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U.S. Department of Energy, Building Technologies Program
 Mailstop EE-2J
 1000 Independence Ave., SW
 Washington, D.C. 20585-0121
 Attn: Ms. Brenda Edwards

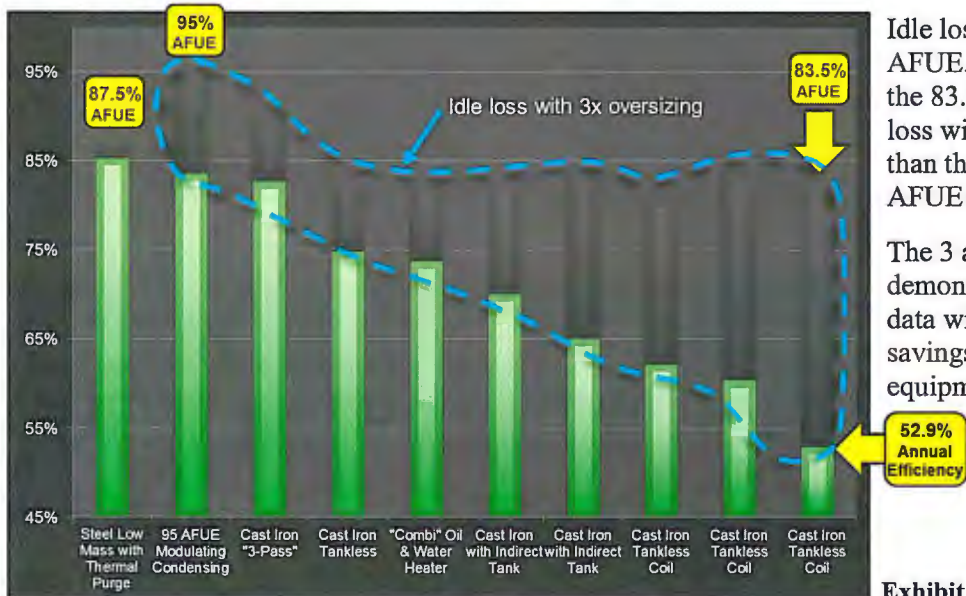
Reference: NODA for Energy Conservation Standards for Residential Boilers
 Docket No: EERE-2012-BT-STD-0047 RIN: RIN 1904-AC88

Annual Fuel Utilization Efficiency (AFUE) for residential boilers was implemented nearly 30 years ago and has gone through little practical revision since that time. Independent studies demonstrate that field savings can be much greater than demonstrated with AFUE alone, and developing a standard that is more representative of annual efficiency and actual installed performance is the right thing to do for consumers and contractors.

Further, by ignoring integrated hydronic systems, the current procedure fails to recognize and act on a very significant energy savings opportunity. Respondents have indicated that up to 75% of boilers provide heat and hot water, and we expect integrated systems to become the norm in the future.

The following statements are derived from the NODA and other information sources as footnoted:

- 1) The average design day heat loss based on Heating Degree Day (HDD) data and the average hot water boiler input indicate an average oversizing factor between 3 and 4 on design day. This exceeds the 0.7 oversizing factor indicated in the AFUE standard by 239% and 187% for gas and oil boilers respectively.¹ This 3 to 4 times oversizing has a clear and direct impact on annual efficiency due to idle losses as shown in the below graph which also reflect a small hot water load.²



Idle loss is virtually ignored in AFUE. It is important to note that the 83.5% AFUE with 4.87% idle loss will consume 63% more fuel than the 0.15% idle loss 87.5 AFUE model shown.

The 3 and 4 oversizing factors demonstrated with the NODA data will show even greater savings potential with an equipment upgrade.

Exhibit 1

- 2) The NODA documents indicate steady state efficiency is between 0.29% and 0.89% higher than AFUE for non-condensing hot water boilers, and is 2.53% lower than AFUE for condensing boilers. Replacing the AFUE metric with steady state efficiency plus a latent heat of condensation component could vastly reduce manufacturer testing burden.³

The development of a “Smart Standard” that incorporates an idle loss metric could be simultaneously implemented, greatly improving the ability of contractors and consumers to make informed decisions about energy savings and equipment selection. Idle loss values for classes of boilers could be prescriptive, further reducing testing burden. For innovative and more highly performing systems, a test method for idle loss could be implemented and the idle loss used instead. This or a similar process would foster innovation and recognize better performing systems, while simultaneously reducing test burden.

