROKE/2.*SIN(ANG0)/CONL)**2 )
HIGH0=VCLEAR/(3.1416*BORE*BORE/4.)+STROKE/2.*(1.-COS(ANG0))
1 *CONL*(1.-HIGH0)
C
C PISTON SPEED
VPIS=SQRT(CONL*CONL-STROKE*STROKE/4.*SIN(ANG)*SIN(ANG))
VPIS=STROKE/2*SIN(ANG)+STROKE*STROKE/4.*SIN(ANG)*COS(ANG)/VPIS
VPIS=VPIS*3.1416/180.*RPM*6.
XX=HIGH/AK1
IF(RJET.LT.RCUP) GO TO 100
IF(HJET.LT.HIGH) GO TO 200
VZGAS=0.
VRGAS=0.
GO TO 400
C
C GAS VELOCITY COMPONENTS ABOVE THE PISTON( FLAT PART)
200 VRGAS=RJET/(2.*AK1)*(RCYL*RCYL/(RJET*RJET)-1.)*VPIS/(XX*(1.+XX))
VZGAS=HJET/HIGH*VPIS
GO TO 400
100 IF(HJET.LT.HIGH) GO TO 300
C
C GAS VELOCITY COMPONENTS INSIDE THE CUP
VRGAS=0.
VZGAS=(AK1-AK2+HJET)/(AK1+HIGH)*VPIS
GO TO 400
C

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C GAS VELOCITY COMPONENTS ABOVE PISTON CJP
300 VRGAS=RJET/(2.*AK1)*(1.-AK2/AK1)/(XX*(XX+AK2/AK1))
    VRGAS=-VRGAS*VPIS
    VZGAS=(1.-RCYL*RCYL/(RCUP*RCUP*(XX+1.)))*HJET/HIGH*VPIS
C
C GAS VELOCITY (TANGENTIAL COMPONENT)
400 WSWIR=(HIGH+AK1)/(HIGH0+AK1)*WSWIR0*(AK2*RCUP**4.
1 +HIGH0*RCYL**4.)/(AK2*RCUP**4.+HIGH*RCYL**4.)
    WSWIR=WSWIR*FSWIR
    RETURN
    END
    SUBROUTINE CONT(XX,YY,X1,Y0,X,Y,N)
    DIMENSION Y(2),Y(2)
    IF(N-2) 10,20,30
10 X(1)=XX
    Y(1)=YY
    XX=XX+X1
    N=2
    RETURN
20 X(2)=XX
    Y(2)=YY
    N=3
    GO TO 60
30 IF((Y(2)-Y0)*(YY-Y0)) 40,50,50
40 X(1)=XX
    Y(1)=YY
    GO TO 60
50 X(2)=XX
    Y(2)=YY
60 XX=(Y0-Y(1))*(X(2)-X(1))/(Y(2)-Y(1))+X(1)
    RETURN
    END

```

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SUBROUTINE JET(RJET,HJET,AJET,RJET,THET,VRJET,V7JET,WJET,TDM,
1 AMJET,ASTEP,JSTEP,RPM,ANGLE,PITCH,FSWIR,WSWIR)

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```

C
C
C THIS SUBROUTINE IS FOR GETTING JET TRAJECTORY AND AIR ENTRAINMENT
COMMON /GEOMT/ STROKE,BORE,CONL,VCLEAR,VCUP,RCUP,CYLN
COMMON /JETDAT/ WSWIR,ALPHA,BETA,RHGAS
C RJET=RADIAL JET LOCATION FROM CENTER(CM) (GIVEN AND RETURNED)
C HJET=VERTICAL JET LOCATION FROM CYL. HEAD (GIVEN AND RETURNED)
C AJET=TANGEN. JET LOCATION FROM NOZZLE HEAD (GIVEN AND RETURNED)
C AJET=TANGEN. JET LOCATION FROM INJECTION NOZZ. (GIVEN AND RETURNED)
C RJET=JET RADIIJS(CM) (GIVEN AND RETURNED)
C THET=CONTACTING ANGLE(PI) (GIVEN AND RETURNED)
C VRJET=JET VELOCITY OF RADIAL DIRECTION(CM/SEC) (GIVEN AND RETURNED)
C V7JET=JET VEL. VERTICAL DIRECTION(CM/SEC) (GIVEN AND RETURNED)
C WJET=JET ANGULAR VELOCITY(RAD/SEC) (GIVEN AND RETURNED)
C TDM=ENTRAINED MASS(G/GIVEN TIME) (RETURNED)
C AMJET=JET MASS(G) (GIVEN AND RETURNED)
C ASTEP=CALCULATION INTERVAL OF MAIN PROGRAM(DEG) (GIVEN)
C JSTEP=HOW MANY TIMES DO YOU WANT TO CALCULATE IN ONE CALCULATION
C INTERVAL OF MAIN PROGRAM (GIVEN)
C RPM=ENGINE SPEED(RPM) (GIVEN)
C ANGLE=CRANK ANGLE (DEG),TDC =360 (GIVEN)
C PITCH=PITCH OF JET ELEMENT(GIVEN INPUT ROUTINE) (GIVEN)
C FSWIR=CONSERVATION RATIO OF SWIRL (GIVEN)
C STROKE ,BORE,CONL,VCLEAR,VCUP,RCUP,WSWIR,ALPHA,BETA,RHGAS ARE
C GIVEN IN COMMON
C (SEE THE INPUT ROUTINE TO CHECK THE MEANINGS OF THESE VALUES)
C SUBROUTINE REQUIRED:
C VGAS
C CONT
DIMENSION YX(2),YY(2)
RCUP=2.463*
PCYL=BORE/2.
TDM=0.
AASTEP=ASTEP/FLOAT(JSTEP)
C1=ASTEP/(6.*RPM)
C2=AASTEP/(6.*RPM)
C3=1./(6.*RPM)
VJET=SQRT(VRJET**2 + V7JET**2 + (RJET*WJET)**2 )
DO 3000 I=1,JSTEP
RHJET=AMJET/((3.1416*BJET**2*(1.-THET/(2.*3.1416)) + BJET**2
1 *SIN(THET)/2.)*VJET*PITCH/(6*RPM))
ANGLJ=ANGLE+FLOAT(I)*AASTEP
ANG=ANGLJ*3.1416/180.
HIGH=SQRT(1.-(STROKE/2.*SIN(ANG)/CONL)**2 )
HIGH=VCLEAR/(3.1416*BORE*BORE/4.)*STROKE/2.*(1.-COS(ANG))
1 +CONL*(1.-HIGH)
C
C GAS VELOCITY SURROUNDING THE JET
CALL VGAS(RJET,HJET,RPM, ANGLJ,VRGAS,V7GAS,WSWIR,VPIS,FSWIR)
1111 FORMAT(7E12.4)
C
C CALCULATION OF THE PARALLEL VELOCITY COMPONENT TO THE JET
VI=SQRT(VRJET**2 + (RJET*WJET)**2 + V7JET**2 )
VI=(VRJET*VRGAS + RJET**2.*WSWIR*WJET + V7JET*V7GAS)/VI
C
C CALCULATION OF THE NORMAL VELOCITY COMPONENT TO THE JET
VN=VRGAS**2 + (RJET*WSWIR)**2 + V7GAS**2
XXX=VN-VI**2

```

```

      IF (XXX.GT.0.) GO TO 50
      VN=0.
      GO TO 60
50  VN=SQRT(VN-VI**2)
60  CONTINUE
C
C  CHANGES OF MASS AND VELOCITY COMPONENTS OF THE JET
C  CHANGE OF MASS (G)
      DM= VJET*SQRT(RHJET/RHGAS)*RHGAS*PITCH/(6.*RPM)
      DM=DM*2.*3.1416*RJET*(ALPHA*ARS(VJET-VI)+BETA*ARS(VN))
      DM=DM*(1.-THER/(2.*3.1416))
      DM=DM*C2
      TDM=TDM+DM
      AMJET=AMJET+DM
C  CHANGE OF VERTICAL VELOCITY
      DVZ=DM/AMJET*(VZGAS-VZJET)
C  CHANGE OF RADIAL VELOCITY
      DVR=AMJET*PJET*(WJET**2 -RHGAS/RHJET*WSWIR**2 )
      DVR=DVR*C2
      DVR=(DVR-(VRJET-VRGAS)*DM)/AMJET
C  CHANGE OF ANGULAR VELOCITY
      DW=((WSWIR-WJET)*RJET*DM-2.*AMJET*VRJET*WJET*C2)/(AMJET*RJET)
C  CHANGE OF TOTAL JET VELOCITY
      DV=(RJET**2.*WJET*DW+RJET*WJET**2 *VRJET*C2+VRJET*DVR+VZJET*DVZ)
      DV=DV/VJET
C
C  NEW STATE CONDITIONS OF THE JET
      VJET=VJET+DV
      WJET=WJET+DW
      VRJET=VRJET+DVR
      VZJET=VZJET+DVZ
      RJET=RJET+VRJET*C2
      ALJET=VJET*PITCH/(6.*RPM)
      VOLJE=DM/RHGAS+(AMJET-DM)/RHJET
C  JET RADIUS WITHOUT CONTACTING ANY WALL
      RJFT0=SQRT(VOLJE/(3.1416*ALJET))
      HJET=HJET+VZJET*C2
      AJFT=AJFT+WJET*C2
C
C  CALCULATION OF THE JET CONFIGURATION
      X1=RJET+RJFT0
      X2=HJET+BJFT0
      X5=HIGH+VCUP/(3.1416*PCUP**RCUP)
      IF (HJET.GT.HIGH) GO TO 100
      IF (PCYL.GT.X1) GO TO 200
      IF (PCYL.GT.RJET) GO TO 300
C
C  JET IS RESTRICTED BY THE CYLINDER WALL (VRJET=0.)
      VRJET=0.
      RJFT=1.414*BJFT0
      THER=3.1416
      RJET=PCYL
      GO TO 2000
C
C  JET IS CONTACTING WITH THE CYLINDER WALL
300  NN=0
      X3=RJET
      X4=PCYL
      GO TO 1500
200  IF (X2.LT.HIGH) GO TO 400

```

```

IF(X1.LT.RCUP) GO TO 400
IF(RJET.LT.RCUP) GO TO 500

```

C

C JET IS CONTACTING WITH THE PISTON HEAD

```

NN=0
X3=HJET
X4=HIGH
GO TO 1500

```

C

C JET IS NOT CONTACTING

```

400 RJET=RJET0
    THER=0.
    GO TO 2000
100 IF(RJET.LT.RCUP) GO TO 600
    SL=ABS(HJET-HIGH)
    SR=ABS(RJET-RCUP)
    IF(SL.GT.SP) GO TO 700

```

C

C JET IS RESTRICTED BY THE PISTON HEAD(VZJET=VPIS)

```

VZJET=VPIS
RJET=1.414*RJET0
THER=3.1416
HJET=HIGH
GO TO 2000

```

```

700 IF(HJET.GT.X5) GO TO 800

```

C

C JET IS RESTRICTED BY THE PISTON CUP(VRJET=0.)

```

VRJET=0.
RJET=1.414*RJET0
THER=3.1416
RJFT=RCUP
GO TO 2000

```

C

C JET IS RESTRICTED BY THE PISTON CUP EDGE(VRJET=0. AND VZJET=VPIS)

```

800 VRJET=0.
    VZJET=VPIS
    RJET=RCUP
    HJET=X5
    RJFT=1.414*RJET0
    GO TO 2000

```

```

600 IF(RCUP.LT.X1) GO TO 900
    IF(X5.LT.X2) GO TO 1000

```

C

C JET IS IN THE PISTON CUP WITHOUT ANY RESTRICTION

```

RJFT=RJET0
THER=0
GO TO 2000

```

```

1000 IF(X5.LT.HJET) GO TO 1100

```

C JET IS CONTACTING WITH THE PISTON BOTTOM

C

```

NN=0
X3=HJET
X4=X5
GO TO 1500

```

C

C JET IS RESTRICTED BY THE PISTON BOTTOM

```

1100 RJET=1.414*RJET0
    THER=3.1416
    HJET=X5
    VZJET=VPIS

```



```

      GO TO 2000
900  IF(RCIJP.LT,RJET) GO TO 1200
C
C  JFT IS CONTACTING WITH THE PISTON CUP WALL
500  N1=0
      X3=RJET
      X4=RCIJP
      GO TO 1500
C
C  JFT IS RESTRICTED BY THE PISTON CUP WALL
1200  RJET=1.414*RJET0
      THER=3.1416
      RJET=RCIJP
      VRJET=0.
      GO TO 2000
C
C  SOLVE THE CONTACTING ANGLE
1500  ITEL=1
1550  IF(RJET.GT.(X4-X3)) GO TO 1500
      RJET=X4-X3
1600  ANGI=2.*ATAN(SORT(RJET*RJET-(X4-X3)*(X4-X3))/(X4-X3))
      THER=ANG1
      AREA1=3.1416*RJET*RJET*(1.-ANG1/(2.*3.1416))+(X4-X3)*SIN(ANG1
1  /2.)*RJET
      AREA2=3.1416*RJET0*RJET0
      ERRO=(AREA1-AEA2)/AREA2
      IF(ABS(ERRO).LT.0.005) GO TO 2000
      Y0=0.
      XNEXT=0.1
      CALL CONT(RJET,ERRO,XNEXT,Y0,XX,YY,NV)
1990  FORMAT(2E12.4)
      ITEL=ITFL+1
      IF(ITFL.GT.30) GO TO 1990
      GO TO 1550
1990  WRITE(6,1999)
1999  FORMAT('STOP DUE TO THE OVER ITELATION')
2000  CONTINUE
3000  CONTINUE
      RETURN
      END

```

SUBROUTINE OPFPA

```

C
C
C THIS SUBROUTINE IS FOR OPERATION OF EACH PROCESS
COMMON/EPKOP/ TECHF(100),EFUEL(100),EANGL(100),FMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DOEMAS(100),
1 ERGAS(100),ECP(100),FENTH(R,100),FMASS(R,100),FEQUI(R,100),
1 EVOL(R,100),FTEMP(R,100),PCYL(R)
COMMON /STATE/ ANGLE1,ANGLE,JCOND,ASTEP,NSTEP,IF
COMMON /FUFLP/ FUEL,FKCAL,ZM,ZN,PSI,S AFC,VACAL
COMMON /TIMIG/ AINT0,AEXT0,AINTC,AEXTC,AFUELS,AFUFLE
COMMON /OSOPE/ LPT,INP,LINECO,LINEST,ITEL,LITEL
COMMON/CONTL/ ASTIN,ASTCL,ASTCR,ASTEX,ASTOV,PRININ,PRINCL,PRINEX,
1 PRINOV
COMMON /STAT1/ RPM,JLAST,NJET,PHIO,EGR
COMMON /HETR1/ PRASF,TRASE,VBASF,COHT1,COHT2,COHT,RHR(10)
COMMON /GEOMT/ STROKF,RORE,CONL,VCLFAR,VCUP,RCUP,CYLN
COMMON /SUWZ/ TTGINT,TTGEXT,TTQEXT,TTQHTR(10),TQHTR(10),PS
COMMON/INFL0/ NINEF,XINFF(20),YINEF(20),AREIND,PINT,TINT,GINT,
1 EQUINT,AREINT,XINV
COMMON /AMPJE/ TAMR,PAMB
COMMON /NOXSP/ CONNO(100),PPMNO(100),WNO(100),SUMNO(100),PPMEXT
ANGLE=ANGLE+ASTEP
IF (ANGLE-AEXTC) 100,120,120
100 IF (JCOND-1) 110,9000,9000
110 IF (JLAST.EQ.1) GO TO 112
C
C SET THE LAST COMPUTING STEP
ASTEP=AINT0-(ANGLE-ASTEP)
ANGLE=AINT0
JLAST=1
GO TO 9000
C
C INTAKE AND EXHAUST VALVE OPEN
112 JCOND=1
JLAST=0
ASTEP=ASTOV
LINEST=FIX(PRINOV)
LINECO=0
WRITE(LPT,7000)
WRITE(LPT,7010)
WRITE(LPT,7020)
WRITE(LPT,7030)
GO TO 9000
120 IF (ANGLE-AINTC) 130,140,140
130 IF (JCOND-2) 150,9000,9000
150 IF (JLAST.EQ.1) GO TO 152
ASTEP=AEXTC-(ANGLE-ASTEP)
ANGLE=AEXTC
JLAST=1
GO TO 9000
C
C INTAKE VALVE OPENS
152 JCOND=2
JLAST=0
ASTEP=ASTIN
LINEST=FIX(PRININ)
LINECO=0
WRITE(LPT,7200)
WRITE(LPT,7010)

```

```

WRITE (LPT,7220)
WRITE (LPT,7230)
GO TO 9000
140 IF (ANGLE-AFUELS) 160,170,170
160 IF (JCOND-3) 180,9000,9000
180 IF (JLAST.EQ.1) GO TO 182
   ASTEP=AINTC-(ANGLE-ASTEP)
   ANGLE=AINTC
   JLAST=1
   GO TO 9000
C
C   COMPRESSION PROCESS
182 JCOND=3
   JLAST=0
   ASTEP=ASTC
   LINEST=IFIX (PRINCL)
   LINECO=0
C
C   CALCULATE THE OVERALL EQUIVALENCE RATIO
C
   PHIO=0.5
   DO 172 I=1.4
   FRES=(EMASS(2,1)-TTGINT)*PHIO/(SAFC+1.)
   FEGR=TTGINT*PHIO*EGR/(SAFC+1.)
   PHIO=(FUEL+FRES+FEGR)/(EMASS(2,1)-FRES+FEGR)*SAFC
172 CONTINUE
C
C   CALCULATE THE VOLUMETRIC EFFICIENCY
   DEL=ZN/ZM
   CALL UPROP (PAMB,TAMB,0.,DEL,PST,0.,DJM,DUM,DUM,DUM,DUM,DUM,
1   DUM,RGAS,DUM)
   DISP=3.1415/4.*BORE*BORE*STROKE
   THMASS=PAMB*DISP/(RGAS*TAMB)
   EFFVOL=TTGINT/THMASS
C
C   CORRECT THE EQUIVALENCE RATIO OF CHARGED AIR
   FEQUI(2,1)=(FRES+FEGR)/(EMASS(2,1)-FRES-FEGR)*SAFC
   TRASE=ETEMP(2,1)
   PRASE=PCYL(2)
   VRASE=EVOL(2,1)
C
C   NOX CONCENTRATION FOR CHARGED AIR
   CONNO(1)=(EMASS(2,1)-TTGINT)/EMWT(1)*PPMEXT/1000000.
1   +TTGINT*FEGR/EMWT(1)*PPMEXT/1000000.
   PPMNO(1)=1000000.*CONNO(1)*EMWT(1)/EMASS(2,1)
   WNO(1)=30.*CONNO(1)
   WRITE (LPT,6100) PHIO
   WRITE (LPT,6120) EFFVOL
   WRITE (LPT,6140) FEQUI(2,1)
   WRITE (LPT,6160) PPMNO(1),WNO(1)
   WRITE (LPT,7400)
   WRITE (LPT,7010)
   WRITE (LPT,7420)
   WRITE (LPT,7430)
   GO TO 9000
170 IF (ANGLE-AFXT0) 190,200,200
190 IF (JCOND-4) 210,192,192
192 IF (ANGLE-AFUELF-20.) 9000,9000,194
194 ASTEP=ASTC
   GO TO 9000

```

```

210 IF(JLAST.EQ.1) GO TO 212
   ASTEP=AFUELS-(ANGLF-ASTEP)
   ANGLE=AFUELS
   JLAST=1
   GO TO 9000
C
C FUEL STARTS TO BE INJECTED
C COMBUSTION AND EXPANSION
212 JLAST=0
   JCOND=4
   ASTEP=ASTCR
   LINFST=IFIX(PRINCL)
   LINECO=0
   WRITE(LPT,7600)
   WRITE(LPT,7010)
   WRITE(LPT,7620)
   WRITE(LPT,7630)
   GO TO 9000
200 IF(ANGLE-AINT0-720.) 220,230,230
220 IF(JCOND-5) 240,9000,9000
240 IF(JLAST.EQ.1) GO TO 242
   ASTEP=AEXT0-(ANGLE-ASTEP)
   ANGLE=AEXT0
   JLAST=1
   GO TO 9000
C
C EXHAUST VALVE OPENS
242 JCOND=5
   JLAST=0
   ASTEP=ASTEY
   LINFST=IFIX(PRINFX)
   LINECO=0
   IF(IE.EQ.1) GO TO 244
C
C CALCULATE THE TOTAL MASS AND ENTHALPY
   TENTH=0.
   TMASS=0.
   DO 7320 I=1,IF
   TENTH=TENTH+EFNTH(2,I)*EMASS(2,I)
   TMASS=TMASS+EMASS(2,I)
7320 CONTINUE
C
C CALCULATE THE AVERAGE ENTHALPY
   AVENT=TENTH/TMASS
C
C CALCULATE THE AVERAGE TEMPERATURE
C FIRST FIND OUT THE ELEMENT WHICH ENTHALPY IS CLOSE TO THE
C AVERAGE ENTHALPY
   DENTM=1000.
   II=1
   DO 7330 I=1,IF
   DENT=ABS(AVENT-EFNTH(2,I))
   IF(DENTM.LT.DENT) GO TO 7330
   DENTM=DENT
   II=I
7330 CONTINUE
C
C DEFINE THE NEW PROPERTIES
   ETEMP(2,1)=ETEMP(2,II)+(AVENT-EFNTH(2,II))/FCP(II)
   EFQUI(2,1)=P-II0

```

```

FENTH(2,1)=AVFNT
FMASS(2,1)=TMASS
CALL HPROD(PCYL(2),FTFMP(2,1),FEQUI(2,1),DEL,PSI,FENTHLP,CCURP,
1 CSUBT,RHO,DRHODT,DRHODP,EMWT(1),ERGAS(1),CVBG,EBGR)
ECP(1)=ERGAS(1)/41.447+CVBG
EVOL(2,1)=VCYLN
IE=1

C
C NOX AMOUNT WILL BE DEFINED.
C FXHAUST NOX ON MASS BASE
  ANOXEX=TWN0*ITGINT/TMASS*RPW/120.
  WRITE(LPT,7340)
  WRITE(LPT,7342) PCYL(2),ETEMP(2,1),FEQUI(2,1),FENTH(2,1),
1 FMASS(2,1)
  WRITE(LPT,7450) TPPM,ANOXEX
7340 FORMAT(/,10X,***EXPANSION PROCESS IS CALCULATED AS ONE ELEMENT**
1**)
7342 FORMAT(15X,18HPRESSURE          = ,F10.2,'ATA',/,
1 15X,18HAV. TEMPERATURE          = ,F10.1,'< DEG',/,
2 15X,18HAV. EQUI. RATIO          = ,F10.3,/,
3 15X,18HAV. ENTHALPY              = ,F10.2,'CAL/G',
4 15X,18HTOTAL MASS                = ,F10.4,'G')
244 CONTINUE
  WRITE(LPT,7800)
  WRITE(LPT,7010)
  WRITE(LPT,7820)
  WRITE(LPT,7230)
  GO TO 9000
230 JCOND=5
9000 ANGLE=ANGLF-ASTEP
  RETURN
6100 FORMAT(/,10X,'OVERALL EQUIVALENCE RATIO =',F12.3)
6120 FORMAT(10X,'VOLUMETRIC EFFICIENCY=',F12.3)
6140 FORMAT(10X,'CORRECT CHARGED AIR PHI =',F12.3)
6160 FORMAT(10X,'INDUCED AIR AND REG. GAS NOX',/,
1 15X,'PPM =',F12.1,5X,'MASS =',E12.3,'G')
7000 FORMAT(1H1,5X,'INTAKE AND EXHAUST VALVE OPEN',/)
7010 FORMAT(1H , 'OUTPUT DATA')
7020 FORMAT(2X ,4HTHEO,3X,4H P ,3X,4H T ,3X,4H V ,3X,4HWORK,
1 3X,7HCYL.MAS,3X,15H HEAT TRANSFER ,6X,4HAEFI,3X,4H GWI,5X,
2 4HAEFF,3X,4H GWE)
7030 FORMAT(2X ,4HDEG ,3X,4HATA ,3X,4H K ,2X,5HCM**3,3X,4H PS ,
1 3X,7H KG ,3X,15H CAL/A CAL ,5X,5HCM**2,3X,4H G/S,4X,
2 5HCM**2,3X,4H G/S,/)
7200 FORMAT(1H1,5X,'INTAKE VALVE OPEN',/)
7220 FORMAT(2X ,4HTHEO,3X,4H P ,3X,4H T ,3X,4H V ,3X,4HWORK,
1 3X,7HCYL.MAS,3X,15H HEAT TRANSFER,5X,4HAEFI,3X,4H GWI)
7230 FORMAT(2X ,4HDEG ,3X,4HATA ,3X,4H K ,2X,5HCM**3,3X,4H PS ,
1 3X,7H G ,3X,15H CAL/A CAL ,5X,5HCM**2,3X,4H G/S,/)
7400 FORMAT(1H1,5X,'COMPRESSION',/)
7420 FORMAT(2X ,4HTHEO,3X,4H P ,3X,4H T ,3X,4H V ,3X,4HWORK,
1 3X,7HCYL.MAS,3X,15H HEAT TRANSFER,7X,4HFSWR,3X,4H S,RT)
7430 FORMAT(2X ,4HDEG ,3X,4HATA ,3X,4H K ,2X,5HCM**3,3X,4H PS ,
1 3X,7H G ,3X,15H CAL/A CAL ,11X,3X,4H*RPW)
7450 FORMAT(/,10X,***EXHAUST NOX***,/,
1 15X,15HNOX CONCEN. = ,F10.1,'PPM',/,
2 15X,15HTOTAL MASS = ,E12.3,'G')
7600 FORMAT(1H1,5X,'COMBUSTION AND EXPANSION',/)
7620 FORMAT(2X,4HTHEO,3X, 'PRESS',9X, 'PSI',7X, 'FS/IR',7X, 'S,RT',7X,
1 'PPM',9X, 'WNO',/,1X.

```

```

1          34N1J4.1X.2H**3X.4HT.4V.3X.4HATEM.3X.4HR.9T.
1 3X.4HMASS.3X.4HPMAS.3X.4HBMAS. 3X.4H V .3X.4HFQUI.3X.4H CP.
2 3X.4HENTH.3X.4H PPM.3X.4H WNO.1X.1RH JET LOCATIONS .1X.
1 4HRJET.1X.5HC.ANG .2X.4HT. )
7630 FORMAT( 6X.4X.4H K .3X.4H K .3X.4X .3X.4H MG .3X.4H MG .
1 3X.4H MG .2X.5HCM**3. 7X.2X.5HCAL/G.2X.5HCAL/G.2X.4H PPM.
2 1X.7HG*10**6.1X.5HR(CM) . 1X.5HZ(CM) .1X.5H DEG .2X.4H CM .2X.
3 4H DEG.2X.4H CAL)
7800 FORMAT(1H1.5X. EXHAUST VALVE OPEN. /)
7820 FORMAT(2X .4HTHEO.3X.4H P .3X.4H T .3X.4H V .3X.4HWRK.
1 3X.7HCYL.MAS.3X.15H HEAT TRANSFER.5X.4HAEFE.3X.4H GWF)
END

```

```

SUBROUTINE EXPOR
C
C EXHAUST PORT CALCULATION
C THIS PROGRAM WAS ORIGINALLY MADE USING METER-KG UNIT
C
COMMON/EPROP/ IECHE(100),EFJEL(100),EANGL(100),EMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DOFMAS(100),
1 ERGAS(100),ECP(100),EFNTH(9,100),FMASS(R,100),FEQUI(R,100),
1 EVOL(R,100),ETEMP(R,100),PCYL(R)
COMMON/EXFLO/NXFEF,XFEF(20),YFEF(20),AREXD,PEXT,TEXT,GEXT,
1 IGLFT,TLEFT,DEXT,OFLOW,ELEFT,AREXT,XEXV
COMMON /STATE/ ANGLE1,ANGLE,JCOND,ASTEP,NSTEP,IE
COMMON /TIMIG/ AINT0,AFXT0,AINTC,AFXC,AFUEL5,AFUELE
COMMON /FUFLP/ FUEL,FKCAL,ZM,7N,PSI,SAFC,VACAL
COMMON /STAT1/ RPM,ILAST,NJET,PHIO,ESR
IF(PEXT-PCYL(1)) 50,50,60
C EXHAUST FLOW IS FROM CYLINDER TO EX. PORT
C P VALUE DOESN'T CHANGE MUCH.
50 CEXT=-1.
   PHIGH=ERGAS(1)*9.8
   PHIGH=PCYL(1)*10000.
   PREX=PEXT/PCYL(1)
   EHIGH=FEQUT(1,1)
   IF(IGLFT) 51,51,52
C IGLFT=1 THERE IS RESIDUAL GAS
C IGLFT=0 THERE IS NOT RESIDUAL GAS
52 THIGH=ETEMP(1,1)
   AKCYL=1.34
   AKHIGH=AKCYL
   GO TO 100
51 THIGH=TLEFT
   PHIGH=ERGAS(1)*9.8
   AKLEF=1.34
   AKHIGH=AKLEF
   GO TO 100
C EXHAUST FLOW IS FROM EX. PORT TO CYLINDER
60 PHIGH=ERGAS(1)*9.8
   CEXT=1.
   EHIGH=FEQUT(1,1)
   PHIGH=PEXT*10000.
   PREX=PCYL(1)/PEXT
   THIGH=ETEMP(1,1)
   AKEXT=1.34
   AKHIGH=AKEXT
C
C CALCULATE THE EFFECTIVE EXHAUST FLOW AREA
100 IF(ANGLE-360.) 150,150,160
150 ANAE=(ANGLE-(AFXT0-720.))/(AEXTC-(AFXT0-720.))
   GO TO 200
160 ANAE=(ANGLE-AFXT0)/(AEXTC-AEXT0+720.)
200 CALL TWODIM(NXFEF,XEFF,YFEF,ANAE,AREFAC)
   AREXT=AREXD*AREFAC
   AREXT=AREXT/10000.
   PRCRIT=(2./(AKHIGH+1.))**(AKHIGH/(AKHIGH-1.))
C
C CHECK WHETHER FLOW IS CHOKED OR NOT
IF(PRCRIT-PREX) 300,300,310
C
C CHOKED

```

```

310 PREX=PRCRIT
C
C NEXT EQUATION IS BASED ON METER -KG UNIT
300 GEXT=          SQRT(ABS(2.*AKHIGH*9.8/RHIGH/THIGH
1 / (AKHIGH-1.)*(PREX**(2./AKHIGH)-PREX**((AKHIGH+1.)
2 /AKHIGH))))
GEXT=PHIGH*AREFXT/6./PPM*GEXT
GEXT=CEXT*GFXT
GFXT=GEXT*1000.
AREFXT=AREFXT*10000.
IF (CEXT) 410,410,400
410 TMANI=THIGH*PREX**((AKHIGH-1.)/AKHIGH)
CEXT=GEXT*ENT(TMANI,PHIGH,SAFC)
GO TO 1000
400 OEXT=GEXT*ENT(TEXT,PHIGH,SAFC)
1000 RETURN
END

```



```

C      SUBROUTINE VOLCY (ANGLE, VCYL, DVCYL, STROKE, BORE, CONL, VCLFAR, VCUP)
      PAI=3.141593
      ANGRAD=ANGLE*PAI/180.
      HSTROK=STROKE/2.
C      DELTA VOLUME CALCULATION
      DVCYL=PAI*BORE**2/4.*(HSTROK*SIN(ANGRAD)+HSTROK**2*SIN(ANGRAD)
1      *COS(ANGRAD)/SQRT(CONL**2-HSTROK**2*SIN(ANGRAD)**2))*PAI/180.
C      VOLUME CALCULATION
      DISTTB=CONL*(1.-SQRT(1.-(HSTROK/CONL)**2*SIN(ANGRAD)**2))
1      +HSTROK*(1.-COS(ANGRAD))
      VCYL=VCLFAR+VCUP+PAI*BORE*BORE/4.*DISTTB
      RETURN
      END
C

```

```
      SUBROUTINE TWOIM(N,X,Y,XIN,YOUT)
      DIMENSION X(1),Y(1)
      XDUMMY=XIN
      IF (XDUMMY-X(1)) 510,810,910
510  XDUMMY=X(1)
810  CONTINUE
      IF (XDUMMY-X(N)) 820,820,520
520  XDUMMY=X(N)
820  CONTINUE
      J=0
      3 J=J+1
      5 IF (XDUMMY-X(J)) 1,2,3
      2 YOUT=Y(J)
      GO TO 4
      1 YOUT=Y(J)-(X(J)-XDUMMY)/(X(J)-X(J-1))*(Y(J)-Y(J-1))
      4 RETURN
      END
```

```

SUBROUTINE HEATR
C
C HEAT TRANSFER CALCULATION
COMMON/ERROR/ IERR(100),EFJEL(100),FANGI(100),FMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DIEMAS(100),
1 FPGAS(100),EQ2(100),FFNTH(R,100),FMASS(R,100),FEQUT(R,100),
1 FVOL(R,100),ETEMP(R,100),PCYL(R)
COMMON /STATE/ ANGLE, JCONO, ASTEP, NSTEP, IF
COMMON /STAT1/ RPM, ILAST, NJET, PHIO, FGR
COMMON /GEOM1/ STROKE, RORE, CONL, VOLCLR, VCUP, RCUP, CYLN
COMMON /HETPS/ NOIVID, TMETAL(10), QHTRC(10), ARF(10)
COMMON /JETS/ RJET(50), HJET(50), AJET(50), RJET(50), THER(50),
1 VJET(50), VZJET(50), WJET(50), VJET(50), ANJ(50), RHJET(50),
2 PITCH(50)
COMMON /SUPGAS/ FSWIR, WSWIR, SDFCAY
COMMON /HETR1/ PRASE, TRASE, VRASE, COHT1, COHT2, COHT, RHR(10)
DIMENSION HTCO2(10), HTI(10)
ANGRAD=ANGLE*3.1416/180.
CALL VOLCY(ANGLE, CYLV, DIM, STROKE, RORE, CONL, VOLCLR, VCUP)
C
C GET THE H.T. AREA OF CYLINDER WALL
C
ARF(2) = (CYLV-VCUP)*4./RORE
DO 200 I=1,NOIVID
QHTRC(I)=0.
200 CONTINUE
DO 202 I=1,IF
DHLOS(I)=0.
202 CONTINUE
HTCON1=COHT*7014.*RORE**(-0.2)*PCYL(1)**0.9/(2.16*10.**R*RPM)
GO TO (210,210,220,220,210),JCONO
C
C ISENTROPIC TEMPERATURE
220 PISEN=PRASE*(VRASE/CYLV)**1.32
HTCON2=COHT1*(STROKE*RPM/30.)*100.*COHT2*CYLV*TRASE/(PRASE
1 *VRASE)*(PCYL(1)-PISEN)
GO TO 230
C
C FOR PROCESS --INTAKE, EXHAUST AND VALVE OVERLAP
210 PISEN=PCYL(1)
HTCON2=COHT1*(STROKE*RPM/30.)
230 CONTINUE
DO 300 I=2,NOIVID
C
C 0.4 IS AN ADJUSTING FACTOR
HTCO2(I)=(HTCON2+WSWIR**RHR(I)*0.4)**0.8
300 CONTINUE
IFILL=0
IF(I,FO,1) GO TO 1000
TAPFA=0.
DO 2400 J=1,IF
I=IF+1-J
IF(I,FO,1) GO TO 2120
C
C THE AREA OF JET CONTACTING WITH WALL
AREFATH=RJET(I)*STH(THER(I)/2.)*2.*VJET(I)*PITCH(I)/(6.*RPM)
TAPFA=TAPFA+AREFATH
C
C CHECK WHETHER THE PISTON CUP IS FULL WITH JET ELEMENTS OR NOT
C

```

```

      IF (TAREA.GT.ARF(4) ) GO TO 2090
      RTAREA=ARF(2)+ARF(3)+ARF(5)+ARF(6)+ARF(7)
C
C  PISTON CUP IS NOT FILLED WITH JET ELEMENTS
      DHLOS(I)=HTCON1*ETEMP(1,I)**(-0.53)*HTCO2(4)*ARFATH*
      1 (ETEMP(1,I)-TMETAL(4))
      QHTPC(4)=QHTRC(4)+DHLOS(I)
      QHTPC(1)=QHTPC(1)+DHLOS(I)
      GO TO 2900
C
C  PISTON CUP IS FILLED WITH JET ELEMENTS
2090 IFILL=1
2100 AA=ARFATH*HTCON1*ETEMP(1,I)**(-0.53)
      RTAREA=ARE(2)+ARE(3)+ARE(5)+ARE(6)+ARE(7)
2110 DO 2110 JJ=2,NDIVID
      IF(IFILL.EQ.0) GO TO 2112
      IF(JJ.EQ.4) GO TO 2110
2112 HTI(JJ)=AA*ARF(JJ) /RTAREA*HTCO2(JJ)*(ETEMP(1,I)-TMETAL(JJ))
      QHTPC(JJ)=QHTPC(JJ)+HTI(JJ)
      DHLOS(I)=DHLOS(I)+HTI(JJ)
      QHTPC(1)=QHTPC(1)+HTI(JJ)
2110 CONTINUE
      GO TO 2900
C
C  FOR ELEMENT 1
2120 XX=ARF(4)
      IF(IFILL.EQ.0) GO TO 2114
      AA=(ARE(2)+ARE(3)+ARE(4)+ARE(5)+ARE(6)+ARE(7)-TARFA)*HTCON1*
      1 ETEMP(1,I)**(-0.53)
      RTAREA=ARE(2)+ARE(3)+ARE(5)+ARE(6)+ARE(7)
      GO TO 2116
2114 RTAREA=ARE(2)+ARE(3)+ARE(5)+ARE(6)+ARE(7)+ARE(4)-TARFA
      XX=ARF(4)
      ARE(4)=ARE(4)-TARFA
      AA=RTARFA*HTCON1*ETEMP(1,I)**(-0.53)
      GO TO 2116
2900 CONTINUE
      ARF(4)=XX
      GO TO 9000
1900 HTCO1=HTCON1*ETEMP(1,I)**(-0.53)
      DO 1900 I=2,NDIVID
      QHTPC(I)=HTCO1*HTCO2(I)*(ETEMP(1,I)-TMETAL(I))*ARE(I)
      QHTPC(1)=QHTPC(1)+QHTPC(I)
1900 CONTINUE
      DHLOS(1)=QHTPC(1)
9000 RETURN
      END

```

```

SUPROUTINE DELTA(DVCYL, FNTINT)
COMMON/PROP/ IECHF(100), EFJF(100), FANGI(100), FMWT(100),
1 DHFUFL(100), DHLOS(100), DIEMAS(100), DOEMAS(100),
1 FBGAS(100), FCP(100), FENTH(8,100), FMASS(8,100), FEQUI(8,100),
1 FVOL(8,100), FTEMP(8,100), PCYL(8)
COMMON /STATE/ ANGLE, ANGLE, ICOND, ASTEP, NSTEP, IE
COMMON/FXFLOW/NXFF, XEXF(20), YEXF(20), AREXD, PEXT, TEXT, GFXT,
1 ILEFT, TLEFT, OFXT, OFLOW, FLEFT, ARFEXT, XEXV
DIMENSION DQ(100), HENTH(100)
AJ=42.68
DO 50 I=1,IE
DQ(I)=DHFUFL(I)-DHLOS(I)
HENTH(I)=FENTH(1,I)
50 CONTINUE
FNTCYL=HENTH(1)
AA=DQ(1)+QFLOW -(DIEMAS(1)-
1 DOEMAS(1))*HENTH(1)
AA=AA*FVOL(1,1)/(EMASS(1,1)*FCP(1)*FTEMP(1,1))+(DIEMAS(1)
1 -DOEMAS(1))/FMASS(1,1)*FVOL(1,1)
RR=FVOL(1,1)*FVOL(1,1)/(EMASS(1,1)*FCP(1)*FTEMP(1,1)*A)
1-FVOL(1,1)/PCYL(1)
IF(IE.LT.2) GO TO 200
DO 100 I=2,IE
AA=AA+FVOL(I,1)/(EMASS(1,I)*FCP(I)*FTEMP(1,I))*(DQ(I)
1 +DIEMAS(I)*HENTH(I) -DOEMAS(I)*HENTH(I) -(DIEMAS(I)
2 -DOEMAS(I))*HENTH(I)) +(DIEMAS(I)-DOEMAS(I))/EMASS(1,I)
3 *FVOL(1,I)
RR=RR+FVOL(1,I)*FVOL(I,I) / (EMASS(1,I)*FCP(I)
1 *FTEMP(1,I)*A)-FVOL(1,I)/PCYL(1)
100 CONTINUE
200 PCYL(3)=(DVCYL-AA)/RR
FVOL(3,1)=FVOL(1,1)/(FMASS(1,1)*FCP(1)*FTEMP(1,1)
1 *(DQ(1)+QFLOW -(DIEMAS(1)
2 -DOEMAS(1))*HENTH(1)) +(DIEMAS(1)-DOEMAS(1))/FMASS(1,1)
3 *FVOL(1,1)+(FVOL(1,1)*FVOL(1,1))/(FMASS(1,1)*FCP(1)*FTEMP(1,1)
4 *A)-FVOL(1,1)/PCYL(1))*PCYL(3)
FTEMP(3,1)=FTEMP(1,1)*(PCYL(3)/PCYL(1)+FVOL(3,1)/FVOL(1,1)
1 -(DIEMAS(1)-DOEMAS(1))/EMASS(1,1))
FMASS(3,1)=DIEMAS(1)-DOEMAS(1)
FENTH(3,1)=FCP(1)*FTEMP(3,1)
IF(IE.LT.2) GO TO 400
DO 300 I=2,IE
FVOL(3,I)=FVOL(1,I)/(FMASS(1,I)*FCP(I)*FTEMP(1,I)
1 *(DQ(I)+DIEMAS(I)*HENTH(I) -DOEMAS(I)*HENTH(I) -(DIEMAS(I)-
2 DOEMAS(I))*HENTH(I)) +(DIEMAS(I)-DOEMAS(I))/FMASS(1,I)
3 *FVOL(1,I)+(FVOL(1,I)*FVOL(1,I))/(FMASS(1,I)*FCP(I)*FTEMP(1,I)*A)
4 -FVOL(1,I)/PCYL(1))*PCYL(3)
FTEMP(3,I)=FTEMP(1,I)*(PCYL(3)/PCYL(1)+FVOL(3,I)/FVOL(1,I)-
1 (DIEMAS(I)-DOEMAS(I))/FMASS(1,I))
FMASS(3,I)=DIEMAS(I)-DOEMAS(I)
FENTH(3,I)=FCP(I)*FTEMP(3,I)
300 CONTINUE
400 RETURN
END

```

FUNCTION ENT(T,E,XXX)

C
 C TO CALCULATE THE ENTHALPY OF GAS AT LOW PRESSURE (T<1800 K)
 C THIS SUBROUTINE IS USED FOR CALCULATING THE ENTHALPY OF INITIAL GAS
 C INSIDE THE CYLINDER, INTAKE AIR AND EXHAUST GAS.
 C IT IS RECOMMENDED THAT W. MARTIN'S PROGRAMS BE USED FOR HIGH PRESSURE
 C OR HIGH TEMPERATURE GASES.
 C

```

    SAFC=14.4
    IF (E.LT.0.01) GO TO 100
    AF=SAFC/E
    GO TO 200
100 AF=0.
200 CONTINUE
    IF (T-200.) 10,10,20
10 ENT =-0.405671 +0.243753*T -0.164508E-4*T**2 +0.157984E-7
   1 *T**3+0.253172E-10*T**4 -0.179767E-13*T**5
    IF (AF) 99,29,40
40 ENT =ENT +(14.9607 -0.126357*T +0.769710E-3*T**2-0.517585E-6
   1 *T**3 +0.595961E-10*T**4 +0.833377E-13*T**5)/(AF+1.)
    RETURN
20 ENT =11.3176 +0.190755*T +0.637625E-4*T**2 -0.182173E-7*T**3
   1 +0.223445E-11*T**4
    IF (AF) 99,09,30
30 ENT =ENT +(-42.2372 +0.199892*T +0.871219E-4*T**2 +0.600828E-7
   1 *T**3 -0.299545E-10*T**4 +0.391857E-14*T**5)/(AF+1.)
99 RETURN
    END
  
```

```

C***** VERSION 1.0 *** 5/29/74 *****
C
C   SUBROUTINE HPROD
C
C   PURPOSE:
C     TO CALCULATE THE SPECIFIC ENTHALPY OF THE PRODUCTS OF HYDRO
C     CARBON-AIR COMBUSTION AS A FUNCTION OF TEMPERATURE AND PRES
C     SURE, USING AN APPROXIMATE CORRECTION FOR DISSOCIATION.
C     THE PARTIAL DERIVATIVES OF H WITH RESPECT TO THESE VARIABLES
C     ARE ALSO CALCULATED, ALONG WITH THE GAS DENSITY AND ITS PAR
C     TIAL DERIVATIVES
C
C   USAGE:
C     CALL HPROD(P,T,PHI,DEL,PSI,ENTHLP,CSURP,CSURT,RHO,DRHODT,
C              DRHODP)
C
C   DESCRIPTION OF PARAMETERS:
C   GIVEN:
C     P   - ABSOLUTE PRESSURE OF PRODUCTS (ATM)
C     T   - TEMPERATURE OF PRODUCTS (DEG K)
C     PHI - EQUIVALENC RATIO (FUEL/AIR RATIO DIVIDED BY THE
C           CHEMICALLY CORRECT FUEL/AIR RATIO)
C     DEL - MOLAR C:H RATIO OF THE PRODUCTS
C     PSI - MOLAR N:O RATIO OF THE PRODUCTS
C   RETURNS:
C     ENTHLP- SPECIFIC ENTHALPY OF THE PRODUCTS (KCAL/G)
C     CSURP - PARTIAL DERIVATIVE OF ENTHLP WITH RESPECT TO T
C             AT CONSTANT P (CAL/G-DEG K)
C     CSURT - PARTIAL DERIVATIVE OF ENTHLP WITH RESPECT TO P
C             AT CONSTANT T (CC/G)
C     RHO   - DENSITY OF THE PRODUCTS (G/CC)
C     DRHODT- PARTIAL DERIVATIVE OF RHO WITH RESPECT TO T AT
C             CONSTANT P (G/CC-DEG K)
C     DRHODP- PARTIAL DERIVATIVE OF RHO WITH RESPECT TO P AT
C             CONSTANT T (G/CC-ATM)
C
C   REMARKS:
C     1) ENTHALPY DATUM STATE IS AT T = 0 ABSOLUTE WITH
C        O2,N2,H2 GASFOUS AND C SOLID GRAPHITE
C     2) IN CASE OF PROBLEMS CONTACT MIKE MARTIN AT 253-2411
C        (ROOM 3-339 D)
C
C   SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED:
C     DERIVS,CLDPRD
C
C   METHOD:
C     SEE MARTIN & HEYWOOD 'APPROXIMATE RELATIONS FOR THE THERMO-
C     DYNAMIC PROPERTIES OF HYDROCARBON-AIR COMBUSTION PRODUCTS'
C*****
C
C   SUBROUTINE HPROD(P,T,PHI,DEL,PSI,ENTHLP,CSURP,CSURT,RHO,DRHODT,
C   DRHODP,AMWT,RAVG,CVAVG,ENERGY)
C   LOGICAL RICH,LEAN,NOTHOT,NOTWRM,NOTC_D
C
C   INITIALIZE PARAMETERS USED IN THE CALCULATION
C
C   DATA AHFCO2,AHFH2O,AHFCO/-93.965,-57.103,-27.200/
C   DATA ROVR2/.99745E-3/
C   DATA TCOLD,THOT /1000.,1100./