- 3) The NODA analysis showing minimal impact on AFUE based on water temperature represents the operation of temperature reset controls. With changes of less than 1% demonstrated, temperature reset controls would be highly ineffective without accounting for idle loss. Simply put, idle loss or energy wasted at the end of the heating cycle, not during the burner operation, greatly impacts annual energy efficiency.⁴

AFUE assumes that the heating boiler is in the conditioned space and heat lost is gained in the conditioned space. In practice, much of this heat energy is wasted in basements, up chimneys, and out draft hoods and draft regulators. Furthermore, if combined heat and hot water boilers are considered to be in the conditioned space, then heat lost in summer time while heating domestic water should have an impact on air conditioning cooling loads.

- 4) Because the AFUE test method states that all controls must be disabled prior to commencing testing, AFUE does not account for the impact of energy saving controls. This limitation in the test method means that consumers and contractors cannot make informed decisions about innovative systems. The combination of controls and equipment design can deliver powerful solutions for energy savings that are not reflected in AFUE alone. This situation is an impediment to the development and marketing of truly innovative and effective controls.
- 5) Although AFUE does not apply to boilers that make hot water, hot water is in effect just a small load that recurs several times a day. System controls and integration are even more critical to deliver high efficiency with combined heat and hot water systems.
- 6) AFUE is used for both boilers and furnaces, implying that these very different appliances may be compared with AFUE ratings. I feel that due to the typically high distribution efficiency of boilers and the significant distribution losses of conventionally ducted systems, these metrics should be separate and distinct. Guidance should be given to consumers reflecting distribution efficiency, especially if a more accurate rating method is employed for boilers.

- 7) Full Fuel Cycle (FFC) efficiency analysis should also be incorporated for cross class comparisons between fossil fired heat and hot water systems and electric grid based systems. Low electric power generation efficiency combined with high transmission and distribution losses creates a false sense of high efficiency regarding vapor compression cycle heating equipment driven by an electric motor when compared to direct fired heating equipment.

In closing, I reference the following response received from the Department of Energy's National Renewable Energy Labs Uniform Methods Project Comment Process for Residential Furnaces and Boilers in section (3) Savings Calculations, October 8, 2012:

Though we agree that AFUE is not the best measure to capture efficiency it is the most widely used measure of efficiency to qualify furnaces and boiler for efficiency rebates.

This needs to be corrected.

I firmly believe these important points and much needed improvements to AFUE are both technologically feasible and economically justified.

Respectfully submitted,



Roger D. Marran
President

¹ Data from EERE-2012-BT-STD-0047-0013.xlsm workbook [Shipment] downloaded on 3/7/2014. 91.9% of shipments are to the states of NY, CT, ME, NH, RI, VT, MA, NJ, PA, DE, DC, MD, IL, IA, MN, ND, SD, WI, IN, OH, OR, WA. Weather data is a summary of these states from the workbook [Weather Data] with an additional column that calculates the design day heating degree days (HDD) as a percentage of the January HDD's; the average for these states is 5.2%. Based on the January Average Monthly Fuel Consumption peak values of 16.8 MMBTU/month (HWGB) and 21.3 MMBTU/month (HWOB) shown in the [Energy Use] workbook, the maximum design day fuel consumption range is 5.2% of the monthly use or from 873,600 BTU/day to 1,107,600 BTU/day or 36,400 BTU/hr to 46,150 BTU/hr; further adjusting for 82% efficiency yields an annual average design day heat loss of 29,848 BTU/hr to 37,843 BTU/hr.

The January average energy consumption per hour can be estimated as 16.8 MMBTU/month / 31 days in January / 24 hours per day. This implies that during the heating season, the typical load averages approximately 23 MBH to 28 MBH for gas and oil water boilers respectively.

Further, table 7-B.2.7 Comparison of Derived Input Capacity to Shipment Weighted Data and Model Data provided in the NODA indicates an average 2013 boiler input rate of 151,000 BTU/hr and 158,000 BTU/hr for gas and oil water boilers respectively. This implies an average oversizing factor of 4 to 5, 298% and 256% higher than the .7 oversizing factor indicated in the AFUE standard.

Location ID	States	Shipments	Running Sum
3	New York	25.6%	25.6%
1	CT, ME, NH, RI, VT	16.0%	41.5%
2	Massachusetts	11.3%	52.8%
4	New Jersey	10.5%	63.3%
5	Pennsylvania	10.1%	73.4%
14	DE, DC, MD	5.5%	78.8%
6	Illinois	3.4%	82.2%
10	IA, MN, ND, SD	3.1%	85.2%
9	Wisconsin	2.4%	87.7%
7	Indiana, Ohio	2.2%	89.9%
27	OR, WA	2.0%	91.9%
8	Michigan	1.8%	93.7%
13	Virginia	1.8%	95.5%
22	Colorado	1.5%	97.0%
23	ID, MT, UT, WY	0.7%	97.8%
26	California	0.5%	98.2%
25	NV, NM	0.4%	98.6%
28	Alaska	0.3%	98.9%
12	Missouri	0.2%	99.1%
16	NC, SC	0.2%	99.3%
20	AR, LA, OK	0.1%	99.4%
19	Tennessee	0.1%	99.5%
11	Kansas, Nebraska	0.1%	99.6%
18	AL, KY, MS	0.1%	99.7%
21	Texas	0.1%	99.8%
24	Arizona	0.1%	99.9%
30	West Virginia	0.1%	100.0%
15	Georgia	0.0%	100.0%
17	Florida	0.0%	100.0%
29	Hawaii	0.0%	100.0%
31	United States	100.0%	

Monthly Fuel (Gas/LPG/Oil) Use		1	2	3	4	5	6	7	8	9	10	11	12			
Annual Fuel Use		Average Monthly Fuel Energy Consumption (MMBtu/month)												(MMBtu/yr)		
Description		Jan	Feb	Mar	Apr	May	Jun	Jul	Aug	Sep	Oct	Nov	Dec	(MMBtu/yr)		
iPC	iEL	31	28	31	30	31	30	31	31	30	31	30	31			
1	1 HWGB	0	82% AFUE - Baseline	16.8	14.3	11.8	6.8	3.5	1.0	0.1	0.2	1.4	5.7	9.0	14.0	84.7
2	1 HWGB	1	83% AFUE - Increased HX Area	16.6	14.1	11.6	6.7	3.5	1.0	0.1	0.2	1.4	5.7	8.9	13.8	83.6
3	1 HWGB	2	84% AFUE - Increased HX Area	16.4	14.0	11.5	6.6	3.4	1.0	0.1	0.2	1.4	5.6	8.8	13.6	82.6
4	1 HWGB	3	85% AFUE - Increased HX Area	16.2	13.8	11.3	6.5	3.4	1.0	0.1	0.2	1.4	5.5	8.7	13.4	81.5
5	1 HWGB	4	90% AFUE - Condensing Baseline	15.6	13.3	10.9	6.3	3.3	1.0	0.1	0.2	1.3	5.3	8.4	13.0	78.6
6	1 HWGB	5	92% AFUE - Increased HX Area	15.4	13.1	10.8	6.2	3.2	1.0	0.1	0.2	1.3	5.3	8.3	12.8	77.7
7	1 HWGB	6	96% AFUE - Max Tech	14.8	12.5	10.3	6.0	3.1	0.9	0.1	0.2	1.2	5.0	7.9	12.3	74.3
8	2 SGB	0	80% AFUE - Baseline	9.0	7.7	6.5	4.0	2.1	0.8	0.4	0.4	0.8	2.8	4.8	7.4	38.6
9	2 SGB	1	82% AFUE - Increased HX Area	8.8	7.5	6.4	3.9	2.1	0.8	0.3	0.4	0.7	2.7	4.7	7.2	37.7
10	2 SGB	2	83% AFUE - Max Tech	8.7	7.5	6.3	3.8	2.1	0.7	0.3	0.4	0.7	2.7	4.6	7.1	37.2
11	3 HWOB	0	84% AFUE - Baseline	6.8	5.9	4.3	2.3	0.9	0.1	0.0	0.0	0.3	2.1	3.8	6.0	32.5
12	3 HWOB	1	85% AFUE - Increased HX Area	6.7	5.8	4.2	2.2	0.9	0.1	0.0	0.0	0.3	2.1	3.7	5.9	32.1
13	3 HWOB	2	86% AFUE - Increased HX Area	6.6	5.7	4.2	2.2	0.9	0.1	0.0	0.0	0.3	2.1	3.7	5.9	31.8
14	3 HWOB	3	91% AFUE - Condensing (Max Tec	6.4	5.6	4.0	2.1	0.9	0.1	0.0	0.0	0.3	2.0	3.6	5.7	30.7
15	4 HWOB	0	82% AFUE - Baseline	21.3	18.1	14.9	8.6	4.5	1.3	0.1	0.3	1.8	7.3	11.4	17.7	107.5
16	4 HWOB	1	84% AFUE - Increased HX Area	20.8	17.7	14.6	8.4	4.4	1.3	0.1	0.3	1.7	7.1	11.2	17.3	104.9
17	4 HWOB	2	85% AFUE - Increased HX Area	20.6	17.5	14.4	8.3	4.3	1.3	0.1	0.3	1.7	7.0	11.0	17.1	103.7
18	4 HWOB	3	86% AFUE - Max Tech	20.4	17.3	14.3	8.2	4.3	1.3	0.1	0.3	1.7	7.0	10.9	16.9	102.5