```

```

C
RICH = PHI .GE. 1.0
LEAN = .NOT. RICH
NOTHOT = T .LT. THOT
NOTCLD = T .GT. TCOLD
NOTWRM = .NOT. (NOTCLD .AND. NOTHOT)
EPS=(4.*DEL)/(1. + 4.*DEL)

C
C
C   USE SIMPLE ROUTINE FOR LOW TEMPERATURE MIXES
C
C   IF (NOTCLD) GO TO 5
CALL CLOPRD(P,T,PHI,DEL,PSI,ENTHLP,CSURP,CSURT,RHO,DRHODT,
1  DRHODP,IER,AMWT)
GO TO 30

C
C   CALCULATE EQUILIBRIUM CONSTANTS FOR DISSOCIATION.
C   (NOTE THAT UNITS ARE INVERSE PRESSURE TO THE 1/2 POWER)
C
5 AK1 = .39E-4 * EXP(-.3*FPS + 34000./T)
AK2 = .14E-3 * EXP(1.3*FPS + 29000./T)

C
C   CALCULATE A, X, Y, AND U AS IN NOTES
C
A = ((2.- FPS + PSI)/(4.*P*AK1*AK1))**.33333333)

C
T1 = 2.- EPS + PSI
T2 = 1. + 2.*T1
T3 = EPS*A
X = A*(3.*T1 + T2*T3)/(3.*(1.+ 2.*T3)*T1 + 2.*T2*T3*T3)

C
Z = (1.- PHI)/X
IF (RICH) Y = X/(1.- .64*Z + .30*Z**2)
IF (LEAN) Y = X*(1.+ 7 + .36*Z**2)/(1. + .36*Z) - (1. - PHI)
U = (2. - FPS + PSI)*(1.- 2.*FPS*X)/(4.*AK1*AK2*P*X)

C
C   CALCULATE THE ENTHALPY OF FORMATION FOR THIS APPROXIMATE
C   COMPOSITION
C
ENTFOR = 1000.*R0VR2*((117. + 30.*FPS)*Y + 135.*EPS*U)
XH2O = 2.*(1.- FPS)*PHI
T1 = 7.*PSI + 5.*Y + 3.*J
T2 = PSI - 3.*Y - 11

C
IF (LEAN) GO TO 10

C
RCVT = 2. + 2.*(7. - 4.*EPS)*PHI + T1
RCVV = 4. + (2.- 3.*FPS)*PHI + T2
XC02 = 2.- (2.- FPS)*PHI
XC0 = 2.*(PHI - 1.)
ENTFOR = ENTFOR - 1000.*R0VR2*13.*(PHI - 1.)/FP.
GO TO 20

C
10 RCVT = 7. + (9. - 8.*EPS)*PHI + T1
RCVV = 1. + (5.- 3.*FPS)*PHI + T2
XC0 = 0.
XC02 = EPS*PHI

C
20 ENTFOR = ENTFOR + (XC02*AHFC02 + XH2O)*AHFH2O + XC0*AHFC0)

C
C   ADD IN TRANSLATIONAL, VIBRATIONAL, AND ROTATIONAL TERMS TO GFT

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```

C   TOTAL ENTHALPY
C
C   TV = (3000.- 2000.*EPS + 300.*PSI)/(1.- .5*EPS + .09*PSI)
C   TV = TV/( EXP(TV/T) - 1.)
C   AMCP = (8.*EPS + 4.)*PHI + 32. + 28.*PSI
C
C   ENTHLP = (POVR2*(RCVT*T + RCVV*TV*2.) + ENTFOR)/AMCP
C
C   CALCULATE AVERAGE MOLECULAR WEIGHT, AND GET DENSITY BY
C   USING THE PERFECT GAS LAW
C
C   IF (LEAN)  AMWT = AMCP/(1. + (1.- EPS)*PHI + PSI + Y + U)
C   IF (RICH)  AMWT = AMCP/((2.- EPS)*PHI + PSI + Y + U)
C   RHO = .012187*AMWT*P/T
C
C   GET PARTIAL DERIVATIVES BY WAY OF A SUBROUTINE CALL
C
C   CALL DERIVS(P,T,PHI,EPS,PSI,A,X,Y,U,AMWT,CSURP,CSURT,DRHODT,
1   DRHODP)
C
C
C   IF CALCULATING FOR AN INTERMEDIATE TEMPERATURE, USE A WEIGHTED
C   AVERAGE OF THE RESULTS FROM THIS ROUTINE AND THOSE FROM THE
C   SIMPLE ROUTINE
C
C   IF (NOTWRM) GO TO 30
C
C   CALL CLDPRD(P,T,PHI,DFL,PSI,TH,TCP,TCT,TRHO,TDRT,TDRP,IER,AMWT)
C   W1 = (T - TCOLD)/(THOT - TCOLD)
C   W2 = 1.0 - W1
C
C   ENTHLP = W1*ENTHLP + W2*TH
C   CSURP = W1*CSURP + W2*TCP
C   CSURT = W1*CSURT + W2*TCT
C   RHO = W1*RHO + W2*TRHO
C   DRHODT = W1*DRHODT + W2*TDRT
C   DRHODP = W1*DRHODP + W2*TDRP
C
C   30  RAVG=R2.057/AMWT
C       CVAVG=CSURP-1.98/AMWT
C       ENERGY=1000.*ENTHLP-1.98*T/AMWT
C
C   RETURN
C   END

```

```

C***** VERSION 1.1 ***
C
C SUBROUTINE CLOPRD
C
C PURPOSE:
C TO CALCULATE THE SPECIFIC ENTHALPY OF THE PRODUCTS OF HC-A19
C COMBUSTION AT TEMPERATURES AND PRESSURES WHERE DISSOCIATION
C OF THE PRODUCT GASES MAY BE IGNORED. THE DENSITY OF THE
C PRODUCT GAS IS ALSO CALCULATED, AS ARE THE PARTIAL
C DERIVATIVES OF BOTH OF THESE QUANTITIES WITH RESPECT TO
C PRESSURE AND TEMPERATURE.
C
C USAGE:
C CALL CLOPRD(P,T,PHI,DEL,PSI,ENTHLP,CSUBP,CSUBT,RHO,DRHODT,
C DRHODP,IER)
C
C DESCRIPTION OF PARAMETERS:
C GIVEN:
C P - ABSOLUTE PRESSURE OF PRODUCTS (ATM)
C T - TEMPERATURE OF PRODUCTS (DEG K)
C PHI - EQUIVALENCE RATIO (FUEL/AIR RATIO DIVIDED BY THE
C CHEMICALLY CORRECT FUEL/AIR RATIO)
C DEL - MOLAR C:H RATIO OF THE PRODUCTS
C PSI - MOLAR N:O RATIO OF THE PRODUCTS
C RETURNS:
C ENTHLP- SPECIFIC ENTHALPY OF THE GAS (KCAL/G)
C CSUBP - PARTIAL DERIVATIVE OF ENTHLP WITH RESPECT TO T
C AT CONSTANT P (CAL/G-DEG K)
C CSUBT - PARTIAL DERIVATIVE OF ENTHLP WITH RESPECT TO P
C AT CONSTANT T (CC/G)
C RHO - DENSITY OF THE MIXTURE (G/CC)
C DRHODT- PARTIAL DERIVATIVE OF RHO WITH RESPECT TO T AT
C CONSTANT P (G/CC-DEG K)
C DRHODP- PARTIAL DERIVATIVE OF RHO WITH RESPECT TO P AT
C CONSTANT T (G/CC-ATM)
C IER - FLAG, SET TO 1 FOR T<100 DEG K
C 2 FOR T> 6000 DEG K
C 0 OTHERWISE
C
C REMARKS:
C 1) ENTHALPY DATUM STATE IS AT T = 0 ABSOLUTE WITH
C O2,N2,H2 GASEOUS AND C SOLID GRAPHITE
C 2) IN CASE OF PROBLEMS CONTACT MIKE MARTIN AT 253-2411
C (ROOM 3-339 D)
C
C SUBROUTINES AND FUNCTION SUBPROGRAMS NEEDED: NONE
C
C METHOD:
C DESCRIBED IN APPENDIX IV OF WRITEJP
C*****
C SUBROUTINE CLOPRD(P,T,PHI,DEL,PSI,ENTHLP,CSUBP,CSUBT,RHO,
C 1 DRHODT,DRHODP,IER,MBAR)
C
C LOGICAL RICH,LEAN
C DIMENSION A(6,6,2),X(6)
C DIMENSION A1(36),A2(36)

```

EQUIVALENC (A1(1),A(1,1,1)),(A2(1),A(1,1,2))
 REAL*4 MRAR,K

C
 C
 C
 C
 C

INITIALIZE PARAMETERS, AND CHECK TO SEE IN WHAT TEMPERATURE
 RANGE WE ARE SO THAT THE CORRECT FITTED COEFFICIENTS WILL BE
 USED. FLAG TEMPERATURES TOO BIG OR TOO SMALL

DATA A1/11.94033,2.088581,-0.47029,.037363,-.589447,-97.1418,
 1 6.139094,4.60783,-.9356009,.06669498,.0335801,-56.62588,
 2 7.099556,1.275957,-.2877457,.022355,-.1598696,-27.73464,
 3 5.555680,1.787191,-.2881342,.01951547,.1611828,.76498,
 4 7.865847,.6883719,-.031944,-.00268708,-.2013873,-.893455,
 5 6.807771,1.453404,-.328985,.02561035,-.1199462,-.331835/
 DATA A2/4.737305,16.65283,-11.23249,2.828001,.00676702,-93.75793,
 7 7.809672,-.2023519,3.418708,-1.179013,.00143629,-57.08004,
 8 6.97393,-.8238319,2.942042,-1.176239,.0004132409,-27.19597,
 9 6.991878,.1617044,-.2182071,.2968197,-.01625234,-.118189,
 6 6.295715,2.388387,-.0314788,-.3267433,.00435925,.103637,
 - 7.092199,-1.295825,3.20688,-1.202212,-.0003457939,-.013967/

C

RICH = PHI .GT. 1.0
 LEAN = .NOT. RICH
 EPS = 4.*DFL/(1. + 4.*DFL)
 IFR = 0
 IF (T .LT. 100.) IFR = 1
 IF (T .GT. 6000.) IFR = 2
 IR = 1
 IF (T .LT. 500.) IR = 2

C
 C
 C

GET THE COMPOSITION IN MOLES/MOLE OXYGEN

IF (RICH) GO TO 10
 Y(1) = EPS*PHI
 X(2) = 2.*(1.- EPS)*PHI
 X(3) = 0.
 X(4) = 0.
 X(5) = 1.- PHI
 GO TO 20
 10 T = 1000./T
 K = EXP(2.743 + 7*(-1.761 + 2*(-1.611 + 7*.2803)))
 ALPHA = 1. - K
 BETA = (2.*(1.- EPS*PHI) + K*(2.*(PHI - 1.) + EPS*PHI))
 GAMMA = 2.*K*EPS*PHI*(PHI - 1.)
 C = (- BETA + SORT(BETA*BETA + 4.*ALPHA*GAMMA))/(2.*ALPHA)
 X(1) = EPS*PHI - C
 X(2) = 2.*(1. - EPS*PHI) + C
 X(3) = C
 X(4) = 2.*(PHI - 1.) - C
 X(5) = 0.
 20 X(6) = PSI

C
 C
 C
 C

CONVERT COMPOSITION TO MOLE FRACTIONS AND CALCULATE AVERAGE
 MOLECULAR WEIGHT

IF (LEAN) TMOLES = 1. + PSI + PHI*(1.-EPS)
 IF (RICH) TMOLES = PSI + PHI*(2.-EPS)
 DO 30 J = 1,6
 30 X(J) = X(J)/TMOLES
 MRAR = ((8.*EPS + 4.)*PHI + 32. + 28.*PSI)/TMOLES

C

C CALCULATE H, CP, AND CT AS IN WRITEUP. USING FITTED
 C COEFFICIENTS FROM JANAF TABLES
 C

```

ENTHLP = 0.
CSURP = 0.
CSURT = 0.
ST = T/1000.
DO 40 J = 1,6
  TH = ((( A(4,J,IR)/4.*ST + A(3,J,IR)/3. ) *ST
1      + A(2,J,IR)/2. ) *ST + A(1,J,IR) ) *ST
  TCP = (( A(4,J,IR)*ST + A(3,J,IR) ) *ST
1      + A(2,J,IR) ) *ST + A(1,J,IR)
  TH = TH - A(5,J,IR)/ST + A(6,J,IR)
  TCP = TCP + A(5,J,IR)/ST**2
  ENTHLP = ENTHLP + TH*X(J)
40  CSURP = CSURP + TCP*X(J)
  ENTHLP = ENTHLP/MBAR
  CSURP = CSURP/MBAR

```

C NOW CALCULATE RHO AND ITS PARTIAL DERIVATIVES
 C USING PERFECT GAS LAW
 C

```

RHO = .012187*MBAR*P/T
DRHODT = -RHO/T
DRHODP = RHO/P

```

C ALL DONE
 C RETURN
 C END

```

SUBROUTINE DERJVS(P,T,PHI,EPS,PSI,A,X,Y,U,AMWT,CSUBP,CSURT,
1  DRHONT,DRHONP)
LOGICAL RICH,LFAN
DATA ROVR2/.99745/
DATA SCALF/41.29287/
C
RICH = PHI .GE. 1.0
LFAN = .NOT. RICH
C
C3 = (117. + 30.*EPS)*1000.
C4 = 1.35E5*EPS
C5 = 2.0 - FPS + PSI
C6 = 5.0 - 2.*FPS + 2.*PSI
C
DUJTPX = 6.3E4*U/T**2
DUJPTX = -U/P
DUJXPT = -U/(X*(1. - 2.*EPS*X))
C
DADTP = (3.4E4*2./3.)*A/T**2
DADPT = -A/(3.*P)
C
AP = EPS*A
T5 = 3.*C5
DXDA = T5*(T5 + 2.*C6*AP)/(T5*(1. + 2.*AP) + 2.*C6*AP**2)**2
C
Z = (1. - PHI)/X
IF (LFAN) DYDX = (1. + .72*Z)/(1. + .36*Z)**2
IF (RICH) DYDX = (1. - 1.25*Z + .90*Z**2)/(1. - .64*Z + .3*Z**2)**2
C
DYDTP = DYDX*DXDA*DADTP
DYDPT = DYDX*DXDA*DADPT
DUJTP = DUJXPT*DXDA*DADTP + DUJTPX
DUJPT = DUJXPT*DXDA*DADPT + DUJPTX
C
DHFDPT = C3*DYDPT + C4*DUJPT
DC2DPT = -2.*(3.*DYDPT + DUJPT)
DC1DPT = 5.*DYDPT + 3.*DUJPT
DHFDTP = C3 * DYDTP + C4*DUJTP
DC2DTP = -2.*(3.*DYDTP + DUJTP)
DC1DTP = 5.*DYDTP + 3.*DUJTP
C
TV0 = (3000. - 2000.*FPS + 300.*PSI)/(1. - .5*EPS + .09*PSI)
FARG = EXP(TV0/T)
TV = TV0/(FARG - 1.)
DTVJTP = TV0*FARG/(T*(FARG - 1.))**2 *TV0
C
AMCP = (8.*EPS + 4.)*PHI + 32. + 28.*PSI
C1 = 7.*PSI + 5.*Y + 3.*J
C2 = 2.*(PSI - 3.*Y - U)
IF (LFAN) C1 = C1 + 7. + (9. - 8.*FPS)*PHI
IF (RICH) C1 = C1 + 2. + 2.*(7. - 4.*EPS)*PHI
IF (LFAN) C2 = C2 + 2.*(1. + (5. - 3.*FPS)*PHI)
IF (RICH) C2 = C2 + 2.*(4. + (2. - 3.*EPS)*PHI)
C
CSUBP = ROVR2/AMCP*(C1 + T*DC1DTP + C2*DTVJTP + TV*DC2DTP
1  + DHFDTP)
CSURT = ROVR2/AMCP*(T*DC1DPT + TV*DC2DPT + DHFDPT)*SCALF
C
IF (LFAN) S = 1. + (1.-FPS)*PHI + PSI + Y + U
IF (RICH) S = (2.- FPS)*PHI + PSI + Y + U

```

```
G = -AMCP/R**2
DMOTP = G*(DYDTP + DUOTP)
DMOPT = G*(DYOPT + DUOPT)
C
DRHODT = .012187*P/T*(DMOTP - AMWT/T)
DRHODP = .012187/T*(AMWT + P*DMOPT)
C
RETURN
END
```

```

SUBROUTINE INPOR
C
C INTAKE PORT CALCULATION
C THIS PROGRAM WAS ORIGINALLY MADE USING METER-KG UNIT
COMMON/EPROP/ TECHF(100),EFUEL(100),EANGL(100),EMWT(100),
1 DHFUEL(100),DHLOS(100),DIEMAS(100),DOEMAS(100),
1 ERGAS(100),ECP(100),FENTH(9,100),FMASS(R,100),FEQUI(R,100),
1 EVOL(R,100),FTEMP(R,100),PCYL(R)
COMMON/INFLO/ NINEF,XINEF(20),YINEF(20),AREIND,PINT,TINT,GINT,
1 EQUINT,AREINT,XINV
COMMON /STATE/ ANGLE1,ANGLE,JCOND,ASTEP,NSTEP,IE
COMMON /TIMIG/ AINT0,AEXT0,AINTC,AEXTC,AFUELS,AFUELE
COMMON /STAT1/ RPM,JLAST,NJET,PHIO,ESR
C
C IF(PINT-PCYL(1)) 50,50,60
C
C INTAKE FLOW IS FROM CYLINDER TO PORT (BACK FLOW)
50 CINT=-1.
RHIGH=ERGAS(1)*9.8
PHIGH=PCYL(1)*10000.
PRIN=PINT/PCYL(1)
THIGH=FTEMP(1,1)
AKHIGH=1.35
GO TO 100
C
C INTAKE FLOW IS FROM PORT TO CYLINDER
C P VALUE DOESN'T CHANGE MUCH.
60 RINT=ERGAS(1)*9.8
CINT=1.
RHIGH=RINT
PHIGH=PCYL(1)*10000.
PRIN=PCYL(1)/PINT
THIGH=TINT
AKHIGH=1.35
C
C CALCULATE THE EFFECTIVE INTAKE FLOW AREA
100 ANAI=(ANGLE-AINT0)/(AINTC-AINT0)
CALL TWODIM(NINEF,XINEF,YINEF,ANAI,AREFAC)
AREINT=AREIND*AREFAC
AREINT=AREINT/10000.
C
C NEXT EQUATION IS BASED ON METER-KG UNIT
GINT= SQRT(ABS(2.*AKHIGH*9.8/RHIGH/THIGH
1 /((AKHIGH-1.)*(PRIN*(2./AKHIGH)-PRIN*((AKHIGH+1.)
2 /AKHIGH))))
GINT=PHIGH*AREINT/6./RPM*GINT
GINT=GINT*CINT
GINT=GINT*1000.
AREINT=AREINT*10000.
RETURN
END

```

```

C      SUBROUTINE UPROP                                UPRP  30
C
C      PURPOSE:                                       UPRP  40
C      TO CALCULATE THE ENTHALPY AND DENSITY OF A HOMOGENEOUS MIXTURE UPRP  50
C      OF AIR, RESIDUAL GAS, AND FUEL AS A FUNCTION OF UPRP  60
C      EQUIVALENCE RATIO, TEMPERATURE, AND PRESSURE UPRP  70
C
C      USAGE:                                         UPRP  80
C      CALL UPROP(P,T,PHI,DEL,PSI,RESFRK,ENTHLP,CSURP,CSURT,RHO, UPRP  90
C      DRHODT,DRHODP,CHI)                             UPRP 100
C
C      DESCRIPTION OF PARAMETERS:                     UPRP 110
C      GIVEN:                                         UPRP 120
C      P      - ABSOLUTE PRESSURE OF PRODUCTS (ATM)   UPRP 130
C      T      - TEMPERATURE OF PRODUCTS (DEG K)      UPRP 140
C      RESFRK- RESIDUAL GAS FRACTION                 UPRP 150
C      PHI    - EQUIVALENCE RATIO (FUEL/AIR RATIO DIVIDED BY THE UPRP 160
C      CHEMICALLY CORRECT FUEL/AIR RATIO)           UPRP 170
C      GIVEN IN COMMON AREA /FUEL/:                  UPRP 180
C      AF(I)  - 6 DIMENSIONAL VECTOR OF ENTHALPY COEFFICIENTS SUCH UPRP 190
C      THAT THE ENTHALPY OF FUEL VAPOR AS A FUNCTION UPRP 200
C      OF TEMPERATURE ( T DEG K ) IS GIVEN BY:      UPRP 210
C       $H(T) = AF(1)*ST + (AF(2)*ST**2)/2 + (AF(3)*ST**3)/3$  UPRP 220
C       $+ (AF(4)*ST**4)/4 - AF(5)/ST + AF(6)$            UPRP 230
C      WHERE ST = T/1000 AND H(T) = <KCAL/MOLE>      UPRP 240
C      FOR MOST APPLICATIONS THE ENTHALPY FUNCTION H(T) SHOULD UPRP 250
C      BE VALID OVER AT LEAST THE FOLLOWING TEMPERATURE RANGE: UPRP 260
C       $300 < T < 1000$                                UPRP 270
C      ENTHALPY DATUM STATE IS AT T = 0 ABSOLUTE WITH O2,N2, UPRP 280
C      AND H2 GASFOUS AND C SOLID GRAPHITE.          UPRP 290
C      ENW    - AVERAGE NUMBER OF NITROGEN ATOMS PER FUEL MOLECULE UPRP 300
C      CX     - AVERAGE NUMBER OF CARBON ATOMS PER FUEL MOLECULE UPRP 310
C      HY     - AVERAGE NUMBER OF HYDROGEN ATOMS PER FUEL MOLECULE UPRP 320
C      OZ     - AVERAGE NUMBER OF OXYGEN ATOMS PER FUEL MOLECULE UPRP 330
C      QLOWER- LOWER HEATING VALUE (KCAL/G)          UPRP 340
C      XI     - MOLAR N:O RATIO OF THE OXIDANT (FOR AIR XI = 3.76) UPRP 350
C
C      RETURNS:                                       UPRP 360
C      DEL    - MOLAR C:H RATIO OF THE PRODUCTS      UPRP 370
C      PSI    - MOLAR N:O RATIO OF THE PRODUCTS      UPRP 380
C      CHI    - EQUIVALENCE RATIO OF THE PRODUCTS FOR AN EQUIVALENT UPRP 390
C      HYDROCARBON-OXIDANT COMBUSTION                UPRP 400
C      ENTHLP- SPECIFIC ENTHALPY OF THE PRODUCTS (KCAL/G) UPRP 410
C      CSURP  - PARTIAL DERIVATIVE OF ENTHLP WITH RESPECT TO T UPRP 420
C      AT CONSTANT P (CAL/G-DEG K)                   UPRP 430
C      CSURT  - PARTIAL DERIVATIVE OF ENTHLP WITH RESPECT TO P UPRP 440
C      AT CONSTANT T (CC/G)                           UPRP 450
C      RHO    - DENSITY OF THE PRODUCTS (G/CC)        UPRP 460
C      DRHODT- PARTIAL DERIVATIVE OF RHO WITH RESPECT TO T AT UPRP 470
C      CONST NT P (G/CC-DEG K)                       UPRP 480
C      DRHODP- PARTIAL DERIVATIVE OF RHO WITH RESPECT TO P AT UPRP 490
C      CONSTANT T (G/CC-ATM)                          UPRP 500
C      RETURNS IN COMMON AREA CMPSTN:                 UPRP 510
C      X(1)   - CARBON DIOXIDE MOLE FRACTION          UPRP 520
C      X(2)   - WATER VAPOR MOLE FRACTION             UPRP 530
C      X(3)   - CARBON MONOXIDE MOLE FRACTION        UPRP 540
C      X(4)   - HYDROGEN MOLE FRACTION                UPRP 550
C      X(5)   - OXYGEN MOLE FRACTION                 UPRP 560
C      X(6)   - NITROGEN MOLE FRACTION                UPRP 570
C      X(7)   - FUEL MOLE FRACTION                   UPRP 580
C

```


C
C
C

GET THE COMPOSITION IN MOLES/MOLE OXYGEN OF OXIDANT

```

PCTRES = RESFRK
PCTNEW = 1.0 - RESFRK
IF (RICH) GO TO 10
X(1) = EPS*PHI*PCTRES
X(2) = (2.*(1. - EPS) + EPS*7)*PHI*PCTRES
X(3) = 0.
X(4) = 0.
X(5) = (1. - PHI)*PCTRES + PCTNEW
DCDT = 0.
GO TO 20
10 ZT = 1000./T
K = EXP(2.743 + ZT*(-1.751 + ZT*(-1.511 + ZT*.2A03)))
DKDT = -K*ZT*(-1.761 + ZT*(-3.222 + ZT*.9409))/T
ALPHA = 1.0 - K
RETA = 2.*(1. - EPS*PHI) + K*(2.*(PHI - 1.) + EPS*PHI)
BETA = RETA + EPS*PHI*7
GAMMA = 2.*K*EPS*PHI*(PHI - 1.)
C = (-BETA + SQRT(RETA*RETA + 4.*ALPHA*GAMMA))/(2.*ALPHA)
DCDT = -DKDT*(C*C + (2.*(PHI - 1.) + EPS*PHI)*C - GAMMA/K)
DCDT = DCDT/(2.*ALPHA*C + BETA)
X(1) = (EPS*PHI - C)*PCTRES
X(2) = (2.0*(1. - EPS*PHI) + EPS*PHI*7 + C)*PCTRES
X(3) = C*PCTRES
X(4) = (2.0*(PHI - 1.) - C)*PCTRES
X(5) = PCTNEW
20 X(6) = XI + EPS*PHI*W/2.*PCTRES
X(7) = PCTNEW * EPS*PHI/CX

```

C
C
C
C

CONVERT COMPOSITION TO MOLE FRACTIONS AND CALCULATE AVERAGE
MOLECULAR WEIGHT

```

IF (LEAN) TMOLES = XI + (1. + EPS*PHI/CX)*PCTNEW
+ (1. + (1. - EPS)*PHI + EPS*PHI*(7 + W/2.))*PCTRES
IF (RICH) TMOLES = XI + (1. + EPS*PHI/CX)*PCTNEW
+ ((2. - EPS)*PHI + EPS*PHI*(7 + W/2.))*PCTRES
DO 30 J = 1,7
30 X(J) = X(J)/TMOLES
MBAR = EPS*PHI*(12. + 1./DEL + 16.*7 + 14.*W) + 32. + 28.*XI
MBAR = MBAR/TMOLES

```

C
C
C
C

CALCULATE H. CP. AND CT AS IN WRITEUP, USING FITTED
COEFFICIENTS FROM JANAF TABLES

```

ENTHLP = 0.
CSURP = 0.
CSURT = 0.
ST = T/1000.
DO 40 J = 1,7
1 TH = ((( A(4,J,IR)/4.*ST + A(3,J,IR)/3. )*ST
+ A(2,J,IR)/2.)*ST + A(1,J,IR) )*ST
TCP = (( A(4,J,IR)*ST + A(3,J,IR) )*ST
+ A(2,J,IR))*ST + A(1,J,IR)
1 TH = TH - A(5,J,IR)/ST + A(6,J,IR)
TCP = TCP + A(5,J,IR)/ST**2
ENTHLP = ENTHLP + TH*X(J)
40 CSURP = CSURP + TCP*X(J) + 1000.*TH*DCDT*PCTRES*TABLE(J)
ENTHLP = ENTHLP/MBAR

```

UPRP1250
UPRP1260
UPRP1270
UPRP1280
UPRP1290
UPRP1300
UPRP1310
UPRP1320
UPRP1330
UPRP1340
UPRP1350
UPRP1360
UPRP1370
UPRP1380
UPRP1390
UPRP1400
UPRP1410
UPRP1420
UPRP1430
UPRP1440
UPRP1450
UPRP1460
UPRP1470
UPRP1480
UPRP1490
UPRP1500
UPRP1510
UPRP1520
UPRP1530
UPRP1540
UPRP1550
UPRP1560
UPRP1570
UPRP1580
UPRP1590
UPRP1600
UPRP1610
UPRP1620
UPRP1630
UPRP1640
UPRP1650
UPRP1660
UPRP1670
UPRP1680
UPRP1690
UPRP1700
UPRP1710
UPRP1720
UPRP1730
UPRP1740
UPRP1750
UPRP1760
UPRP1770
UPRP1780
UPRP1790
UPRP1800
UPRP1810
UPRP1820
UPRP1830
UPRP1840

C	CSURP = CSURP/MRBP	IJRP1850
C		IJRP1860
C	NOW CALCULATE RHO AND ITS PARTIAL DERIVATIVES	UPRP1870
C	USING PERFECT GAS LAW	UPRP1880
C		IJRP1890
	RHO = .012187*MBAR*P/T	IJRP1900
	DRHODT = -RHO/T	IJRP1910
	DRHODP = RHO/P	IJRP1920
C		UPRP1930
C	CALCULATE PSI AND CHI FOR MIXED GASES	UPRP1940
C		UPRP1950
	PSI = (XI + EPS*PHI*W/2.)/(1. + EPS*Z*PHI/2.)	IJRP1960
	CHI = PHI*(1. + EPS*Z/2.)/(1. + EPS*Z*PHI/2.)	UPRP1970
C		
C		
	RAVG=82.057/MRBP	
	CVAVG=CSUBP*-1.98/MRBP	
C		IJRP1980
C	ALL DONE	UPRP1990
	RETURN	IJRP2000
	END	IJRP2010

```
SUBROUTINE EQIR(TTTT,PPPP,P-HIC,IFLAG,Z)
```

```
C  
C  
C
```

```
VARIABLES LIST
```

```
REAL*8 DPPATM,DPTEMP,DEQIB,XMOFR  
DIMENSION DEQIR(4),XMOFR(14),YMOFR(14),Z(5)  
C CALCULATE RATIOS OF H/C,N/C,O/C,AND C/C WHERE C=1.  
TEMP=TTTT  
PATM=PPPP  
SLR=15.1  
ERURN=1./P-HIC  
IF(TEMP.LT.1800.) GO TO 101  
IFLAG=0  
WHY=(12.*SLR-137.4)/(-SLR+.25*137.4)  
XWHY=1.+25*WHY  
DEQIB(1)=WHY  
DEQIB(2)=1.  
DEQIB(4)=2.*XWHY*ERURN  
DEQIB(3)=3.764*DEQIB(4)
```

```
C
```

```
C CALCULATE N AND NO EQIR. AND EQIB. MOLEFRACTIONS  
DPPATM=PATM  
DPTEMP=TEMP  
CALL PTCHEM(DPTEMP,DPPATM,DEQIB,XMOFR,ISENT)  
DO 100 I=1,14  
YMOFR(I)=XMOFR(I)  
100 CONTINUE  
GO TO 103  
101 IFLAG=1  
DO 102 J=1,14  
102 YMOFR(J)=0.  
103 Z(1)=YMOFR(2)  
Z(2)=YMOFR(5)  
Z(3)=YMOFR(8)  
Z(4)=YMOFR(9)  
Z(5)=YMOFR(14)  
RETURN  
END
```

SUBROUTINE DNONT(TTTT,PPPP,TOTMOL,ZMOFR,IFLAG,XNO,F)

C
C VARIABLE LIST

C
C F FINAL EVALUATION OF R.H.SIDE RATE EQUATION
C W,X,Y,Z DUMMY VARIABLES
C VOL VOLUME OF ELEMENT
C XNO AMOUNT OF NITRIC OXIDE
C RC1F RATE CONSTANT FOR FIRST REACTION, FORWARD
C RC1R RATE CONSTANT FOR FIRST REACTION, REVERSE
C RC2F RATE CONSTANT FOR SECOND REACTION, FORWARD
C RC2R RATE CONSTANT FOR SECOND REACTION, REVERSE
C TEMP TEMPERATURE OF ELEMENT
C RTEMP RECIPROCAL OF TEMP
C

DIMENSION ZMOFR(5)

TEMP=TTTT

PATM=PPPP

VOL=B2.06*TEMP*TOTMOL/PATM

IF(TEMP.LT.1800.) GO TO 2000

IF(IFLAG.GT.0) GO TO 2000

IF(TOTMOL.LF.0.0) GO TO 2000

C 4. COMPUTE RATE CONSTANTS (FUNCTIONS OF TEMPERATURE ONLY)

RTEMP=1./TEMP

RC1F=7.6E+13*EXP(-75460./RTEMP)

RC1R=1.6E+13

RC2F=6.4E+09*EXP(-6255./RTEMP)*TEMP

RC2R=1.5E+09*EXP(-38720./RTEMP)*TEMP

RCOH=4.1E+13

C

TMOLNO=ZMOFR(4)*TOTMOL

TMOLN=ZMOFR(3)*TOTMOL

XMOLCO=ZMOFR(1)

C 5. CALCULATE $dnO/dt = F(VOL, TEMP, PATM, ERURN, ZMOFR, XNO)$

ZR=RC1R*ZMOFR(4)

WR=RC2F*ZMOFR(5)+RCOH*ZMOFR(2)

CAPL=ZR/WR

ALPHA=XNO/TMOLNO

C

TERM1=2.*RC1R*TMOLN*TMOLNO/VOL*(1.-ALPHA*ALPHA)

TERM2=1.+ALPHA*CAPL

F=TERM1/TERM2

C

GO TO 3000

2000 F=0.

3000 CONTINUE

RETURN

END


```

C     FOLLOWING ARRAYS ARE 'NELEM' *2 = 'INXNSL'
C
C     DIMENSION R(6), G(6,6)
C
C     FOLLOWING ARRAYS ARE 'ITOTSP'
C
C     DIMENSION CMN(14), CP(14), CPT(14), X(14), XMAX(14)
C     DIMENSION HORT(14), SR(14), C(14)
C     DIMENSION YHORT(14)
C
C     COMMON / HORT / HORTSM,SMOHDY,XMU,RTP,SUMNY
C
C     NSAME = INITIALIZATION FLAG
C     NFLEM = NO OF ELEMENTS INVOLVED
C     ITOTSP = TOTAL NR OF SPECIES INVOLVED
C
C     DATA NSAME / 0 /
C     DATA NELEM / 4 /
C     DATA INXNSL / 6 /
C     DATA ITOTSP / 14 /
C
C     DATA DD,XNII,E,F / 28*0.000 /
C     DATA R, G / 42 * 0.000 /
C     DATA A, CP, CPT, X, XMAX / 112 * 0.000 /
C CMN IS NOT INITIALIZED WHEN IT LIES WITHIN ARRAY G
C
C     ARRAY ZL(K,J) CONTAINS THE HI-TEMP DATA FOR EACH OF 14 SPECIES
C
C     K = SPECIES NO ( 1 - 14 )
C     J = DATA ( ) - 8 EACH SPECIES )
C
C     DATA WAS TAKEN FROM ORIGINAL DATA SET ( CARDS), ARRANGED IN
C     ROW,COL (14,8), BUT DATA STATEMENT INTERNALLY STORES
C     BY COL,ROW: THEREFORE, DATA MUST BE REVERSED FOR PROG EXEC.
C     ORIGINAL DATA ARRANGMENT MAINTAINED FOR CONVENIENCE IN MAKING
C     CHANGES TO SPECIES DATA.
C
C     NR OF CONTINUATION CARDS LIMITED TO 19
C     CONTINUATION COL #6 CONTAINS THE HEXIDECIMAL SPECIES NO
C
C     DATA ZL1 /
1  9.3434439D 00, 2.9512196D 00,-6.7088366D-01, 5.0901942D-02,
1 -4.1140777D-01,-6.9903498D 00, 6.2335554D 01, 2.3009987D 00,
2  7.0995646D 00, 1.2759552D 00,-2.8774744D-01, 2.2756123D-02,
2 -1.5986955D-01,-2.9023636D 01, 5.4823700D 01, 1.2890015D 00,
3  1.1940331D 01, 2.0885811D 00,-4.7029203D-01, 3.7363116D-02,
3 -5.8944769D-01,-9.9468796D 01, 6.2207169D 01, 2.3269949D 00,
4  4.9679995D 00,-5.9678716D-13, 1.9345219D-13,-1.8732195D-14,
4 -7.4194935D-14, 5.0619217D 01, 3.3404419D 01, 1.0119925D 00,
5  5.6197815D 00, 1.9668446D 00,-3.8645178D-01, 2.7364515D-02,
5  1.3418680D-01, 8.0923529D 00, 5.0777405D 01, 2.0489998D 00,
6  5.5556803D 00, 1.7871914D 00,-2.8813416D-01, 1.9515470D-02,
6  1.6118270D-01,-1.2590199D 00, 3.8126053D 01, 2.0239983D 00,
7  6.1390944D 00, 4.6078291D 00,-9.3550094D-01, 6.6694975D-02,
7  3.3580098D-02,-5.9687856D 01, 5.1456772D 01, 7.0619965D 00 /
C     DATA ZL2 /
8  5.1632023D 00,-1.8977487D-01, 3.6921006D-02, 3.6241105D-03,
8 -3.2719156D-02, 1.1134157D 02, 4.2733066D 01, 1.0359793D 00,
9  7.5180626D 00, 1.0245209D 00,-2.3053735D-01, 1.7926671D-02,
9 -1.9369543D-01, 1.8756821D 01, 5.8378296D 01, 2.0729971D 00,

```

```

A 1.21232820 01, 1.25449930 00, -3.01172960-01, 2.36587490-02, 00002570
A -6.22562890-01, 2.38795980 00, 6.88109590 01, 3.11099820 00, 00002580
R 6.80776990 00, 1.45340350 00, -3.28985750-01, 2.56103460-02, 00002590
R -1.18946190-01, -2.40383530 00, 5.31551610 01, 2.07199860 00, 00002600
C 1.23659610 01, 1.72617820 00, -4.05288460-01, 3.14184350-02, 00002610
C -5.81208770-01, 1.41054700 01, 6.43314670 01, 3.10999970 00, 00002620
D 5.10063090 00, -1.51772200-01, 4.89531380-02, -2.88143520-03, 00002630
D 8.92994180-03, 5.80809480 01, 4.47525480 01, 1.03799440 00, 00002640
E 7.86584570 00, 6.88371900-01, -3.19441000-02, -2.68708170-03, 00002650
F -2.01387290-01, -2.96845440 00, 5.74246370 01, 2.07499890 00 / 00002660

```

C
C
C

```
IF (NSAME) 2000, 2000, 2004
```

```
2000 NSAME=1
```

```
AP=1.9872600
```

```
ITMAX = 500
```

```
ITW = 400
```

```
DIF = 15.000
```

```
DIF1 = DIF
```

```
T = 1.000
```

```
TLUB = 6000.
```

```
XN=N
```

```
TOL1 = .0100
```

```
TOL3 = .0000100
```

```
TOL5 = 1.00-5
```

```
TOL6 = .100
```

```
TOL7 = 10.000
```

C
C
C
C
C
C

```
ARRAY A(I,K) CONTAINS NR OF ATOMS PER ELEMENT ( IN SPECIES )
```

```
I = ELEMENTS H, C, N, O RESPECTIVELY
```

```
K = SPECIES NR
```

```
A(1,1) = 1.
```

```
A(1,4) = 1.
```

```
A(1,5) = 1.
```

```
A(1,6) = 2.
```

```
A(1,7) = 2.
```

```
A(2,1) = 1.
```

```
A(2,2) = 1.
```

```
A(2,3) = 1.
```

```
A(3,8) = 1.
```

```
A(3,9) = 1.
```

```
A(3,10) = 1.
```

```
A(3,11) = 2.
```

```
A(3,12) = 2.
```

```
A(4,1) = 1.
```

```
A(4,2) = 1.
```

```
A(4,3) = 2.
```

```
A(4,5) = 1.
```

```
A(4,7) = 1.
```

```
A(4,9) = 1.
```

```
A(4,10) = 2.
```

```
A(4,12) = 1.
```

```
A(4,13) = 1.
```

```
A(4,14) = 2.
```

C
C
C

```
LOAD DATA INTO SPECIES ARRAY 7L
```

```

00002670
00002680
00002690
00002700
00002710
00002720
00002730
00002740
00002750
00002760
00002770
00002780
00002790
00002800
00002810
00002820
00002830
00002840
00002850
00002860
00002870
00002880
00002890
00002900
00002910
00002920
00002930
00002940
00002950
00002960
00002970
00002980
00002990
00003000
00003010
00003020
00003030
00003040
00003050
00003060
00003070
00003080
00003090
00003100
00003110
00003120
00003130
00003140
00003150
00003160

```



```

      TT = 0
      DO 1050 K = 1,7
      I = K + 7
      DO 1050 J = 1, 8
      TT = TT + 1
      ZL(K,J) = ZL1(TT)
1050  ZL(I,J) = ZL2(TT)
      DO 3010 I = 1,N
      XMU(I) = 0.0
3010  CONTINUE
C
C
C      END OF ONE-TIME INITIALIZATION
C
2004  IF ( TMP - 200.0 ) 2112, 2112, 2355
2355  CONTINUE
      IF ( TMP - TL09 ) 2012, 2012, 2009
2009  ISENT=1
      GO TO 5000
2112  ISENT=2
      GO TO 5000
2012  CONTINUE
      TK = TMP / 1.00+3
      XLP = DLOG ( PRS )
C
C      START OF ORIGINAL 'HS' SUBROUTINE
C
      DO 5004 K10 = 1, M
      IF ( ZL(K10,1) ) 5003, 5002, 5003
5002  HORT(K10) = 0.1111111100
      SR (K10) = -1.006
      GO TO 5004
5003  HORT(K10) = ( ( ( ZL(K10,4) * TK / 4.00 + ZL(K10,3) / 3.000 ) *
1      TK + ZL(K10,2) / 2.00 ) * TK + ZL(K10,1) ) * TK -
2      ZL(K10,5) / TK + ZL(K10,6) ) / ( 42 * TK )
      SR(K10) = ( ZL(K10,1) * DLOG(TK) + TK * ( ZL(K10,2) +
1      ZL(K10,3) * 0.500 * TK + ZL(K10,4) * TK **2 / 3.00 ) -
2      ZL(K10,5) * 0.500 / TK ** 2 + ZL(K10,7) ) / 42
5004  C(K10) = HORT(K10) - SR(K10) + XLP
C
C      END OF ORIGINAL 'HS' SUBROUTINE
C
      ISENT=0
      ITER=0
      ITTPDG=0
      TOL4 = 0.100
      YMAX = 0.
      DO 412 J=1,M1
      XMAX(J) = 1.0010
      DO 412 I=1,N
      IF ( A(I,J) ) 412,412,413
413  IF ( D(I)/A(I,J) -XMAX(J) ) 414,412,412
414  XMAX(J) = D(I)/A(I,J)
412  CONTINUE
      X0=0.000
      DO 90 I=1,N
90  X0=X0+D(I)
      AVD = X0 / XN
      Y0=0.000
      DO 93 J=1,M

```

```

00003170
00003180
00003190
00003200
00003210
00003220
00003230
00003240
00003250
00003260
00003270
00003280
00003290
00003300
00003310
00003320
00003330
00003340
00003350
00003360
00003370
00003380
00003390
00003400
00003410
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00003480
00003490
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00003580
00003590
00003600
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00003670
00003680
00003690
00003700
00003710
00003720
00003730
00003740
00003750
00003760

```

```

93   Y0=Y0+ XMAX(J)          00003770
    X0= DMIN1(X0,Y0)         00003780
    X0=X0*1.0500            00003790
    DO 825 J=1,M1           00003800
825   XMAX(J)=XMAX(J)*1.0500 00003810
C                                         00003820
C   CAUTION*** ASSUMPTION: THAT EACH ELEMENT MOLE RATIO DIVIDED BY 00003830
C   THE SUM OF THE RATIOS WILL BE GREATER THAN 0.01: 00003840
C   IF = OR < 0.01. THEN MUST USE FOLLOWING ROUTINE: 00003850
C   I.E., REMOVE THE 'C' COMMENT FROM COL 1 FROM HERE TO LABEL 530 & 00003860
C   FROM LABEL 503 TO 475 00003870
C   NOTE: ARRAY K2 IS DIMENSIONED (2) 00003880
C                                         00003890
C   SUM IS NOT DEFINED IN ORIGINAL PROGRAM 00003900
    SUM = 0.0                00003910
    DO 525 I = 1,N           00003920
525   SUM = SUM + D(I)      00003930
C                                         00003940
C   L1=0 00003950
    NG05=1 00003960
    RATIO = .0100 00003970
    DO 526 I=1,N 00003980
    IF (D(I)/SUM - RATIO) 527,527,526 00003990
527   L1=L1+1 00004000
    K2(L1) =I 00004010
526   CONTINUE 00004020
    NL= L1 00004030
    IF (NL) 528,528,529 00004040
528   NG05 =1 00004050
    GO TO 530 00004060
529   NG05=2 00004070
530   CONTINUE 00004080
C                                         00004090
C   ITER4=0 00004100
    NG06=1 00004110
    RH2=1.000 00004120
C                                         00004130
C   END OF ENTRY INITIALIZATION & CHECKING OF TEMPERATURE 00004140
C                                         00004150
C   BEGINNING OF MAIN PROGRAM LOOP 00004160
C                                         00004170
C                                         00004180
99   NG0=1 00004190
    ITER2=0 00004200
    IF (ITER - 1) 473,473,474 00004210
474   RH2=DSORT(42) 00004220
    RH2=DMIN1(RH2,1.000) 00004230
473   NG01=1 00004240
    DO 101 J=1,M1 00004250
    SUM= C(J) 00004260
    DO 1002 I=1,N 00004270
1002   SUM=SUM+ XMU(I)*A(I,J) 00004280
    101   CP(J)= SUM 00004290
    DO 1012 J=1,M1 00004300
1012   CPT(J)= CP(J) 00004310
    DO 1001 I=1,N 00004320
1001   D(I)= D(I) 00004330
    L1= 0 00004340
    SUMEX=0.000 00004350
    DO 102 J=1,M 00004360

```

```

          J1=J
          IF (CPT(J)+30.000) 420,420,421
421 XJ=DEXP(-CPT(J))
          SUMFX=SUMFX+XJ
102 X(J)=XJ
          IF (SUMFX-1.000-TOL1*R+2) 103,103,107
103 IF (SUMEX-1.000+TOL1*R+2) 112,104,104
104 L1=1
          DO 106 I=1,N
          SUM=0.0
          DO 105 J=1,M
105 SUM=SUM+ A(I,J)*X(J)
106 F(I,1) = SUM
          YMAX = X0
          GO TO 123
420 CPT1 = CPT(J1)
          DO 423 J=1,M
          J1= J
          CPT2 = CPT(J1)-CPT1
          IF (CPT2 +30.000) 420,420,422
422 X(J) =DEXP(-CPT2)
423 CONTINUE
107 SUM=0.000
          DO 416 J=1,M
416 SUM= SUM+ X(J)
          SUM= X0/SUM
          DO 108 J= 1,M
108 X(J)=X(J)*SUM
          DO 109 I=1,N
          SUM= 00(I)
          DO 110 J=1,M
110 SUM = SUM -A(I,J)*X(J)
109 00(I)= SUM
          GO TO 123
112 DO 113 J=1,M
113 X(J)= 0.000
123 L=L1
C
C L = 0 OR 1 ONLY
C
          IF (L) 131,131,200
131 DO 132 I=1,N
132 XNU(I)=-00(I)
          GO TO 23
430 DO 431 J=1,M1
          CMN(J)= 0.000
          DO 431 I=1,N
431 CMN(J)=CMN(J)+XNU(I)*A(I,J)
          COMF=TOL5*TOL5/H2
          XNU0=0.000
          DO 432 I=1,N
432 XNU0= XNU0+ XNU(I)*0(I)
446 H= -XNU0
          ITTRDG=ITTRDG+1
          IF (ITTRDG-5000) 235R,235R,235R
235R ISENT = 8
          GO TO 5000
235R CONTINUE
          DO 433 J=1,M1
433 CPT(J)= CP(J)+ T*CMN(J)

```

```

00004370
00004380
00004390
00004400
00004410
00004420
00004430
00004440
00004450
00004460
00004470
00004480
00004490
00004500
00004510
00004520
00004530
00004540
00004550
00004560
00004570
00004580
00004590
00004600
00004610
00004620
00004630
00004640
00004650
00004660
00004670
00004680
00004690
00004700
00004710
00004720
00004730
00004740
00004750
00004760
00004770
00004780
00004790
00004800
00004810
00004820
00004830
00004840
00004850
00004860
00004870
00004880
00004890
00004900
00004910
00004920
00004930
00004940
00004950
00004960

```

```

      EXMIN= 1.0010
      DO 510 I=1,M
      IF (CPT(J) - EXMIN) 511,510,510
511 CONTINUE
      EXMIN= CPT(J)
510 CONTINUE
      SUMEX =0.000
      DO 513 I=1,M
      IF (CPT(J)-EXMIN-DIF1) 517,517,516
516 GO TO 513
517 X(J) =DEXP(EXMIN -CPT(J))
      SUMEX =SUMEX +X(J)
513 CONTINUE
      IF (EXMIN ) 521,521 , 519
519 IF (SUMEX*DEXP(-EXMIN)-1.000) 443,521,521
521 PROD = X0 /SUMEX
      DO 522 I=1,M
522 H= H+ PROD* X(J)*CMN(J)
543 CONTINUE
      IF (H) 458,458,457
557 T0=T
      H0= H
      GO TO (448,459) , 'GO
548 T= T+T
      IF (T - 1.0015 ) 446, 446, 908
908 ISFNT = 7
      GO TO 5000
558 T1=T
      NG0=2
559 IF (DABS(H/42)-TOL3) 453,453,460
560 IF ((T1-T0)**2-COMP) 453,453,452
552 T=0.500*(T0+T1)
      GO TO 446
553 DO 454 I=1,N
554 XMU(I) = XMU(I) + T* XNU(I)
      GO TO 99
C
C L = 1 AT THIS POINT
C
200 SDUM = 0.000
      SUM = 0.000
      DO 2 I = 1, N
      SDUM = SDUM + F(I,1) * F(I,1)
      SUM = SUM + F(I,1) * DJ(I)
      E(I) = SUM / SDUM
      G(1,1) = 0.000
      Y1 = 0.000
      Z1 = 0.000
      71 SUM = E(I) + G(1,1) * Z1
      IF ( SUM )10, 10, 11
      10 Z1 = 0.000
      GO TO 8
      11 IF ( SUM - YMAX ) 471, 471, 472
572 Z1 = YMAX
      GO TO 8
571 Z1 = SUM
      9 IF ( DABS( Z1 - Y1 ) - TOL3 ) 15, 15, 13
      13 Y1 = Z1
      GO TO 71