Table 7-B.2.7 Comparison of Derived Input Capacity to Shipment Weighted Data and Model Data

Product Class	DOE Derived Average		2013 DOE Boiler Models*		Shipment Weighted Output Capacity for Cast Iron Boilers**			
	Input Capacity	Output Capacity	Input Capacity	Output Capacity	1970	1980	1990	2000
Gas-Fired Hot Water Boilers	164	145	151	131	124	122	113	116
Gas-Fired Steam Boilers	179	135	172	142	153	153	138	139
Oil-Fired Hot Water Boilers	176	144	158	137	148	154	147	144
Oil-Fired Steam Boilers	179	148	190	162	1	1	1	1
Electric Boiler	164	147	NA	NA	NA	NA	NA	NA

* Based on October 2013 AHRI directory as well as other sources (see Appendix 7-D).

** Based on AHRI (formerly GAMA) shipment data provided in April 2002.

² Butcher, T. 2007. "Performance of Integrated Hydronic Systems." BNL Report BNL-79814-IR.
<http://www.osti.gov/bridge/purl.cover.jsp?purl=/924431-kio3fU/> . Table 3 and appendices for AFUE values.

Table 3. Results of Analysis of Annual Performance with Each Unit

Unit	Description	Steady State Thermal Efficiency (%)	Idle Loss (%)	Annual Efficiency ¹ (%) Oversize = 2	Annual Efficiency ¹ (%) Oversize = 3	Summer DHW oil use (gal) Oversize = 2	Annual Oil Use (gal) ² Oversize =2
1	Oil, cast iron boiler with tankless	83.7	1.2	77.9	74.9	.54	897
2	Oil, cast iron boiler with indirect	78.4	2.1	72.9	65.1	.74	1007
3	Oil, steel boiler with purge control	86.5	.15	85.7	85.3	.36	816
4	Oil condensing boiler	92.0	1.5	84.2	80.3	.54	830
5	Oil, well insulated cast iron	87.5	.69	84.4	82.7	.42	828
6	Oil, water heater used also for heating	81.5	1.2	75.9	73.0	.56	921
7	Oil, combi System	79.5	.8	75.8	73.8	.51	923
8	Gas atmospheric with tankless	72.5	1.7	65.6	62.2	.72	1065
9	Gas atmospheric water heater	74.5	.65			.51	976
8+9	Gas boiler + separate gas water heater			66.6	64.7	.51	1081
10	Old cast iron boiler	72.8	2.1	64.5	60.4	.79	1085
11	Gas cond. modulating	88.5	.60	85.3	83.6	.42	819
12a	tankless mode	78.0	4.87	60.0	52.9	1.22	1165
12b	indirect mode	78.0	1.16	72.8	70.1	.57	960

1. Based on oversize factor stated, not actual firing rate tested.

³ EERE-2012-BT-STD-0047-0012.xlsm workbook [Energy Use].