```

```

00004970
00004980
00004990
00005000
00005010
00005020
00005030
00005040
00005050
00005060
00005070
00005080
00005090
00005100
00005110
00005120
00005130
00005140
00005150
00005160
00005170
00005180
00005190
00005200
00005210
00005220
00005230
00005240
00005250
00005260
00005270
00005280
00005290
00005300
00005310
00005320
00005330
00005340
00005350
00005360
00005370
00005380
00005390
00005400
00005410
00005420
00005430
00005440
00005450
00005460
00005470
00005480
00005490
00005500
00005510
00005520
00005530
00005540
00005550
00005560

```

```

15 DO 16 I = 1, N
16 XNU(I) = -D(I) * F(I,1) * Z1
   DO 19 J = 1, M
19 X(J) = X(J) * 71
C
C   L = 0 OR 1 BELOW THIS POINT
C
23   DO 24 I=1,N
   IF (DABS(XNU(I)) - TOL4 * AVD) 24,24,25
24   CONTINUE
   NG01=2
   GO TO 330
25   GO TO (550,551), NG06
551   ITER4=ITER4+1
   IF (L) 550,550,814
814  IF (ITER4-20)550,552,552
552  ITER4=0
   NG01=2
   GO TO 330
550  H0=0.000
   DO 26 I=1,N
26  H0=H0+XNU(I)**2
   H2= H0
   T0=0.000
   T=1.000
   GO TO 330
C
C   SEE COMMENTS ABOVE ( LABEL 527 )
C
503 CONTINUE
C
   GO TO (475,476),NG05
475 K = ITER - ( ITER / N ) * N
   IF (K) 531,532,531
532  K=N
531  DO 540 L1=1,NL
   IF (K2(L1)-K) 540,541,540
540  CONTINUE
   GO TO 475
541 IF (DABS(XNU(K))-TOL5*(K)) 475,475,542
542 IF (DABS(XNU(K))-1.0-15)475,475,8000
8000 SUM=XNU(K)
   DO 543 I=1,N
543  XNU(I)=0.000
   XNU(K)=SUM
   H0=SUM*SUM
   DIF1=100.000
   H2=H0
   T0=0.000
   T=1.000
   GO TO 430
475  DIF1=DIF
C
   GO TO 430
330  ITER= ITER+1
   IF (ITER - ITW) 333,670,670
670 ISFN = 4
   GO TO 5079
C
333  GO TO (503,500),NG01

```

```

00005570
00005580
00005590
00005600
00005610
00005620
00005630
00005640
00005650
00005660
00005670
00005680
00005690
00005700
00005710
00005720
00005730
00005740
00005750
00005760
00005770
00005780
00005790
00005800
00005810
00005820
00005830
00005840
00005850
00005860
00005870
00005880
00005890
00005890
00005900
00005910
00005920
00005930
00005940
00005950
00005960
00005970
00005980
00005990
00006000
00006010
00006020
00006030
00006040
00006050
00006060
00006070
00006080
00006090
00006100
00006110
00006120
00006130
00006140
00006150
00006160

```

```

C
500      NTOT=N+1.                                00006170
      DO 820 I=1,N                                00006180
820      DO(I)=XMU(I)                              00006190
      DO 821 J=1,M1                               00006200
821      CPT(J)=X(J)                              00006210
      IF (L) 727,727,502                          00006220
502      XBAR=Z1                                   00006230
      ITER2 = 1                                    00006240
750 IF (XBAR) 727, 727, 701                       00006250
701      G(N+1,N+1) = 0.000                       00006260
      DO 704 K=1,M                                00006270
      SUM=- C(K)                                   00006280
      DO 705 I=1,N                                00006290
705      SUM=SUM-XMU(I)*A(I,K)                    00006300
      IF (SUM -30.000)425,727, 727               00006310
425      X(K) = XBAR*DEXP(SUM)                    00006320
704      CONTINUE                                  00006330
      DO 706 I=1,N                                00006340
      SUM=0.000                                    00006350
      DO 707 K=1,M                                00006360
707      SUM=SUM +A(I,K)*X(K)                     00006370
      SUM=SUM/XBAR                                 00006380
      G(I,N+1) =SUM                                00006390
      G(N+1,I) =SUM                                00006400
      R(I) = D(I)-SUM*XBAR                         00006410
      DO 708 J=1,N                                00006420
      SUM= 0.000                                    00006430
      DO 709 K=1,M                                00006440
709      SUM=SUM - A(I,K)*A(J,K)*X(K)             00006450
706      G(I,J)=SUM                                00006460
      SUM=-XBAR                                     00006470
      DO 710 K=1,M                                00006480
710      SUM=SUM+X(K)                              00006490
      R(N+1) =SUM/XBAR                             00006500
      DO 728 I=1,N                                00006510
      IF(DABS(R(I)) - TOL6*AVD) 728,728,727       00006520
728      CONTINUE                                  00006530
      GO TO 730                                     00006540
727      DO 903 I=1,N                              00006550
903      IF(DABS(XN(I))-TOL5*D(I)) 903,903,801    00006560
      CONTINUE                                     00006570
      DO 822 I=1,N                                 00006580
822      XMU(I)=D(I)                               00006590
      DO 823 J=1,M1                               00006600
823      X(J)=CPT(J)                              00006610
      GO TO 5079                                    00006620
801      TOL4=0.100*TOL4                          00006630
      NGOK=2                                        00006640
      H2=1.000                                     00006650
      GO TO 99                                      00006660
730 IPP=NXNSOL/INXNSL,NTAT,G,R)                  00006670
      IF(IPP-2)8001,727,8001                       00006680
8001 ITER2=ITER2+1                                00006690
      DO 465 I=1,N                                 00006700
      IF(DABS(R(I)) -TOL7) 465,465,727           00006710
465      CONTINUE                                  00006720
      DO 731 I=1,NTOT                              00006730
      IF(DABS(R(I)) - TOL5) 731, 731, 739        00006740
731      CONTINUE                                  00006750
00006760

```

```

GO TO 737
737 DO 734 I=1,N
734 XMU(I)=XMU(I)+R(I)
      XBAR =XRAR+ R(N+1)
GO TO 750
739 CONTINUE
      IF (ITER2- ITMAX) 733,740,740
740 ISENT=5
      GO TO 5000
C
C TEST IF X(J) NEGATIVE
C
737 DO 470 J=1,M
      IF (X(J)) 801,470,470
470 CONTINUE
C
C END OF MAIN PROGRAM LOOP
C
5070 SUMNI =0.000
      DO 5010 I = 1. M
      SUMNI = SUMNI + x(I)
5010 CONTINUE
      IF (SUMNI) 751, 752, 751
752 ISENT=6
      GO TO 5000
751 CONTINUE
      DO 5011 I = 1. M
5011 XMOFR(I) = x(I) / SUMNI
C
C NOTE: ROUTINE TO CHECK FOR A SINGULAR DERIVATIVE MATRIX
C IN RTP & RDP ARRAYS HAS BEEN REMOVED.
C
C ADDITIONAL CALCULATIONS REQUIRED BY HPCHEM:
C
NP11 = N + 1
      DO 6000 I= 1,N
      DO 6000 J= 1,N
      G(I,J) = 0.000
      DO 6000 K= 1,M
      G(I,J) = G(I,J) - A(I,K)*A(J,K)* x(K)
6000 CONTINUE
      DO 6001 I = 1,N
      G(I, NP11) = 0.000
      G(NP11, I) = 0.000
      DO 6002 K = 1,M
      G(I, NP11) = G(I, NP11) + A(I,K)* x(K)
      G(NP11, I) = G(NP11, I) + A(I,K)* x(K)
6002 CONTINUE
      G(I, NP11) = G(I, NP11) / XBAR
      G(NP11, I) = G(NP11, I) / XBAR
6001 CONTINUE
      G(NP11,NP11) = 0.0
      XBAR = SUMNI
      YHPRIM = 0.0
      HORTSM = 0.0
      DO 6010 I = 1,N
      RTP(I) = 0.0
6010 CONTINUE
      DO 6030 K = 1,M
      YHPRIM = YHPRIM + x(K) * ( (7L(K,4)*TK + 7L(K,3))*TK + 7L(K,2) )

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00006770
00006780
00006790
00006800
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00007000
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00007270
00007275
00007280
00007290
00007300
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00007340
00007350

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1      * TK + ZL(K,1) + ZL(K,5)/(TK*TK) ) 00007360
XHORT(K) = X(K) * HORT(K) 00007370
HORTSM = HORTSM + XHORT(K) 00007380
DO 6020 I = 1,N 00007390
RTP(I) = RTP(I) - A(I,K) * XHORT(K) 00007400
6020 CONTINUE 00007410
6030 CONTINUE 00007420
RTP(NP11) = HORTSM / YBAR 00007430
NPL = N + L 00007440
DO 6040 I = 1,NPL 00007450
RTP(I) = RTP(I) / TMP 00007460
6040 CONTINUE 00007470
IPP = NXNSOL(INXNSL,NPL,3,RTP) 00007510
CSURP = 0.0 00007540
DO 6060 K = 1,M 00007550
RTPA = 0.0 00007560
DO 6050 I = 1,N 00007570
RTPA = RTPA + RTP(I) * A(I,K) 00007580
6050 CONTINUE 00007590
CSURP = CSURP + (HORT(K)/TMP - RTPA)*XHORT(K) 00007600
6060 CONTINUE 00007610
SMDHDT = XHPRIM + AR*TMP*(CSURP + RTP(NP11)*HORTSM/SUMNI) 00007620
C 00007630
RETURN 00007640
C 00007650
C ERROR: SET MOLE FRACTIONS TO 0. 00007660
C 00007670
5000 DO 5100 I = 1, ITOTSP 00007680
5100 XMOFR(I) = 0. 00007690
RETURN 00007700
END 00007710

```


FUNCTION	NXNSOL (IM,IN,A,9)	00007720
CNXNSOL	IMPLICIT REAL*8(A-H,O-Z)	00007730
	DIMENSION A(2),B(2)	00007740
	INTEGER XROW	00007750
	XROW(K000FX,K001FX)=K001FX*4-M+K000FX	00007760
	M=IM	00007770
	N=IN	00007780
	N1=N-1	00007790
	DO 44 J=1,N1	00007800
	K=J	00007810
	J1=J+1	00007820
	JJ=XROW(J,J)	00007830
	WS1=DABS(A(JJ))	00007840
C	LOOP TO FIND LARGEST	00007850
	DO 11 L=J1,N	00007860
	LJ=XROW(L,J)	00007870
	WWS1=DABS(A(LJ))	00007880
	IF (WS1-WWS1) 12,11,11	00007890
	12 WS1=WWS1	00007900
	K=L	00007910
	11 CONTINUE	00007920
	IF (J-K) 13,31,31	00007930
C	5 IF DIAG NOT LARGEST INTERCHANGE ROWS	00007940
	13 DO 26 L=1,N	00007950
	JL=XROW(J,L)	00007960
	KL=XROW(K,L)	00007970
	WS1=A(JL)	00007980
	A(JL)=A(KL)	00007990
26	A(KL)=WS1	00008000
	WS1=B(J)	00008010
	B(J)=B(K)	00008020
	B(K)=WS1	00008030
31	DO 33 L=J1,N	00008040
	JL=XROW(J,L)	00008050
	IF (A(JJ)) 33,54,33	00008060
33	A(JL)=A(JL)/A(JJ)	00008070
	B(J)=B(J)/A(JJ)	00008080
	DO 43 L=1,N	00008090
	IF (L-J) 37,43,37	00008100
37	LJ=XROW(L,J)	00008110
38	DO 41 L2=J1,N	00008120
	LL2=XROW(L,L2)	00008130
	JL2=XROW(J,L2)	00008140
41	A(LL2)=A(LL2)-A(LJ)*A(JL2)	00008150
	B(L)=B(L)-A(LJ)*B(J)	00008160
43	CONTINUE	00008170
44	CONTINUE	00008180
C	LAST COLUMN HAS NOT BEEN DONE YET	00008190
	NN=XROW(N,N)	00008200
	IF (A(NN)) 45,54,46	00008210
45	B(N)=B(N)/A(NN)	00008220
	DO 50 L=1,N1	00008230
	LN=XROW(L,N)	00008240
50	B(L)=B(L)-A(LN)*B(N)	00008250
	NXNSOL=1	00008260
	RETURN	00008270
54	NXNSOL=2	00008280
	RETURN	00008290
	END	00008300
		00008310

```

-----C20049.DAT  BIN #28  JUL. 28 1977-----
9.842  9.842  16.8275  12.7  57.29  2.46  1.
-20.  480.  200.  14.  335.  355.
16.5  4.0  14.45  10020.  64.4
0.0196  2000.  0.
650.  500.  700.  500.  650.
2.28  0.0032  1.6
34.  15.  2.  0.02227
3.65  0.08  0.75  0.80  0.012  0.0000065
0.1  0.22  200.  50.
4.57  1.625  2.95  1.625  1.  1.
7.  1.  3.
18
9
0.  0.058  0.096  0.135  0.173  0.211  0.25  0.288
0.308  0.692  0.712  0.75  0.789  0.827  0.865  0.904
0.942  1.0
0.  0.029  0.096  0.304  0.544  0.755  0.911  0.991
1.0  1.0  0.991  0.911  0.755  0.544  0.304  0.096
0.029  0.0
0.0  0.05  0.1  0.15  0.2  0.25  0.3  0.35
0.4
0.0  0.08  0.135  0.19  0.25  0.3  0.35  0.41
0.47
18
10
0.  0.058  0.096  0.135  0.173  0.211  0.25  0.288
0.308  0.692  0.712  0.75  0.789  0.827  0.865  0.904
0.942  1.0
0.  0.029  0.096  0.304  0.544  0.755  0.911  0.991
1.0  1.0  0.991  0.911  0.755  0.544  0.304  0.096
0.029  0.0
0.0  0.05  0.1  0.15  0.2  0.25  0.3  0.35
0.4  0.46
0.0  0.15  0.25  0.35  0.44  0.52  0.57  0.61
0.63  0.65
18
310.  1.
0.960  1.035  310.
2.  2.  2.  2.
1.  1.  1.  1.
1.27  640.  0.5  540.
0.420.
-10.  710.
1

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TABLE OF INPUT DATA AND INITIAL ENGINE CONDITIONS

```

*****
INITIAL JET SPEED
VINIT= 9530.9CM/SEC
-----
ENGINE GEOMETRY * TIMING * FUEL PROPERTIES * ESTIMATE TEMPERATURES
STROKE = 9.842CM * INT.VAL.OPEN = -20.0DEG * Z(MND. OF H) = 16.5 * PISTON TLP(TP) = 650.OK
CCK.RCD = 16.83CM * EXH.VAL.OPEN = 480.0DEG * Z(MND. OF C) = 9.0 * CYL.WALL(TW) = 600.OK
BCRE = 9.842CM * INT.VAL.CLOSE = 200.0DEG * SAFC(STGICH.A/F) = 14.4 * CYL.HEAD(TCYLH) = 650.OK
CLEAR.VOL. = 12.70CM**3 * EXH.VAL.CLOSE = 14.0DEG * LCHER HEATING = 10020. CAL/G* INT.VALVE(TINV) = 500.OK
CLP VCL. = 57.25CM**3 * FUEL INJ.START = 330.0DEG * VAPORIZING HEAT = 64. CAL/G* EXH.VALVE(TEXV) = 700.OK
* FUEL INJ.END = 360.0DEG * DENSITY = 0.800G/CM**3*
-----
INITIAL JET DATA * JET MIXING PARAMETER * PLUME MODEL
JET DIRECTION** * SWIRL RATIO AT BDC = 3.650* RPM * INITIAL PLUME RADIUS = 0.1000 CM *
ANGLE(VERTICAL) = 34.00 DEG * AIR ENTRAIN. PARAMETER * ENTRAIN. PARAMETER = 0.220 *
ANGLE(TANGENTIAL) = 15.00 DEG * ALPHA = 0.070 * LAM. FLAME SPEED = 200.00 CM/SEC *
JET RADIUS = 0.02227 CM * BETA = 0.350 * SPARK PLUG LOCATION = 50.00 DEG *
JET LOCATION(R) = 2.000 CM * DENSITY OF SURROUND. GAS = 0.01200 G/CC * (FROM INJ. NOZZLE)
PISTON CUP RADIUS = 2.460 CM * SWIRL DECAY FACTOR = 0.650E-05 * TAYLOR MICRO SCALE = 0.0236 CM
-----
ENGINE CONDITION * INTAKE CONDITIONS * COMPUTING INTERVALS * PRINT CONTROL
ENGINE SPEED = 2000. RPM * PRESSURE = 0.960 ATA * VALVE OVERLAP = 2.00 DEG * VALVE OVERLAP = 1.00
FUEL AMOUNT = 0.02970 G/SHOT * TEMPERATURE = 310.00 K * INTAKE = 2.00 DEG * INTAKE = 1.00
EGR = 0.0 * EQUI. RATIO = 0.0 * COMPRESSION = 2.00 DEG * COMPRESSION = 1.00
* AMBIENT CONDITIONS * EXHAUST CONDITIONS(ASSUME) * COMBUSTION = 3.75 DEG * COMBUSTION = 1.00
PRESSURE = 1.00 ATA * PRESSURE = 1.03 ATA * EXPANSION = 2.00 DEG * EXPANSION = 1.00
TEMPERATURE = 310.00 K * TEMPERATURE = 786.00 K * EXHAUST = 2.00 DEG * EXHAUST = 1.00
* EQUI. RATIO = 0.500 *
-----
INTAKE VALVE * HEAT TRANS. AREA * HEAT TRANS. COEFF * INITIAL CONDITION
DIAMETER = 4.570 CM * INTAKE VALVE = 16.403 CM**2 * (WCSCHNI'S EQ.) * CYL. PRESS. = 1.27 ATA
LIFT = 1.625 CM * EXHAUST VALVE = 6.835 CM**2 * COHT = 1.60000 * CYL. TEMP. = 786.0 DEG
NUMBER = 1.0 * PISTON TOP = 57.066 CM**2 * COHT1 = 2.28000 * CYL. EQUI. = 0.500
EXHAUST VALVE * PISTON TOP = 46.576 CM**2 * COHT2 = 0.00320 * EXHAUST TEMP. = 786.0 DEG *
DIAMETER = 2.950 CM * CYL. HEAD = 52.840 CM**2 *
LIFT = 1.625 CM * CYL. WALL = VARIABLE*
NUMBER = 1.0 *
-----
FRICTION LOSS *
PISTON SKIRT = 7.000 CM *
JCU. BEAR. COE = 1.000 *
NC. OF RINGS = 3.0 *
-----

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INTAKE VALVE--EFFECTIVE FLOW AREA VS. CRANK ANGLE
 NORMALIZED CRANK ANGLE

0.0	0.058	0.096	0.135	0.173	0.211	0.250	0.288	0.308	0.692
0.712	0.750	0.789	0.827	0.865	0.904	0.942	1.000		
NORMALIZED EFFECTIVE FLOW AREA									
0.0	0.000	0.005	0.044	0.132	0.240	0.345	0.409	0.417	0.417
0.409	0.345	0.240	0.132	0.044	0.005	0.000	0.0		

EXHAUST VALVE--EFFECTIVE FLOW AREA VS. CRANK ANGLE
 NORMALIZED CRANK ANGLE

0.0	0.058	0.096	0.135	0.173	0.211	0.250	0.288	0.308	0.692
0.712	0.750	0.789	0.827	0.865	0.904	0.942	1.000		
NORMALIZED EFFECTIVE FLOW AREA									
0.0	0.001	0.015	0.116	0.310	0.480	0.592	0.644	0.650	0.650
0.644	0.592	0.480	0.310	0.116	0.015	0.001	0.0		

INTAKE AND EXHAUST VALVE OPEN

OUTPUT DATA		T	V	WCRK	CYL.MAS	HEAT TRANSFER		AEFI	GWI	AEFE	GWE
THEO	P	K	CM**3	PS	KG	CAL/A	CAL	CM**2	G/S	CM**2	G/S
DEG	ATA										
-8.0	1.31	790.	75.	-0.00	0.043	-0.862E-02	-0.345E-01	0.01	-0.16	0.17	-2.99
-6.0	1.33	792.	73.	-0.00	0.043	-0.887E-02	-0.700E-01	0.03	-0.28	0.13	-2.47
-4.0	1.35	793.	71.	-0.00	0.043	-0.901E-02	-0.106E+00	0.05	-0.73	0.09	-1.85
-2.0	1.35	791.	70.	-0.00	0.042	-0.901E-02	-0.142E+00	0.08	-1.22	0.05	-1.17
0.0	1.34	788.	70.	-0.01	0.042	-0.886E-02	-0.177E+00	0.11	-1.69	0.02	-0.49
2.0	1.31	783.	70.	-0.00	0.041	-0.854E-02	-0.212E+00	0.21	-2.35	0.02	-0.30
4.0	1.25	773.	71.	-0.00	0.040	-0.793E-02	-0.243E+00	0.42	-5.27	0.01	-0.24
6.0	1.16	758.	73.	-0.00	0.039	-0.698E-02	-0.271E+00	0.63	-7.71	0.01	-0.16
8.0	1.06	741.	75.	-0.00	0.038	-0.583E-02	-0.295E+00	0.84	-8.55	0.01	-0.09
10.0	0.98	725.	77.	-0.00	0.037	-0.471E-02	-0.313E+00	1.09	-6.60	0.01	0.02
12.0	0.95	721.	81.	-0.00	0.037	-0.397E-02	-0.329E+00	1.59	3.43	0.00	0.04
14.0	0.95	724.	84.	-0.00	0.039	-0.407E-02	-0.346E+00	2.08	10.18	0.0	0.01