Difference Between AFUE and Steady State Efficiency			
HWGB	non,condensing	0.29	82
HWGB	90-91 AFUE	0.14	90
HWGB	92+ AFUE	-2.53	92
SGB		0.31	100
HWOB		0.89	
SOB		1.22	

Energy Use Calculations							
		Level	Description	AFUE	Average Return Temp	AFUE Adj	Diff Between AFUE and Eff _{ss,M} (%)
1	HWGB	0	82% AFUE - Baseline	82%	150	81.4%	0.29%
2	HWGB	1	83% AFUE - Increased HX Area	83%	150	82.4%	0.29%
3	HWGB	2	84% AFUE - Increased HX Area	84%	150	83.4%	0.29%
4	HWGB	3	85% AFUE - Increased HX Area	85%	150	84.4%	0.29%
5	HWGB	4	90% AFUE - Condensing Baseline	90%	150	87.0%	0.14%
6	HWGB	5	92% AFUE - Increased HX Area	92%	150	89.0%	-2.53%
7	HWGB	6	96% AFUE - Max Tech	96%	150	93.0%	-2.53%
8	SGB	0	80% AFUE - Baseline	80%	150	80.0%	0.31%
9	SGB	1	82% AFUE - Increased HX Area	82%	150	82.0%	0.31%
10	SGB	2	83% AFUE - Max Tech	83%	150	83.0%	0.31%
11	HWOB	0	84% AFUE - Baseline	84%	150	83.4%	0.89%
12	HWOB	1	85% AFUE - Increased HX Area	85%	150	84.4%	0.89%
13	HWOB	2	86% AFUE - Increased HX Area	86%	150	85.4%	0.89%
14	HWOB	3	91% AFUE - Condensing (Max Te	91%	150	88.0%	0.89%
15	SOB	0	82% AFUE - Baseline	82%	150	82.0%	1.22%
16	SOB	1	84% AFUE - Increased HX Area	84%	150	84.0%	1.22%
17	SOB	2	85% AFUE - Increased HX Area	85%	150	85.0%	1.22%
18	SOB	3	86% AFUE - Max Tech	86%	150	86.0%	1.22%

⁴ EERE-2012-BT-STD-0047-0012.xlsm in workbook [AFUE Existing] and in "TSD_NODA_Appendix_7-B_Determination_of_Boiler_Energy_Use_in_the_LCC_Analysis.pdf"; excerpts shown below.

Import Parameters	1	2	3	4	5	6
From Bldg Data	HWGB	SGB	HWOB	SOB	HWEB	SEB
Household Region	1	1	5	1	3	5
Age of Boiler (in 2009)	15	6	26	26	0	26

Intermediate Parameters	1	2	3	4
Percentile (for AFUE)	86%			
AFUE for Existing Unit	83%	82%	84%	82%
Return Temperature	150	150	150	150

Export Parameters	1	2	3	4	5	6
To Energy Use, Installation Cost						
Adjusted AFUE	82%	81%	83%	81%	98%	98%

Table 7-B.2.12 summarizes the AFUE adjustments used in the analysis. The adjusted AFUE values reflect the efficiency of boilers for the fraction of households and buildings that utilize hydronic heat distribution systems which operate at return water temperatures different from 120°F.

Table 7-B.2.12 Adjustment to AFUE Based on RWT Application

	Low RWT (100°F)	Medium RWT (120°F) ¹	High RWT (150°F)
Non-Condensing	-	As Report	-0.6%
Condensing	+3%	As Report	-3%