INTAKE VALVE OPEN

OUTPUT DATA										
T-EC	P	T	V	WRK	CYL.MAS	HEAT TRANSFER		AEFI	GVI	
DEG	ATA	K	CM**3	PS	G	CAL/A	CAL	CM**2	G/S	
16.0	0.95	726.	89.	-0.00	0.041	-0.426E-02	-0.363E+00	2.57	11.68	
18.0	0.95	727.	94.	-0.00	0.043	-0.445E-02	-0.381E+00	3.06	13.06	
20.0	0.93	692.	99.	-0.01	0.047	-0.376E-02	-0.396E+00	3.66	22.59	
22.0	0.92	657.	105.	-0.01	0.052	-0.190E-02	-0.403E+00	4.27	33.03	
24.0	0.92	616.	111.	-0.01	0.059	0.416E-04	-0.403E+00	4.88	39.28	
26.0	0.92	582.	118.	-0.01	0.066	0.195E-02	-0.395E+00	5.48	43.92	
28.0	0.93	555.	126.	-0.01	0.074	0.379E-02	-0.390E+00	6.06	47.65	
30.0	0.93	530.	134.	-0.01	0.083	0.551E-02	-0.358E+00	6.63	50.90	
32.0	0.93	509.	142.	-0.01	0.092	0.716E-02	-0.329E+00	7.20	54.04	
34.0	0.93	491.	151.	-0.01	0.101	0.871E-02	-0.295E+00	7.76	57.05	
36.0	0.93	475.	161.	-0.02	0.111	0.102E-01	-0.254E+00	8.23	59.46	
38.0	0.93	462.	170.	-0.02	0.121	0.115E-01	-0.208E+00	8.59	61.29	
40.0	0.93	450.	180.	-0.02	0.132	0.128E-01	-0.156E+00	8.94	63.59	
42.0	0.93	440.	191.	-0.02	0.143	0.141E-01	-0.100E+00	9.30	65.94	
44.0	0.93	431.	202.	-0.02	0.154	0.153E-01	-0.390E-01	9.57	68.11	
46.0	0.93	423.	213.	-0.02	0.166	0.164E-01	0.265E-01	9.65	69.36	
48.0	0.93	415.	224.	-0.02	0.178	0.175E-01	0.956E-01	9.72	70.87	
50.0	0.93	407.	236.	-0.03	0.190	0.186E-01	0.171E+00	9.72	72.26	
52.0	0.93	403.	248.	-0.03	0.202	0.196E-01	0.249E+00	9.72	73.41	
54.0	0.93	399.	261.	-0.03	0.214	0.206E-01	0.331E+00	9.72	74.55	
56.0	0.93	393.	273.	-0.03	0.227	0.215E-01	0.419E+00	9.72	76.21	
58.0	0.93	389.	285.	-0.03	0.240	0.225E-01	0.508E+00	9.72	77.35	
60.0	0.93	385.	299.	-0.04	0.253	0.234E-01	0.601E+00	9.72	78.39	
62.0	0.93	382.	312.	-0.04	0.266	0.243E-01	0.699E+00	9.72	79.30	
64.0	0.92	379.	325.	-0.04	0.280	0.252E-01	0.800E+00	9.72	80.10	
66.0	0.92	376.	339.	-0.04	0.293	0.261E-01	0.904E+00	9.72	80.79	
68.0	0.92	374.	352.	-0.05	0.307	0.270E-01	0.101E+01	9.72	81.36	
70.0	0.92	371.	366.	-0.05	0.320	0.279E-01	0.112E+01	9.72	81.82	
72.0	0.92	369.	379.	-0.05	0.334	0.287E-01	0.124E+01	9.72	82.16	
74.0	0.92	367.	393.	-0.05	0.348	0.295E-01	0.136E+01	9.72	82.40	
76.0	0.92	365.	406.	-0.05	0.361	0.303E-01	0.148E+01	9.72	82.53	
78.0	0.92	364.	420.	-0.06	0.375	0.311E-01	0.160E+01	9.72	82.55	
80.0	0.92	362.	434.	-0.06	0.389	0.319E-01	0.173E+01	9.72	82.46	
82.0	0.92	361.	447.	-0.06	0.403	0.327E-01	0.186E+01	9.72	82.27	
84.0	0.92	360.	461.	-0.06	0.416	0.335E-01	0.199E+01	9.72	81.98	
86.0	0.92	359.	474.	-0.07	0.430	0.343E-01	0.213E+01	9.72	81.59	
88.0	0.92	358.	487.	-0.07	0.443	0.350E-01	0.227E+01	9.72	81.10	
90.0	0.93	357.	500.	-0.07	0.457	0.357E-01	0.242E+01	9.72	80.50	
92.0	0.93	356.	513.	-0.07	0.470	0.365E-01	0.256E+01	9.72	79.82	
94.0	0.93	355.	526.	-0.08	0.483	0.372E-01	0.271E+01	9.72	79.03	
96.0	0.93	355.	539.	-0.08	0.496	0.379E-01	0.286E+01	9.72	78.15	
98.0	0.93	354.	551.	-0.08	0.509	0.385E-01	0.302E+01	9.72	77.18	
100.0	0.93	353.	564.	-0.08	0.522	0.392E-01	0.317E+01	9.72	76.12	
102.0	0.93	353.	576.	-0.08	0.534	0.399E-01	0.333E+01	9.72	74.97	
104.0	0.93	353.	588.	-0.08	0.547	0.405E-01	0.349E+01	9.72	73.73	
106.0	0.93	352.	599.	-0.09	0.559	0.411E-01	0.366E+01	9.72	72.40	
108.0	0.93	352.	611.	-0.09	0.571	0.417E-01	0.382E+01	9.72	71.00	
110.0	0.94	352.	622.	-0.09	0.582	0.423E-01	0.399E+01	9.72	69.51	
112.0	0.94	351.	633.	-0.09	0.593	0.429E-01	0.417E+01	9.72	67.95	
114.0	0.94	351.	643.	-0.09	0.604	0.434E-01	0.434E+01	9.72	66.31	
116.0	0.94	351.	653.	-0.09	0.615	0.439E-01	0.451E+01	9.72	64.60	
118.0	0.94	351.	664.	-0.10	0.626	0.444E-01	0.469E+01	9.72	62.82	
120.0	0.94	351.	673.	-0.10	0.636	0.449E-01	0.487E+01	9.72	60.98	
122.0	0.94	351.	683.	-0.10	0.646	0.454E-01	0.505E+01	9.72	59.07	
124.0	0.94	351.	692.	-0.10	0.655	0.459E-01	0.524E+01	9.72	57.11	
126.0	0.95	351.	701.	-0.10	0.664	0.463E-01	0.542E+01	9.72	55.10	
128.0	0.95	351.	709.	-0.10	0.673	0.467E-01	0.561E+01	9.72	53.03	

130.0	0.95	351.	715.	-0.10	0.682	0.471E-01	0.580E+01	9.72	50.91
132.0	0.95	351.	725.	-0.10	0.690	0.475E-01	0.599E+01	9.72	48.76
134.0	0.95	351.	733.	-0.10	0.698	0.478E-01	0.614E+01	9.65	46.43
136.0	0.95	352.	740.	-0.10	0.705	0.481E-01	0.637E+01	9.57	43.99
138.0	0.95	352.	747.	-0.11	0.712	0.484E-01	0.657E+01	9.30	41.40
140.0	0.95	352.	754.	-0.11	0.718	0.487E-01	0.676E+01	8.94	38.40
142.0	0.95	352.	760.	-0.11	0.724	0.490E-01	0.696E+01	8.59	35.78
144.0	0.95	352.	766.	-0.11	0.730	0.492E-01	0.715E+01	8.23	33.40
146.0	0.95	353.	772.	-0.11	0.735	0.494E-01	0.735E+01	7.76	31.09
148.0	0.95	353.	777.	-0.11	0.740	0.495E-01	0.755E+01	7.20	28.37
150.0	0.95	353.	782.	-0.11	0.744	0.498E-01	0.775E+01	6.63	25.68
152.0	0.95	353.	787.	-0.11	0.748	0.499E-01	0.795E+01	6.06	23.54
154.0	0.95	354.	791.	-0.11	0.751	0.500E-01	0.815E+01	5.48	21.28
156.0	0.95	354.	795.	-0.11	0.755	0.501E-01	0.835E+01	4.88	18.99
158.0	0.95	354.	799.	-0.11	0.757	0.502E-01	0.855E+01	4.27	16.72
160.0	0.95	355.	803.	-0.11	0.760	0.502E-01	0.875E+01	3.65	14.48
162.0	0.95	355.	806.	-0.11	0.762	0.503E-01	0.895E+01	3.06	12.26
164.0	0.95	356.	809.	-0.11	0.764	0.503E-01	0.915E+01	2.57	10.26
166.0	0.95	356.	811.	-0.11	0.765	0.503E-01	0.935E+01	2.08	8.42
168.0	0.95	357.	813.	-0.11	0.766	0.502E-01	0.955E+01	1.59	6.56
170.0	0.95	357.	815.	-0.11	0.767	0.502E-01	0.975E+01	1.09	4.72
172.0	0.95	358.	816.	-0.11	0.767	0.501E-01	0.995E+01	0.84	3.27
174.0	0.96	359.	817.	-0.11	0.768	0.501E-01	0.102E+02	0.63	2.40
176.0	0.96	359.	818.	-0.11	0.768	0.500E-01	0.104E+02	0.42	1.56
178.0	0.96	359.	819.	-0.11	0.768	0.499E-01	0.106E+02	0.21	0.80
180.0	0.96	360.	819.	-0.11	0.768	0.498E-01	0.108E+02	0.11	0.23
182.0	0.96	361.	819.	-0.11	0.768	0.497E-01	0.110E+02	0.08	-0.08
184.0	0.96	362.	818.	-0.11	0.768	0.495E-01	0.112E+02	0.05	-0.16
186.0	0.97	363.	817.	-0.11	0.768	0.494E-01	0.113E+02	0.03	-0.14
188.0	0.97	363.	816.	-0.11	0.768	0.493E-01	0.115E+02	0.01	-0.07
190.0	0.98	364.	815.	-0.11	0.768	0.491E-01	0.117E+02	0.01	-0.05
192.0	0.98	365.	813.	-0.11	0.768	0.490E-01	0.119E+02	0.01	-0.05
194.0	0.99	367.	811.	-0.11	0.768	0.488E-01	0.121E+02	0.01	-0.04
196.0	0.99	368.	808.	-0.11	0.768	0.487E-01	0.123E+02	0.00	-0.03
198.0	1.00	369.	806.	-0.11	0.768	0.485E-01	0.125E+02	0.00	-0.02
200.0	1.01	370.	803.	-0.11	0.768	0.483E-01	0.127E+02	0.00	-0.01

OVERALL EQUIVALENCE RATIO = 0.590
 VOLUMETRIC EFFICIENCY = 0.855
 CORRECT CHARGED AIR PHI = 0.031
 INDUCED AIR AND REG. GAS NOX
 PPM = 52.4 MASS = 0.418E-04G

COMPRESSION

OUTPUT	DATA				WCHK	CYL.MAS	HEAT TRANSFER		FSWR	S.RT
TRFD	P	T	V	PS			CAL/A	CAL		
DFD	ATA	K	CM**3						#RPM	
202.0	1.01	371.	799.	-0.11	C.768	0.482E-01	0.129E+02	0.999	3.65	
204.0	1.02	373.	795.	-0.11	C.768	0.480E-01	0.131E+02	0.999	3.65	
206.0	1.03	374.	791.	-0.11	C.768	0.478E-01	0.133E+02	0.998	3.65	
208.0	1.04	375.	787.	-0.11	C.768	0.476E-01	0.135E+02	0.997	3.65	
210.0	1.05	377.	782.	-0.11	C.768	0.473E-01	0.137E+02	0.997	3.65	
212.0	1.06	379.	777.	-0.11	C.768	0.471E-01	0.139E+02	0.996	3.65	
214.0	1.07	380.	772.	-0.12	C.768	0.469E-01	0.140E+02	0.996	3.65	
216.0	1.09	382.	766.	-0.12	C.768	0.466E-01	0.142E+02	0.995	3.65	
218.0	1.10	384.	760.	-0.12	C.768	0.464E-01	0.144E+02	0.994	3.65	
220.0	1.12	386.	754.	-0.12	C.768	0.461E-01	0.146E+02	0.994	3.64	
222.0	1.13	388.	747.	-0.12	C.768	0.459E-01	0.148E+02	0.993	3.64	
224.0	1.15	390.	740.	-0.12	C.768	0.456E-01	0.150E+02	0.992	3.64	
226.0	1.17	392.	733.	-0.13	C.768	0.453E-01	0.151E+02	0.992	3.64	
228.0	1.18	394.	725.	-0.13	C.768	0.450E-01	0.153E+02	0.991	3.64	
230.0	1.20	396.	718.	-0.13	C.768	0.447E-01	0.155E+02	0.991	3.64	
232.0	1.23	399.	709.	-0.14	C.768	0.444E-01	0.157E+02	0.990	3.64	
234.0	1.25	401.	701.	-0.14	C.768	0.440E-01	0.159E+02	0.989	3.64	
236.0	1.27	406.	692.	-0.15	C.768	0.437E-01	0.160E+02	0.988	3.64	
238.0	1.30	406.	683.	-0.15	C.768	0.433E-01	0.162E+02	0.988	3.65	
240.0	1.33	409.	673.	-0.16	C.768	0.430E-01	0.164E+02	0.987	3.65	
242.0	1.35	412.	664.	-0.17	C.768	0.426E-01	0.166E+02	0.987	3.65	
244.0	1.38	415.	653.	-0.17	C.768	0.422E-01	0.167E+02	0.986	3.65	
246.0	1.42	418.	643.	-0.18	C.768	0.418E-01	0.169E+02	0.985	3.65	
248.0	1.45	421.	633.	-0.19	C.768	0.415E-01	0.171E+02	0.985	3.65	
250.0	1.49	424.	622.	-0.20	C.768	0.409E-01	0.172E+02	0.984	3.65	
252.0	1.53	428.	611.	-0.22	C.768	0.404E-01	0.174E+02	0.984	3.66	
254.0	1.57	431.	599.	-0.23	C.768	0.397E-01	0.175E+02	0.983	3.66	
256.0	1.62	435.	588.	-0.25	C.768	0.394E-01	0.177E+02	0.982	3.66	
258.0	1.66	439.	576.	-0.26	C.768	0.388E-01	0.179E+02	0.982	3.67	
260.0	1.71	443.	564.	-0.28	C.768	0.383E-01	0.180E+02	0.981	3.67	
262.0	1.77	447.	551.	-0.30	C.768	0.377E-01	0.182E+02	0.980	3.67	
264.0	1.83	451.	539.	-0.32	C.768	0.370E-01	0.183E+02	0.980	3.68	
266.0	1.89	456.	526.	-0.35	C.768	0.364E-01	0.184E+02	0.979	3.68	
268.0	1.95	461.	513.	-0.37	C.768	0.357E-01	0.186E+02	0.978	3.69	
270.0	2.03	466.	500.	-0.40	C.768	0.350E-01	0.187E+02	0.978	3.69	
272.0	2.11	471.	487.	-0.43	C.768	0.342E-01	0.189E+02	0.977	3.70	
274.0	2.19	476.	474.	-0.47	C.768	0.334E-01	0.190E+02	0.976	3.71	
276.0	2.28	481.	461.	-0.50	C.768	0.325E-01	0.191E+02	0.976	3.71	
278.0	2.37	487.	447.	-0.54	C.768	0.316E-01	0.193E+02	0.975	3.72	
280.0	2.47	493.	434.	-0.59	C.768	0.307E-01	0.194E+02	0.975	3.73	
282.0	2.57	499.	420.	-0.63	C.768	0.297E-01	0.195E+02	0.974	3.74	
284.0	2.71	506.	406.	-0.68	C.768	0.286E-01	0.196E+02	0.973	3.75	
286.0	2.85	512.	393.	-0.74	C.768	0.274E-01	0.197E+02	0.973	3.76	
288.0	2.99	520.	379.	-0.80	C.768	0.261E-01	0.198E+02	0.972	3.78	
290.0	3.14	527.	366.	-0.86	C.768	0.249E-01	0.199E+02	0.971	3.79	
292.0	3.31	534.	352.	-0.92	C.768	0.234E-01	0.200E+02	0.970	3.81	
294.0	3.49	542.	339.	-1.00	C.768	0.218E-01	0.201E+02	0.970	3.82	
296.0	3.67	551.	325.	-1.07	C.768	0.202E-01	0.202E+02	0.969	3.84	
298.0	3.91	559.	312.	-1.16	C.768	0.184E-01	0.203E+02	0.968	3.86	
300.0	4.15	569.	299.	-1.24	C.768	0.165E-01	0.203E+02	0.968	3.88	
302.0	4.40	577.	286.	-1.34	C.768	0.144E-01	0.204E+02	0.967	3.91	
304.0	4.69	587.	273.	-1.44	C.768	0.122E-01	0.204E+02	0.966	3.94	
306.0	5.00	597.	261.	-1.55	C.768	0.0973E-02	0.205E+02	0.965	3.97	
308.0	5.34	607.	248.	-1.65	C.768	0.0706E-02	0.205E+02	0.965	4.00	
310.0	5.71	618.	235.	-1.73	C.768	0.0414E-02	0.205E+02	0.964	4.03	
312.0	6.12	629.	224.	-1.91	C.768	0.0045E-03	0.205E+02	0.963	4.07	
314.0	6.57	641.	213.	-2.05	C.768	-0.257E-02	0.205E+02	0.962	4.12	
316.0	7.06	653.	202.	-2.19	C.768	-0.644E-02	0.205E+02	0.961	4.17	

318.0	7.60	665.	191.	-2.34	C.768	-0.107E-01	0.204E+02	0.961	4.23
320.0	8.20	678.	180.	-2.51	C.768	-0.154E-01	0.204E+02	0.960	4.29
322.0	8.86	691.	170.	-2.57	C.768	-0.207E-01	0.203E+02	0.959	4.36
324.0	9.58	705.	161.	-2.85	C.768	-0.264E-01	0.202E+02	0.958	4.44
326.0	10.37	719.	151.	-3.04	C.768	-0.329E-01	0.201E+02	0.957	4.52
328.0	11.24	733.	142.	-3.23	C.768	-0.400E-01	0.199E+02	0.956	4.62
330.0	12.19	748.	134.	-3.43	C.768	-0.479E-01	0.197E+02	0.955	4.74

COMBUSTION AND EXPANSION

OUTPUT DATA		THEO PRESS	PS	FSWIR	S.R.T	PPM	WNO	JET LOCATIONS		RJET	C.ANG	H.T.							
NUM	**	T.AV K	DIAM K	W.R.T	MASS MG	PMAS MG	UMAS MG	V CM**3	EQUI	CP CAL/G	ENTH CAL/G	PPM PPM	WNO G*10**6	R(CM)	Z(CM)	DEG	CM	DEG	CAL
333.7		PRESS.= 14.27	PS= -3.8	FSWIR= 0.953	S.R.T= 5.146	SUM.H.REJ=0.192E+02													
TOTAL			C.0		119.35	C.59						52.	42.						-0.62E-01
1	1	777.			758.6			117.31	C.03	0.267	193.8	52.	41.						-0.23E+00
2	1	777.	777.	C.0	13.2	0.0	0.0	2.03	5.69	0.432	193.8	48.	0.52	2.0	1.15	32.	0.463	0.	0.0
337.5		PRESS.= 16.61	PS= -4.2	FSWIR= 0.950	S.R.T= 5.506	SUM.H.REJ=0.186E+02													
TOTAL			-0.022		106.53	C.59						52.	42.						-0.83E-01
1	1	805.			742.1			102.19	C.03	0.268	201.4	52.	40.						-0.31E+00
2	2	737.	-182.	-0.012	20.1	1.8	1.5	2.51	3.32	0.349	178.5	31.	0.89	2.2	1.77	55.	0.783	137.	-0.13E-02
3	1	805.	805.	C.0	13.4	0.0	0.0	1.81	5.59	0.433	201.4	48.	0.53	2.0	1.16	33.	0.463	0.	0.0
341.2		PRESS.= 19.31	PS= -4.7	FSWIR= 0.948	S.R.T= 5.953	SUM.H.REJ=0.178E+02													
TOTAL			C.001		95.52	C.59						51.	42.						-0.11E+00
1	1	834.			717.7			88.01	C.03	0.269	209.1	52.	39.						-0.40E+00
2	2	903.	1214.	C.019	27.0	6.3	5.7	3.56	2.34	0.337	235.7	37.	1.27	2.4	2.22	74.	1.131	170.	-0.34E-02
3	2	770.	-88.	-0.012	27.7	1.8	1.5	2.30	3.20	0.349	187.7	31.	0.92	2.2	1.77	57.	0.790	138.	-0.20E-02
4	1	834.	834.	C.0	13.7	0.0	0.0	1.64	5.40	0.433	209.1	48.	0.55	2.0	1.15	34.	0.467	0.	0.0
345.0		PRESS.= 22.66	PS= -5.1	FSWIR= 0.945	S.R.T= 6.491	SUM.H.REJ=0.167E+02													
TOTAL			C.017		86.42	C.59						50.	42.						-0.14E+00
1	1	865.			685.3			74.25	C.03	0.271	217.3	52.	37.						-0.52E+00
2	2	1246.	1830.	C.123	34.2	14.7	13.7	5.29	1.80	0.347	353.7	41.	1.66	2.4	2.57	92.	1.367	177.	-0.99E-02
3	2	950.	1314.	C.024	27.9	6.6	6.0	3.26	2.25	0.337	247.9	37.	1.32	2.4	2.22	76.	1.122	168.	-0.49E-02
4	2	807.	61.	-0.011	21.5	1.9	1.6	2.11	3.06	0.348	196.6	32.	0.97	2.2	1.75	58.	0.798	138.	-0.30E-02
5	1	865.	865.	C.0	14.2	0.0	0.0	1.49	5.14	0.431	217.3	48.	0.57	2.0	1.14	34.	0.476	0.	0.0
349.7		PRESS.= 27.32	PS= -5.4	FSWIR= 0.941	S.R.T= 7.098	SUM.H.REJ=0.153E+02													
TOTAL			C.067		79.27	C.59						50.	42.						-0.20E+00
1	1	901.			647.4			60.27	C.03	0.272	227.3	52.	35.						-0.68E+00
2	2	1462.	2291.	C.374	41.8	29.8	28.0	8.03	1.44	0.351	570.0	44.	2.06	2.3	2.85	111.	1.420	170.	-0.23E-01
3	2	1321.	1919.	C.140	35.4	16.0	14.9	4.77	1.73	0.348	378.3	41.	1.72	2.4	2.56	96.	1.314	172.	-0.13E-01
4	2	1004.	1437.	C.030	28.9	7.0	6.3	2.94	2.17	0.338	267.3	38.	1.37	2.3	2.19	79.	1.105	166.	-0.71E-02
5	2	851.	252.	-0.059	22.4	2.0	1.7	1.91	2.91	0.348	212.1	33.	1.01	2.2	1.72	60.	0.807	138.	-0.44E-02
6	1	901.	901.	C.0	14.9	0.0	0.0	1.33	4.85	0.429	227.3	49.	0.61	2.0	1.12	35.	0.487	24.	-0.11E-02
352.5		PRESS.= 34.40	PS= -5.8	FSWIR= 0.936	S.R.T= 7.694	SUM.H.REJ=0.132E+02													
TOTAL			C.102		74.12	C.59						50.	42.						-0.28E+00
1	1	948.			591.2			46.24	C.03	0.274	240.0	52.	32.						-0.93E+00
2	1	2325.	2325.	C.632	50.3	41.8	41.8	9.58	1.18	0.356	732.8	48.	2.60	2.1	3.06	134.	1.341	148.	-0.39E-01
3	3	2158.	2138.	C.441	43.4	35.4	35.2	7.53	1.34	0.354	666.5	44.	2.15	2.2	2.83	117.	1.320	159.	-0.32E-01
4	3	1745.	1745.	C.277	36.7	28.9	27.6	5.14	1.66	0.357	528.0	42.	1.79	2.3	2.52	100.	1.250	165.	-0.23E-01
5	2	1247.	1510.	C.049	30.0	18.3	15.8	2.98	2.08	0.356	352.8	38.	1.43	2.3	2.14	83.	1.083	162.	-0.13E-01
6	2	823.	486.	-0.026	23.3	7.8	6.3	1.51	2.78	0.343	197.5	34.	1.06	2.2	1.67	62.	0.813	138.	-0.57E-02
7	1	948.	948.	C.0	15.5	0.0	0.0	1.14	4.58	0.427	240.0	49.	0.64	2.0	1.08	36.	0.500	42.	-0.26E-02
356.2		PRESS.= 41.54	PS= -6.0	FSWIR= 0.931	S.R.T= 8.140	SUM.H.REJ=0.102E+02													
TOTAL			C.314		71.02	C.59						75.	63.						-0.39E+00
1	1	933.			529.5			35.60	C.03	0.276	249.9	52.	29.						-0.12E+01
2	3	2532.	2532.	C.830	58.9	50.3	50.3	10.11	1.00	0.477	814.2	385.	23.71	1.8	3.24	164.	1.263	120.	-0.52E-01
3	3	2403.	2403.	C.674	51.7	43.4	43.4	8.38	1.15	0.365	760.6	56.	3.09	2.0	3.05	144.	1.240	133.	-0.49E-01
4	3	2217.	2217.	C.516	44.9	36.7	36.7	6.61	1.34	0.354	696.5	45.	2.22	2.1	2.79	125.	1.230	147.	-0.43E-01
5	3	1978.	1978.	C.362	37.3	30.0	30.0	4.93	1.61	0.359	615.4	42.	1.85	2.2	2.46	106.	1.189	158.	-0.34E-01
6	3	1471.	1470.	C.148	31.0	23.3	23.1	2.77	2.00	0.361	429.8	39.	1.48	2.3	2.07	87.	1.062	159.	-0.20E-01
7	2	878.	631.	-0.019	24.1	8.3	6.7	1.39	2.67	0.345	221.3	34.	1.11	2.2	1.60	65.	0.820	139.	-0.92E-02
8	1	933.	933.	C.0	16.1	0.0	0.0	1.01	4.36	0.425	249.8	49.	0.68	2.0	1.04	37.	0.513	55.	-0.46E-02

360.C PRESS.= 47.53 PS= -6.1 FSWIR= 0.925 S.RT= 8.275 SUM.H.REJ=0.656E+01															
TOTAL			C.464				69.99	0.59			284.	241.			-0.49E+00
1 1	1005.			443.1			27.80	0.03	0.277	254.7	52.	25.			-0.15E+01
2 3	2633.	2630.	C.973	65.3	58.9	58.9	10.18	0.90	0.478	861.0	2293.	156.40	1.7	3.40	202. 1.281 111.-0.65E-01
3 3	2584.	2584.	C.867	57.5	51.9	51.9	9.03	0.99	0.488	835.6	765.	47.67	1.8	3.23	179. 1.215 111.-0.59E-01
4 3	2457.	2457.	C.726	52.7	44.8	44.8	7.53	1.13	0.376	781.7	73.	4.69	1.9	3.01	155. 1.179 121.-0.56E-01
5 3	2272.	2272.	C.546	45.5	37.8	37.8	5.96	1.32	0.354	720.4	45.	2.26	2.1	2.73	133. 1.166 139.-0.52E-01
6 3	2046.	2046.	C.386	38.5	31.0	31.0	4.49	1.58	0.359	637.6	43.	1.88	2.2	2.38	112. 1.151 154.-0.44E-01
7 3	1553.	1553.	C.168	31.7	24.1	23.9	2.77	1.95	0.362	456.8	39.	1.52	2.3	1.98	90. 1.054 159.-0.26E-01
8 2	956.	843.	-0.012	24.7	8.6	7.0	1.32	2.59	0.346	239.3	35.	1.14	2.2	1.52	66. 0.832 142.-0.13E-01
9 1	1005.	1005.	C.0	16.6	0.0	0.0	0.93	4.22	0.423	255.7	49.	0.70	2.0	0.99	37. 0.525 66.-0.68E-02
363.7 PRESS.= 51.36 PS= -6.0 FSWIR= 0.919 S.RT= 8.035 SUM.H.REJ=0.231E+01															
TOTAL			C.589				71.02	0.59			616.	524.			-0.57E+00
1 1	1010.			410.4			22.91	0.03	0.277	257.1	52.	22.			-0.17E+01
2 3	2560.	2560.	C.986	71.1	65.3	65.3	9.99	0.83	0.424	829.7	3297.	244.12	1.9	3.54	238. 1.501 139.-0.93E-01
3 3	2638.	2638.	C.972	64.6	59.5	59.5	9.26	0.91	0.480	862.0	2712.	183.00	1.9	3.40	217. 1.328 125.-0.82E-01
4 3	2624.	2624.	C.895	58.6	52.7	52.7	8.28	1.01	0.499	853.3	1029.	63.22	1.8	3.21	192. 1.219 117.-0.70E-01
5 3	2486.	2486.	C.723	52.2	45.5	45.5	6.92	1.14	0.378	797.3	84.	4.65	1.9	2.95	166. 1.162 122.-0.63E-01
6 3	2319.	2319.	C.562	45.2	38.5	38.5	5.53	1.33	0.356	735.1	45.	2.25	2.1	2.64	140. 1.154 139.-0.60E-01
7 3	2097.	2097.	C.403	38.6	31.7	31.7	4.22	1.57	0.359	653.3	42.	1.88	2.2	2.28	116. 1.155 157.-0.51E-01
8 3	1608.	1608.	C.184	32.1	24.7	24.5	2.67	1.93	0.362	474.5	40.	1.54	2.3	1.87	92. 1.075 153.-0.31E-01
9 2	973.	952.	-0.005	25.0	8.6	7.0	1.28	2.55	0.347	251.2	35.	1.16	2.2	1.44	67. 0.854 148.-0.16E-01
367.6 PRESS.= 52.91 PS= -5.7 FSWIR= 0.913 S.RT= 7.505 SUM.H.REJ=-.227E+01															
TOTAL			C.713				74.12	0.59			882.	744.			-0.61E+00
1 1	1001.			357.7			19.22	0.03	0.277	254.7	52.	19.			-0.17E+01
2 3	2419.	2419.	C.000	80.2	71.1	71.1	10.33	0.73	0.357	774.3	3254.	271.29	2.3	3.68	265. 1.928 167.-0.11E+00
3 3	2504.	2504.	C.985	72.8	64.6	64.6	9.61	0.81	0.400	803.4	3117.	236.11	2.2	3.56	247. 1.701 160.-0.10E+00
4 3	2607.	2607.	C.961	64.6	58.6	58.6	8.80	0.91	0.466	844.9	2409.	162.37	2.1	3.39	226. 1.466 148.-0.96E+00
5 3	2619.	2619.	C.874	57.0	52.2	52.2	7.73	1.04	0.481	852.8	752.	45.05	2.0	3.16	201. 1.280 136.-0.82E-01
6 3	2504.	2504.	C.719	50.6	45.2	45.2	6.49	1.18	0.374	801.9	83.	4.51	2.0	2.87	172. 1.202 136.-0.71E-01
7 3	2345.	2345.	C.566	44.3	38.6	38.6	5.25	1.35	0.357	741.9	45.	2.20	2.2	2.53	144. 1.200 151.-0.66E-01
8 3	2123.	2123.	C.412	38.4	32.1	32.1	4.09	1.58	0.360	660.8	42.	1.87	2.3	2.15	118. 1.213 147.-0.56E-01
9 3	1600.	1600.	C.191	32.2	25.0	24.8	2.62	1.92	0.363	480.9	40.	1.55	2.4	1.75	92. 1.127 171.-0.34E-01
371.2 PRESS.= 51.57 PS= -5.1 FSWIR= 0.908 S.RT= 6.852 SUM.H.REJ=-.680E+01															
TOTAL			C.807				79.27	0.59			1037.	870.			-0.60E+00
1 1	978.			303.0			16.36	0.04	0.276	248.3	52.	17.			-0.16E+01
2 3	2278.	2278.	C.000	87.8	80.2	80.2	10.72	0.67	0.341	724.5	3045.	277.46	2.4	3.81	286. 2.253 178.-0.10E+00
3 3	2345.	2345.	C.000	81.9	72.8	72.8	10.39	0.72	0.353	743.8	2917.	248.31	2.5	3.70	270. 2.138 179.-0.11E+00
4 3	2453.	2453.	C.971	73.9	64.6	64.6	9.71	0.80	0.384	780.1	2541.	195.07	2.4	3.56	252. 1.912 176.-0.11E+00
5 3	2557.	2557.	C.949	64.2	57.0	57.0	8.72	0.92	0.450	823.6	1569.	105.02	2.3	3.35	230. 1.626 167.-0.10E+00
6 3	2573.	2573.	C.841	55.5	50.6	50.6	7.51	1.07	0.440	829.2	350.	20.47	2.2	3.08	203. 1.392 156.-0.87E-01
7 3	2431.	2431.	C.700	49.0	44.3	44.3	6.31	1.22	0.367	791.2	65.	3.47	2.2	2.76	173. 1.296 156.-0.76E-01
8 3	2335.	2335.	C.562	43.5	38.4	38.4	5.23	1.38	0.358	736.6	44.	2.14	2.3	2.39	143. 1.308 168.-0.69E-01
9 3	2110.	2110.	C.414	38.2	32.2	32.1	4.14	1.59	0.360	656.9	42.	1.86	2.5	2.01	116. 1.317 179.-0.57E-01
375.0 PRESS.= 48.00 PS= -4.3 FSWIR= 0.904 S.RT= 6.213 SUM.H.REJ=-.109E+02															
TOTAL			C.872				86.42	0.59			1140.	953.			-0.55E+00
1 1	944.			263.0			14.66	0.03	0.274	239.0	52.	14.			-0.14E+01
2 3	2199.	2199.	C.000	87.7	87.8	87.7	11.68	0.65	0.334	697.9	2969.	279.61	2.5	3.93	306. 2.339 180.-0.92E-01
3 3	2246.	2246.	C.000	85.7	81.9	81.9	11.19	0.68	0.340	709.6	2825.	251.99	2.4	3.38	290. 2.188 177.-0.91E-01
4 3	2312.	2312.	C.000	79.3	73.9	73.8	10.63	0.74	0.354	735.8	2483.	204.59	2.4	3.38	272. 2.084 179.-0.90E-01
5 3	2446.	2446.	C.976	71.5	64.2	64.2	9.96	0.82	0.389	777.0	1789.	132.95	2.5	3.39	251. 1.950 180.-0.91E-01
6 3	2522.	2522.	C.934	62.7	55.5	55.5	8.91	0.94	0.449	805.7	859.	56.14	2.5	3.28	220. 1.761 180.-0.95E-01
7 3	2511.	2511.	C.806	53.9	45.0	49.0	7.53	1.10	0.399	802.3	142.	8.09	2.4	2.97	159. 1.517 173.-0.84E-01
8 3	2434.	2434.	C.681	47.0	43.5	43.5	6.44	1.25	0.361	772.0	53.	2.75	2.4	2.60	168. 1.438 174.-0.75E-01
9 3	2306.	2306.	C.557	42.0	38.2	38.2	5.44	1.40	0.357	725.0	44.	2.10	2.5	2.22	139. 1.415 180.-0.66E-01
378.7 PRESS.= 43.57 PS= -3.4 FSWIR= 0.901 S.RT= 5.657 SUM.H.REJ=-.145E+02															
TOTAL			C.910				95.52	0.59			1224.	1019.			-0.47E+00
1 1	900.			236.2			13.93	0.03	0.273	228.3	52.	13.			-0.12E+01
2 3	2121.	2121.	C.000	90.7	90.7	90.7	12.74	0.63	0.330	672.0	2906.	280.35	2.4	4.06	325. 2.304 176.-0.82E-01
3 3	2151.	2151.	C.000	89.1	85.9	85.9	12.27	0.64	0.333	677.7	2736.	253.08	2.3	3.47	311. 2.119 177.-0.80E-01
4 3	2200.	2200.	C.000	81.5	79.3	79.3	11.63	0.72	0.343	707.1	2457.	207.90	2.4	3.48	293. 2.043 175.-0.79E-01

5	3	2389.	2389.	1.000	72.5	71.5	71.5	10.89	0.81	0.376	755.3	1952.	146.97	2.4	3.49	273.	1.950	178.-0.81E-01
6	3	2480.	2480.	0.995	66.9	62.7	62.7	10.32	0.88	0.417	787.4	1246.	86.77	2.5	3.47	249.	1.845	180.-0.80E-01
7	3	2502.	2502.	0.909	59.4	53.9	53.9	9.12	1.00	0.463	798.3	393.	24.38	2.5	3.16	222.	1.699	180.-0.79E-01
8	3	2460.	2460.	0.781	52.4	47.9	47.9	7.85	1.13	0.376	781.5	78.	4.37	2.5	2.80	191.	1.582	180.-0.72E-01
9	3	2385.	2385.	0.667	47.1	42.9	42.9	6.91	1.27	0.358	753.4	48.	2.45	2.5	2.42	162.	1.494	180.-0.66E-01
390.7		PRESS.= 41.27		PS=	-2.9	FSWIR=	0.899	S.RT=	5.402	SUM.H.REJ=-.162E+02								
TOTAL				C.950				101.16	0.59			1270.	1057.					-0.42E+00
1	1	894.			223.8			13.62	0.03	0.272	227.7	52.	12.					-0.53E+00
2	3	2073.	2073.	0.999	94.5	92.9	92.8	13.37	0.62	0.328	656.3	2800.	280.59	2.4	4.13	336.	1.285	174.-0.40E-01
3	3	2101.	2101.	0.998	90.7	86.1	89.0	12.87	0.65	0.370	661.1	2691.	253.40	2.3	3.51	323.	2.113	169.-0.39E-01
4	3	2206.	2206.	0.979	82.6	81.5	81.4	12.20	0.71	0.379	692.1	2436.	208.88	2.3	3.53	304.	2.043	173.-0.39E-01
5	3	2353.	2353.	0.979	73.1	72.5	72.4	11.41	0.81	0.369	741.9	1999.	151.73	2.4	3.54	284.	1.950	178.-0.40E-01
6	3	2446.	2446.	0.997	67.6	66.9	66.7	10.85	0.87	0.405	773.5	1392.	97.78	2.5	3.57	260.	1.849	180.-0.39E-01
7	3	2524.	2524.	0.977	61.8	59.4	59.1	10.10	0.96	0.461	808.4	628.	40.47	2.5	3.26	234.	1.739	180.-0.40E-01
8	3	2537.	2537.	0.874	54.7	52.4	52.2	8.92	1.09	0.423	812.2	149.	8.61	2.5	2.90	203.	1.624	180.-0.37E-01
9	3	2487.	2487.	0.758	49.2	47.1	46.8	7.84	1.21	0.371	790.4	60.	3.19	2.5	2.52	174.	1.535	180.-0.35E-01
382.7		PRESS.= 38.64		PS=	-2.4	FSWIR=	0.898	S.RT=	5.174	SUM.H.REJ=-.177E+02								
TOTAL				C.964				107.33	0.59			1304.	1083.					-0.38E+00
1	1	862.			209.5			13.27	0.03	0.271	214.7	52.	11.					-0.47E+00
2	3	2021.	2021.	0.999	96.2	94.5	94.4	14.18	0.61	0.324	639.2	2810.	280.74	2.3	4.20	348.	2.285	173.-0.37E-01
3	3	2052.	2052.	0.995	92.2	90.7	90.6	13.65	0.64	0.328	645.0	2649.	253.58	2.3	3.56	336.	2.113	169.-0.36E-01
4	3	2160.	2160.	0.979	83.7	82.6	82.5	12.92	0.70	0.336	676.4	2410.	209.41	2.3	3.58	316.	2.043	173.-0.36E-01
5	3	2307.	2307.	1.000	74.0	73.1	73.0	12.10	0.80	0.359	725.3	2011.	154.46	2.4	3.59	295.	1.974	179.-0.37E-01
6	3	2374.	2374.	0.979	70.0	67.6	67.5	11.65	0.84	0.380	745.5	1421.	103.29	2.5	3.66	271.	1.888	180.-0.37E-01
7	3	2466.	2466.	0.948	64.1	61.8	61.6	10.94	0.92	0.429	782.6	767.	51.17	2.5	3.36	246.	1.779	180.-0.38E-01
8	3	2525.	2525.	0.918	56.9	54.7	54.6	9.88	1.04	0.456	807.1	247.	14.79	2.5	3.00	215.	1.665	180.-0.36E-01
9	3	2494.	2494.	0.808	51.3	49.2	49.1	8.76	1.16	0.382	793.1	79.	4.33	2.5	2.61	186.	1.576	180.-0.34E-01
384.7		PRESS.= 36.06		PS=	-1.8	FSWIR=	0.896	S.RT=	4.973	SUM.H.REJ=-.191E+02								
TOTAL				C.975				114.01	0.59			1328.	1102.					-0.34E+00
1	1	840.			194.3			12.86	0.03	0.270	210.8	52.	11.					-0.41E+00
2	3	1956.	1956.	0.999	98.2	96.2	96.1	15.08	0.60	0.324	621.4	2755.	280.87	2.3	4.28	359.	2.292	172.-0.34E-01
3	3	1999.	1999.	0.999	93.9	92.2	92.1	14.52	0.63	0.326	627.7	2602.	253.71	2.3	3.61	348.	2.159	171.-0.34E-01
4	3	2107.	2107.	0.979	85.1	83.7	83.6	13.74	0.69	0.333	659.0	2374.	209.70	2.4	3.63	328.	2.084	175.-0.34E-01
5	3	2252.	2252.	0.979	75.3	74.0	73.9	12.88	0.78	0.350	705.5	1993.	155.85	2.5	3.65	306.	2.007	180.-0.34E-01
6	3	2302.	2302.	0.998	72.1	70.0	69.8	12.47	0.82	0.362	718.8	1414.	105.80	2.5	3.74	283.	1.922	180.-0.34E-01
7	3	2405.	2405.	0.998	66.4	64.1	64.0	11.85	0.89	0.401	757.5	833.	57.54	2.5	3.45	258.	1.819	180.-0.35E-01
8	3	2501.	2501.	0.954	59.1	56.9	56.8	10.89	1.00	0.471	795.9	363.	22.43	2.5	3.09	227.	1.705	180.-0.34E-01
9	3	2484.	2484.	0.853	53.4	51.3	51.2	9.74	1.11	0.396	790.6	103.	5.86	2.5	2.70	198.	1.616	180.-0.32E-01
345.7		PRESS.= 33.55		PS=	-1.3	FSWIR=	0.895	S.RT=	4.795	SUM.H.REJ=-.203E+02								
TOTAL				C.983				121.19	0.59			1345.	1115.					-0.31E+00
1	1	818.			177.8			12.31	0.03	0.269	204.8	52.	10.					-0.36E+00
2	3	1902.	1908.	0.999	100.4	98.2	98.1	16.09	0.59	0.322	602.8	2595.	281.00	2.3	4.35	371.	2.337	173.-0.32E-01
3	3	1939.	1939.	0.999	96.3	93.9	93.8	15.51	0.61	0.324	608.0	2540.	253.85	2.4	3.66	360.	2.249	174.-0.32E-01
4	3	2047.	2047.	0.999	87.0	85.1	85.0	14.67	0.68	0.330	639.0	2324.	209.88	2.4	3.66	339.	2.161	178.-0.31E-01
5	3	2193.	2193.	0.979	76.8	75.3	75.3	13.75	0.77	0.343	685.2	1963.	156.53	2.5	3.70	317.	2.030	180.-0.32E-01
6	3	2242.	2242.	0.999	74.2	72.1	72.0	13.37	0.79	0.350	693.8	1389.	106.93	2.5	3.83	294.	1.957	180.-0.31E-01
7	3	2329.	2329.	0.999	68.7	66.4	66.3	12.76	0.86	0.375	728.2	848.	60.54	2.5	3.54	270.	1.858	180.-0.33E-01
8	3	2459.	2459.	0.975	61.3	59.1	59.0	11.93	0.96	0.449	776.4	457.	29.19	2.5	3.18	239.	1.745	180.-0.32E-01
9	3	2476.	2476.	0.895	56.4	53.4	53.3	10.82	1.07	0.418	785.9	136.	7.91	2.5	2.79	210.	1.655	180.-0.31E-01
389.7		PRESS.= 31.17		PS=	-0.7	FSWIR=	0.894	S.RT=	4.637	SUM.H.REJ=-.214E+02								
TOTAL				C.990				128.87	0.59			1357.	1125.					-0.27E+00
1	1	797.			160.0			11.61	0.03	0.268	199.1	52.	9.					-0.31E+00
2	3	1848.	1848.	0.999	103.0	100.4	100.3	17.20	0.57	0.321	583.2	2628.	281.14	2.4	4.42	382.	2.417	176.-0.29E-01
3	3	1871.	1871.	0.979	99.2	96.3	96.1	16.61	0.59	0.322	586.4	2465.	254.02	2.4	3.71	371.	2.380	179.-0.29E-01
4	3	1984.	1984.	0.979	89.2	87.0	86.9	15.69	0.66	0.327	618.4	2268.	210.03	2.5	3.73	349.	2.235	180.-0.28E-01
5	3	2135.	2135.	0.979	78.3	76.8	76.7	14.69	0.75	0.336	665.4	1930.	156.87	2.5	3.75	329.	2.054	180.-0.29E-01
6	3	2163.	2163.	0.999	76.2	74.2	74.1	14.34	0.77	0.341	669.9	1358.	107.44	2.5	3.90	305.	1.991	180.-0.29E-01
7	3	2254.	2254.	0.998	71.0	68.7	68.6	13.73	0.83	0.358	700.7	840.	61.87	2.5	3.62	281.	1.897	180.-0.30E-01
8	3	2411.	2411.	0.992	63.5	61.3	61.2	13.04	0.93	0.421	755.6	518.	34.22	2.5	3.27	251.	1.785	180.-0.30E-01
9	3	2459.	2459.	0.933	57.4	55.4	55.3	11.98	1.03	0.447	778.7	178.	10.70	2.5	2.87	221.	1.694	180.-0.29E-01

390.7 PRESS.= 28.93 PS= -0.2 FSWIR= 0.892 S.RT= 4.49A SUM.H.REJ=-.223E+02

TOTAL			0.994				137.01	C.59			1366.	1132.							-0.24E+00
1	1	776.		143.3			10.91	C.01	0.266	193.4	52.	8.							-0.26E+00
2	3	1784.	1794.	0.999	105.1	103.0	102.9	19.43	C.56	0.319	563.9	2554.	281.31	2.5	4.50	393.	2.528	179.	-0.27E-01
3	3	1817.	1817.	0.999	101.3	99.2	99.1	17.75	C.58	0.321	569.0	2415.	254.13	2.5	3.76	381.	2.427	180.	-0.26E-01
4	3	1932.	1932.	0.999	90.7	89.2	89.1	16.75	C.65	0.325	601.5	2231.	210.12	2.5	3.78	360.	2.258	180.	-0.26E-01
5	3	2077.	2077.	0.999	70.8	78.3	78.2	15.70	C.74	0.334	646.1	1895.	157.06	2.5	3.80	340.	2.078	180.	-0.27E-01
6	3	2096.	2096.	0.999	78.3	76.2	76.1	15.37	C.75	0.336	647.1	1325.	107.71	2.5	3.98	316.	2.025	180.	-0.27E-01
7	3	2190.	2190.	0.999	73.2	71.0	70.9	14.76	C.80	0.346	674.7	822.	62.47	2.5	3.71	293.	1.936	180.	-0.28E-01
8	3	2347.	2347.	0.999	65.6	63.5	63.4	14.13	C.90	0.393	729.6	544.	37.07	2.5	3.35	262.	1.824	180.	-0.28E-01
9	3	2434.	2434.	0.955	59.4	57.4	57.3	13.22	1.00	0.456	767.2	227.	14.07	2.5	2.95	233.	1.732	180.	-0.27E-01

392.7 PRESS.= 26.82 PS= 0.3 FSWIR= 0.892 S.RT= 4.37A SUM.H.REJ=-.232E+02

TOTAL			0.997				145.52	C.59			1371.	1136.							-0.21E+00
1	1	755.		127.1			10.16	C.03	0.265	187.9	52.	7.							-0.22E+00
2	3	1722.	1722.	0.999	109.3	106.1	105.9	19.77	C.54	0.317	543.1	2480.	281.48	2.5	4.16	403.	2.596	180.	-0.25E-01
3	3	1771.	1771.	0.999	102.9	101.3	101.2	18.96	C.57	0.320	554.3	2378.	254.21	2.5	3.81	391.	2.449	180.	-0.24E-01
4	3	1882.	1882.	0.999	92.3	90.7	90.6	17.89	C.64	0.324	585.2	2194.	210.21	2.5	3.83	370.	2.281	180.	-0.24E-01
5	3	2021.	2021.	0.999	81.4	75.8	79.8	14.79	C.72	0.331	627.4	1861.	157.13	2.5	3.85	351.	2.103	180.	-0.25E-01
6	3	2031.	2031.	0.999	80.4	78.3	78.2	14.49	C.73	0.332	625.4	1293.	107.88	2.5	4.05	327.	2.059	180.	-0.24E-01
7	3	2109.	2109.	0.999	75.4	73.2	73.1	15.86	C.78	0.338	650.1	802.	62.77	2.5	3.79	305.	1.974	180.	-0.25E-01
8	3	2274.	2274.	0.999	67.7	65.6	65.5	15.24	C.87	0.370	701.8	546.	38.42	2.5	3.43	274.	1.862	180.	-0.26E-01
9	3	2322.	2322.	0.983	61.4	59.4	59.3	14.48	C.96	0.433	748.4	266.	16.98	2.5	3.03	245.	1.770	180.	-0.25E-01

394.7 PRESS.= 24.87 PS= 0.8 FSWIR= 0.891 S.RT= 4.267 SUM.H.REJ=-.239E+02

TOTAL			0.999				154.57	C.59			1374.	1139.							-0.18E+00
1	1	735.		111.3			7.31	C.03	0.264	182.7	52.	6.							-0.18E+00
2	3	1664.	1664.	0.999	112.4	109.3	109.2	21.19	C.52	0.316	524.6	2412.	281.65	2.5	4.27	413.	2.639	180.	-0.22E-01
3	3	1724.	1724.	0.999	104.5	102.9	102.8	20.23	C.55	0.318	540.0	2343.	254.31	2.5	3.86	402.	2.472	180.	-0.22E-01
4	3	1833.	1833.	0.999	93.8	92.3	92.2	19.11	C.64	0.323	545.3	2158.	210.30	2.5	3.89	381.	2.305	180.	-0.22E-01
5	3	1956.	1956.	0.999	87.9	81.4	81.3	17.95	C.71	0.329	609.4	1828.	157.28	2.5	3.90	361.	2.128	180.	-0.23E-01
6	3	1957.	1957.	0.999	82.4	80.4	80.3	17.67	C.71	0.329	604.9	1263.	108.01	2.5	4.13	338.	2.093	180.	-0.22E-01
7	3	2040.	2040.	0.999	77.6	75.4	75.3	17.03	C.76	0.334	627.1	781.	62.95	2.5	3.87	316.	2.011	180.	-0.23E-01
8	3	2202.	2202.	0.998	69.8	67.7	67.6	16.41	C.84	0.354	675.7	539.	39.06	2.5	3.51	285.	1.900	180.	-0.24E-01
9	3	2344.	2344.	0.998	63.3	61.4	61.3	15.79	C.93	0.407	728.2	290.	19.08	2.5	3.11	256.	1.867	180.	-0.24E-01

396.7 PRESS.= 23.05 PS= 1.3 FSWIR= 0.890 S.RT= 4.170 SUM.H.REJ=-.245E+02

TOTAL			0.999				164.14	C.59			1376.	1140.							-0.16E+00
1	1	715.		94.9			8.37	C.04	0.263	177.6	52.	5.							-0.15E+00
2	3	1608.	1608.	0.999	115.5	112.4	112.3	22.70	C.51	0.314	507.2	2349.	281.82	2.5	4.37	423.	2.682	180.	-0.20E-01
3	3	1682.	1682.	0.999	106.1	104.5	104.5	21.60	C.56	0.317	526.0	2308.	254.39	2.5	3.90	412.	2.495	180.	-0.20E-01
4	3	1785.	1785.	0.999	95.4	93.8	93.8	20.41	C.62	0.321	553.9	2124.	210.38	2.5	3.93	391.	2.328	180.	-0.20E-01
5	3	1913.	1913.	0.999	84.5	82.9	82.8	19.17	C.70	0.327	591.9	1795.	157.37	2.5	3.95	372.	2.153	180.	-0.21E-01
6	3	1903.	1903.	0.999	84.4	82.4	82.3	18.93	C.70	0.327	585.7	1234.	108.13	2.5	4.20	349.	2.127	180.	-0.20E-01
7	3	1974.	1974.	0.999	79.8	77.6	77.5	18.28	C.74	0.330	605.2	762.	63.09	2.5	3.94	327.	2.048	180.	-0.21E-01
8	3	2132.	2132.	0.999	71.9	69.8	69.7	17.64	C.82	0.344	651.1	528.	39.38	2.5	3.59	297.	1.937	180.	-0.22E-01
9	3	2276.	2276.	0.999	65.3	63.3	63.3	17.04	C.90	0.381	701.5	298.	20.18	2.5	3.18	268.	1.844	180.	-0.22E-01

398.7 PRESS.= 21.39 PS= 1.8 FSWIR= 0.890 S.RT= 4.084 SUM.H.REJ=-.250E+02

TOTAL			0.999				174.02	C.59			1377.	1140.							-0.13E+00
1	1	697.		79.0			7.31	C.03	0.262	172.7	52.	4.							-0.12E+00
2	3	1556.	1556.	0.999	119.4	115.5	115.4	24.29	C.50	0.313	490.9	2290.	281.99	2.5	4.48	433.	2.724	180.	-0.18E-01
3	3	1640.	1640.	0.999	107.8	106.1	106.1	23.03	C.55	0.316	517.5	2274.	254.48	2.5	3.95	422.	2.518	180.	-0.18E-01
4	3	1739.	1739.	0.999	97.0	95.4	95.3	21.78	C.61	0.320	539.1	2090.	210.47	2.5	3.98	401.	2.352	180.	-0.18E-01
5	3	1852.	1852.	0.999	85.0	84.5	84.4	20.50	C.68	0.325	575.2	1753.	157.46	2.5	3.99	383.	2.178	180.	-0.19E-01
6	3	1851.	1851.	0.999	84.4	84.4	84.3	20.27	C.68	0.325	566.6	1206.	108.24	2.5	4.27	360.	2.160	180.	-0.19E-01
7	3	1912.	1912.	0.999	82.0	79.8	79.7	19.59	C.72	0.328	584.7	743.	63.22	2.5	4.02	339.	2.085	180.	-0.20E-01
8	3	2064.	2064.	0.999	73.9	71.9	71.8	18.93	C.80	0.337	628.0	516.	39.57	2.5	3.66	308.	1.974	180.	-0.20E-01
9	3	2207.	2207.	0.999	67.2	65.3	65.2	18.33	C.88	0.362	675.9	297.	20.74	2.5	3.25	279.	1.880	180.	-0.20E-01

400.7 PRESS.= 19.88 PS= 2.2 FSWIR= 0.889 S.RT= 4.007 SUM.H.REJ=-.255E+02

TOTAL			0.999				184.29	C.59			1377.	1141.							-0.11E+00
1	1	680.		63.2			5.13	C.03	0.261	168.1	52.	3.							-0.88E-01
2	3	1507.	1507.	0.999	121.6	118.6	118.4	25.96	C.49	0.311	475.5	2235.	282.15	2.5	4.57	443.	2.765	180.	-0.17E-01
3	3	1599.	1599.	0.999	109.4	107.8	107.7	24.53	C.54	0.315	499.5	2241.	254.57	2.5	4.00	433.	2.541	180.	-0.17E-01
4	3	1695.	1695.	0.999	98.6	97.0	96.9	23.22	C.60	0.319	524.9	2057.	210.56	2.5	4.03	412.	2.376	180.	-0.17E-01

5	3	1813.	1813.	0.999	97.6	86.0	85.9	21.97	0.67	0.324	559.2	1733.	157.54	2.5	4.04	394.	2.203	180.-0.17E-01
6	3	1797.	1797.	0.999	89.4	86.4	86.3	21.67	0.67	0.323	549.1	1180.	108.35	2.5	4.33	370.	2.193	180.-0.17E-01
7	3	1853.	1853.	0.999	84.1	82.0	81.9	20.97	0.70	0.326	565.4	725.	63.33	2.5	4.09	350.	2.121	180.-0.18E-01
8	3	1949.	1949.	0.999	76.0	73.9	73.8	20.28	0.77	0.333	606.4	504.	39.71	2.5	3.73	319.	2.011	180.-0.18E-01
9	3	2132.	2132.	0.999	69.1	67.2	67.1	19.67	0.85	0.349	652.0	293.	21.03	2.5	3.32	290.	1.916	180.-0.19E-01

EXPANSION PROCESS IS CALCULATED AS ONE ELEMENT

PRESSURE = 15.88ATA
 AV. TEMPERATURE = 1663.9K DEG
 AV. EQUI. RATIO = 0.590
 AV. ENTHALPY = 515.11CAL/G
 TOTAL MASS = 0.7978G

EXHAUST NOX

NOX CONCEN. = 1377.0PPM
 TOTAL MASS = 0.173E-01G

THED DEG	P ATA	T K	V CM**3	WCRK PS	CYL.MAS G	HEAT TRANSFER CAL/A CAL	FSWR	S.RY *RPM
402.7	18.47	1536.	155.	2.53	0.798	-0.500E+00 -0.275E+02	0.888	4.01
404.7	17.20	1509.	206.	3.04	0.798	-0.470E+00 -0.294E+02	0.887	3.94
406.7	16.04	1543.	217.	3.43	0.798	-0.443E+00 -0.312E+02	0.887	3.88
408.7	14.99	1558.	229.	3.81	0.798	-0.419E+00 -0.328E+02	0.886	3.82
410.7	14.03	1534.	241.	4.17	0.798	-0.376E+00 -0.344E+02	0.885	3.77
412.7	13.15	1511.	253.	4.51	0.798	-0.376E+00 -0.359E+02	0.885	3.73
414.7	12.36	1489.	265.	4.83	0.798	-0.357E+00 -0.373E+02	0.884	3.68
416.7	11.63	1468.	278.	5.14	0.798	-0.340E+00 -0.387E+02	0.883	3.65
418.7	10.96	1448.	291.	5.43	0.798	-0.324E+00 -0.400E+02	0.883	3.61
420.7	10.35	1427.	304.	5.71	0.798	-0.310E+00 -0.412E+02	0.882	3.58
422.7	9.79	1410.	317.	5.98	0.798	-0.297E+00 -0.424E+02	0.881	3.55
424.7	9.28	1392.	330.	6.23	0.798	-0.284E+00 -0.436E+02	0.881	3.51
426.7	8.81	1375.	344.	6.47	0.798	-0.273E+00 -0.447E+02	0.880	3.50
428.7	8.38	1359.	357.	6.70	0.798	-0.263E+00 -0.457E+02	0.880	3.48
430.7	7.98	1344.	371.	6.91	0.798	-0.253E+00 -0.467E+02	0.879	3.46
432.7	7.61	1329.	384.	7.12	0.798	-0.244E+00 -0.477E+02	0.879	3.44
434.7	7.27	1314.	398.	7.31	0.798	-0.235E+00 -0.486E+02	0.878	3.42
436.7	6.95	1300.	412.	7.50	0.798	-0.228E+00 -0.495E+02	0.877	3.40
438.7	6.66	1286.	425.	7.67	0.798	-0.220E+00 -0.504E+02	0.877	3.39
440.7	6.39	1274.	439.	7.84	0.798	-0.213E+00 -0.513E+02	0.876	3.37
442.7	6.14	1261.	452.	8.00	0.798	-0.207E+00 -0.521E+02	0.876	3.36
444.7	5.91	1250.	466.	8.15	0.798	-0.201E+00 -0.529E+02	0.875	3.35
446.7	5.69	1238.	479.	8.29	0.798	-0.196E+00 -0.537E+02	0.875	3.33
448.7	5.49	1227.	492.	8.42	0.798	-0.190E+00 -0.545E+02	0.874	3.32
450.7	5.30	1217.	505.	8.55	0.798	-0.185E+00 -0.552E+02	0.874	3.31
452.7	5.13	1207.	519.	8.67	0.798	-0.181E+00 -0.559E+02	0.873	3.30
454.7	4.96	1197.	531.	8.79	0.798	-0.176E+00 -0.566E+02	0.873	3.29
456.7	4.81	1188.	544.	8.90	0.798	-0.172E+00 -0.573E+02	0.872	3.28
458.7	4.67	1179.	556.	9.00	0.798	-0.168E+00 -0.580E+02	0.872	3.28
460.7	4.53	1170.	568.	9.10	0.798	-0.164E+00 -0.586E+02	0.871	3.27
462.7	4.41	1162.	580.	9.19	0.798	-0.161E+00 -0.593E+02	0.871	3.26
464.7	4.29	1154.	592.	9.28	0.798	-0.158E+00 -0.599E+02	0.870	3.25
466.7	4.18	1147.	603.	9.36	0.798	-0.155E+00 -0.605E+02	0.870	3.24
468.7	4.08	1139.	615.	9.44	0.798	-0.152E+00 -0.611E+02	0.869	3.24
470.7	3.99	1132.	625.	9.51	0.798	-0.147E+00 -0.617E+02	0.869	3.23
472.7	3.89	1125.	637.	9.58	0.798	-0.146E+00 -0.623E+02	0.868	3.23
474.7	3.81	1117.	647.	9.65	0.798	-0.144E+00 -0.629E+02	0.868	3.22
476.7	3.73	1111.	657.	9.71	0.798	-0.141E+00 -0.635E+02	0.867	3.21
478.7	3.65	1107.	667.	9.77	0.798	-0.139E+00 -0.640E+02	0.867	3.21
480.0	3.51	1100.	673.	9.81	0.798	-0.137E+00 -0.644E+02	0.866	3.20

EXHAUST VALVE OPEN

OUTPUT DATA		P ATA	T K	V CM**3	WCRK PS	CYL.MAS G	HEAT CAL/A	TRANSFR CAL	AEFE CM**2	GWE G/S
THFC DEG	DEG									
492.0	3.54	1098.	683.	9.96	0.798	-0.127E+03	-0.649E+02	0.00	-0.06	
494.0	3.48	1092.	692.	9.91	0.798	-0.125E+00	-0.654E+02	0.01	-0.18	
496.0	3.42	1087.	701.	9.96	0.798	-0.123E+00	-0.659E+02	0.01	-0.30	
498.0	3.36	1082.	709.	10.01	0.798	-0.121E+00	-0.663E+02	0.01	-0.42	
490.0	3.31	1078.	718.	10.05	0.798	-0.120E+00	-0.668E+02	0.01	-0.53	
492.0	3.26	1073.	725.	10.09	0.797	-0.118E+00	-0.673E+02	0.02	-0.63	
494.0	3.21	1069.	733.	10.13	0.797	-0.117E+00	-0.678E+02	0.02	-0.74	
496.0	3.16	1065.	740.	10.16	0.797	-0.115E+00	-0.682E+02	0.05	-1.15	
498.0	3.12	1060.	747.	10.20	0.797	-0.114E+00	-0.687E+02	0.09	-2.69	
500.0	3.08	1055.	754.	10.23	0.796	-0.113E+00	-0.691E+02	0.13	-4.30	
502.0	3.04	1052.	760.	10.26	0.795	-0.111E+00	-0.696E+02	0.17	-5.87	
504.0	3.00	1049.	766	10.28	0.794	-0.110E+00	-0.700E+02	0.22	-7.43	
506.0	2.96	1045.	772.	10.31	0.792	-0.109E+00	-0.705E+02	0.47	-12.25	
508.0	2.91	1040.	777.	10.33	0.788	-0.107E+00	-0.709E+02	0.78	-23.11	
510.0	2.86	1035.	782.	10.35	0.782	-0.105E+00	-0.713E+02	1.09	-33.92	
512.0	2.80	1029.	787.	10.37	0.775	-0.103E+00	-0.717E+02	1.39	-44.31	
514.0	2.73	1023.	791.	10.39	0.766	-0.100E+00	-0.721E+02	1.70	-54.22	
516.0	2.66	1017.	795.	10.40	0.755	-0.976E-01	-0.725E+02	2.26	-67.46	
518.0	2.58	1009.	799.	10.42	0.740	-0.944E-01	-0.729E+02	2.87	-85.75	
520.0	2.49	1000.	803.	10.43	0.723	-0.906E-01	-0.732E+02	3.47	-102.86	
522.0	2.39	990.	806.	10.44	0.703	-0.864E-01	-0.736E+02	4.08	-118.37	
524.0	2.28	980.	808.	10.45	0.681	-0.818E-01	-0.739E+02	4.68	-132.14	
526.0	2.17	968.	811.	10.45	0.658	-0.770E-01	-0.742E+02	5.21	-142.99	
528.0	2.05	956.	813.	10.46	0.632	-0.720E-01	-0.745E+02	5.74	-151.15	
530.0	1.94	943.	815.	10.46	0.606	-0.670E-01	-0.748E+02	6.27	-157.73	
532.0	1.82	930.	816.	10.46	0.579	-0.619E-01	-0.750E+02	6.80	-162.68	
534.0	1.70	916.	817.	10.47	0.551	-0.568E-01	-0.753E+02	7.29	-165.24	
536.0	1.59	902.	818.	10.47	0.524	-0.519E-01	-0.755E+02	7.64	-162.64	
538.0	1.49	888.	819.	10.47	0.498	-0.472E-01	-0.757E+02	7.98	-156.54	
540.0	1.39	874.	819.	10.47	0.474	-0.429E-01	-0.758E+02	8.32	-148.26	
542.0	1.31	861.	819.	10.47	0.451	-0.390E-01	-0.760E+02	8.66	-137.03	
544.0	1.23	849.	818.	10.47	0.430	-0.355E-01	-0.761E+02	8.96	-124.37	
546.0	1.17	835.	817.	10.47	0.412	-0.325E-01	-0.763E+02	9.12	-107.27	
548.0	1.11	823.	816.	10.47	0.397	-0.300E-01	-0.764E+02	9.28	-87.53	
550.0	1.07	812.	815.	10.47	0.386	-0.281E-01	-0.765E+02	9.45	-67.28	
552.0	1.05	817.	813.	10.47	0.378	-0.267E-01	-0.764E+02	9.61	-45.67	
554.0	1.04	814.	811.	10.47	0.375	-0.259E-01	-0.767E+02	9.72	-22.98	
556.0	1.04	814.	808.	10.47	0.374	-0.255E-01	-0.768E+02	9.75	-5.01	
558.0	1.04	813.	806.	10.47	0.373	-0.254E-01	-0.769E+02	9.79	-7.20	
560.0	1.04	813.	803.	10.47	0.371	-0.253E-01	-0.770E+02	9.79	-7.33	
562.0	1.04	813.	799.	10.47	0.370	-0.251E-01	-0.771E+02	9.79	-8.14	
564.0	1.04	812.	795.	10.46	0.369	-0.250E-01	-0.772E+02	9.79	-8.94	
566.0	1.04	812.	791.	10.46	0.367	-0.249E-01	-0.773E+02	9.79	-9.83	
568.0	1.04	811.	787.	10.46	0.365	-0.247E-01	-0.774E+02	9.79	-10.82	
570.0	1.04	811.	782.	10.46	0.363	-0.246E-01	-0.775E+02	9.79	-12.56	
572.0	1.04	810.	777.	10.46	0.361	-0.244E-01	-0.776E+02	9.79	-13.45	
574.0	1.04	810.	772.	10.46	0.358	-0.242E-01	-0.777E+02	9.79	-13.74	
576.0	1.04	809.	766.	10.46	0.356	-0.241E-01	-0.778E+02	9.79	-14.31	
578.0	1.04	809.	760.	10.46	0.354	-0.239E-01	-0.779E+02	9.79	-15.08	
580.0	1.04	808.	754.	10.46	0.351	-0.237E-01	-0.780E+02	9.79	-15.93	
582.0	1.04	808.	747.	10.46	0.348	-0.235E-01	-0.781E+02	9.79	-16.82	
584.0	1.04	807.	740.	10.46	0.345	-0.233E-01	-0.782E+02	9.79	-17.72	
586.0	1.04	807.	733.	10.46	0.342	-0.231E-01	-0.783E+02	9.79	-18.62	
588.0	1.04	807.	725.	10.46	0.339	-0.229E-01	-0.783E+02	9.79	-19.52	
590.0	1.04	806.	718.	10.46	0.335	-0.227E-01	-0.784E+02	9.79	-20.42	
592.0	1.04	805.	709.	10.46	0.332	-0.225E-01	-0.785E+02	9.79	-21.31	
594.0	1.04	805.	701.	10.46	0.328	-0.222E-01	-0.786E+02	9.79	-22.20	

596.0	1.04	805.	692.	10.46	0.324	-0.220E-01	-0.787E+02	9.79	-23.09
598.0	1.04	805.	683.	10.46	0.320	-0.219E-01	-0.788E+02	9.79	-23.56
600.0	1.04	804.	673.	10.45	0.316	-0.215E-01	-0.789E+02	9.79	-24.84
602.0	1.04	804.	664.	10.45	0.312	-0.213E-01	-0.790E+02	9.79	-25.70
604.0	1.04	804.	653.	10.45	0.307	-0.210E-01	-0.790E+02	9.79	-26.55
606.0	1.04	803.	643.	10.45	0.303	-0.208E-01	-0.791E+02	9.79	-27.39
608.0	1.04	803.	633.	10.45	0.298	-0.205E-01	-0.792E+02	9.79	-28.22
610.0	1.04	802.	622.	10.45	0.293	-0.202E-01	-0.793E+02	9.79	-29.03
612.0	1.05	802.	611.	10.45	0.288	-0.199E-01	-0.794E+02	9.79	-29.82
614.0	1.05	802.	599.	10.45	0.283	-0.197E-01	-0.795E+02	9.79	-30.59
616.0	1.05	801.	588.	10.45	0.278	-0.194E-01	-0.795E+02	9.79	-31.35
618.0	1.05	801.	576.	10.44	0.273	-0.191E-01	-0.796E+02	9.79	-32.07
620.0	1.05	800.	564.	10.44	0.267	-0.188E-01	-0.797E+02	9.79	-32.77
622.0	1.05	800.	551.	10.44	0.262	-0.185E-01	-0.798E+02	9.79	-33.45
624.0	1.05	800.	539.	10.44	0.256	-0.182E-01	-0.798E+02	9.79	-34.09
626.0	1.05	799.	526.	10.44	0.250	-0.179E-01	-0.799E+02	9.79	-34.70
628.0	1.05	799.	513.	10.44	0.244	-0.176E-01	-0.800E+02	9.79	-35.27
630.0	1.05	798.	500.	10.44	0.238	-0.172E-01	-0.800E+02	9.79	-35.80
632.0	1.05	798.	487.	10.44	0.232	-0.169E-01	-0.801E+02	9.79	-36.29
634.0	1.05	797.	474.	10.43	0.226	-0.166E-01	-0.802E+02	9.79	-36.74
636.0	1.05	797.	461.	10.43	0.220	-0.163E-01	-0.802E+02	9.79	-37.14
638.0	1.05	795.	447.	10.43	0.214	-0.160E-01	-0.803E+02	9.79	-37.49
640.0	1.05	795.	434.	10.43	0.207	-0.156E-01	-0.804E+02	9.79	-37.78
642.0	1.05	795.	420.	10.43	0.201	-0.153E-01	-0.804E+02	9.79	-38.03
644.0	1.05	795.	406.	10.43	0.195	-0.150E-01	-0.805E+02	9.79	-38.21
646.0	1.05	794.	393.	10.42	0.188	-0.146E-01	-0.805E+02	9.79	-38.34
648.0	1.05	794.	379.	10.42	0.182	-0.143E-01	-0.806E+02	9.79	-38.41
650.0	1.05	793.	366.	10.42	0.176	-0.140E-01	-0.807E+02	9.79	-38.41
652.0	1.05	793.	352.	10.42	0.169	-0.137E-01	-0.807E+02	9.79	-38.34
654.0	1.05	792.	339.	10.42	0.163	-0.133E-01	-0.808E+02	9.79	-38.21
656.0	1.05	792.	325.	10.42	0.156	-0.130E-01	-0.808E+02	9.79	-38.01
658.0	1.05	791.	312.	10.42	0.150	-0.127E-01	-0.809E+02	9.75	-37.70
660.0	1.05	791.	299.	10.41	0.144	-0.124E-01	-0.809E+02	9.72	-37.35
662.0	1.05	790.	286.	10.41	0.138	-0.121E-01	-0.810E+02	9.61	-36.89
664.0	1.05	789.	273.	10.41	0.132	-0.118E-01	-0.810E+02	9.45	-36.26
666.0	1.05	789.	261.	10.41	0.126	-0.115E-01	-0.811E+02	9.28	-35.72
668.0	1.05	788.	248.	10.41	0.120	-0.112E-01	-0.811E+02	9.12	-35.13
670.0	1.05	787.	236.	10.41	0.114	-0.109E-01	-0.811E+02	8.96	-34.48
672.0	1.05	787.	224.	10.41	0.109	-0.106E-01	-0.812E+02	8.66	-33.61
674.0	1.05	786.	211.	10.40	0.103	-0.104E-01	-0.812E+02	8.32	-32.69
676.0	1.05	785.	200.	10.40	0.098	-0.101E-01	-0.813E+02	7.98	-31.83
678.0	1.05	785.	191.	10.40	0.093	-0.988E-02	-0.813E+02	7.64	-30.92
680.0	1.05	784.	180.	10.40	0.088	-0.966E-02	-0.813E+02	7.29	-29.95
682.0	1.05	783.	170.	10.40	0.083	-0.944E-02	-0.814E+02	6.80	-28.74
684.0	1.05	783.	161.	10.40	0.078	-0.925E-02	-0.814E+02	6.27	-27.47
686.0	1.05	782.	151.	10.40	0.074	-0.907E-02	-0.815E+02	5.74	-26.29
688.0	1.05	782.	142.	10.40	0.070	-0.891E-02	-0.815E+02	5.21	-25.02
690.0	1.05	781.	134.	10.39	0.066	-0.877E-02	-0.815E+02	4.68	-23.69
692.0	1.05	781.	126.	10.39	0.062	-0.865E-02	-0.816E+02	4.08	-22.16
694.0	1.07	781.	118.	10.39	0.059	-0.857E-02	-0.816E+02	3.47	-20.51
696.0	1.07	781.	111.	10.39	0.056	-0.852E-02	-0.816E+02	2.87	-18.75
698.0	1.07	781.	105.	10.39	0.053	-0.853E-02	-0.817E+02	2.26	-16.76
700.0	1.10	782.	99.	10.39	0.050	-0.862E-02	-0.817E+02	1.70	-14.46
702.0	1.12	784.	94.	10.39	0.048	-0.877E-02	-0.817E+02	1.39	-12.87
704.0	1.13	785.	89.	10.39	0.046	-0.896E-02	-0.818E+02	1.07	-11.41
706.0	1.15	787.	84.	10.38	0.045	-0.920E-02	-0.818E+02	0.78	-9.41
708.0	1.18	790.	81.	10.38	0.044	-0.953E-02	-0.818E+02	0.47	-6.94
710.0	1.22	793.	77.	10.38	0.043	-0.100E-01	-0.819E+02	0.22	-4.02

EXHAUST TEMPERATURE= 811.9KDEG

*PREDICTION OF THE FRICTION LOSS**
MISC. AND PUMPS= 77.6420/CV**2
CAM BEAR = 101.3100/CV**2
BEARINGS = 140.8456/CV**2

BLEWBY = 43.481G/CM**2
VISCIOUS PISTON = 366.137G/CM**2
RING TENSION = 115.044G/CM**2
RING GAS PRESS = 7.335G/CM**2

TOTAL LOSS = 851.834G/CM**2

PI MEP= -67.65G/CM**2 PSIA= -0.11HP
PE MEP= 345.07G/CM**2 PSEX= 0.57HP

AI MEP= 6240.0/CM**2 IHP= 10.38HP AT SFC= 171.64G/HP.HR

BM EP= 5388.0/CM**2 BHP= 8.96HP BSFC= 198.78G/BHP.HR

HEAT REJECTION= CAL/MIN
TOTAL H. RCJ =0.8189E+05
CYL. LINER =0.2081E+05
PISTON TOP =0.1673E+05
PISTON CLP =0.2044E+05
CYL. HEAD =0.1517E+05
INT. VALVE =0.7221E+04
EXH. VALVE =0.1322E+04

HEAT REJ./FUEL = 0.277

PEAK CYL PRESS= 52.9AT367.5DEG. CA